

Good

800127

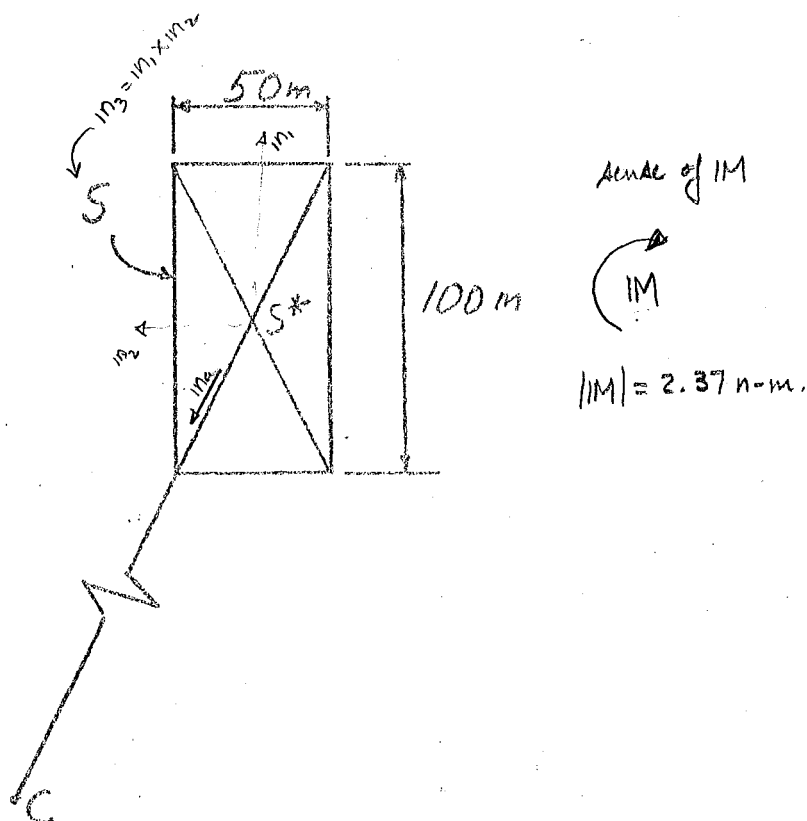
When the mass center S^* of an Earth satellite S moves on a circular orbit, the moment $\underline{M}$ about S^* of all gravitational forces exerted on S by the Earth is given approximately by

$$\underline{M} = 3\Omega^2 \underline{n}_a \times \underline{I}_a$$

where Ω is the angular speed of the line joining S^* to the center C of the Earth, $\underline{n}_a$ is a unit vector parallel to this line, and $\underline{I}_a$ is the second moment of S relative to S^* for $\underline{n}_a$.

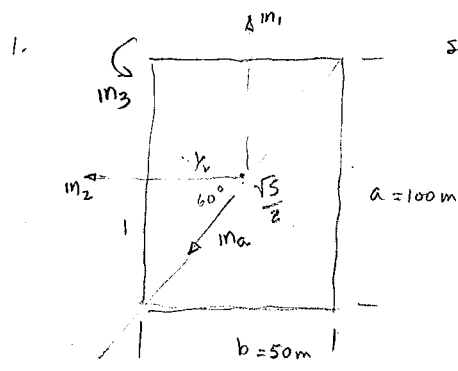
The sketch shows a satellite consisting of a uniform rectangular plate having a mass of 1000 kg and having sides of length 50 m and 100 m. This satellite has a period of 90 minutes.

Determine the magnitude of $\underline{M}$ and indicate the sense of $\underline{M}$ on a sketch showing C , S^* , and S .





Very good



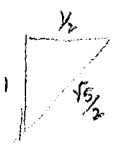
Since n_1, n_2, n_3 are unit vectors in direction of principal axes

then
$$I = \frac{mb^2}{12} n_1 n_1 + \frac{ma^2}{12} n_2 n_2 + \frac{m}{12} (a^2 + b^2) n_3 n_3$$

$$n_a = \frac{1}{\sqrt{5}} n_2 - \frac{2}{\sqrt{5}} n_1$$

$$I \cdot n_a = I_a = \frac{mb^2}{12} \left(-\frac{2}{\sqrt{5}} n_1 \right) + \frac{1}{\sqrt{5}} \frac{ma^2}{12} n_2$$

$$\begin{aligned} n_a \times I_a &= \left(\frac{1}{\sqrt{5}} n_2 - \frac{2}{\sqrt{5}} n_1 \right) \times \left(-\frac{mb^2}{12} \cdot \frac{2}{\sqrt{5}} n_1 + \frac{1}{\sqrt{5}} \frac{ma^2}{12} n_2 \right) \\ &= \frac{mb^2}{30} n_3 - \frac{1}{15} ma^2 n_3 \\ &= \frac{m}{30} (b^2 - 2a^2) n_3 = \frac{1000}{30} (-17500) n_3 = -\frac{1.75}{30} \times 10^7 n_3 \text{ kg} \cdot \text{m}^2 \cdot \text{s}^{-2} \\ &= \left(-\frac{1.75}{3} \times 10^6 \text{ kg} \cdot \text{m}^2 \right) n_3 \end{aligned}$$



hence
$$M = 3\Omega^2 \left(-\frac{1.75 \times 10^6}{3} \right) n_3 \frac{\text{kg} \cdot \text{m}^2}{\text{sec}^2}$$

$$= -1.75 \times 10^6 \Omega^2 n_3 \frac{\text{kg} \cdot \text{m}^2}{\text{sec}^2}$$

where Ω is in rads/sec .

$$n_a = \frac{1}{\sqrt{5}} n_2 - \frac{2}{\sqrt{5}} n_1$$

1 period = 90 min

$$\Omega t = 2\pi \text{ rad.}$$

$\Omega = \frac{2\pi}{90 \times 60 \text{ sec}}$

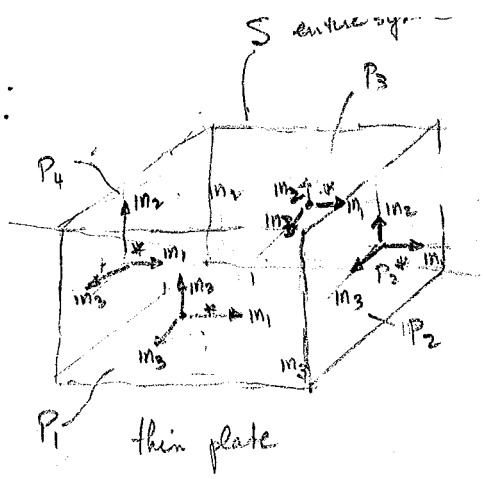
$$\Omega = \frac{2\pi}{54 \times 10^3}$$

$$M = \left(\frac{2\pi}{5.4 \times 10^3} \right)^2 \left(-1.75 \times 10^6 \right) n_3 \text{ N-m} = \frac{-4\pi^2 (1.75)}{(5.4)^2} n_3 \text{ N-m}$$

$$|M| = \frac{4\pi^2 (1.75)}{(5.4)^2} \text{ N-m} = 2.369 \text{ N-m}$$

Very good - see p. 2

LEVY



let m_1, m_2, m_3 be fixed in center of box

let each of these vectors be defined for each plate. $\therefore m_i \parallel m_i$

$$I_{P_1/P_1^*} = \frac{ma^2}{12} (m_1 m_1 + m_2 m_2 + 2m_3 m_3)$$

$$I_{P_2/P_2^*} = \frac{ma^2}{12} (2m_1 m_1 + m_2 m_2 + m_3 m_3)$$

$$I_{P_3/P_3^*} = \frac{ma^2}{12} (m_1 m_1 + m_2 m_2 + 2m_3 m_3)$$

$$I_{P_4/P_4^*} = \frac{ma^2}{12} (2m_1 m_1 + m_2 m_2 + m_3 m_3)$$

now $m_a = \frac{1}{\sqrt{3}} (m_2 + m_3 - m_1)$

then $I_{P_1/P_1^*} \cdot m_a = \frac{ma^2}{12\sqrt{3}} (-m_1 + m_2 + 2m_3)$

$I_{aa} = \frac{ma^2}{12 \cdot 3} \cdot 4 = \frac{ma^2}{9}$

$I_{P_2/P_2^*} \cdot m_a = \frac{ma^2}{12 \cdot \sqrt{3}} (-2m_1 + m_2 + m_3)$

$I_{aa} = \frac{ma^2}{12 \cdot 3} \cdot 4 = \frac{ma^2}{9}$

$I_{P_3/P_3^*} \cdot m_a = \frac{ma^2}{12 \cdot \sqrt{3}} (-m_1 + m_2 + 2m_3)$

$I_{aa} = \frac{ma^2}{12 \cdot 3} \cdot 4 = \frac{ma^2}{9}$

$m_a = I_{P_4/P_4^*} \cdot m_a = \frac{ma^2}{9}$

let $S^* =$ center of box

now $\rightarrow I_{aa}^{P_1/S^*} = I_{aa}^{P_1/P_1^*} + I_{aa}^{P_1^*/S^*}$ w/ $P_1^*/S^* = \frac{1}{2} a m_3$

But $\rightarrow I_{aa}^{P_1^*/S^*} = m \left[\frac{1}{4} a^2 - \frac{a \cdot a}{2\sqrt{3} \cdot 2\sqrt{3}} \right] = \frac{a^2}{6} = m(p \cdot p - (p \cdot m_a)(p \cdot m_a))$

also: $I_{aa}^{P_1/S^*} = \frac{ma^2}{9} + \frac{ma^2}{6} = \frac{10ma^2}{36}$

$I_{aa}^{P_2/S^*} = I_{aa}^{P_2/P_2^*} + I_{aa}^{P_2^*/S^*}$ w/ $I_{aa}^{P_2^*/S^*} = m(p \cdot p - (p \cdot m_a)(p \cdot m_a))$

$= \frac{10a^2 m}{36}$

thus $P_3^*/S^* = \frac{1}{2} a m_1 \Rightarrow$

$I_{aa}^{P_3/S^*} = m \left(\frac{a^2}{4} - \left(\frac{-a}{2\sqrt{3}} \right) \left(\frac{-a}{2\sqrt{3}} \right) \right) = \frac{ma^2}{6}$

thus we can similarly show the same results for

$I_{aa}^{P_3/S^*} = \frac{10ma^2}{36}$

$I_{aa}^{P_4/S^*} = \frac{10ma^2}{36}$

$S =$ box thus $I_{aa}^{S/S^*} = \sum I_{aa}^{P_i/S^*} = \frac{40ma^2}{36}$

but $I = M k^2 = 4m k^2 = \frac{40ma^2}{36}$

$\therefore k^2 = \frac{10a^2}{9} \Rightarrow k = \frac{a\sqrt{10}}{3}$

I think it's over please

on ball $I^{P/S^*} = \frac{ma^2}{12} (n_1 n_1 + 2n_2 n_2 + n_3 n_3)$

$$I_{aa} = \frac{ma^2}{9}$$

& we can do same to show $I^{P/S^*} = \sqrt{\frac{10ma^2}{36}}$

let $S = \text{box}$

hence $I^{S/B^*} = \sum I = \frac{50ma^2}{36} = Mk^2$

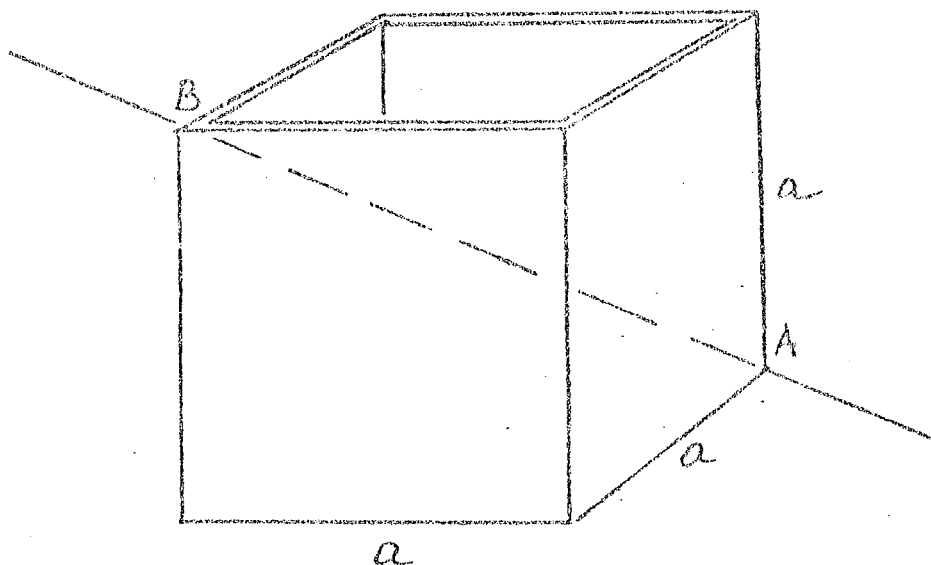
$$5 \neq Mk^2 = \frac{50ma^2}{36}$$

$$k^2 = \frac{50a^2}{144}$$

$$k = \frac{a\sqrt{50}}{12} \times \frac{a \cdot 5\sqrt{2}}{12} = \frac{\sqrt{10}}{6} a$$

860128

The sketch shows an open box whose base and sides consist of identical, thin, uniform plates. Determine the radius of gyration of the box with respect to line AB.



800129

A deformable body α consists of two rigid bodies β and γ which are connected to each other at their respective mass centers but are otherwise free to move relative to each other.

At a certain instant t^* , the central principal axes of inertia of β , called Y_1, Y_2, Y_3 , coincide respectively with Z_1, Z_2, Z_3 , the central principal axes of γ ; and the angular velocity of γ relative to β at this instant is given by

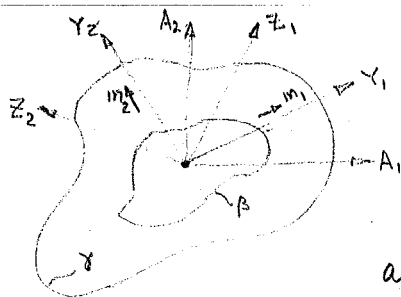
$$\underline{\omega}^{\beta\gamma} = \omega_1 \underline{n}_1 + \omega_2 \underline{n}_2 + \omega_3 \underline{n}_3$$

where $\underline{n}_1, \underline{n}_2, \underline{n}_3$ are unit vectors respectively parallel to Y_1, Y_2, Y_3 (and Z_1, Z_2, Z_3).

Letting A be a reference frame in which the central principal axes of α are fixed, determine the angular velocity of A relative to β for the instant t^* . Express the result in the form

$$\underline{\omega}^{\beta A} = f_1 \underline{n}_1 + f_2 \underline{n}_2 + f_3 \underline{n}_3$$

where f_1, f_2, f_3 are functions of $\omega_1, \omega_2, \omega_3$ and of the central principal moments of inertia $\beta_1, \beta_2, \beta_3$ and $\gamma_1, \gamma_2, \gamma_3$ of β and γ , respectively; that is, determine these functions.



Let Y_1, Y_2, Y_3 be the principal axes of inertia of β

Let Z_1, Z_2, Z_3 be the principal axes of inertia of α

Let A_1, A_2, A_3 be the principal axes of inertia of α in a reference frame A which is fixed

at time $t=t^*$ the Z_i 's coincide with the Y_i 's. Furthermore

Let them coincide with the A_i 's. Let the m_i 's be unit vectors $\parallel$ to Y_i 's.

If @ $t=t^*$ ${}^A\omega^\alpha = \omega_1 m_1 + \omega_2 m_2 + \omega_3 m_3$ & β_i 's and δ_i 's are principal moments of inertia of β

Find: ${}^A\omega^A = f_1 m_1 + f_2 m_2 + f_3 m_3$

In the reference frame A

$${}^A\omega^\alpha = {}^A\omega^\delta + {}^\delta\omega^\alpha \quad (1)$$

where δ is a fictitious rigid body having the mass distribution of α but the instantaneous motion of δ . Hence ${}^A\omega^\delta = \mathbb{I}^\delta \omega^\delta$ (2)

yet we may also write

$${}^\delta\omega^\alpha = {}^\delta\omega^\beta + {}^\beta\omega^\alpha \quad (3)$$

But ${}^\delta\omega^\alpha = m_\beta \mathbb{I}^{\beta/\alpha} \times \omega^\beta = 0$ since β^* and α^* coincide (4)

$$\text{Thus } {}^\delta\omega^\alpha = {}^\delta\omega^\beta = \mathbb{I}^{\beta/\alpha} \omega^\beta \quad (5)$$

(4,3)

Now at the instant of coincidence ${}^A\omega^\alpha = 0$ because ${}^A\omega^\alpha = 0$ (6)

Also at that instant from (2)

$${}^A\omega^\delta = \mathbb{I}^\delta \omega^\delta = (\mathbb{I}^{\beta/\alpha} + \mathbb{I}^{\delta/\alpha}) \omega^\delta$$

α is not a rigid body. ${}^A\omega^\alpha$ is not meaningful.

due to all axes being principal axes and coinciding

Now we also know that at that instant

$$\mathbb{I}^{\beta/\alpha} = \beta_1 m_1 m_1 + \beta_2 m_2 m_2 + \beta_3 m_3 m_3 \quad (8)$$

and

$$\mathbb{I}^{\delta/\alpha} = (\beta_1 + \delta_1) m_1 m_1 + (\beta_2 + \delta_2) m_2 m_2 + (\beta_3 + \delta_3) m_3 m_3 \quad (9)$$

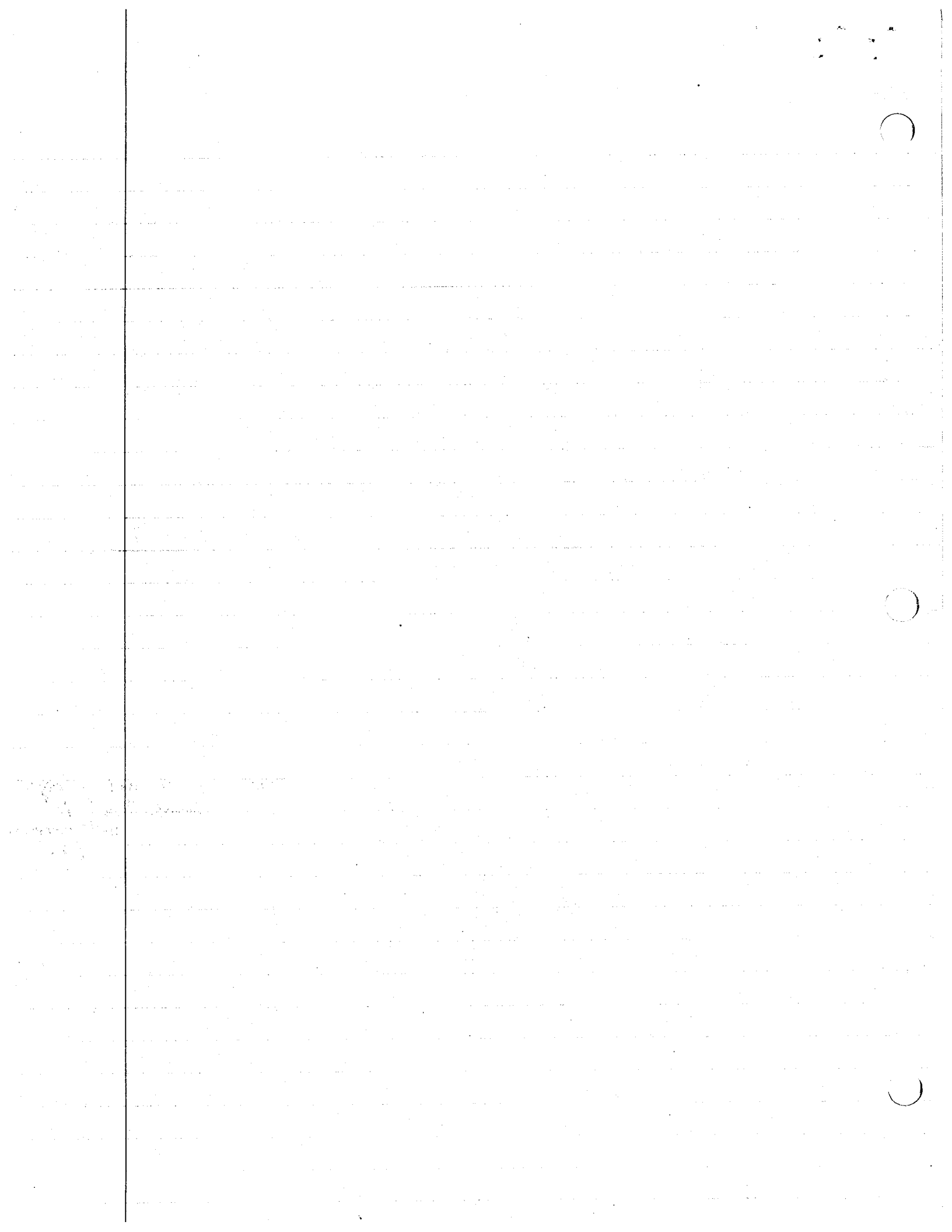
but also

$${}^A\omega^\delta = {}^A\omega^\beta + {}^\beta\omega^\delta = (-f_1 + \omega_1) m_1 + (-f_2 + \omega_2) m_2 + (-f_3 + \omega_3) m_3 \quad (10)$$

Thus using (5-10) in (3) we obtain

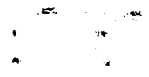
$$\begin{aligned} 0 &= (\beta_1 + \delta_1)(-f_1 + \omega_1) m_1 + (\beta_2 + \delta_2)(-f_2 + \omega_2) m_2 + (\beta_3 + \delta_3)(-f_3 + \omega_3) m_3 - [\beta_1 \omega_1 m_1 + \beta_2 \omega_2 m_2 + \beta_3 \omega_3 m_3] \\ &= (\omega_1 \delta_1 - f_1 [\beta_1 + \delta_1]) m_1 + (\omega_2 \delta_2 - f_2 [\beta_2 + \delta_2]) m_2 + (\omega_3 \delta_3 - f_3 [\beta_3 + \delta_3]) m_3 \end{aligned} \quad (11)$$

Solving the 3 scalar equations obtained from (11) yields



$$f_1 = \frac{\gamma_1 \omega_1}{\gamma_1 + \beta_1} \quad f_2 = \frac{\gamma_2 \omega_2}{\gamma_2 + \beta_2} \quad f_3 = \frac{\gamma_3 \omega_3}{\gamma_3 + \beta_3} \quad \times$$

(12, 13, 14)



Christine Miller

$$\text{Body } \beta \begin{cases} Y_1 & Y_2 & Y_3 & \text{prin. axes} \\ b_1 & b_2 & b_3 & \text{unit vectors along prin. axes} \\ \beta_1 & \beta_2 & \beta_3 & \text{prin. moments} \end{cases}$$

$$\text{Body } \gamma \begin{cases} Z_1 & Z_2 & Z_3 & \text{prin. axes} \\ c_1 & c_2 & c_3 & \text{unit vectors along prin. axes} \\ \gamma_1 & \gamma_2 & \gamma_3 & \text{prin. moments} \end{cases}$$

Let A be a co-ord. system in which the central prin. moments of $\alpha = \beta + \gamma$ are fixed, with unit vectors a_1, a_2, a_3

At t^* , X, Y, Z, A correspond and:

$$\beta \omega^\gamma = \omega_1 n_1 + \omega_2 n_2 + \omega_3 n_3 = \omega_1 b_1 + \omega_2 b_2 + \omega_3 b_3$$

$$\beta \omega^A = f_1 n_1 + f_2 n_2 + f_3 n_3 = f_1 b_1 + f_2 b_2 + f_3 b_3$$

Let " $\sim$ " denote quantities at t^* . We will work in a reference frame β where b_i^0 are fixed. At a very small time t after t^* ,

$$a_i = \tilde{a}_i + \left(\frac{da_i}{dt} \right)_{t^*} t + \dots \approx \tilde{a}_i + (\beta \omega^A \times a_i)_{t^*} t$$

where we will drop all terms of order two or higher in t .

$$\begin{aligned} a_1 &= b_1 + [(f_1 b_1 + f_2 b_2 + f_3 b_3) \times b_1] t \\ &= b_1 + f_3 t b_2 - f_2 t b_3 \end{aligned}$$

$$\begin{aligned} a_2 &= b_2 + [(f_1 b_1 + f_2 b_2 + f_3 b_3) \times b_2] t \\ &= -f_3 t b_1 + b_2 + f_1 t b_3 \end{aligned}$$

$$\begin{aligned} a_3 &= b_3 + [(f_1 b_1 + f_2 b_2 + f_3 b_3) \times b_3] t \\ &= f_2 t b_1 - f_1 t b_2 + b_3 \end{aligned}$$

For the vectors a_i , for the reference frame $\mathcal{S}$:

$${}^{\mathcal{S}}\omega^A = {}^{\mathcal{S}}\omega^B + \beta^A \omega^B = \beta^A \omega^B - \beta^B \omega^A$$

$${}^A\omega^{\mathcal{S}} = (f_1 - \omega_1) a_1 + (f_2 - \omega_2) a_2 + (f_3 - \omega_3) a_3$$

$$a_i = \tilde{a}_i + \left(\frac{da_i}{dt} \right)_{t^x} t + \dots \equiv \tilde{a}_i + ({}^{\mathcal{S}}\omega^A \times a_i)_{t^x} t$$

$$a_1 = e_1 + (f_3 - \omega_3) t e_2 - (f_2 - \omega_2) t e_3$$

$$a_2 = -(f_3 - \omega_3) t e_1 + e_2 + (f_1 - \omega_1) t e_3$$

$$a_3 = (f_2 - \omega_2) t e_1 - (f_1 - \omega_1) t e_2 + e_3$$

Summarizing,

	b_1	b_2	b_3	e_1	e_2	e_3
a_1	1	$f_3 t$	$-f_2 t$	1	$(f_3 - \omega_3) t$	$-(f_2 - \omega_2) t$
a_2	$-f_3 t$	1	$f_1 t$	$-(f_3 - \omega_3) t$	1	$(f_1 - \omega_1) t$
a_3	$f_2 t$	$-f_1 t$	1	$(f_2 - \omega_2) t$	$-(f_1 - \omega_1) t$	1

At time t we can calculate the products of inertia of bodies $\beta \in \mathcal{S}$ about their mass centers for axes a_i :

$$I_{ab} = \sum_{j=1}^3 \sum_{k=1}^3 a_j I_{jk} b_k$$

$$I_{12}^{\beta/\beta^*} = -\beta_1 f_3 t + \beta_2 f_3 t + \beta_3 f_2 t^2 f_1$$

$$I_{12}^{\mathcal{S}/\mathcal{S}^*} = -\gamma_1 (f_3 - \omega_3) t + \gamma_2 (f_3 - \omega_3) t - \gamma_3 (f_2 - \omega_2) (f_1 - \omega_1) t^2$$

$$I_{23}^{\beta/\beta^*} = -\beta_1 f_3 t^2 f_2 + \beta_2 f_1 t + \beta_3 f_1 t$$

$$I_{23}^{\mathcal{S}/\mathcal{S}^*} = -\gamma_1 (f_3 - \omega_3) (f_2 - \omega_2) t^2 - \gamma_2 (f_1 - \omega_1) t + \gamma_3 (f_1 - \omega_1) t$$

$$I_{13}^{\beta/\beta^*} = \beta_1 f_2 t - \beta_2 f_3 t^2 f_1 - \beta_3 f_2 t$$

$$I_{13}^{\gamma/\gamma^*} = \gamma_1 (f_2 - \omega_2) t - \gamma_2 (f_3 - \omega_3) (f_1 - \omega_1) t^2 - \gamma_3 (f_2 - \omega_2) t$$

Since the a_i are the principal axes, the products of inertia for $\beta + \gamma$ must disappear.

$$I_{12}^{\beta/\beta^*} + I_{12}^{\gamma/\gamma^*} = 0$$

$$-\beta_1 f_3 t + \beta_2 f_3 t - \gamma_1 (f_3 - \omega_3) t + \gamma_2 (f_3 - \omega_3) t = 0$$

$$f_3 (-\beta_1 + \beta_2 - \gamma_1 + \gamma_2) = \omega_3 (\gamma_2 - \gamma_1)$$

$$f_3 = \frac{\omega_3 (\gamma_2 - \gamma_1)}{\beta_2 - \beta_1 + \gamma_2 - \gamma_1}$$

$$I_{23}^{\beta/\beta^*} + I_{23}^{\gamma/\gamma^*} = 0$$

$$-\beta_2 f_1 t + \beta_3 f_1 t - \gamma_2 (f_1 - \omega_1) t + \gamma_3 (f_1 - \omega_1) t = 0$$

$$f_1 (-\beta_2 + \beta_3 - \gamma_2 + \gamma_3) = \omega_1 (\gamma_3 - \gamma_2)$$

$$f_1 = \frac{\omega_1 (\gamma_3 - \gamma_2)}{\beta_3 - \beta_2 + \gamma_3 - \gamma_2}$$

$$I_{13}^{\beta/\beta^*} + I_{13}^{\gamma/\gamma^*} = 0$$

$$\beta_1 f_2 t - \beta_3 f_2 t + \gamma_1 (f_2 - \omega_2) t - \gamma_3 (f_2 - \omega_2) t = 0$$

$$f_2 (\beta_1 - \beta_3 + \gamma_1 - \gamma_3) = \omega_2 (\gamma_1 - \gamma_3)$$

$$f_2 = \frac{\omega_2 (\gamma_1 - \gamma_3)}{\beta_1 - \beta_3 + \gamma_1 - \gamma_3}$$

Very nice

23
LEVY

800130

A rigid body A and a rigid axisymmetric rotor B whose mass center B* and axis of symmetry are fixed in A form a "gyrostat" C. Letting C* designate the mass center of C, express the inertia torque of C in a reference frame R solely in terms of the inertia dyadics $\underline{I}^{C/C*}$ and $\underline{I}^{B/B*}$, the angular velocities $\underline{\omega}^{R/A}$ and $\underline{\omega}^{A/B}$, and the angular accelerations $\underline{\alpha}^{R/A}$ and $\underline{\alpha}^{A/B}$.

Problem #1

From Class we had been given the following:

$$\mathbf{H}^{C/C^*} = \mathbf{H}^{D/D^*} + \mathbf{H}^{B/C^*} \quad (1)$$

where C makes up the system of the rotor B and the rigid body A; D is a fictitious rigid body whose mass distribution is the same as C but whose motion at any instant is that of A. The asterisk (*) denotes the mass center

In problem 7(l) we had showed that for a system of particles S

$$\mathbf{H}^{S/O} = \mathbf{H}^{S/S^*} + \mathbf{H}^{S^*/O} \quad (2a)$$

where it was shown that $\mathbf{H}^{S/O} = m_S \mathbf{R}^{S^*/O} \times \mathbf{V}^{S^*}$ (2b)

where $\mathbf{H}^{S/O}$ is the angular momentum of S with respect to point O and $m_S = \sum_{i=1}^N m_i$

Thus $\mathbf{H}^{B/C^*} = \mathbf{H}^{B/B^*} + \mathbf{H}^{B^*/C^*}$ (2c)

where $\mathbf{H}^{B^*/C^*} = m_B \mathbf{R}^{B^*/C^*} \times \mathbf{V}^{B^*}$ (2d)

But because B^* is fixed in A

$$\mathbf{V}^{B^*} = 0 \Rightarrow \mathbf{H}^{B^*/C^*} \neq 0 \quad (2e, f)$$

Thus $\mathbf{H}^{B/C^*} = \mathbf{H}^{B/B^*}$ using (2f) & (2c). (2g)

By problem 8(b) $\frac{d}{dt} \mathbf{H}^{C/C^*} = \frac{d}{dt} \mathbf{H}^{D/D^*}$ (3a)

Thus by differentiation of $\mathbf{H}^{C/C^*} = \mathbf{H}^{D/D^*} + \mathbf{H}^{B/B^*}$ (3b)

we will find

$$\frac{d}{dt} \mathbf{H}^{C/C^*} = \frac{d}{dt} \mathbf{H}^{D/D^*} + \frac{d}{dt} \mathbf{H}^{B/B^*} \quad (4)$$

Now $\frac{d}{dt} \mathbf{H}^{B/B^*} = \frac{d}{dt} \mathbf{H}^{B/B^*} + \omega^A \times \mathbf{H}^{B/B^*} = -\mathbf{H}^{B/B^*} + \omega^A \times \mathbf{H}^{B/B^*}$ (5a)

$$= -\mathbf{H}^{B/B^*} - (\mathbf{H}^{B/B^*} \cdot \omega^A) \times \omega^A \quad \text{using results of problem (7i).} \quad (6)$$

Thus $\frac{d}{dt} \mathbf{H}^{B/B^*} = - \left[(\mathbf{H}^{B/B^*} \cdot \omega^A) \times \omega^A - \mathbf{H}^{B/B^*} \cdot \omega^A + (\mathbf{H}^{B/B^*} \cdot \omega^A) \times \omega^A \right]$ (7)

Now we look at $\frac{d}{dt} \mathbf{H}^{D/D^*} = - \left[(\mathbf{H}^{D/D^*} \cdot \omega^A) \times \omega^A - \mathbf{H}^{D/D^*} \cdot \omega^A \right]$ (8)

but since D represents a rigid body (fictitious though) whose motion is that of A at any instant and whose mass distribution is that of C, then

$$\omega^D = \omega^A \quad \omega^D = \omega^A \quad \text{and} \quad \mathbf{H}^{D/D^*} = \mathbf{H}^{C/C^*} \quad (9a, b, c)$$

Placing (9a-c) into (8) yields



$$\left[\frac{d}{dt} \mathbb{I}^{R/D*} = - \left[\mathbb{I}^{C/C*, R/A} \times \omega^A - \mathbb{I}^{C/C*, P/A} \right] \right] \quad (10)$$

Now using (10) & (7) in (4). then

$$\mathbb{I}^{R/C} = \left(\mathbb{I}^{C/C*, R/A} \right) \times \omega^A - \mathbb{I}^{C/C*, P/A} + \left(\mathbb{I}^{B/B*, A/B} \right) \times \omega^B - \mathbb{I}^{B/B*, A/B} \times \omega^B + \left(\mathbb{I}^{B/B*, A/B} \right) \times \omega^B \quad (11)$$

which was to be obtained

To show (10)

$$\frac{d}{dt} \mathbb{I}^{R/D*} = \frac{d}{dt} \sum m_i r_i^{P_i/C*} (\omega^A \times r_i^{P_i/C*}) = \frac{d}{dt} \sum m_i r_i^{P_i/C*} \times (\omega^A \times r_i^{P_i/C*}) + \omega^A \times \mathbb{I}^{R/D*}$$

but $\frac{d}{dt} \sum m_i r_i^{P_i/C*} \times (\omega^A \times r_i^{P_i/C*}) = \sum m_i r_i^{P_i/C*} \times \left(\frac{d}{dt} \omega^A \right) \times r_i^{P_i/C*}$ since $r_i^{P_i/C*}$ is fixed wrt body C as well as body D.

$$\text{but } \frac{d}{dt} \omega^A = \frac{d}{dt} \omega^A + \omega^A \times \omega^A = \alpha^A + \omega^A \times \omega^A$$

but $\omega^R = -\omega^A$ by definition of body D hence $\frac{d}{dt} \omega^A = \alpha^A$

thus $\frac{d}{dt} \mathbb{I}^{R/D*} = \mathbb{I}^{C/C*, P/A} + \omega^A \times (\mathbb{I}^{C/C*, R/A})$ as required

11/11/11



$$A \frac{\alpha/\alpha^*}{H} = I \frac{\alpha/\alpha^*}{\omega} \cdot A \gamma + II \frac{\beta/\beta^*}{\omega} \gamma \beta$$

$$R \quad C/C^+ \quad R \quad P/D^+ \quad A \quad B/C^+$$

Since $A \begin{array}{|c|} \hline B/C^H \\ \hline \end{array} = A \begin{array}{|c|} \hline B/B^* \\ \hline \end{array} + A \begin{array}{|c|} \hline B^*/C^H \\ \hline \end{array}$

$m_B IR^{B^*} \times W^{B^*}$

$$R \text{ II } C^+ = R \text{ II } D^+ + \begin{matrix} A & B/B^+ \\ | & | \\ H & H \end{matrix} + \begin{matrix} B/B^+, AB \\ | \\ H \end{matrix}$$

$$R \text{---} C/C^* = \text{II}^{C/C^*, R, A} + \text{II}^{B/B^*, A, B}$$

if $R = \mathbb{C}$

$${}^C \text{II} \text{---} {}^C/C^* = 0 = \text{II}^{C/C^*, CA} + \text{II}^{B/B^*, AB}$$

Given ${}^B \omega^A = \omega_1 n_1 + \omega_2 n_2 + \omega_3 n_3$
 need to find ${}^B \omega^C$

Now $I^{c/c^*} = (\beta_1 + \gamma_1)m_1n_1 + (\beta_2 + \gamma_2)m_2n_2 + (\beta_3 + \gamma_3)m_3n_3$

$$C_w^A = \Omega_1 \ln_1 + \Omega_2 \ln_2 + \Omega_3 \ln_3$$

$$C \quad B \quad B \quad A \quad C$$

$$(w) \quad + \quad (w) \quad = \quad (w)$$

$$\text{II}^{\text{B/B}^*} = (\beta_1) m_1 n_1 + \beta_2 m_2 n_2 + \beta_3 m_3 n_3$$

$$A \otimes B = (\omega_1 n_1 + \omega_2 n_2 + \omega_3 n_3)$$

hence,

$$(\beta_1 + \gamma_1) \Omega_1 m_1 + (\beta_2 + \gamma_2) \Omega_2 m_2 + (\beta_3 + \gamma_3) \Omega_3 m_3 + \beta_1 \omega_1 m_1 + \beta_2 \omega_2 m_2 + \beta_3 \omega_3 m_3 \stackrel{\text{new}}{=} 0$$

thus $(\beta_i + \gamma_i) \Omega_i + \beta_i \omega_i = 0$ thus $\Omega_i = \frac{-\beta_i \omega_i}{(\beta_i + \gamma_i)}$ hence $\Omega_i = \frac{-\beta_i \omega_i}{\beta_i + \gamma_i}$ i not

$$R \quad \alpha/\alpha^* \quad R \quad \delta/\delta^* \quad \gamma \quad \beta/\alpha^*$$

$$A \text{ || } H \text{ || } \alpha/\alpha^* = A \text{ || } H \text{ || } \delta/\delta^* + \gamma \text{ || } H \text{ || } \beta/\beta^*$$

$$= \text{II}^{\alpha/\alpha^*} \cdot A \cdot \gamma + \text{II}^{\beta/\beta^*} \cdot \gamma \cdot \beta$$

now if $R=A$ & when θ 's are zero

$$\gamma \parallel \beta / \alpha^* = \gamma \parallel \beta / \beta^* + \gamma \parallel \beta^* / \alpha^*$$

$$x_{IH}^{B^*/C^*} = m_B \cdot x_{IR}^{B^*/A^*} \cdot x_{IV}^{B^*/C^*}$$

since $IR^{\beta/\alpha} = 0$

$$\gamma + \beta = \alpha$$

δ is a fictitious body w/ distib of α but instantaneous motion of δ

now $\frac{d}{dt} \alpha^{\mu} = 0$ since at the instant the momentum of α wrt α^{μ} in Reference frame $A = 0$

now $\gamma \omega \beta = -\omega \gamma = -[\omega_1 n_1 + \omega_2 n_2 + \omega_3 n_3]$

at that instant $\Pi^{P/\rho^*} = \beta_1 m_1 m_1 + \beta_2 m_2 m_2 + \beta_3 m_3 m_3$

$$\prod \alpha^{\gamma} = (\beta_1 + \gamma_1) m_1 m_1 + (\beta_2 + \gamma_2) m_2 m_2 + (\beta_3 + \gamma_3) m_3 m_3 \quad \text{since } \prod \alpha^{\gamma} = \prod \beta^{\gamma} = \prod \beta^{\gamma} + \prod \gamma^{\gamma}$$

and let ${}^A \omega^\gamma = -\gamma \omega^A = -(\gamma \omega^\beta + \omega^\beta A) = -\gamma \omega^\beta - \omega^\beta A$
 $= (\beta \omega^\gamma - \beta \omega^A)$

$$\Omega_1 m_1 + \Omega_2 m_2 + \Omega_3 m_3 = {}^A \omega^\gamma = \omega_1 m_1 + \omega_2 m_2 + \omega_3 m_3 - [\eta_1 m_1 + \eta_2 m_2 + \eta_3 m_3]$$

$$\text{II } \omega^\alpha \cdot {}^A \omega^\gamma + \text{II } \beta/\beta^\alpha \cdot \gamma \omega^\beta$$

$$\gamma \omega^\beta = -\omega_1 m_1 - \omega_2 m_2 - \omega_3 m_3$$

$$(\gamma_1 + \beta_1) \Omega_1 m_1 + (\gamma_2 + \beta_2) \Omega_2 m_2 + (\gamma_3 + \beta_3) \Omega_3 m_3 - \beta_1 \omega_1 m_1 - \beta_2 \omega_2 m_2 - \beta_3 \omega_3 m_3$$

$$(\gamma_1 + \beta_1) [\omega_1 - \eta_1] m_1 + (\gamma_2 + \beta_2) (\omega_2 - \eta_2) m_2 + (\gamma_3 + \beta_3) (\omega_3 - \eta_3) m_3 - \beta_1 \omega_1 m_1 - \beta_2 \omega_2 m_2 - \beta_3 \omega_3 m_3 = 0$$

$$[\omega_1 \gamma_1 - (\gamma_1 + \beta_1) \eta_1] m_1 + [\omega_2 \gamma_2 - (\gamma_2 + \beta_2) \eta_2] m_2 + [\omega_3 \gamma_3 - (\gamma_3 + \beta_3) \eta_3] m_3 = 0$$

$$\Rightarrow \frac{\gamma_i \omega_i}{\gamma_i + \beta_i} = \eta_i \quad \therefore \beta \omega^A = \eta_i m_i$$

$${}^A \omega^\alpha \cdot {}^A \beta/\beta^\alpha \cdot \gamma \omega^\gamma + \omega^\beta \cdot \omega^\alpha$$

$$\beta \omega^A = \frac{\gamma_1 \omega_1}{\gamma_1 + \beta_1} m_1 + \frac{\gamma_2 \omega_2}{\gamma_2 + \beta_2} m_2 + \frac{\gamma_3 \omega_3}{\gamma_3 + \beta_3} m_3$$

$${}^A \text{II} \omega^\alpha = (\text{II } \beta/\beta^\alpha + \text{II } \gamma/\gamma^\alpha) \cdot ({}^A \omega^\beta + \omega^\beta) = \text{II } \omega^\alpha \cdot {}^A \omega^\gamma + \text{II } \beta/\beta^\alpha \cdot \gamma \omega^\beta$$

$${}^R \mathbb{H} \dot{C}/C^* = {}^R \mathbb{H} \dot{D}/D^* + {}^A \mathbb{H} \dot{B}/B^* \quad \text{where } C \text{ makes up the rotor } B \text{ and the rigid body } A$$

$$(1) \quad D \text{ is a fictitious rigid body whose mass distrib is equal to } C \text{ but whose motion at any instant is that of } A.$$

in 7(c)
$${}^R \mathbb{H} \dot{S}/S^* = {}^R \mathbb{H} \dot{S}^*/S^* + {}^R \mathbb{H} \dot{S}^*/S^*$$
 was proven where ${}^R \mathbb{H} \dot{S}^*/S^* = m_S \dot{R} S^*/S^* \times {}^R W S^*$

$$m_S = \sum_{i=1}^n m_i$$

Thus
$${}^A \mathbb{H} \dot{B}/B^* = {}^A \mathbb{H} \dot{B}/B^* + {}^A \mathbb{H} \dot{B}^*/B^*$$
 where ${}^A \mathbb{H} \dot{B}^*/B^* = m_B \dot{R} B^*/B^* \times {}^A V B^*$

now since the axis of symmetry of B is fixed in A and B^* lies on the axis of symmetry and is also fixed in A then ${}^A V B^* \equiv 0$

Thus
$${}^A \mathbb{H} \dot{B}/B^* = {}^A \mathbb{H} \dot{B}/B^* \quad (2)$$

Placing (2) in (1) leads to

$${}^R \mathbb{H} \dot{C}/C^* = {}^R \mathbb{H} \dot{D}/D^* + {}^A \mathbb{H} \dot{B}/B^* \quad (3)$$

but by problem 8(b)
$${}^R \mathbb{T}^* C = - \frac{d}{dt} {}^R \mathbb{H} \dot{C}/C^*$$

Thus we need to differentiate (3) wrt time in reference frame R .

thus
$$\frac{d}{dt} {}^R \mathbb{H} \dot{C}/C^* = - {}^R \mathbb{T}^* C = \frac{d}{dt} {}^R \mathbb{H} \dot{D}/D^* + \frac{d}{dt} {}^A \mathbb{H} \dot{B}/B^* \quad (4)$$

but
$$\frac{d}{dt} {}^A \mathbb{H} \dot{B}/B^* = \frac{d}{dt} {}^A \mathbb{H} \dot{B}/B^* + {}^R \omega^A \times {}^A \mathbb{H} \dot{B}/B^* \quad (5)$$

$$= - {}^A \mathbb{T}^* B + {}^R \omega^A \times ({}^A \mathbb{H} \dot{B}/B^* \cdot {}^A \omega^B) \quad (6)$$

problem 7(i)

$$\frac{d}{dt} {}^R \mathbb{H} \dot{B}/B^* = {}^A \mathbb{H} \dot{B}/B^* \cdot {}^A \omega^B + {}^R \omega^A \times ({}^A \mathbb{H} \dot{B}/B^* \cdot {}^A \omega^B) + {}^R \omega^A \times ({}^A \mathbb{H} \dot{B}/B^* \cdot {}^A \omega^B) \quad (7)$$

(5,6) (3,43)

Now
$$\frac{d}{dt} ({}^R \mathbb{H} \dot{D}/D^*) = {}^A \mathbb{H} \dot{D}/D^* \cdot {}^A \omega^D + {}^R \omega^D \times ({}^A \mathbb{H} \dot{D}/D^* \cdot {}^A \omega^D) = - {}^R \mathbb{T}^* D \quad (8)$$

3.43

but since D represents a fictitious rigid body whose mass distribution is that of C but whose motion at any instant is that of A , then

$${}^R \omega^D = {}^R \omega^A, \quad {}^R \omega^D = {}^R \omega^A \text{ and } {}^A \mathbb{H} \dot{D}/D^* = {}^A \mathbb{H} \dot{C}/C^* \quad (9, 10, 11)$$

Placing these into (8) give

$${}^R \mathbb{H} \dot{D}/D^* = {}^A \mathbb{H} \dot{C}/C^* \cdot {}^R \omega^A + {}^R \omega^A \times ({}^A \mathbb{H} \dot{C}/C^* \cdot {}^R \omega^A) \quad (12)$$

Using (12), (7) into (4)

$${}^R \mathbb{T}^* C = {}^A \mathbb{T}^* B + {}^R \mathbb{T}^* D + ({}^A \mathbb{H} \dot{B}/B^* \cdot {}^A \omega^B) \times {}^R \omega^A$$

$$= ({}^A \mathbb{H} \dot{C}/C^* \cdot {}^R \omega^A) \times {}^R \omega^A + {}^A \mathbb{H} \dot{C}/C^* \cdot {}^R \omega^A + ({}^A \mathbb{H} \dot{B}/B^* \cdot {}^A \omega^B) \times {}^R \omega^B - {}^A \mathbb{H} \dot{B}/B^* \cdot {}^A \omega^B + ({}^A \mathbb{H} \dot{B}/B^* \cdot {}^A \omega^B) \times {}^R \omega^B$$

We can actually show that

$$\frac{d}{dt} \mathbb{I}^{R/C} = \frac{d}{dt} \sum m_i \mathbf{r}_i^{P/C} \times \omega^A \times \mathbf{r}_i^{P/C} \\ = \sum m_i \frac{d}{dt} \mathbf{r}_i^{P/C} \times \omega^A \times \mathbf{r}_i^{P/C} + \mathbb{I}^{C/A} \frac{d\omega^A}{dt} + \sum m_i \mathbf{r}_i \times \omega^A \times \frac{d}{dt} \mathbf{r}_i$$

and that this becomes

$$\sum m_i \left\{ (\dot{\mathbf{r}}_i \cdot \mathbf{r}_i) \omega^A - (\dot{\mathbf{r}}_i \cdot \omega) \mathbf{r}_i + (\mathbf{r}_i \cdot \dot{\mathbf{r}}_i) \omega - (\mathbf{r}_i \cdot \omega) \dot{\mathbf{r}}_i \right\} \mathbb{I}^{C/A}$$

$$\sum m_i \left\{ 2(\dot{\mathbf{r}}_i \cdot \mathbf{r}_i) \omega^A - \omega \cdot (\dot{\mathbf{r}}_i \mathbf{r}_i + \mathbf{r}_i \dot{\mathbf{r}}_i) \right\}$$

now $\frac{d}{dt} \mathbf{r}_i = \dot{\mathbf{r}}_i = \frac{d}{dt} \mathbf{r}_i + \omega^C \times \mathbf{r}_i$ but $\mathbf{r}_i = \mathbf{r}_i^{P/C}$ & are fixed in C thus

$$\dot{\mathbf{r}}_i \cdot \mathbf{r}_i = (\omega^C \times \mathbf{r}_i) \cdot \mathbf{r}_i = (\mathbf{r}_i \times \mathbf{r}_i) \cdot \omega^C = 0$$

thus we have $-\sum m_i \omega^A \cdot \frac{d}{dt} (\mathbf{r}_i \mathbf{r}_i) = + \omega^A \cdot \frac{d}{dt} \sum m_i (\mathbf{r}_i^2 U - \mathbf{r}_i \mathbf{r}_i)$

$$\omega^A \cdot \frac{d}{dt} \mathbb{I}^{C/A} =$$

Numerical Solution of Initial Value Problems in Dynamics

1. Introduction

The initial value problem in dynamics consists of determining certain quantities related to the motion of a system for time $t > t_1$ when the values of these quantities for time t_1 are known. Frequently, the solution of this problem involves the numerical integration of a set of ordinary differential equations. For example, Fig. 1 shows a system composed of a rigid body B and a particle P that moves in a tube T and is attached to B with a spring S and a dashpot D . For $t = 0$, the orientation of B , the angular velocity of

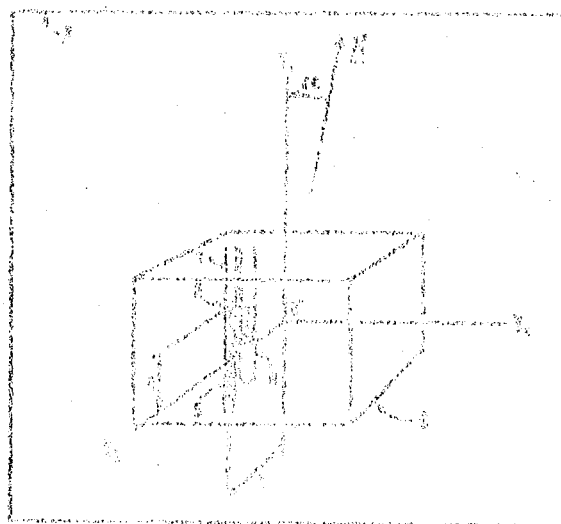


Fig. 1. Schematic representation of a system.

B , the position of P in T , and the velocity of P in T are known, and it is desired to find α and $\dot{\alpha}$ for $t > 0$. Here α is the angle between T_1 and

of the central principal axes of inertia of B , and an inertially fixed vector, namely $\underline{H}$, the central angular momentum of the system, and q is the distance from P to γ_2 , another central principal axis of inertia of B . It is the purpose of what follows to discuss the numerical solution of such problems.

The sequel is arranged as follows: Sec. 2 deals with the construction of a computational algorithm. In Sec. 3, a computer program based on the algorithm is discussed, and Sec. 4 contains a sample listing of the program.

2. Construction of an algorithm

Use of principles of mechanics leads to equations called dynamical equations (often differential equations). These govern certain scalar variables and involve certain system parameters. For example, use of $\underline{F}_x + \underline{F}_x^* = 0$ (see DYNAMICS, p.177) in connection with the system mentioned in Sec. 1 leads to four differential equations governing u_1, u_2, u_3 , and u_4 , the first three of which are the $\underline{b}_1, \underline{b}_2, \underline{b}_3$ measure numbers of the angular velocity of B , where $\underline{b}_1, \underline{b}_2, \underline{b}_3$ are unit vectors parallel to $\gamma_1, \gamma_2, \gamma_3$, while u_4 is the time-rate of change of q , the distance between P and γ_2 ; the four differential equations involve the system parameters $b, \sigma, \delta, m_P, m_B, I_1, I_2, I_3$ which characterize S, D, P , and B . They are

$$\begin{aligned} \ddot{u}_1 = & \left\{ - \left(\frac{m_P m_B}{m_P + m_B} \right) q^2 + I_2^2 \right\} \left[\frac{m_P m_B}{m_P + m_B} b (u_1 u_3 q + u_2 u_3 b \right. \\ & - 2u_2 u_4) - u_2 u_3 (I_2 - I_3) \left. \right] + \frac{m_P m_B}{m_P + m_B} b q \left[\frac{m_P m_B}{m_P + m_B} q (u_1 u_3 q \right. \\ & + u_2 u_3 b - 2u_2 u_4) + u_3 u_1 (I_3 - I_1) \left. \right] \left\{ \frac{m_P m_B}{m_P + m_B} (q^2 I_1 \right. \\ & \left. + b^2 I_2) + I_1 I_2 \right\}^{-1} \end{aligned} \quad (1)$$

$$\begin{aligned} \dot{u}_2 = & \left\{ - \frac{m_P m_B}{m_P + m_B} b q \left(\frac{m_P m_B}{m_P + m_B} b (u_1 u_3 q + u_2 u_3 b - 2 u_2 u_4) \right. \right. \\ & - u_2 u_3 (I_2 - I_3) + \left. \left(\frac{m_P m_B}{m_P + m_B} b^2 + I_1 \right) \left(\frac{m_P m_B}{m_P + m_B} q (u_1 u_3 q \right. \right. \\ & + u_2 u_3 b - 2 u_2 u_4) + u_3 u_1 (I_2 - I_1) \left. \left. \right) \right\} \left(\frac{m_P m_B}{m_P + m_B} (q^2 I_1 \right. \\ & + b^2 I_2) + I_1 I_2 \left. \right)^{-1} \end{aligned} \quad (2)$$

$$\begin{aligned} \dot{u}_3 = & - (q q + \delta u_4) b + \frac{m_P m_B}{m_P + m_B} q (2 u_3 u_4 - u_3^2 b - u_1^2 b + u_1 u_2 q) \\ & - u_1 u_2 (I_1 - I_2) \left(\frac{m_P m_B}{m_P + m_B} q^2 + I_3 \right)^{-1} \end{aligned} \quad (3)$$

$$\dot{u}_4 = - (q q + \delta u_4) \left(\frac{m_P m_B}{m_P + m_B} \right)^{-1} + \dot{u}_3 b - u_2 (u_1 b - u_2 q) + u_3^2 q \quad (4)$$

Kinematical considerations furnish additional equations, called kinematical equations, some of which may be differential equations. For example,

$$\dot{q} = u_4 \quad (5)$$

in such an equation and, after introducing Euler parameters $\epsilon_1, \dots, \epsilon_4$, one can write, in addition,

$$\epsilon_1 = \frac{1}{2} (u_1 \epsilon_4 - u_2 \epsilon_3 + u_3 \epsilon_2) \quad (5)$$

$$\epsilon_2 = \frac{1}{2} (u_1 \epsilon_3 + u_2 \epsilon_4 - u_3 \epsilon_1) \quad (7)$$

$$\epsilon_3 = \frac{1}{2} (-u_1 \epsilon_2 + u_2 \epsilon_1 + u_3 \epsilon_4) \quad (8)$$

$$\epsilon_4 = \frac{1}{2} (-u_1 \epsilon_1 - u_2 \epsilon_2 - u_3 \epsilon_3) \quad (9)$$

Furthermore, after defining c and s as

$$c \triangleq 1 - 2\epsilon_2^2 - 2\epsilon_3^2 \quad (10)$$

and

$$s \triangleq (1 - c^2)^{1/2} \quad (11)$$

one can express a as

$$a = \tan^{-1}(s/c) \quad (12)$$

Simultaneous solution of (1)-(9) and subsequent use of (10)-(12) leads to the desired values of q and a . However, a program based on the equations as written would be inefficient because it would involve certain repetitions in calculations. For example, since the expression $m_P m_B / (m_P + m_B)$ appears repeatedly, it is desirable to introduce a symbol to replace this expression, i.e., to let

$$\mu \triangleq m_P m_B / (m_P + m_B) \quad (13)$$

Similarly, one may introduce $z_1, \dots, z_9$ as

$$z_1 = u_1 u_3 q + u_2 u_3 b - 2 u_2 u_4 \quad (14)$$

$$z_2 = u_2 u_3 (I_2 - I_3) \quad (15)$$

$$z_3 = u_3 u_1 (I_3 - I_1) \quad (16)$$

$$z_4 = \mu b q \quad (17)$$

$$z_5 = [\mu(q^2 I_1 + b^2 I_2) + I_1 I_2]^{-1} \quad (18)$$

$$z_6 = \mu b z_1 - z_2 \quad (19)$$

$$z_7 = \mu q z_1 + z_3 \quad (20)$$

$$z_8 = \sigma q + \delta u_4 \quad (21)$$

$$z_9 = 2u_3 u_4 - u_3^2 b - u_1^2 b + u_1 u_2 q \quad (22)$$

after which one can replace (1) - (4) with

$$\dot{u}_1 = [-(\mu q^2 + I_2) z_6 + z_4 z_7] z_5 \quad (23)$$

$$\dot{u}_2 = [-z_4 z_6 + (\mu b^2 + I_1) z_7] z_5 \quad (24)$$

$$\dot{u}_3 = -[z_8 b + \mu q z_9 - u_1 u_2 (I_1 - I_2)] (\mu q^2 + I_3)^{-1} \quad (25)$$

$$\dot{u}_6 = -z_8/\mu + u_3 b - u_2(u_1 b - u_2 q) + u_3^2 q \quad (26)$$

3. Computer program

If a set of ordinary differential equations can be expressed as

$$\frac{dy_i}{dx} = F_i(z_1, \dots, z_m; y_1, \dots, y_n, x) \quad (i=1, \dots, n)$$

where z_j ($j = 1, \dots, m$) is a function of $y_1, \dots, y_n$ and y_k ($k=1, \dots, n$) is a function of x , then the FORTRAN subroutine DFEQKM can be used to integrate the equations. (This subroutine is based on the KUTTA MERTON method as described in Fox, L., Numerical Solution of Ordinary and Partial Differential Equations, 1962, p. 24.) For example, (5)-(9) and (23)-(26) form such a set with

$$y_1 = \begin{cases} u_1 & i = 1, \dots, 4 \\ q & i = 5 \\ \epsilon_{1-5} & i = 6, \dots, 9 \end{cases} \quad (27)$$

and

$$F_1 = [-(\mu y_5^2 + I_2)z_6 + z_4 z_7]z_5 \quad (28)$$

To use DFEQKM, one creates a FORTRAN program like the one in Sec. 4. An explanation of this program and a sample output can be obtained from LOTS by giving the TOPS-20 command

```
@ PRINT PS:(K.KANE)DOC..1
```

Execution of the program is initiated by giving the command

```
@ EXEC PS:(K. KANE)DYNEX.FOR, NAL:NALIB/LIB
```

After execution, printed output is obtained by giving the command

```
@ PRINT FOR20.DAT
```

To create a program intended for the solution of an initial value problem other than the one involving (5)-(9) and (23)-(26), one can modify DYNEX.FOR by giving the commands

```
@ EDIT PS:(K.KANE)DYNEX.FOR
```

and then using editing commands to change or delete symbols underlined in the listing in Sec. 4.

FORTRAN program

THE NAME OF THIS FILE IS DYNEX.FOR

EXAMPLE PROBLEM

IMPLICIT DOUBLE PRECISION (A-Z)

INTEGER J, NCUTS, NEO, NSTEPS

LOGICAL STPSZ

DIMENSION Y(9) — 4

EXTERNAL SENS

COMMON/DFOLST/X, STEP, RELERR, ARBERR, NCUTS, NEO, STPSZ

COMMON/PLIST/MP, MB, I1, I2, I3, SIG, DEL, D, MI

DATA YES, NO, 'YES', 'NO'

PLIST1/ CEE, BCE, BEE, CO, CI, B, C

PLIST2/ B1, B2, B3, B3C1, B2C1, B1C1

PLIST3/ TI3, TI2, TI1, AJI3, AJI2, AJI1

WRITE(20, 600)

WRITE(20, 601)

TYPE 602

ACCEPT *, MP, MB, AMU, RM, T/JI

TYPE 603

ACCEPT *, ~~I1, I2, I3~~ ALP, BET

~~IF(I1+I2.LT.I3) TYPE 618~~

~~IF(I1+I2.LT.I3) GO TO 110~~

~~IF(I2+I3.LT.I1) TYPE 619~~

~~IF(I2+I3.LT.I1) GO TO 110~~

~~IF(I3+I1.LT.I2) TYPE 620~~

~~IF(I3+I1.LT.I2) GO TO 110~~

TYPE 604

ACCEPT *, SIG, DEL, D

B, C, D, ALI

TYPE 605

ACCEPT *, W1+U2, U3 W1, W2, W3, AWB

IF (D.GT. AMIN(B,C)) TYPE 6041

IF (D.GT. AMIN(B,C)) WRITE(20, 6041)

IF (D.GT. AMIN(B,C)) D=AMIN(B,C)

TYPE 606

ACCEPT *, Q, QDOT

TYPE 607

ACCEPT *, EPS1, EPS2, EPS3

~~EPS4=QDOT*(1.0-EPS1-EPS2-EPS3-EPS4)~~

TYPE 608

ACCEPT *, TINITL, TFINAL, STEP

TYPE 609

TYPE 609, ~~MP, MB, I1, I2, I3, SIG, DEL, D~~ AMU, RM, T/JI, ALP, BET, B, C, D, ALI

WRITE(20, 609) ~~MP, MB, I1, I2, I3, SIG, DEL, D~~ AMU, RM, T/JI, ALP, BET, B, C, D, ALI

TYPE 610, ~~W1, W2, W3, Q, QDOT, EPS1, EPS2, EPS3, EPS4~~ TINITL, TFINAL, STEP

WRITE(20, 610) ~~W1, W2, W3, Q, QDOT, EPS1, EPS2, EPS3, EPS4~~ TINITL,

TFINAL, STEP

PI=4.0*DATAN(1.000)

DEG=180.0/PI

NSTEPS=IDINT((TFINAL-TINITL)/STEP+0.1)+1

NEO=X4

RELERR=1.00-4

ARBERR=1.00-4

NCUTS=20

STPSZ=.FALSE.

X=TINITL


```

05500 Y(1)=U1 W1
05600 Y(2)=U2 W2
05700 Y(3)=U3 W3
05800 Y(4)=QDOT ANP
05900 Y(5)=Q
06000 Y(6)=EPS1
06100 Y(7)=EPS2
06200 Y(8)=EPS3
06300 Y(9)=EPS4
06400 NU=MP+MB*(MP+MB)
06500 TYPE 611
06600 WRITE(20,611)
06700
06800
06900 DO 200 J=1,NSTEPS
07000 C=1 C=2 C=(Y(7)+Y(7)+Y(8)+Y(8))
07100 S=DEGR7(1,0-C+C)
07200 A=DEG+DATAN2(S,C)
07300 TYPE 612,X,Y(1),Y(2),Y(3),Y(4)
07400 WRITE(20,612) X,Y(1),Y(2),Y(3),Y(4)
07500 IF(J.EQ.NSTEPS) GO TO 200
07600 CALL DFEGM(EQNS,Y,815)
07700 CONTINUE
07800
07900
08000
08100
08200
08300
08400
08500 15 TYPE 615,NCUTS,X
08600 WRITE(20,615) NCUTS,X
08700
08800
08900
09000
09100
09200
09300
09400
09500
09600
09700
09800
09900
10000
10100
10200
10300
10400
10500
10600
10700
10800
10900
11000

```

put in ** my notes

↑
↓

```

C 6041 FORMAT(1X,' ROTOR DIAM EXCEEDS MIN. BODY LENGTH - WILL MAKE ROTOR DIAM = MIN BODY LENGTH ')
600 FORMAT(1X,' OUTPUT FROM PROGRAM DYNEX. FOR ')
601 FORMAT(1X,' (THIS IS FILE FOR20.DAT) ')
602 FORMAT(1X,' INPUT VALUES OF MP,MB ROTOR/BODY MASS RATIO, RMASS, RTORQUE/J ')
603 FORMAT(1X,' INPUT VALUES OF H1,H2,H3 ROTOR AXIS ANGLES ALPHA, BETA (DEG) ')
604 FORMAT(1X,' INPUT VALUES OF SIG,BEL,B BODY LENGTH, ROTOR DIA. RATIOS - B,C,D AND LONGEST LI (M) ')
605 FORMAT(1X,' INPUT INITIAL VALUES OF U1,U2,U3 (RAD/SEC) ')
606 FORMAT(1X,' INPUT INITIAL VALUES OF G (M), QDOT (M/SEC) ')
607 FORMAT(1X,' INPUT INITIAL VALUES OF EPSILON1, EPSILON2, EPSILON3 ')
608 FORMAT(1X,' INPUT TINITL, TFINAL, STEP (SEC) ')
609 FORMAT(1X,' SYSTEM PARAMETERS ')
1 2X, 'RM= ',F9.3, ' '
2 2X, 'G= ',F9.3, ' '
3 T/1 2X, 'T/1 = ',F9.3, ' ' RAD/SEC**2 '
4 ALP 2X, 'ALP = ',F9.3, ' ' DEG/
5 BET 2X, 'BET = ',F9.3, ' ' DEG/
6 B 1X, 'SIG = ',F9.3, ' ' N=SEC/M ' 1X, 'C = ',F9.3, ' 1X, 'D = ',F9.3, '
7 LL 1X, 'DEL = ',F9.3, ' ' N=SEC/M ' M/ '
8 1X, 'B = ',F9.3, ' '
610 FORMAT(1X,' INITIAL CONDITIONS ')
1 7X, 'U1 = ',1PD13.5, ' RAD/SEC '
2 7X, 'U2 = ',1PD13.5, ' RAD/SEC '
3 7X, 'U3 = ',1PD13.5, ' RAD/SEC '

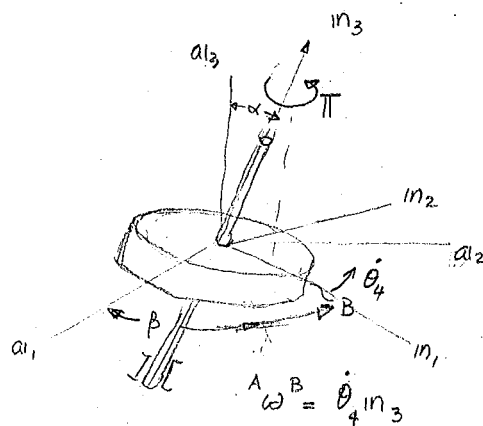
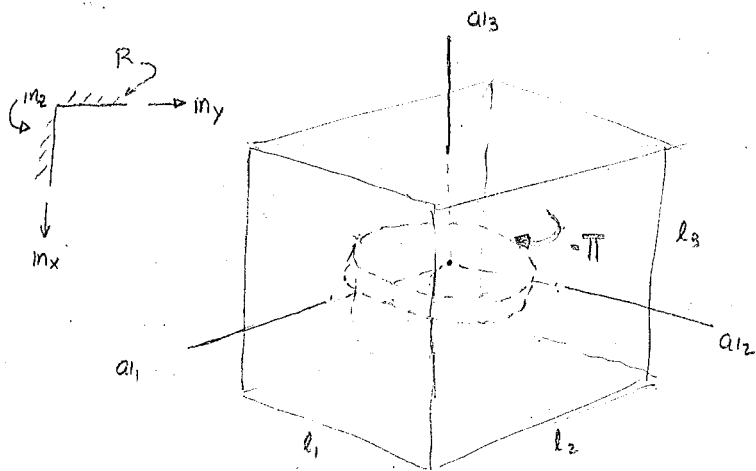
```

'AND'


```

11100 5 5X, 'GDDT = ', IPD13.5, ' M/SEC'//
11200 6 1X, 'EPSILON1 = ', IPD13.5//
11300 7 1X, 'EPSILON2 = ', IPD13.5//
11400 8 1X, 'EPSILON3 = ', IPD13.5//
11500 9 1X, 'EPSILON4 = ', IPD13.5//
11600 1 1X, 'TINITL = ', OPF9.3, ' SEC'//
11700 2 1X, 'TFINAL = ', OPF9.3, ' SEC'//
11800 3 3X, 'STEP = ', F9.3, ' SEC'///)
11900 611 FORMAT(1X, 'SIMULATION RESULTS'//X = TIME IN (SEC), .OMEGA IN (RAD/SEC)'//
12000 1 1X, 'T (SEC)'//7X, 'G (M)'//5X, 'ALPHA (DEG)'// 6X, 'T', 12X, 'W1', 12X, 'W2',
12100 612 FORMAT(1X, F7.3, 4X, F10.7, 3X, F10.7) 1X, 5(F10.7, 4X)) 12X, 'W3', 12X, 'AWB'//
12200 613 FORMAT(//1X, 'IF YOU WANT TO RUN ANOTHER CASE TYPE YES.'//
12300 1 2X, 'ELSE TYPE NO.'//
12400 614 FORMAT(A3)
12500 615 FORMAT(1X, 'STEP SIZE HAS BEEN HALVED THE ALLOTTED '.12, ' TIMES.
12600 1 1X, 'SIMULATION TERMINATED AT X = ', IPD13.5//
12700 2 3X, 'AND DFECKM IS UNABLE TO PROCEED.'//)
12800 616 FORMAT(//1X, '*** PRINTED OUTPUT IS ON FILE FOR20.DAT'//)
12900 617 FORMAT(//40(' ')//)
13000 618 FORMAT(1X, 'I1 + I2 < I3. THIS IS NOT PHYSICALLY POSSIBLE.'//)
13100 619 FORMAT(1X, 'I2 + I3 < I1. THIS IS NOT PHYSICALLY POSSIBLE.'//)
13200 620 FORMAT(1X, 'I3 + I1 < I2. THIS IS NOT PHYSICALLY POSSIBLE.'//)
13300 621
13400 C 621 FORMAT (5X, 'I1', 12X, 'I2', 12X, 'I3', 12X, 'I4', 10X, 'TORQUE')
13500 C END
13600 C
13700 SUBROUTINE EONS(X,Y,DY)
13800 IMPLICIT DOUBLE PRECISION (A-Z)
13900 DIMENSION Y(9), DY(9)
14000 COMMON/PLIST/MP, MB, I1, I2, I3, SIG, DEL, B, MU
14100 C PLIST1/ CEE, BCE, BEC, CO, CI, B, C
14200 PLIST2/ B1, B2, B3, B3C1, B2C1, B1C1
14300 PLIST3/ TI3, T12, T11, AJI3, AJI2, AJI1
14400 Z1=Y(1)*Y(3)*Y(5)+Y(2)*Y(3)*B-2.0*Y(2)*Y(4)
14500 Z2=Y(2)*Y(3)*(I2-I3)
14600 Z3=Y(3)*Y(1)*(I3-I1)
14700 Z4=MU*B*Y(5)
14800 Z5=1.0/(MU*(Y(5)+Y(5)+I1*B*B*I2)+I1*I2)
14900 Z6=MU*B*Z1-Z2
15000 Z7=MU*Y(5)*Z1+Z3
15100 Z8=SIG*Y(3)+DEL*Y(4)
15200 Z9=2.0*Y(2)*Y(4)-(Y(3)*Y(3)+Y(1)*Y(1))*B*Y(1)*Y(2)*Y(5)
15300 C
15400 DY(1)=(-MU*Y(5)*Y(3)+I2)*Z6+Z4*Z7)*Z5
15500 DY(2)=(-Z4*Z6+(MU*B*B+I1)*Z7)*Z5
15600 DY(3)=-((Z8*B*MU+Y(5)*Z9-Y(1)*Y(2)*(I1-I2))/(MU*Y(5)+Y(5)+I3)
15700 DY(4)=-Z8/MU+DY(3)*B-Y(2)*(Y(1)*B-Y(2)*Y(5))+7/3*Y(3)*Y(5)
15800 DY(5)=Y(4)
15900 DY(6)=0.5*(Y(1)*Y(9)-Y(2)*Y(8)+Y(3)*Y(7))
16000 DY(7)=0.5*(Y(1)*Y(9)+Y(2)*Y(8)-Y(3)*Y(7))
16100 DY(8)=0.5*(-Y(1)*Y(7)+Y(2)*Y(6)+Y(3)*Y(9))
16200 DY(9)=0.5*(-Y(1)*Y(6)-Y(2)*Y(7)-Y(3)*Y(9))
16300 C
16400 RETURN
16500 END

```

Let x, y, z be the measure nos to the center of mass of the body A. Thus $r^A = x m_x + y m_y + z m_z$

$$V^A = \dot{x} m_x + \dot{y} m_y + \dot{z} m_z$$

$$\Pi^A = -T m_3$$

let a_1, a_2, a_3 be the central axes of the body A fixed in A

let m_x, m_y, m_z be a set of axes fixed in reference frame R

thus we can define a set of intermediate reference planes

A_1, A_2 in which we can define simple angular motions

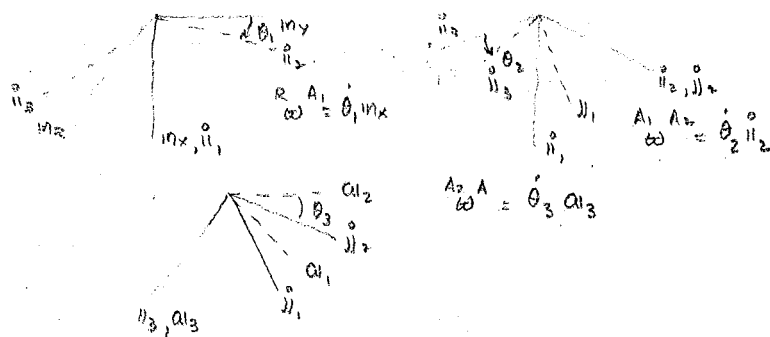
$$\text{so that } {}^R \omega^A = \dot{\theta}_1 {}^R a_1 + \dot{\theta}_2 {}^R a_2 + \dot{\theta}_3 {}^R a_3 = \dot{\theta}_1 m_x + \dot{\theta}_2 m_y + \dot{\theta}_3 m_z$$

$$m_3 = a_3 \cos \alpha + a_1 \sin \alpha \cos \beta + a_2 \sin \alpha \sin \beta$$

$$\Pi^B = T m_3$$

$$\Pi^{B/B^*} = J_1 m_3 m_3 + J_2 (m_1 m_1 + m_2 m_2)$$

let us also assume that the rotor is made of the same material as the body A and that its center of mass lies at the same point of body A, i.e. that a cut out is made in body A of same size as body B in which we may place body B.



We will assume a parallel gravitational field. let body A have mass $(M-m)$ and let body B have mass m .

$$\text{thus } F_r = V_{gr}^A \cdot IF^A + \omega_{gr}^A \cdot \Pi^A + V_{gr}^B \cdot IF^B + \omega_{gr}^B \cdot \Pi^B$$

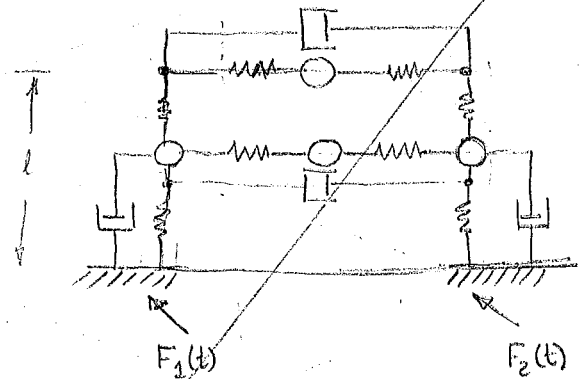
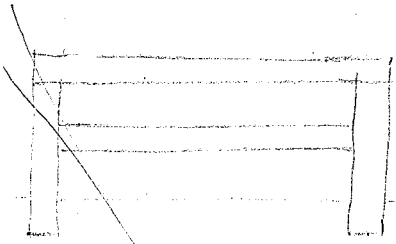
$$F_r^* = V_{gr}^{A^*} \cdot IF^{A^*} + \omega_{gr}^{A^*} \cdot \Pi^{A^*} + V_{gr}^{B^*} \cdot IF^{B^*} + \omega_{gr}^{B^*} \cdot \Pi^{B^*}$$

$$\text{where } IF^{A^*} = -M a_1^*$$

let C be the composite of A & B.

let D

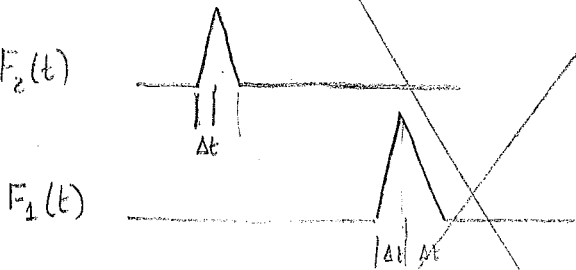
$${}^A \omega^B = {}^A \omega^B \beta = {}^A \omega^B \beta_i a_i$$



A representation of a building by means of springs and masses ^{& dampers} to which forces at the bases are ~~imposed~~ ^{applied} (representation of an earthquake wave passing the building & setting the building in motion). The attempt here is to find what values

of the spring constants for the lower & upper floors necessary to minimize the motion. What I'm attempting to do is model a "shake" table with intent on finding the "building code" for this model.

$$\sum_{i=1}^5 \delta(t) f_i(t-t_i) = F_2(t)$$



This problem is quite practical in nature especially for an area such as California. The forces are modelled as semi δ -functions which can be in fact periodic, yet of decreasing amplitude, to model reflections of P & S waves whose energy is dissipated as their travel paths through the earth's crust increase. $\left[\begin{array}{l} d(t) = e^{-\alpha t} \\ \text{(dissipation fr)} \end{array} \right]$

- My questions are: ⁽¹⁾ should I model the floors by adding a light inextensible string if I want not only longitudinal but also transverse motion of the floors? As a first approximation I guess I can (to simplify model somewhat). ⁽²⁾ Is model too complicated for the purposes you intended?
- This is my project. This seems more like a linear vibrations problem than like an initial value problem to be solved numerically. I don't think it is suitable for our purposes.

0 if A^*

$$F_r = \sum_{i=1}^{N_1+N_2} W_{q_r}^{P_i} \cdot IF^i = \sum_{i=1}^{N_1} W_{q_r}^{P_i} \cdot IF^i + \sum_{i=1}^{N_2} W_{q_r}^{P_i} \cdot IF^i$$

for body A $W_{q_r}^{P_i} = W_{q_r}^{A^*} + (\omega_{q_r}^A \times P_i) \cdot P_i$

for body B $\Rightarrow W_{q_r}^{P_i} = W_{q_r}^{B^*} + \omega_{q_r}^B \times P_i$

$$\begin{aligned} W_{q_r}^{A^*} \cdot IF^i &= \sum IF^i = (W \cdot w) \cdot m_y \\ W_{q_r}^{A^*} \cdot (M \cdot w) \cdot m_y &= (M \cdot w) \cdot m_y \\ (\omega_{q_r}^A \times P_i) \cdot IF^i &= (P_i \times IF^i) \cdot \omega_{q_r}^A = \omega_{q_r}^A \cdot \Pi \end{aligned}$$

$$\begin{aligned} W_{q_r}^{A^*} \cdot \sum m_i g m_y + \sum m_i g m_y + \sum m_i g m_y + \sum m_i g m_y &= \sum m_i g m_y + \sum m_i g m_y + \sum m_i g m_y + \sum m_i g m_y \\ W_{q_r}^{A^*} \cdot m g m_y + \sum (P_i \times IF^i) \cdot \omega_{q_r}^A &= \sum m_i g m_y + \sum m_i g m_y + \sum m_i g m_y + \sum m_i g m_y \end{aligned}$$

$$\begin{aligned} \sum_{i=1}^{N_1+N_2} W_{q_r}^{P_i} \cdot IF_i^* &\Rightarrow \sum_{i=1}^{N_1} W_{q_r}^{P_i} \cdot IF_i^* \quad \text{where } IF_i^* = -m_i a_i \\ &= -m_i [a_i^* + \omega_{q_r}^A \times P_i + \omega_{q_r}^A \times (\omega_{q_r}^A \times P_i)] \\ (W_{q_r}^{A^*} + \omega_{q_r}^A \times P_i) \cdot IF_i^* &= W_{q_r}^{A^*} \cdot IF_i^* + (P_i \times IF_i^*) \cdot \omega_{q_r}^A \\ &= W_{q_r}^{A^*} \cdot [(M \cdot w)] a_i^* + \omega_{q_r}^A \cdot \Pi^A \quad \text{where } \Pi^A = -\sum m_i P_i \times a_i^* \\ &= [(\Pi^{A/A^*} \cdot \omega_{q_r}^A) \times \omega_{q_r}^A - \Pi^{A/A^*} \cdot \omega_{q_r}^A] \end{aligned}$$

$$-W_{q_r}^{A^*} \cdot [(M \cdot w)] a_i^* + \omega_{q_r}^A \cdot \Pi^A \quad \text{where } \Pi^A = -\sum m_i P_i \times a_i^*$$

$$\sum_{i=1}^{N_2} W_{q_r}^{P_i} \cdot IF_i^* = -W_{q_r}^{B^*} \cdot [w a_i^*] + \omega_{q_r}^B \cdot [(\Pi^{B/B^*} \cdot \omega_{q_r}^B) \times \omega_{q_r}^B - \Pi^{B/B^*} \cdot \omega_{q_r}^B]$$

Thus we get $-W_{q_r}^{A^*} \cdot M a_i^* + \omega_{q_r}^A \cdot \Pi^A + \omega_{q_r}^B \cdot \Pi^B$

but look at $\omega_{q_r}^A \cdot (\Pi^A + \Pi^B) + \omega_{q_r}^B \cdot \Pi^B$

$$= \frac{d}{dt} \Pi^{A/A^*} + \frac{d}{dt} \Pi^{B/B^*} = -\frac{d}{dt} (\Pi^{A/A^*} + \Pi^{B/B^*}) = \Pi^A + \Pi^B$$

but $\Pi^{A/A^*} = \Pi^{A/A^*} \cdot \omega_{q_r}^A$; $\Pi^{B/B^*} = \Pi^{B/B^*} \cdot \omega_{q_r}^B = \Pi^{B/B^*} \cdot \omega_{q_r}^A + \Pi^{B/B^*} \cdot \omega_{q_r}^B$

$$\Pi^{A/A^*} + \Pi^{B/B^*} = [\Pi^{C/C^*} \cdot \omega_{q_r}^A] + \Pi^{B/B^*} \cdot \omega_{q_r}^B$$

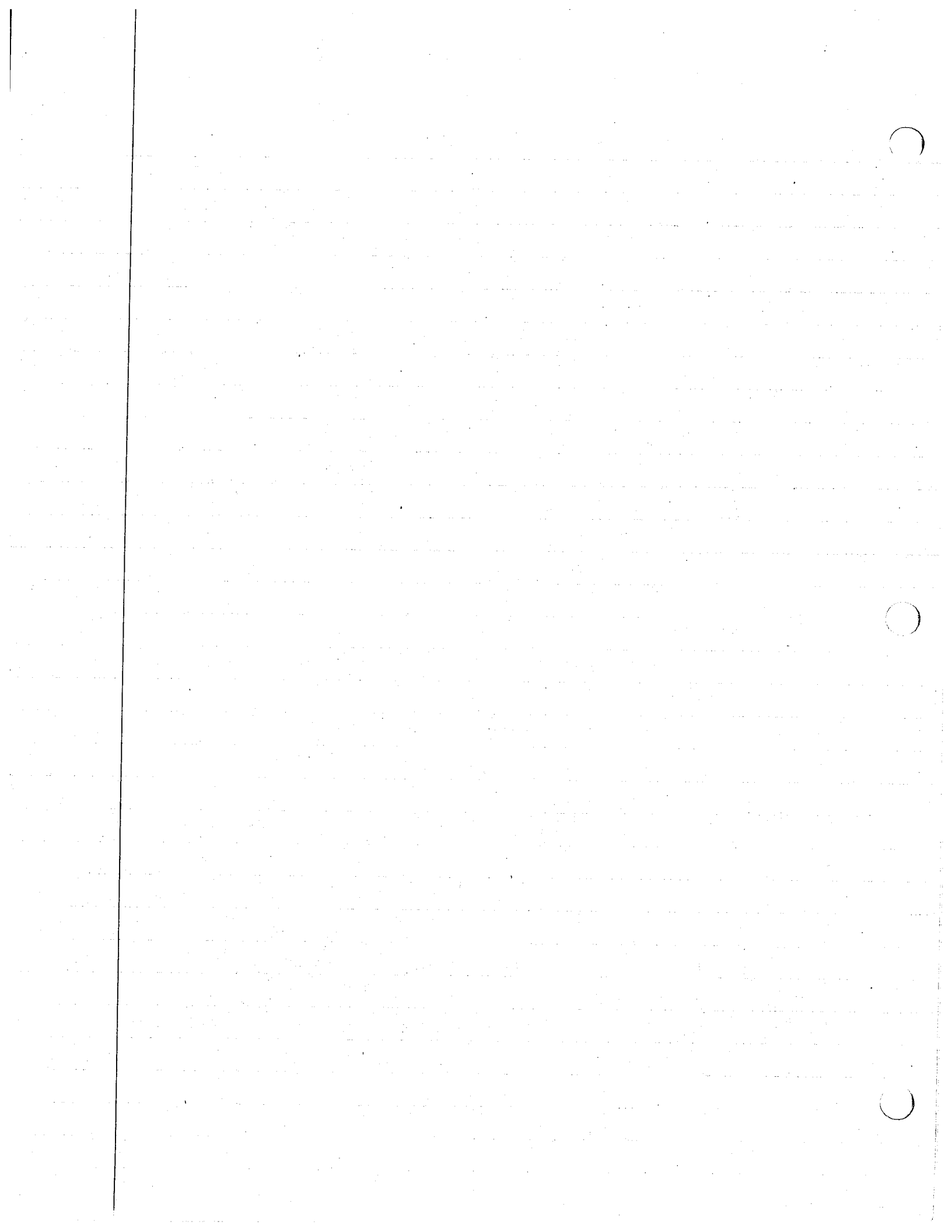
Now define a fictitious rigid body D whose mass distribution is that of C but whose motion is that of A. Thus $\Pi^{A/A^*} = \Pi^{D/D^*} + \Pi^{A/D^*}$

$$\frac{d}{dt} (\Pi^{A/A^*} + \Pi^{B/B^*}) = \frac{d}{dt} \Pi^{D/D^*} + \frac{d}{dt} \Pi^{A/D^*} = \frac{d}{dt} \Pi^{D/D^*} + \frac{d}{dt} \Pi^{A/D^*} + \omega_{q_r}^A \times \Pi^{A/D^*}$$

$$= [\Pi^A + \Pi^B] = \frac{d}{dt} \Pi^{D/D^*} + \frac{d}{dt} \Pi^{A/D^*} + \omega_{q_r}^A \times (\Pi^{A/D^*} \cdot \omega_{q_r}^A)$$

but $\frac{d}{dt} \Pi^{D/D^*} = \Pi^{C/C^*} \cdot \omega_{q_r}^A + \omega_{q_r}^A \times \Pi^{C/C^*} \cdot \omega_{q_r}^A$

thus $\Pi^A + \Pi^B = (\Pi^{C/C^*} \cdot \omega_{q_r}^A) \times \omega_{q_r}^A - \Pi^{C/C^*} \cdot \omega_{q_r}^A + \Pi^B + \Pi^{A/D^*} \times \omega_{q_r}^A$



$\pi^{1/2}$

$$\text{Thus } F_r^* = -W \dot{q}_r^A \cdot \frac{W a_1^A}{g} + \dot{\omega}_{\dot{q}_r}^A \cdot \frac{R^B}{g} + \dot{\omega}_{\dot{q}_r}^A \cdot \left[\left(\frac{R^A}{g} \cdot \frac{R^B}{g} \right) \times \omega^A - \frac{R^A}{g} \cdot \frac{R^B}{g} + \frac{R^A}{g} \cdot \frac{R^B}{g} \right]$$

$$\text{Now back to } F_r = W \dot{q}_r^A \cdot \frac{R^A}{g} + W \dot{q}_r^B \cdot \frac{R^B}{g} + \omega_{\dot{q}_r}^A \cdot \frac{R^A}{g} + \omega_{\dot{q}_r}^B \cdot \frac{R^B}{g} = W \dot{q}_r^A \cdot \frac{R^A}{g} + \omega_{\dot{q}_r}^A \cdot \frac{R^A}{g}$$

$$R \omega_x^A = \dot{m}_x, \quad R \omega_y^A = \dot{m}_y, \quad R \omega_z^A = \dot{m}_z, \quad R \omega_{\theta_1}^A = \dot{m}_x, \quad R \omega_{\theta_2}^A = \dot{m}_y, \quad R \omega_{\theta_3}^A = \dot{m}_z$$

$$\pi^A = -\dot{m}_3, \quad \pi^B = \dot{m}_3$$

$$R \omega_x^A = \dot{m}_x = \dot{\omega}_{\theta_1}^A = \dot{\omega}_{\theta_2}^A = \dot{\omega}_{\theta_3}^A = 0, \quad R \omega_{\theta_1}^A = \dot{m}_x, \quad R \omega_{\theta_2}^A = \dot{m}_y, \quad R \omega_{\theta_3}^A = \dot{m}_z$$

$$R \omega^B = \dot{\omega}^A + \dot{\omega}^B = \dot{\omega}^A + \dot{\theta}_4 m_3 = \dot{\theta}_1 m_x + \dot{\theta}_2 m_y + \dot{\theta}_3 m_z + \dot{\theta}_4 m_3$$

$$R \omega_x^B = \dot{\omega}_x^B = \dot{\omega}_y^B = \dot{\omega}_z^B = 0, \quad R \omega_{\theta_1}^B = \dot{m}_3, \quad R \omega_{\theta_2}^B = \dot{m}_y, \quad R \omega_{\theta_3}^B = \dot{m}_z$$

	m_x	m_y	m_z	$\dot{m}_1$	$\dot{m}_2$	$\dot{m}_3$	$\dot{m}_1$	$\dot{m}_2$	$\dot{m}_3$	$\dot{m}_1$	$\dot{m}_2$	$\dot{m}_3$
$\dot{m}_1$	1	0	0	$\dot{m}_1$	$\dot{m}_2$	$\dot{m}_3$	$\dot{m}_1$	$\dot{m}_2$	$\dot{m}_3$	$\dot{m}_1$	$\dot{m}_2$	$\dot{m}_3$
$\dot{m}_2$	0	$\dot{m}_1$	$\dot{m}_2$	$\dot{m}_1$	$\dot{m}_2$	$\dot{m}_3$	$\dot{m}_1$	$\dot{m}_2$	$\dot{m}_3$	$\dot{m}_1$	$\dot{m}_2$	$\dot{m}_3$
$\dot{m}_3$	0	$\dot{m}_1$	$\dot{m}_2$	$\dot{m}_1$	$\dot{m}_2$	$\dot{m}_3$	$\dot{m}_1$	$\dot{m}_2$	$\dot{m}_3$	$\dot{m}_1$	$\dot{m}_2$	$\dot{m}_3$

$$q_1 = x$$

$$F_1 = W_1 + 0 + 0 = W$$

$$q_2 = y$$

$$F_2 = 0 + 0 + 0 = 0$$

$$q_3 = z$$

$$F_3 = 0 + 0 + 0 = 0$$

$$q_4 = \theta_1$$

$$F_4 = 0 + -T m_x m_3 + T m_x m_3 = T (c_2 (c_3 a_1 - s_3 a_2) + s_2 a_3) = T (c_1 \dot{m}_2 - s_1 (-s_2 \dot{m}_1 + c_2 \dot{m}_3)) = T (c_1 \dot{m}_2 + s_1 s_2 \dot{m}_1 - s_1 c_2 \dot{m}_3)$$

$$q_5 = \theta_2$$

$$F_5 = 0 + -T \dot{m}_2 m_3 + T \dot{m}_2 m_3 = T (s_3 a_1 + c_3 a_2) \cdot m_3 = T (s_3 s_\alpha c_\beta + c_3 s_\alpha s_\beta)$$

$$q_6 = \theta_3$$

$$F_6 = 0 + -T a_3 m_3 + T a_3 m_3 = T \cos \alpha = 0$$

$$q_7 = \theta_4$$

$$F_7 = 0 + 0 + T m_3 m_3 = T$$

$$F_4 = T (c_1 \dot{m}_2 + s_1 s_2 \dot{m}_1 - s_1 c_2 \dot{m}_3) = T (c_1 [s_3 a_1 + c_3 a_2] + s_1 s_2 [c_3 a_1 - s_3 a_2] - s_1 c_2 a_3) \cdot m_3$$

$$= T (c_1 s_3 s_\alpha c_\beta + c_1 c_3 s_\alpha s_\beta + s_1 s_2 c_3 s_\alpha c_\beta - s_1 s_2 s_3 s_\alpha s_\beta - s_1 c_2 c_\alpha)$$

$$a_1^* = \ddot{x} m_x + \ddot{y} m_y + \ddot{z} m_z$$

$$a_1^B = \dot{\theta}_4 m_3 \Rightarrow a_1^B = a_2^B = a_3^B = a_4^B = a_5^B = a_6^B = a_7^B = 0, \quad a_8^B = \dot{\theta}_4 m_3$$

$$R \omega_{\theta_1}^A = c_2 (c_3 a_1 - s_3 a_2) + s_2 a_3, \quad R \omega_{\theta_2}^A = s_3 a_1 + c_3 a_2, \quad R \omega_{\theta_3}^A = a_3$$

$$\pi^{1/2} = I_1 a_1 a_1 + I_2 a_2 a_2 + I_3 a_3 a_3$$

$$R \omega^A = \dot{\theta}_1 (c_2 (c_3 a_1 - s_3 a_2) + s_2 a_3) + \dot{\theta}_2 (s_3 a_1 + c_3 a_2) + \dot{\theta}_3 a_3$$

$$= (\dot{\theta}_1 c_2 c_3 + \dot{\theta}_2 s_3) a_1 + (-\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 c_3) a_2 + (\dot{\theta}_1 s_2 + \dot{\theta}_3) a_3$$

$$= (\dot{\theta}_1 c_2 c_3 + \dot{\theta}_2 s_3) [c_\beta c_\alpha m_1 - s_\beta m_2 + s_\alpha c_\beta m_3] + (-\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 c_3) [s_\beta c_\alpha m_1 + c_\beta m_2 + s_\alpha s_\beta m_3]$$

$$+ (\dot{\theta}_1 s_2 + \dot{\theta}_3) [-s_\alpha m_1 + c_\alpha m_3]$$

$$= [(\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 s_3) c_\beta c_\alpha + (-\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 c_3) s_\beta c_\alpha - (\dot{\theta}_1 s_2 + \dot{\theta}_3) s_\alpha] m_1 + [(\dot{\theta}_1 c_2 c_3 + \dot{\theta}_2 s_3) s_\beta + (-\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 c_3) c_\beta] m_2 + [(\dot{\theta}_1 s_2 + \dot{\theta}_3) c_\alpha + (\dot{\theta}_1 c_2 c_3 + \dot{\theta}_2 s_3) s_\alpha] m_3$$

$$F_r^* = -N \dot{q}_r^{A^*} \cdot \frac{W}{g} a_i^* + \omega \dot{q}_r^{A^*} \cdot \Pi^{B^*} + \omega \dot{q}_r^{A^*} \cdot \left[(\Pi^{C^*} \cdot \omega^R) \times \omega^A - \Pi^{C^*} \cdot \alpha^A - \Pi^{B/B^*} \cdot \alpha^A + \Pi^{B/B^*} \cdot \alpha^A \right]$$

$$F_1^* = -\frac{W}{g} \ddot{x} + 0 + 0$$

$$F_2^* = -\frac{W}{g} \ddot{y} + 0 + 0$$

$$F_3^* = -\frac{W}{g} \ddot{z} + 0 + 0$$

$$F_4^* = 0 + 0 +$$

$$F_5^* = 0 + 0 +$$

$$F_6^* = 0 + 0 +$$

$$F_7^* = 0 + m_3 \cdot \Pi^{B^*} + 0$$

$$F_1 + F_1^* = -M\ddot{x} + Mg = 0 \quad \ddot{x} = g$$

$$F_2 + F_2^* = -M\ddot{y} = 0 \quad \ddot{y} = 0$$

$$F_3 + F_3^* = -M\ddot{z} = 0 \quad \ddot{z} = 0$$

$$F_7 + F_7^* = T + J_1(\dot{\omega}_3 + \ddot{\theta}_4) = 0$$

$$\Pi^{B/B^*} = \Pi^{B/B^*} \cdot \alpha^A = J_1 \ddot{\theta}_4 m_3$$

$$\text{Let } \omega^A = \tilde{\omega}_1 a_1 + \tilde{\omega}_2 a_2 + \tilde{\omega}_3 a_3 = \omega_1 m_1 + \omega_2 m_2 + \omega_3 m_3$$

$$\Pi^{B/B^*} \times \omega^A = J_1 \ddot{\theta}_4 (\omega_1 m_2 - \omega_2 m_1) = J_1 \ddot{\theta}_4 \omega_1 (-s_\beta a_1 + c_\beta a_2) - J_1 \ddot{\theta}_4 \omega_2 (c_\beta c_\alpha a_1 + s_\beta c_\alpha a_2 - s_\alpha a_3) =$$

$$R \alpha^A = \tilde{\omega}_1 a_1 + \tilde{\omega}_2 a_2 + \tilde{\omega}_3 a_3 = (\dot{\omega}_1 - \dot{\omega}_2 \dot{\theta}_4) m_1 + (\dot{\omega}_2 + \dot{\theta}_4 \omega_1) m_2 + \dot{\omega}_3 m_3 \quad x_1 a_1 + x_2 a_2 + x_3 a_3$$

$$\Pi^{C^*} \cdot R \alpha^A = I_1 \tilde{\omega}_1 a_1 + I_2 \tilde{\omega}_2 a_2 + I_3 \tilde{\omega}_3 a_3$$

$$(\Pi^{C^*} \cdot \omega^A) = I_1 \tilde{\omega}_1 a_1 + I_2 \tilde{\omega}_2 a_2 + I_3 \tilde{\omega}_3 a_3$$

$$\Pi^{B/B^*} = (\Pi^{C^*} \cdot \omega^A) \times \omega^A - \Pi^{C^*} \cdot R \alpha^A = [(I_2 - I_3) \tilde{\omega}_2 \tilde{\omega}_3 - I_1 \tilde{\omega}_1] a_1 + [(I_3 - I_1) \tilde{\omega}_3 \tilde{\omega}_1 - I_2 \tilde{\omega}_2] a_2 + [(I_1 - I_2) \tilde{\omega}_1 \tilde{\omega}_2 - I_3 \tilde{\omega}_3] a_3 = T_1^D a_1 + T_2^D a_2 + T_3^D a_3$$

$$A \alpha^B = \frac{d}{dt} (\ddot{\theta}_4 m_3) = \frac{d}{dt} \ddot{\theta}_4 m_3 + \omega^B \times (\omega^B) = \ddot{\theta}_4 m_3 \quad \therefore \Pi^{B/B^*} \cdot \alpha^B = J_1 \ddot{\theta}_4 m_3$$

$$\text{and } \Pi^{B/B^*} \cdot \alpha^B = J_1 \ddot{\theta}_4 c_\alpha a_1 + J_1 \ddot{\theta}_4 s_\alpha s_\beta a_2 + J_1 \ddot{\theta}_4 s_\alpha c_\beta a_3$$

$$R \omega^B = R \omega^A + A \omega^B = (\tilde{\omega}_1 + \dot{\theta}_4 s_\alpha c_\beta) a_1 + (\tilde{\omega}_2 + \dot{\theta}_4 s_\alpha s_\beta) a_2 + (\tilde{\omega}_3 + \dot{\theta}_4 c_\alpha) a_3 = x_1 a_1 + x_2 a_2 + x_3 a_3$$

$$= \omega_1 m_1 + \omega_2 m_2 + (\omega_3 + \dot{\theta}_4) m_3$$

$$R \alpha^B = \dot{\omega}_1 m_1 + \dot{\omega}_2 m_2 + (\dot{\omega}_3 + \ddot{\theta}_4) m_3 = \frac{d}{dt} \omega^B + \omega^B \times \omega^B$$

$$= (\tilde{\omega}_1 + \dot{\theta}_4 s_\alpha c_\beta) a_1 + (\tilde{\omega}_2 + \dot{\theta}_4 s_\alpha s_\beta) a_2 + (\tilde{\omega}_3 + \ddot{\theta}_4 c_\alpha) a_3 + \omega^A \times \omega^B$$

$$R \omega^A \times \omega^B = \omega^A \times \dot{\theta}_4 (s_\alpha c_\beta a_1 + s_\alpha s_\beta a_2 + c_\alpha a_3) = \dot{\theta}_4 [(\tilde{\omega}_2 c_\alpha - \tilde{\omega}_3 s_\alpha s_\beta) a_1 + (\tilde{\omega}_3 s_\alpha c_\beta - \tilde{\omega}_1 c_\alpha) a_2 + (\tilde{\omega}_1 s_\alpha s_\beta - \tilde{\omega}_2 s_\alpha c_\beta) a_3]$$

$$R \alpha^B = [\tilde{\omega}_1 + \dot{\theta}_4 s_\alpha c_\beta + \dot{\theta}_4 (\tilde{\omega}_2 c_\alpha - \tilde{\omega}_3 s_\alpha s_\beta)] a_1 + [\tilde{\omega}_2 + \dot{\theta}_4 s_\alpha s_\beta + \dot{\theta}_4 (\tilde{\omega}_3 s_\alpha c_\beta - \tilde{\omega}_1 c_\alpha)] a_2 + [\tilde{\omega}_3 + \ddot{\theta}_4 c_\alpha + \dot{\theta}_4 (\tilde{\omega}_1 s_\alpha s_\beta - \tilde{\omega}_2 s_\alpha c_\beta)] a_3$$

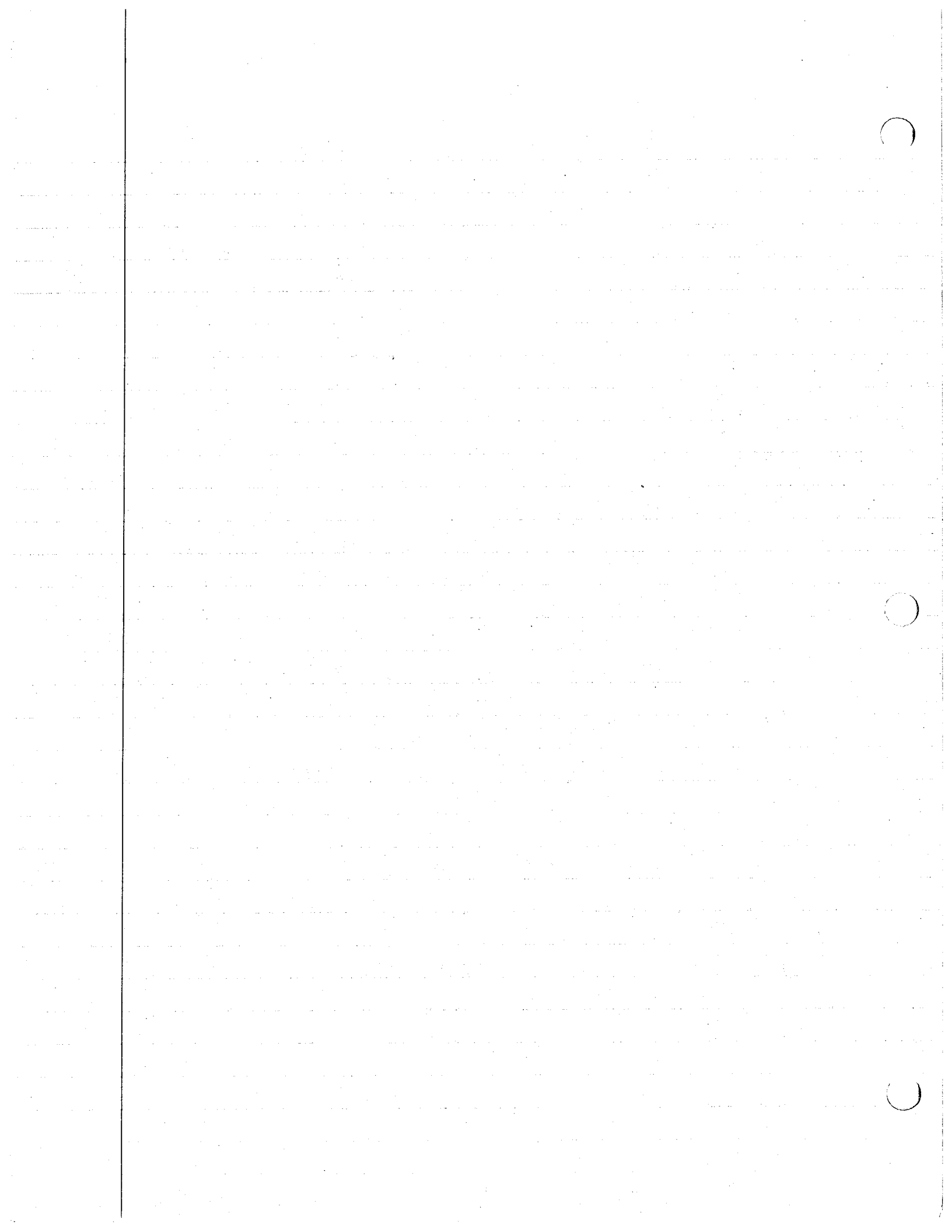
$$R \Pi^{B^*} = (\Pi^{B/B^*} \cdot R \omega^B) \times \omega^B - \Pi^{B/B^*} \cdot R \alpha^B$$

$$\Pi^{B/B^*} \cdot R \alpha^B = J_1 (\dot{\omega}_3 + \ddot{\theta}_4) m_3 + J_2 (\dot{\omega}_2 m_2 + \dot{\omega}_1 m_1)$$

$$(\Pi^{B/B^*} \cdot \omega^B) \times \omega^B = [J_1 (\omega_3 + \dot{\theta}_4) m_3 + J_2 (\omega_2 m_2 + \omega_1 m_1)] \times [\omega_1 m_1 + \omega_2 m_2 + (\omega_3 + \dot{\theta}_4) m_3]$$

$$= J_1 (\omega_3 + \dot{\theta}_4) (\omega_1 m_2 - \omega_2 m_1) + J_2 \omega_2 (-\omega_1 m_3 + (\omega_3 + \dot{\theta}_4) m_1) + J_2 \omega_1 (\omega_2 m_3 - (\omega_3 + \dot{\theta}_4) m_2)$$

$$R \Pi^{B^*} = [(J_2 - J_1) \omega_3 \omega_2 + \dot{\theta}_4 \omega_2 (J_2 - J_1) + J_2 \dot{\omega}_1] m_1 + [(J_1 - J_2) \omega_3 \omega_1 + \dot{\theta}_4 \omega_1 (J_1 - J_2) - J_2 \dot{\omega}_2] m_2 +$$



$$[J_2 \omega_1 \omega_2 - J_2 \dot{\omega}_1 \dot{\omega}_2 - J_2 (\dot{\omega}_3 + \ddot{\theta}_4)] m_3$$

$$\begin{matrix} A & B \\ \omega & \dot{\theta} \end{matrix} \begin{matrix} A & B \\ \omega & \dot{\theta} \end{matrix} \begin{matrix} A & B \\ \omega & \dot{\theta} \end{matrix} \begin{matrix} A & B \\ \omega & \dot{\theta} \end{matrix} \begin{matrix} A & B \\ \omega & \dot{\theta} \end{matrix} \begin{matrix} A & B \\ \omega & \dot{\theta} \end{matrix}$$

$$\begin{matrix} R & A \\ \omega & \dot{\theta} \end{matrix} \begin{matrix} R & A \\ \omega & \dot{\theta} \end{matrix} \begin{matrix} R & A \\ \omega & \dot{\theta} \end{matrix} \begin{matrix} R & A \\ \omega & \dot{\theta} \end{matrix} \begin{matrix} R & A \\ \omega & \dot{\theta} \end{matrix} \begin{matrix} R & A \\ \omega & \dot{\theta} \end{matrix}$$

Only contrib to 2nd term is F_7 term of $J_1(\dot{\omega}_3 + \ddot{\theta}_4) = 0$

$$m_x = c_2 c_3 a_{11} - c_2 s_3 a_{12} + s_2 a_{13}$$

$$I_{12} = s_3 a_{11} + c_3 a_{12}$$

$$\text{thus } m_x \cdot [\quad] = c_2 c_3 T_1^D - c_2 s_3 T_2^D + s_2 T_3^D - J_1 \ddot{\theta}_4 (c_2 c_3 s_\alpha c_\beta - c_2 s_3 s_\alpha s_\beta + s_2 c_\alpha) + c_2 c_3 \kappa_1 - c_2 s_3 \kappa_2 + s_2 \kappa_3$$

$$\text{thus } I_{12} \cdot [\quad] = s_3 T_1^D + c_3 T_2^D - J_1 \ddot{\theta}_4 (s_3 s_\alpha c_\beta + c_3 s_\alpha s_\beta) + s_3 \kappa_1 + c_3 \kappa_2$$

$$\text{thus } a_{13} \cdot [\quad] = T_3^D - J_1 \ddot{\theta}_4 c_\alpha + \kappa_3$$

$$F_4 + F_4^* = 0 + c_2 c_3 (T_1^D + \kappa_1) - c_2 s_3 (T_2^D + \kappa_2) + s_2 (T_3^D + \kappa_3) - J_1 \ddot{\theta}_4 (c_2 s_\alpha (c_3 c_\beta - s_3 s_\beta) + s_2 c_\alpha) = 0$$

$$F_5 + F_5^* = 0 + s_3 (T_1^D + \kappa_1) + c_3 (T_2^D + \kappa_2) - J_1 \ddot{\theta}_4 s_\alpha (s_3 c_\beta + c_3 s_\beta) = 0$$

$$F_6 + F_6^* = 0 + (T_3^D + \kappa_3) - J_1 \ddot{\theta}_4 c_\alpha = 0$$

$$F_7 + F_7^* = T^* J_1 (\dot{\omega}_3 + \ddot{\theta}_4) = 0$$

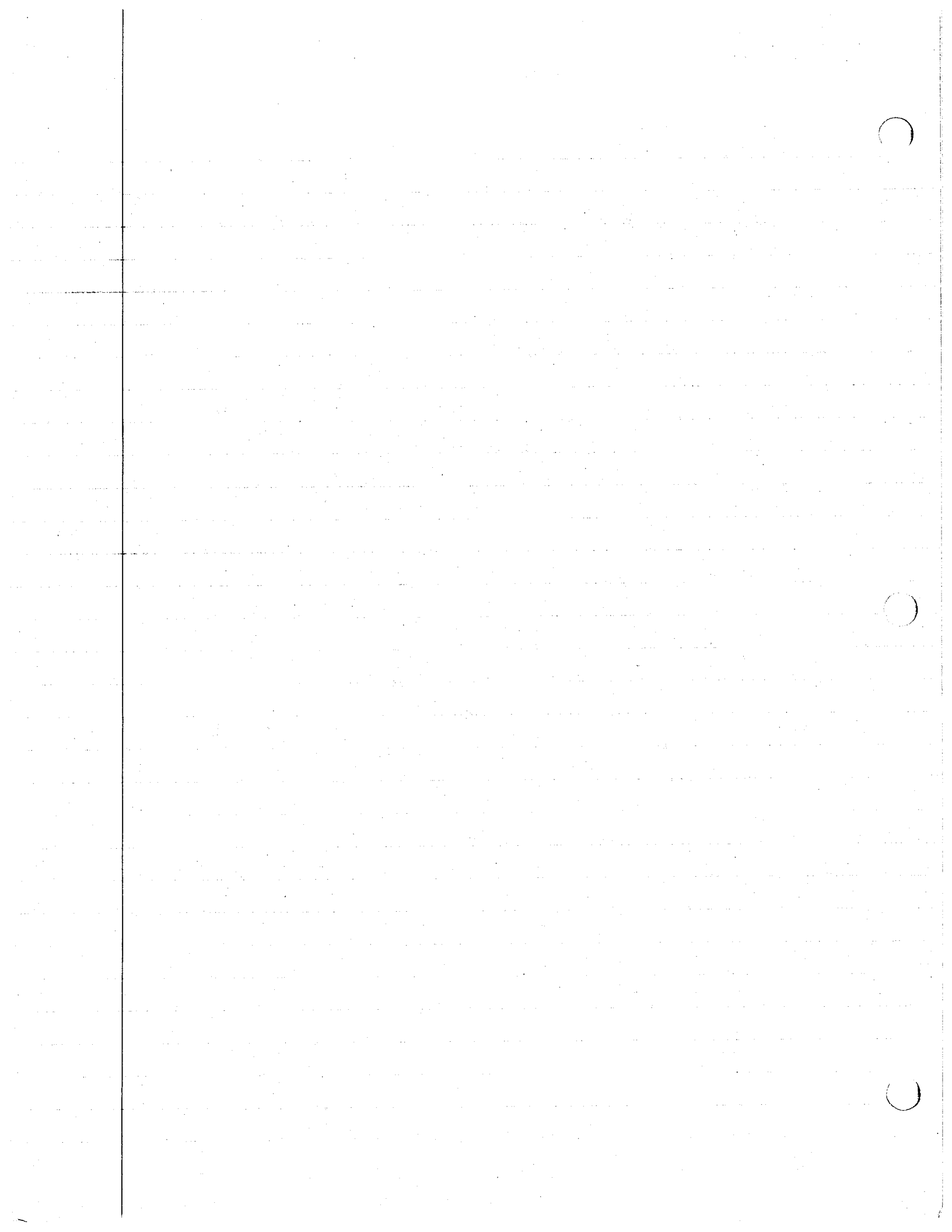
$$T_1^D = (I_2 - I_3) \ddot{\omega}_2 \ddot{\omega}_3 - I_1 \ddot{\omega}_1$$

$$T_2^D = (I_3 - I_1) \ddot{\omega}_3 \ddot{\omega}_1 - I_2 \ddot{\omega}_2$$

$$T_3^D = (I_1 - I_2) \ddot{\omega}_1 \ddot{\omega}_2 - I_3 \ddot{\omega}_3$$

$$\kappa_1 = -J_1 \ddot{\theta}_4 (\omega_1 s_\beta + \omega_2 c_\beta c_\alpha) \quad \kappa_2 = J_1 \ddot{\theta}_4 (\omega_1 c_\beta - \omega_2 s_\beta c_\alpha)$$

$$\kappa_3 = J_1 \ddot{\theta}_4 \omega_2 s_\alpha$$



$$\delta_1 s_\alpha s_\beta a_{13} + \delta_1 c_\alpha a_{12} - \delta_2 s_\alpha c_\beta a_{13} + \delta_2 c_\alpha a_{11} + \delta_3 s_\alpha c_\beta a_{12} - \delta_3 s_\alpha s_\beta a_{11}$$

$$(\tilde{\omega}_2 + \dot{\theta}_4 s_\alpha s_\beta) c_\alpha - (\tilde{\omega}_3 + \dot{\theta}_4 c_\alpha) s_\alpha s_\beta$$

$$(\tilde{\omega}_2 c_\alpha - \tilde{\omega}_3 s_\alpha c_\beta) a_{11}$$

$$(\tilde{\omega}_3 + \dot{\theta}_4 c_\alpha) s_\alpha c_\beta - (\tilde{\omega}_1 + \dot{\theta}_4 s_\alpha c_\beta) c_\alpha$$

$$\tilde{\omega}_3 s_\alpha c_\beta - \tilde{\omega}_1 c_\alpha$$

$$(\tilde{\omega}_1 + \dot{\theta}_4 s_\alpha c_\beta) s_\alpha s_\beta = (\tilde{\omega}_2$$

$$(\tilde{\omega}_1 + \dot{\theta}_4 s_\alpha c_\beta) s_\alpha s_\beta = (\tilde{\omega}_2 + \dot{\theta}_4 s_\alpha s_\beta) s_\alpha c_\beta$$

$$- J_1 (\omega_3 + \dot{\theta}_4) \omega_2 + J_2 \omega_2 (\omega_3 + \dot{\theta}_4) + J_2 \omega_1$$

$$J_1 (\omega_3 + \dot{\theta}_4) \omega_1 - J_2 \omega_1 (\omega_3 + \dot{\theta}_4) - J_2 \omega_2$$

$$- J_2 \omega_2 \omega_1 + J_2 \omega_1 \omega_2$$

$$c_2 c_3 (c_\alpha c_\beta m_1 - s_\beta m_2 + s_\alpha c_\beta m_3) - c_2 s_3 (s_\beta c_\alpha m_1 + c_\beta m_2 + s_\alpha s_\beta m_3) + s_2 (-s_\alpha m_1 + c_\alpha m_3)$$

$$m_x = (c_2 c_3 s_\alpha c_\beta - c_2 s_3 s_\alpha s_\beta + s_2 c_\alpha) m_3 + (c_2 c_3 c_\alpha c_\beta - c_2 s_3 s_\beta c_\alpha - s_2 s_\alpha) m_1 + (-c_2 c_3 s_\beta - c_2 s_3 c_\beta) m_2$$

$$p_z = s_3 (c_\beta c_\alpha m_1 - s_\beta m_2 + s_\alpha c_\beta m_3) + c_3 (s_\beta c_\alpha m_1 + c_\beta m_2 + s_\alpha s_\beta m_3) = (s_3 c_\beta c_\alpha + c_3 s_\beta c_\alpha) m_1 (s_3 s_\alpha c_\beta + c_3 s_\alpha s_\beta) m_3$$

$$- J_1 \dot{\theta}_4 (\omega_1 s_\beta + \omega_2 c_\beta c_\alpha) a_{11} + J_1 \dot{\theta}_4 (\omega_1 c_\beta - \omega_2 s_\beta c_\alpha) a_{12} + J_1 \dot{\theta}_4 \omega_2 s_\alpha a_{13} = \vec{H} \times \vec{p}$$

$$- J_1 \dot{\theta}_4 (\omega_1 s_\beta + \omega_2 c_\beta c_\alpha) c_2 c_3 + J_1 \dot{\theta}_4 (\omega_1 c_\beta - \omega_2 s_\beta c_\alpha) c_2 s_3 + J_1 \dot{\theta}_4 \omega_2 s_2 s_\alpha$$



now $T_3^D + K_3 = J_1 \ddot{\theta}_4 c_\alpha$ put into $F_4 + F_4^* = 0$

$$c_2 c_3 (T_1^D + K_1) - c_2 s_3 (T_2^D + K_2) + s_2 c_\alpha J_1 \ddot{\theta}_4 - \cancel{J_1 \ddot{\theta}_4 s_2 c_\alpha} - J_1 \ddot{\theta}_4 (c_2 s_\alpha (c_3 c_\beta - s_3 s_\beta)) = 0$$

$\Rightarrow$ if $c_2 \neq 0$

$$c_3^2 (T_1^D + K_1) - s_3^2 (T_2^D + K_2) - J_1 \ddot{\theta}_4 s_\alpha (c_3 c_\beta - s_3 s_\beta) = 0$$

$$s_3^2 (T_1^D + K_1) + c_3^2 (T_2^D + K_2) + J_1 \ddot{\theta}_4 s_\alpha (s_3 c_\beta + c_3 s_\beta) = 0$$

$$-(T_2^D + K_2) + J_1 \ddot{\theta}_4 s_\alpha [s_3 c_\beta + c_3^2 s_\beta - s_3 c_\beta + s_3^2 s_\beta] = 0$$

$$\boxed{(T_2^D + K_2) - J_1 \ddot{\theta}_4 s_\alpha s_\beta = 0}$$

$$(T_1^D + K_1) - J_1 \ddot{\theta}_4 s_\alpha (c_3^2 c_\beta - s_3^2 s_\beta + s_3^2 c_\beta + s_3^2 s_\beta)$$

$$\boxed{(T_1^D + K_1) - J_1 \ddot{\theta}_4 s_\alpha c_\beta = 0}$$

$$\boxed{(T_3^D + K_3) - J_1 \ddot{\theta}_4 c_\alpha = 0}$$

$$T = J_1 (\dot{\omega}_3 + \ddot{\theta}_4) = 0$$

$$(T_3^D + K_3) - J_1 \ddot{\theta}_4 c_\alpha = [(I_1 - I_2) \tilde{\omega}_1 \tilde{\omega}_2 - I_3 \dot{\tilde{\omega}}_3] + J_1 \dot{\theta}_4 \omega_2 s_\alpha - J_1 \ddot{\theta}_4 c_\alpha = 0$$

$$[-J_1 (\dot{\omega}^B \beta_3 - \dot{\omega}^B (\tilde{\omega}_2 \beta_1 - \tilde{\omega}_1 \beta_2))] = 0$$

is $\omega_2 s_\alpha = \beta_1 \tilde{\omega}_2 - \beta_2 \tilde{\omega}_1$ $\beta_1 = s_\alpha c_\beta$ $\beta_2 = s_\alpha s_\beta$

$$\tilde{\omega}_2 = -\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 c_3 \quad \tilde{\omega}_1 = \dot{\theta}_1 c_2 c_3 + \dot{\theta}_2 s_3 \quad \omega_2 = -(\dot{\theta}_1 c_2 c_3 + \dot{\theta}_2 s_3) s_\beta + (-\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 c_3) c_\beta$$

$$\tilde{\omega}_2 \beta_1 - \tilde{\omega}_1 \beta_2 = s_\alpha [(-\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 c_3) c_\beta - (\dot{\theta}_1 c_2 c_3 + \dot{\theta}_2 s_3) s_\beta] = \omega_2 s_\alpha$$

$$\rightarrow [(I_1 - I_2) \tilde{\omega}_1 \tilde{\omega}_2 - I_3 \dot{\tilde{\omega}}_3] - J_1 (\dot{\omega}^B \beta_3 - \dot{\omega}^B (\tilde{\omega}_2 \beta_1 - \tilde{\omega}_1 \beta_2)) = 0$$

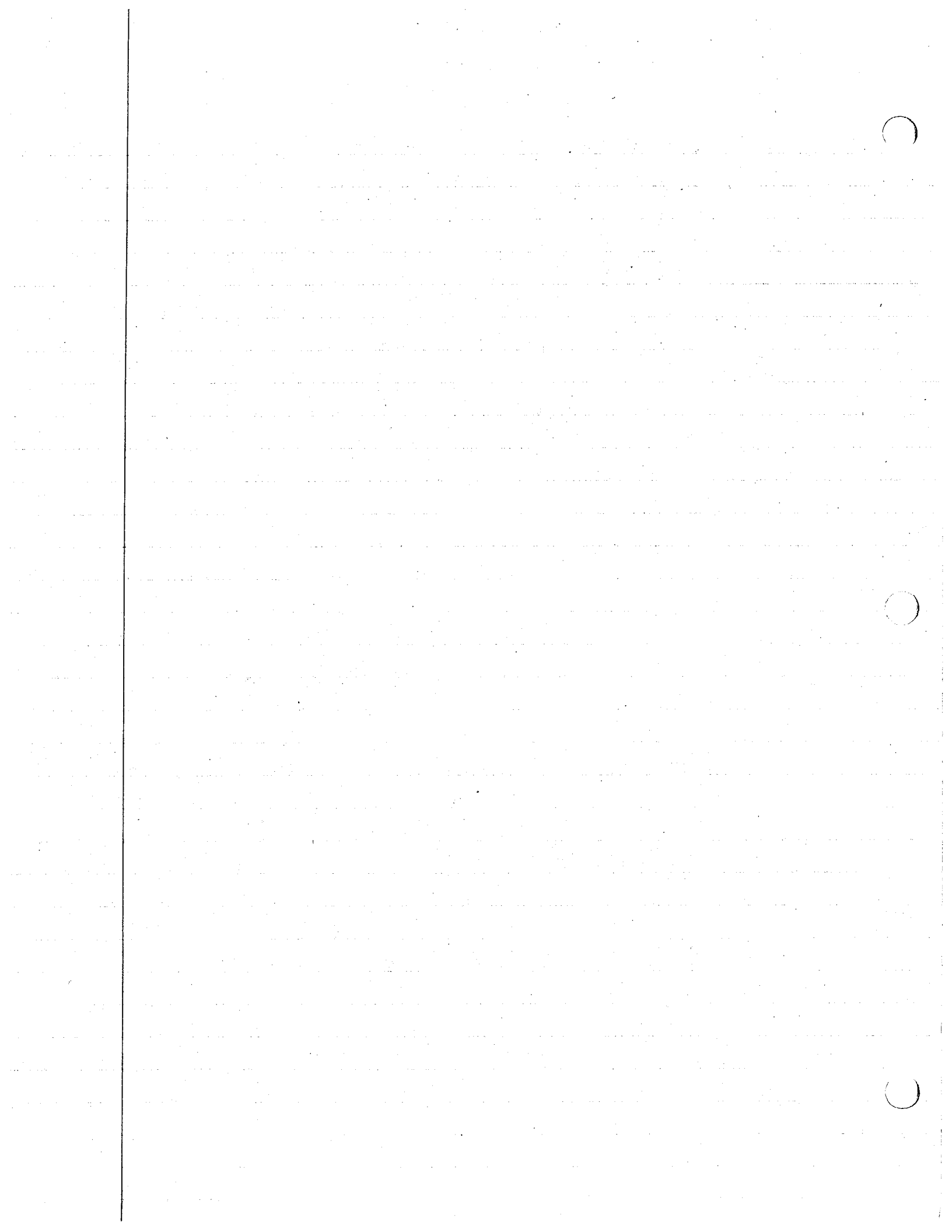
Now

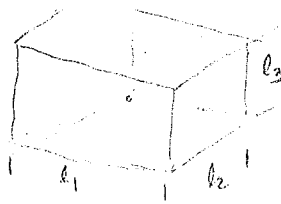
$$T_1^D + K_1 - J_1 \ddot{\theta}_4 s_\alpha c_\beta = [(I_2 - I_3) \tilde{\omega}_2 \tilde{\omega}_3 - I_1 \dot{\tilde{\omega}}_1] - J_1 \dot{\theta}_4 (\omega_1 s_\beta + \omega_2 c_\beta c_\alpha) - J_1 \ddot{\theta}_4 s_\alpha c_\beta = 0$$

$$= [(I_2 - I_3) \tilde{\omega}_2 \tilde{\omega}_3 - I_1 \dot{\tilde{\omega}}_1] - J_1 (\dot{\omega}^B \beta_1 + \dot{\omega}^B [\beta_2 \tilde{\omega}_3 - \beta_3 \tilde{\omega}_2]) = 0$$

is $-\omega_1 s_\beta - \omega_2 c_\beta c_\alpha = \beta_2 \tilde{\omega}_3 - \beta_3 \tilde{\omega}_2 = s_\alpha s_\beta [\dot{\theta}_1 s_2 + \dot{\theta}_3] - c_\alpha [-\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 c_3]$ ✓

$$-s_\beta [\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 s_3] c_\beta c_\alpha + (-\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 c_3) s_\beta c_\alpha - (\dot{\theta}_1 s_2 + \dot{\theta}_3) s_\alpha [-\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 c_3] s_\beta + (-\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 c_3) c_\alpha (-\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 c_3) + (\dot{\theta}_1 s_2 + \dot{\theta}_3) s_\alpha s_\beta \checkmark$$





$$\beta = (s \alpha c \beta, s \alpha s \beta, c \alpha) \quad J_1 = \frac{m R^2}{2} \quad I_1 = \frac{M}{12} (l_1^2 + l_3^2) \quad I_2 = \frac{M}{12} (l_3^2 + l_2^2)$$

$$I_3 = \frac{(l_1^2 + l_2^2) M}{12} \quad {}^A \dot{\omega}^B = \dot{\theta}_4 \quad {}^A \dot{\omega}^B = \dot{\theta}_4 ; \quad \text{let } l_2 = b l_1, \quad l_3 = c l_1, \quad 0 < b, c < 1$$

$$\begin{aligned} \tilde{\omega}_1 &= \dot{\theta}_1 c_2 c_3 + \dot{\theta}_2 s_3 & \tilde{\omega}_2 &= -\dot{\theta}_1 c_2 s_3 + \dot{\theta}_2 c_3 & \tilde{\omega}_3 &= \dot{\theta}_1 s_2 + \dot{\theta}_3 \\ \tilde{\omega}_1 s_3 + \tilde{\omega}_2 c_3 &= \dot{\theta}_2 & \text{and } (\tilde{\omega}_1 c_3 - \tilde{\omega}_2 s_3)/c_2 &= \dot{\theta}_1 & \dot{\theta}_3 &= \tilde{\omega}_3 - (\tilde{\omega}_1 c_3 - \tilde{\omega}_2 s_3) s_2/c_2 \end{aligned}$$

$$I_1 = \frac{M l_1^2}{12} (1 + c^2) \quad I_2 = \frac{M l_1^2}{12} (b^2 + c^2) \quad I_3 = \frac{M l_1^2}{12} (1 + b^2) \quad \text{let } R = \frac{d l_1}{2}, \quad d \leq \min(l_1, c)$$

$$J_1 = \frac{m l_1^2 d^2}{4 \cdot 2} \quad \text{let } C_1 = 1 - J_1 \left(\frac{\beta_1^2}{I_1} + \frac{\beta_2^2}{I_2} + \frac{\beta_3^2}{I_3} \right) \quad I = \frac{1}{C_1} - \frac{J_1}{C_1} \quad)$$

$${}^A \dot{\omega}^B = \frac{I}{J_1} \frac{1}{C_1} - \frac{1}{C_1} \left[\frac{\beta_1 F_1}{I_1} + \frac{\beta_2 F_2}{I_2} + \frac{\beta_3 F_3}{I_3} \right]$$

$$\tilde{\omega}_3 = \frac{1}{I_3} \left[\frac{F_3}{I_3} - \frac{\beta_3}{C_1} \left\{ \frac{I}{I_3} - \frac{J_1}{I_3} \left[\frac{\beta_1 F_1}{I_1} + \frac{\beta_2 F_2}{I_2} + \frac{\beta_3 F_3}{I_3} \right] \right\} \right] \quad \text{OK.}$$

$$\tilde{\omega}_2 = \frac{1}{I_2} \left[F_2 - \beta_2 \left\{ \frac{I}{C_1} - \frac{J_1}{C_1} \left[\frac{\beta_1 F_1}{I_1} + \frac{\beta_2 F_2}{I_2} + \frac{\beta_3 F_3}{I_3} \right] \right\} \right] \quad \text{OK.}$$

$$\tilde{\omega}_1 = \frac{1}{I_1} \left[F_1 - \beta_1 \left\{ \frac{I}{C_1} - \frac{J_1}{C_1} \left[\frac{\beta_1 F_1}{I_1} + \frac{\beta_2 F_2}{I_2} + \frac{\beta_3 F_3}{I_3} \right] \right\} \right] \quad \text{OK.}$$

$$F_1 = (I_2 - I_3) \tilde{\omega}_2 \tilde{\omega}_3 + J_1 {}^A \dot{\omega}^B [\tilde{\omega}_3 \beta_2 - \tilde{\omega}_2 \beta_3]$$

$$F_2 = (I_3 - I_1) \tilde{\omega}_3 \tilde{\omega}_1 + J_1 {}^A \dot{\omega}^B [\tilde{\omega}_1 \beta_3 - \tilde{\omega}_3 \beta_1]$$

$$F_3 = (I_1 - I_2) \tilde{\omega}_1 \tilde{\omega}_2 + J_1 {}^A \dot{\omega}^B [\tilde{\omega}_2 \beta_1 - \tilde{\omega}_1 \beta_2]$$

$$\dot{\theta}_1 = (\tilde{\omega}_1 c_3 - \tilde{\omega}_2 s_3)/c_2$$

$$\dot{\theta}_2 = \tilde{\omega}_1 s_3 + \tilde{\omega}_2 c_3$$

$$\dot{\theta}_3 = \tilde{\omega}_3 - s_2 (\tilde{\omega}_1 c_3 - \tilde{\omega}_2 s_3)/c_2$$

$$\text{let } y_i = \begin{cases} \tilde{\omega}_i & i=1,2,3 \\ \theta_{i-3} & i=4,5,6,7 \\ \omega^B = \theta_8 & i=8 \end{cases} \quad \begin{matrix} \tilde{\omega}_1, \tilde{\omega}_2, \tilde{\omega}_3 \\ \theta_1, \theta_2, \theta_3, \theta_4 \\ \theta_4 \end{matrix}$$

define first

ALP, BET

$$\text{let } I_{23} = I_2 - I_3$$

$$B_{11} = B_1 / I_1$$

$$I_{31} = I_3 - I_1$$

$$B_{22} = B_2 / I_2$$

$$I_{12} = I_1 - I_2$$

$$B_{33} = B_3 / I_3$$

$$J_{11} = J_1 * B_1$$

$$C_1 = 1 - J_{11} * (B_{11} * B_1 + B_{22} * B_2 + B_{33} * B_3)$$

$$J_{12} = J_1 * B_2$$

$$J_C = J_1 / C_1$$

$$J_{13} = J_1 * B_3$$

$$T_C = T / C_1$$

$$Z_1 = I_{23} * Y(2) * Y(3) + Y(8) * [J_{12} * Y(3) - J_{13} * Y(2)]$$

$$Z_2 = I_{31} * Y(3) * Y(1) + Y(8) * [J_{13} * Y(1) - J_{11} * Y(3)]$$

$$Z_3 = I_{12} * Y(1) * Y(2) + Y(8) * [J_{11} * Y(2) - J_{12} * Y(1)]$$

$$Z_4 = B_{11} * Z_1 + B_{22} * Z_2 + B_{33} * Z_3$$

$$\tilde{\omega}_1 \quad DY(1) = [Z_1 - B_1 * (T_C - J_C * Z_4)] / I_1$$

$$\tilde{\omega}_2 \quad DY(2) = [Z_2 - B_2 * (T_C - J_C * Z_4)] / I_2$$

$$\tilde{\omega}_3 \quad DY(3) = [Z_3 - B_3 * (T_C - J_C * Z_4)] / I_3$$

$$\theta_1 \quad DY(4) = Z_4$$

$$\theta_2 \quad DY(5) = Y(1) * Z_6 + Y(2) * Z_5$$

$$\theta_3 \quad DY(6) = Y(3) - Z_8 * Z_9$$

$$\theta_4 \quad DY(7) = Y(8)$$

$$\omega^B \quad DY(8) = T_C / J_1 - Z_4 / C_1$$

Read L1, B, C, D, MB, MR

A = AMIN(B, C)

IF (D.GT.A) D = A

R = A * L1 / 2

J1 = MR * R * R / 2

A = MB * L1 * L1 / 12

I1 = A * (1 + C * C)

I2 = A * (B * B + C * C)

I3 = A * (1 + B * B)

PI04 = ATANH(1.00)

AL = ALP * PI04 / 45

BE = BET * PI04 / 45

B1 = SIN(AL) * COS(BE)

B2 = SIN(AL) * SIN(BE)

Z5 = COS(Y(6)) B3 = COS(AL)

Z6 = SIN(Y(6))

Z7 = COS(Y(5))

Z8 = SIN(Y(5))

Z9 = (Y(1) * Z5 - Y(2) * Z6) / Z7



$$\beta_3 \omega_1 - \beta_1 \omega_3) a_{12} + [\omega_1 \omega_2 (I_1 - I_2) - \omega_3 I_3 - J \dot{\omega}^B \beta_3 + J \dot{\omega}^B (\beta_1 \omega_2 - \beta_2 \omega_1)] a_{13}$$

$$F_4^* = [\omega_2 \omega_3 (I_2 - I_3) - I_1 \dot{\omega}_1 - J \dot{\omega}^B \beta_1 + J \dot{\omega}^B (\beta_2 \omega_3 - \beta_3 \omega_2)] = F_{\omega_1}^*$$

$$F_5^* = [\omega_3 \omega_1 (I_3 - I_1) - I_2 \dot{\omega}_2 - J \dot{\omega}^B \beta_2 + J \dot{\omega}^B (\beta_3 \omega_1 - \beta_1 \omega_3)] = F_{\omega_2}^*$$

$$F_6^* = [\omega_1 \omega_2 (I_1 - I_2) - I_3 \dot{\omega}_3 - J \dot{\omega}^B \beta_3 + J \dot{\omega}^B (\beta_1 \omega_2 - \beta_2 \omega_1)] = F_{\omega_3}^*$$

$$\Pi^{*B} = (\Pi^{B/B^*}, \dot{\omega}^B) \times \dot{\omega}^B - \Pi^{B/B^*} \cdot \dot{\alpha}^B$$

$$\dot{\alpha}^B = \frac{d}{dt} \dot{\omega}^B = \frac{d}{dt} \dot{\omega}^A + \dot{\omega}^A \times \dot{\omega}^A + \frac{d}{dt} \dot{\omega}^B + \dot{\omega}^A \times \dot{\omega}^B$$

$$= \dot{\omega}_i^A a_{1i} + \dot{\omega}_j^B a_{1j} + \dot{\omega}^A \times \dot{\omega}^B$$

$${}^R \omega^A \times {}^A \omega^B = (\omega_i a_{1i}) \times (\dot{\omega}_j^B a_{1j}) = \dot{\omega}^B [(\omega_2 \beta_3 - \omega_3 \beta_2) a_{11} + (\omega_3 \beta_1 - \omega_1 \beta_3) a_{12} + (\omega_1 \beta_2 - \omega_2 \beta_1) a_{13}]$$

$$\begin{aligned} \text{Now } \beta \cdot \Pi^{*B} &= \beta \cdot (\Pi^{B/B^*}, \dot{\omega}^B) \times \dot{\omega}^B - \beta \cdot \Pi^{B/B^*} \cdot \dot{\alpha}^B \\ &= (\Pi^{B/B^*}, \dot{\omega}^B) \cdot (\dot{\omega}^B \times \beta) - \beta \cdot \Pi^{B/B^*} \cdot \dot{\alpha}^B \\ &\quad (\dot{\omega}^A + {}^A \dot{\omega}^B) \times \beta - \beta \cdot \Pi^{B/B^*} \cdot \dot{\alpha}^A - \dot{\omega}^B \beta \cdot \Pi^{B/B^*} \cdot \beta - \beta \cdot \Pi^{B/B^*} \cdot ({}^R \omega^A \times {}^A \omega^B) \\ &= (\Pi^{B/B^*}, \dot{\omega}^B) \cdot (\dot{\omega}^A \times \beta) - J \beta \cdot \dot{\alpha}^A - \dot{\omega}^B J - J \beta \cdot ({}^R \omega^A \times {}^A \omega^B) \quad {}^A \omega^B = \dot{\omega}^B \beta \\ &\quad [I \cdot (\dot{\omega}^A + {}^A \dot{\omega}^B)] - J [\dot{\omega}_1 \beta_1 + \dot{\omega}_2 \beta_2 + \dot{\omega}_3 \beta_3 + {}^A \dot{\omega}^B] \\ &= (\Pi^{B/B^*}, \dot{\omega}^B) + J \dot{\omega}^B \beta \cdot (\dot{\omega}^A \times \beta) - J [\dot{\omega}_1 \beta_1 + \dot{\omega}_2 \beta_2 + \dot{\omega}_3 \beta_3 + {}^A \dot{\omega}^B] \\ \beta \cdot \Pi^{*B} &= \uparrow \cdot (\dot{\omega}^A \times \beta) - J [\dot{\omega}_1 \beta_1 + \dot{\omega}_2 \beta_2 + \dot{\omega}_3 \beta_3 + {}^A \dot{\omega}^B] \\ &\quad - J [\dots] + T = 0 \quad \frac{I}{J} = \dots \end{aligned}$$

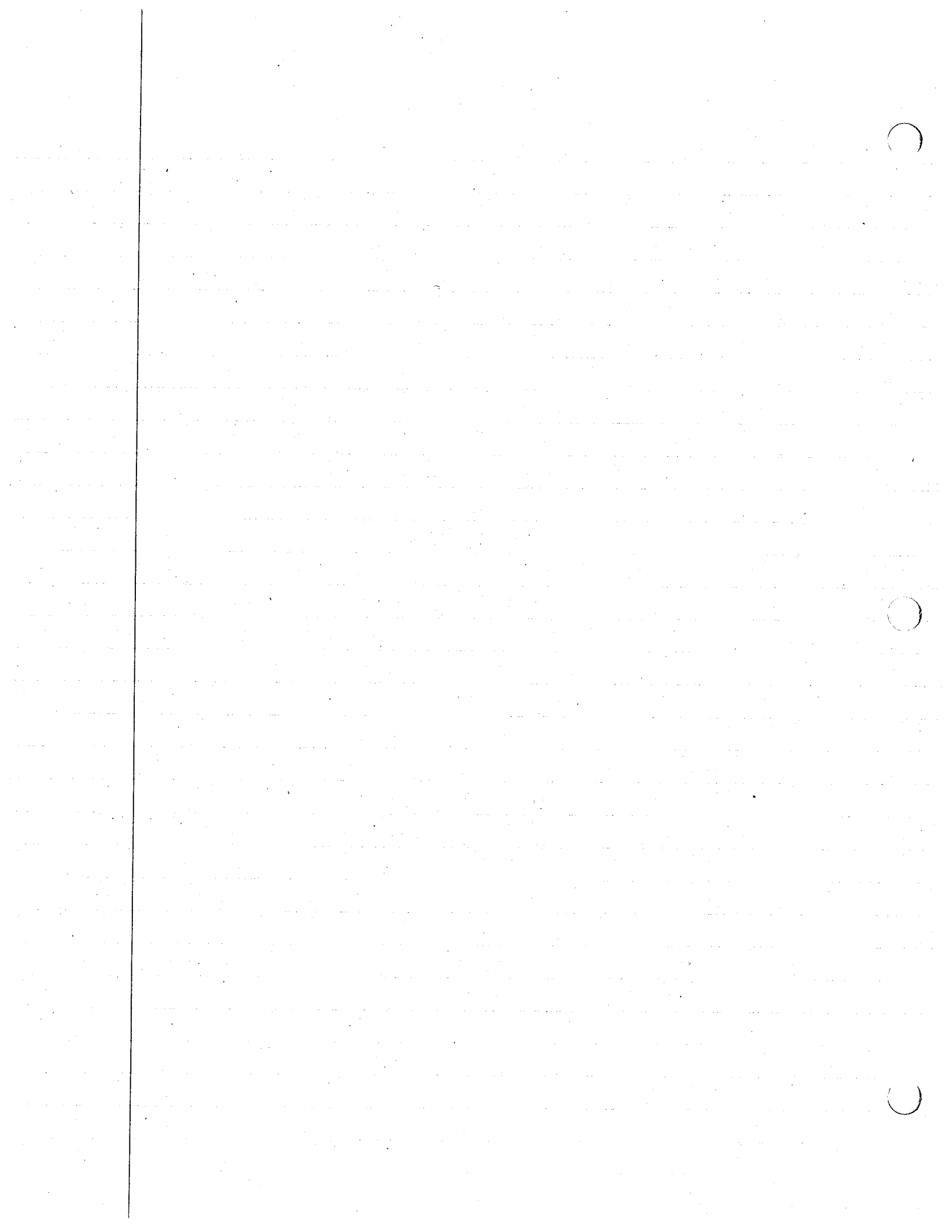
$$\frac{I}{J} - [\dot{\omega}_1 \beta_1 + \dot{\omega}_2 \beta_2 + \dot{\omega}_3 \beta_3] = {}^A \dot{\omega}^B$$

$$\beta \dot{\omega}_1 = \frac{R}{I_1} [\omega_2 \omega_3 (I_2 - I_3) + J \dot{\omega}^B (\beta_2 \omega_3 - \beta_3 \omega_2)] - \frac{J \beta_1^2}{I_1} {}^A \dot{\omega}^B$$

$$= \frac{\beta_1 F_1}{I_1} - \frac{J \beta_1^2}{I_1} {}^A \dot{\omega}^B = \frac{\beta_1 F_1}{I_1} - \frac{I}{I_1} \beta_1^2 + \frac{J}{I_1} \beta_1^2 [$$

$$\frac{I}{J} - [\frac{\beta_1 F_1}{I_1} + \frac{\beta_2 F_2}{I_2} + \frac{\beta_3 F_3}{I_3} + J \dot{\omega}^B (\frac{\beta_1^2}{I_1} + \frac{\beta_2^2}{I_2} + \frac{\beta_3^2}{I_3})] = {}^A \dot{\omega}^B$$

$$\frac{I}{J} - [\frac{\beta_1 F_1}{I_1} + \dots] = {}^A \dot{\omega}^B [1 - J (\frac{\beta_1^2}{I_1} + \dots)]$$



Derivation

let $x, y, z, \omega_1, \omega^A, \omega^B$

$$F_r = \dot{V}_{\dot{q}_r}^A \cdot IF^A + \dot{\omega}_{\dot{q}_r}^A \cdot \Pi^A + \dot{V}_{\dot{q}_r}^B \cdot IF^B + \dot{\omega}_{\dot{q}_r}^B \cdot \Pi^B = \dot{V}_{\dot{q}_r} [IF^A + IF^B] + \dot{\omega}_{\dot{q}_r}^A [\Pi^A + \Pi^B] + \dot{\omega}_{\dot{q}_r}^B \Pi^B$$

$$F_r^* = \dot{V}_{\dot{q}_r}^{A*} \cdot IF^* + \dot{\omega}_{\dot{q}_r}^{A*} \cdot \Pi^{*A} + \dot{\omega}_{\dot{q}_r}^{B*} \cdot \Pi^{*B} \quad \text{where } IF^* = -M\ddot{\alpha}^* \quad M = (\text{mass of rotor} + \square)$$

$$\Pi^A = -T\beta = T\beta_i \alpha_i; \quad \Pi^B = +T\beta_i \alpha_i; \quad {}^R\omega^A = \omega_i \alpha_i; \quad {}^R\omega^B = {}^R\omega^A + {}^A\omega^B = (\omega_i + \dot{\omega}^B \beta_i) \alpha_i$$

$$\dot{V}^A = \dot{x} m_x + \dot{y} m_y + \dot{z} m_z \quad IF^A + IF^B = Mg \ln x \quad {}^R\omega_{\omega_i}^B = \alpha_i; \quad {}^R\omega_{\omega_i}^B = \beta; \quad {}^A\omega_{\omega_i}^B = 0$$

$$\dot{\alpha}^A = \dot{x} m_x + \dot{y} m_y + \dot{z} m_z \quad \Pi^A + \Pi^B = 0; \quad {}^A\omega_{\omega_i}^B = \beta$$

$$F_1 = Mg + 0 + 0$$

$$F_1^* = -M\ddot{x}$$

$$F_2 = 0 + 0 + 0$$

$$F_2^* = -M\ddot{y}$$

$$F_3 = 0 + 0 + 0$$

$$F_3^* = -M\ddot{z}$$

ω_1

$$F_4 = 0 + \alpha_1 \cdot 0 + 0$$

$$F_4^* = \dot{V}_{\omega_1}^{A*} \cdot IF^* + \dot{\omega}_{\omega_1}^{A*} \cdot \Pi^{*A} + \dot{\omega}_{\omega_1}^{B*} \cdot \Pi^{*B}$$

ω_2

$$F_5 = 0 + \alpha_2 \cdot 0 + 0$$

$$0 + \alpha_1 \cdot \Pi^{*A} + \alpha_1 \cdot \Pi^{*B}$$

ω_3

$$F_6 = 0 + \alpha_3 \cdot 0 + 0$$

$$F_5^* = 0 + \alpha_2 \cdot (\Pi^{*A} + \Pi^{*B})$$

${}^A\omega^B$

$$F_7 = 0 + 0 \cdot 0 + T$$

$$F_6^* = 0 + \alpha_3 \cdot (\Pi^{*A} + \Pi^{*B})$$

$$F_7^* = 0 + 0 + \beta \cdot \Pi^{*B}$$

$$\Pi^{*A} + \Pi^{*B} = -\frac{d}{dt} \frac{\partial H^{A/A^*}}{\partial \dot{\alpha}^A} - \frac{d}{dt} \frac{\partial H^{B/B^*}}{\partial \dot{\alpha}^B} = -\frac{d}{dt} (H^{A/A^*} + H^{B/B^*})$$

$$H^{A/A^*} = \Pi^{A/A^*} \cdot {}^R\omega^A \quad H^{B/B^*} = \Pi^{B/B^*} \cdot {}^R\omega^B = \Pi^{B/B^*} \cdot {}^R\omega^A + \Pi^{B/B^*} \cdot {}^A\omega^B$$

$$\therefore H^{A/A^*} + H^{B/B^*} = (\Pi^{A/A^*} + \Pi^{B/B^*}) \cdot {}^R\omega^A + \Pi^{B/B^*} \cdot {}^A\omega^B$$

Define a fictitious body (rigid D) whose mass distrib is that of G but whose motion is that of A.

$$\text{Thus } H^{A/A^*} + H^{B/B^*} = H^{D/D^*} + {}^A H^{B/B^*}$$

$$\frac{d}{dt} () = \frac{d}{dt} H^{D/D^*} + \frac{d}{dt} H^{B/B^*} = \frac{d}{dt} H^{D/D^*} + \frac{d}{dt} H^{B/B^*} + \dot{\omega}^A \times H^{B/B^*} - \Pi^{D/D^*} - \Pi^{B/B^*} + {}^R\omega^A \times (\Pi^{B/B^*} \cdot {}^A\omega^B)$$

$$\therefore \Pi^{*A} + \Pi^{*B} = \Pi^{D/D^*} + \Pi^{*B} - \dot{\omega}^A \times H^{B/B^*}$$

$$= [(\Pi^{C/C^*} \cdot {}^R\omega^A) \times {}^R\omega^A - \Pi^{C/C^*} \cdot {}^R\alpha^A + (\Pi^{B/B^*} \cdot {}^A\omega^B) \times {}^A\omega^B - \Pi^{B/B^*} \cdot {}^A\alpha^B + (\Pi^{B/B^*} \cdot {}^A\omega^B) \times {}^R\omega^A]$$

$${}^R\alpha^A = \frac{d}{dt} {}^R\omega^A = \frac{d}{dt} \omega_i \alpha_i + \omega_i \alpha_i \times \omega_i \alpha_i = \dot{\omega}_i \alpha_i$$

$${}^A\alpha^B = \frac{d}{dt} {}^A\omega^B = \dot{\omega}^B \beta$$

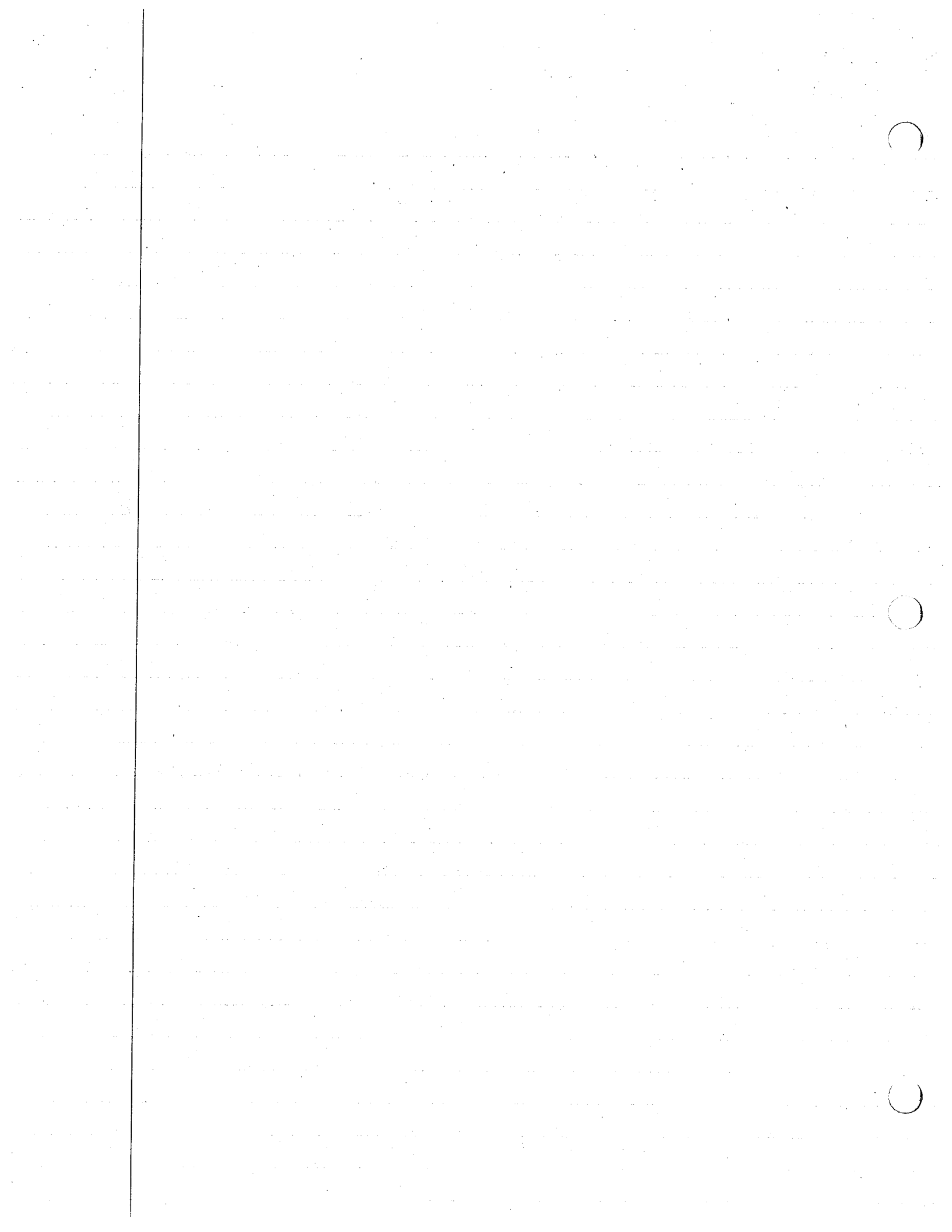
$$\Pi^{C/C^*} \cdot {}^R\omega^A = I_i \omega_i \alpha_i; \quad \Pi^{C/C^*} \cdot {}^R\alpha^A = I_i \dot{\omega}_i \alpha_i; \quad \Pi^{B/B^*} \cdot {}^A\omega^B = J \dot{\omega}^B \beta \quad \text{and } (\Pi^{B/B^*} \cdot {}^A\omega^B) \times {}^R\omega^A =$$

$$\Pi^{B/B^*} \cdot {}^A\alpha^B = J \dot{\omega}^B \beta; \quad (\Pi^{B/B^*} \cdot {}^A\omega^B) \times {}^R\omega^A = (J \dot{\omega}^B \beta_i \alpha_i \times \omega_j \alpha_j) = J \dot{\omega}^B [(\beta_1 \omega_2 - \beta_2 \omega_1) \alpha_3 +$$

$$(\beta_2 \omega_3 - \beta_3 \omega_2) \alpha_1 + (\beta_3 \omega_1 - \beta_1 \omega_3) \alpha_2]$$

$$\Pi^{*A} + \Pi^{*B} = [I_1 \omega_1 (\omega_2 \alpha_3 - \omega_3 \alpha_2) + I_2 \omega_2 (\omega_3 \alpha_1 - \omega_1 \alpha_3) + I_3 \omega_3 (\omega_1 \alpha_2 - \omega_2 \alpha_1) - I_1 \dot{\omega}_1 \alpha_1 - I_2 \dot{\omega}_2 \alpha_2 - I_3 \dot{\omega}_3 \alpha_3 - J \dot{\omega}^B [\beta_1 \alpha_1 + \beta_2 \alpha_2 + \beta_3 \alpha_3] + J \dot{\omega}^B (\beta_1 \omega_2 \alpha_3 - \beta_1 \omega_3 \alpha_2 + \beta_2 \omega_3 \alpha_1 - \beta_2 \omega_1 \alpha_3 + \beta_3 \omega_1 \alpha_2 - \beta_3 \omega_2 \alpha_1)]$$

$$= [I_1 \omega_1 (\omega_2 \alpha_3 - \omega_3 \alpha_2) - I_2 \omega_2 (\omega_3 \alpha_1 - \omega_1 \alpha_3) - I_3 \omega_3 (\omega_1 \alpha_2 - \omega_2 \alpha_1) - J \dot{\omega}^B \beta_1 + J \dot{\omega}^B (\beta_2 \omega_3 - \beta_3 \omega_2)] \alpha_1 + [I_3 \omega_3 (\omega_1 \alpha_2 - \omega_2 \alpha_1) - I_2 \omega_2 (\omega_3 \alpha_1 - \omega_1 \alpha_3) - J \dot{\omega}^B \beta_2 + J \dot{\omega}^B (\beta_3 \omega_1 - \beta_1 \omega_3)] \alpha_2 + [I_1 \omega_1 (\omega_2 \alpha_3 - \omega_3 \alpha_2) - I_3 \omega_3 (\omega_1 \alpha_2 - \omega_2 \alpha_1) - J \dot{\omega}^B \beta_3 + J \dot{\omega}^B (\beta_1 \omega_2 - \beta_2 \omega_1)] \alpha_3$$



Thus our 3 eqns reduce to

$$\begin{aligned} \rightarrow (I_1 - I_2) \tilde{\omega}_2 \tilde{\omega}_1 - J_1 (\dot{\omega}^B \beta_3 - \dot{\omega}^B [\tilde{\omega}_2 \beta_1 - \tilde{\omega}_1 \beta_2]) &= I_3 \dot{\tilde{\omega}}_3 = F_3 - J_1 \beta_3 \dot{\omega}^B \\ \rightarrow (I_2 - I_3) \tilde{\omega}_3 \tilde{\omega}_1 - J_1 (\dot{\omega}^B \beta_1 - \dot{\omega}^B [\tilde{\omega}_3 \beta_2 - \tilde{\omega}_2 \beta_3]) &= I_1 \dot{\tilde{\omega}}_1 = F_1 - J_1 \beta_1 \dot{\omega}^B \\ \rightarrow (I_3 - I_1) \tilde{\omega}_3 \tilde{\omega}_2 - J_1 (\dot{\omega}^B \beta_2 - \dot{\omega}^B [\tilde{\omega}_1 \beta_3 - \tilde{\omega}_3 \beta_1]) &= I_2 \dot{\tilde{\omega}}_2 = F_2 - J_1 \beta_2 \dot{\omega}^B \end{aligned}$$

$$\rightarrow \dot{\omega}^B = \ddot{\theta}_4 \quad \dot{\omega}^B = \dot{\theta}_4 \quad \beta_3 = \cos \alpha \quad \beta_1 = s_\alpha c_\beta \quad \beta_2 = s_\alpha s_\beta$$

$$T + J_1 (\dot{\omega}_3 + \dot{\omega}^B) = 0 \quad \dot{\omega}^B = -\frac{T}{J_1} - \dot{\omega}_3$$

$$\begin{aligned} \text{Now } \tilde{\omega}_1 [c_\beta c_\alpha m_1 - s_\beta m_2 + s_\alpha c_\beta m_3] + \tilde{\omega}_2 [s_\beta c_\alpha m_1 + s_\beta m_2 + s_\alpha s_\beta m_3] + \tilde{\omega}_3 [-s_\alpha m_1 + c_\alpha m_3] &= \dot{\omega}^B \\ \tilde{\omega}_1 s_\alpha c_\beta + \tilde{\omega}_2 s_\alpha s_\beta + \tilde{\omega}_3 c_\alpha &= \dot{\omega}_3 \\ \tilde{\omega}_1 \beta_1 + \tilde{\omega}_2 \beta_2 + \tilde{\omega}_3 \beta_3 &= \dot{\omega}_3 \end{aligned}$$

Solve the above for the derivatives $\dot{\omega}_1, \dot{\omega}_2, \dot{\omega}_3$ & put into eqn. for $\dot{\omega}_3$

$$\begin{aligned} \frac{\beta_1}{I_1} (F_1 - J_1 \dot{\omega}^B \beta_1) + \frac{\beta_2}{I_2} (F_2 - J_1 \dot{\omega}^B \beta_2) + \frac{\beta_3}{I_3} (F_3 - J_1 \dot{\omega}^B \beta_3) &= \dot{\omega}_3 \\ \frac{\beta_1 F_1}{I_1} + \frac{\beta_2 F_2}{I_2} + \frac{\beta_3 F_3}{I_3} &= \dot{\omega}_3 + J_1 \dot{\omega}^B \left(\frac{\beta_1^2}{I_1} + \frac{\beta_2^2}{I_2} + \frac{\beta_3^2}{I_3} \right) \end{aligned}$$

$$\text{Now } \dot{\omega}^B + \dot{\omega}_3 = -\frac{T}{J_1} \quad ; \quad \dot{\omega}^B + \frac{\beta_1 F_1}{I_1} + \frac{\beta_2 F_2}{I_2} + \frac{\beta_3 F_3}{I_3} - J_1 \dot{\omega}^B \left(\frac{\beta_1^2}{I_1} + \frac{\beta_2^2}{I_2} + \frac{\beta_3^2}{I_3} \right) = \dot{\omega}_3 + \dot{\omega}^B = -\frac{T}{J_1}$$

$$\dot{\omega}^B \left[1 - J_1 \left(\frac{\beta_1^2}{I_1} + \frac{\beta_2^2}{I_2} + \frac{\beta_3^2}{I_3} \right) \right] = -\frac{T}{J_1} - \left(\frac{\beta_1 F_1}{I_1} + \frac{\beta_2 F_2}{I_2} + \frac{\beta_3 F_3}{I_3} \right)$$

$$\begin{aligned} -F_1 &= (I_3 - I_2) \tilde{\omega}_2 \tilde{\omega}_3 - J_1 (\tilde{\omega}_3 \beta_2 - \tilde{\omega}_2 \beta_3) \\ -F_2 &= (I_1 - I_3) \tilde{\omega}_3 \tilde{\omega}_1 - J_1 (\tilde{\omega}_1 \beta_3 - \tilde{\omega}_3 \beta_1) \\ -F_3 &= (I_2 - I_1) \tilde{\omega}_2 \tilde{\omega}_1 - J_1 (\tilde{\omega}_2 \beta_1 - \tilde{\omega}_1 \beta_2) \end{aligned}$$

then,

$$\begin{aligned} \rightarrow \dot{\omega}^B \left[1 - J_1 \left(\frac{\beta_1^2}{I_1} + \frac{\beta_2^2}{I_2} + \frac{\beta_3^2}{I_3} \right) \right] &= -\frac{T}{J_1} + \frac{\beta_1}{I_1} [(I_3 - I_2) \tilde{\omega}_2 \tilde{\omega}_3 - J_1 (\tilde{\omega}_3 \beta_2 - \tilde{\omega}_2 \beta_3)] \\ &+ \frac{\beta_2}{I_2} [(I_1 - I_3) \tilde{\omega}_3 \tilde{\omega}_1 - J_1 (\tilde{\omega}_1 \beta_3 - \tilde{\omega}_3 \beta_1)] \\ &+ \frac{\beta_3}{I_3} [(I_2 - I_1) \tilde{\omega}_2 \tilde{\omega}_1 - J_1 (\tilde{\omega}_2 \beta_1 - \tilde{\omega}_1 \beta_2)] \end{aligned}$$

with the following definitions

4300 ✓

4400 ✓

5960 $CO = 1.5 * r * amu$ ✓

6220 ✓

$(1. + \underline{b+b})$

8250

8775 } cont.

9200

9250 } cont.

9700

1 } contin

10300

11900

11950

12000

	W1	W2	W3	AWB
I1	I2	I3	J1	TORQUE

ROTOR MASS $T/J1$

input AMU, RM, B, C, D, AL1, , TJ1, ALP, BET.

input AWB, W1, W2, W3

$$CEE = (C * C - 1.) / (C * C + 1.)$$

$$BCE = (B * B - C * C) / (B * B + C * C)$$

$$BEE = (1. - B * B) / (1. + B * B)$$

~~$$IF (D > GT. AMIN(B, C)) D = AMIN(B, C)$$~~

$$R = D * AL1 / 2. \quad \text{radius of rotor}$$

$$AL3 = C * AL1 \quad l_3$$

$$AL2 = B * AL1 \quad l_2$$

$$AJ1 = RM * R * R / 2.$$

$$J1 = m R^2 / 2 \quad \text{polar mom of inertia of rotor,}$$

$$J = \frac{m}{2} \frac{d^2 l_1^2}{4}$$

$$C0 = 1.5 * AMU * D * D$$

$$\frac{3}{2} \mu d^2$$

**

$$AJ1I1 = C0 / (1. + C * C)$$

$$J1/I1 = \frac{\frac{3}{2} \mu d^2}{1 + C^2}$$

$$AJ1I2 = C0 / (B * B + C * C)$$

$$J1/I2$$

$$AJ1I3 = C0 / (1. + B * B)$$

$$J1/I3$$

$$AI1 = AJ1 / AJ1I1$$

$$\frac{m R^2}{2} \cdot \frac{2}{3} \frac{1 + C^2}{d^2} \frac{M}{m} = \frac{m}{2} \frac{d^2 l_1^2}{4} \cdot \frac{2}{3} \frac{(1 + C^2)}{d^2} \frac{M}{m} = \frac{M l_1^2 (1 + C^2)}{12} = I_1 \quad \checkmark$$

$$AI2 = AJ1 / AJ1I2$$

$$I_2$$

$$AI3 = AJ1 / AJ1I3$$

$$I_3$$

$$T = TJ1 * AJ1$$

$$T = T/J1 \cdot J1$$

IF (AI1 + AI2. LT. AI3) TYPE 618
GO TO 110

IF (AI2 + AI3. LT. AI1) TYPE 619
GO TO 110

IF (AI3 + AI1. LT. AI2) TYPE 620
GO TO 110

$$AL = ALP * ATAN(1.00) / 45.0$$

$$\alpha$$

$$BE = BET * ATAN(1.00) / 45.0$$

$$\beta$$

$$B1 = SIN(AL) * COS(BE)$$

$$\beta_1$$

$$B2 = SIN(AL) * SIN(BE)$$

$$\beta_2$$

$$B3 = COS(AL)$$

$$\beta_3$$

$$C1 = 1 - C0 * (B1 * B1 / (1. + C * C) + B2 * B2 / (B * B + C * C) + B3 * B3 / (1. + B * B))$$

↓

$$B1C1 = B1/C1$$

$$\beta_1/C1$$

$$B3C1 = B3/C1$$

$$\beta_3/C3$$

$$B2C1 = B2/C1$$

$$\beta_2/C2$$



$$T I 3 = T J 1 + J I I 3$$

$$T / I 3$$

$$T I 2 = T J 1 + J I I 2$$

$$T / I 2$$

$$T I 1 = T J 1 + J I I 1$$

$$T / I 1$$

$$G 1 = C E E * \frac{Y(2) Y(3)}{W 2 * W 3} + C 0 * \left(\frac{Y(3)}{W 3 * B 2} - \frac{Y(2)}{W 2 * B 3} \right) * \frac{Y(4)}{A W B} / (1 + C * C)$$

$$G 2 = B C E * \frac{Y(3) Y(1)}{W 3 * W 1} + C 0 * \left(\frac{Y(1)}{W 1 * B 3} - \frac{Y(3)}{W 3 * B 1} \right) * \frac{Y(4)}{A W B} / (B * B + C * C)$$

$$G 3 = B E E * \frac{Y(1) Y(2)}{W 1 * W 2} + C 0 * \left(\frac{Y(2)}{W 2 * B 1} - \frac{Y(1)}{W 1 * B 2} \right) * \frac{Y(4)}{A W B} / (1 + B * B)$$

$$H = B 1 * G 1 + B 2 * G 2 + B 3 * G 3$$

$$D Y(1) \quad \overline{W 3 D O T} = G 3 - [B 3 C 1 + (T I 3 - A J I 3 * H)]$$

$$D Y(2) \quad \overline{W 2 D O T} = G 2 - [B 2 C 1 + (T I 2 - A J I 2 * H)]$$

$$D Y(3) \quad \overline{W 1 D O T} = G 1 - [B 1 C 1 + (T I 1 - A J I 1 * H)]$$

$$D Y(4) \quad \overline{A W B D O T} = [T J 1 - H] / C 1$$

$$\overline{D Y(1)} = \overline{W 3 D O T}$$

$$\overline{D Y(2)} = \overline{W 2 D O T}$$

$$\overline{D Y(3)} = \overline{W 1 D O T}$$

$$\overline{D Y(4)} = \overline{A W B D O T}$$

need to pass $\check{C E E}, \check{E O}, \check{C}, \check{B 1}, \check{B 2}, \check{B 3}, \check{B C E}, \check{B}, \check{C}, \check{B E E}$

$\check{B B C 1}, \check{B 2 C 1}, \check{B 1 C 1}, \check{T I 3}, \check{T I 2}, \check{T I 1}, \check{A J I 3}, \check{A J I 2}, \check{A J I 1}, \check{T J 1}, \check{C 1}$

or $\check{T}, \check{A 1}, \check{A 1}, \check{A 2}, \check{A 3} \leftarrow \check{B 1}, \check{B 2}, \check{B 3}, \check{C 1}$

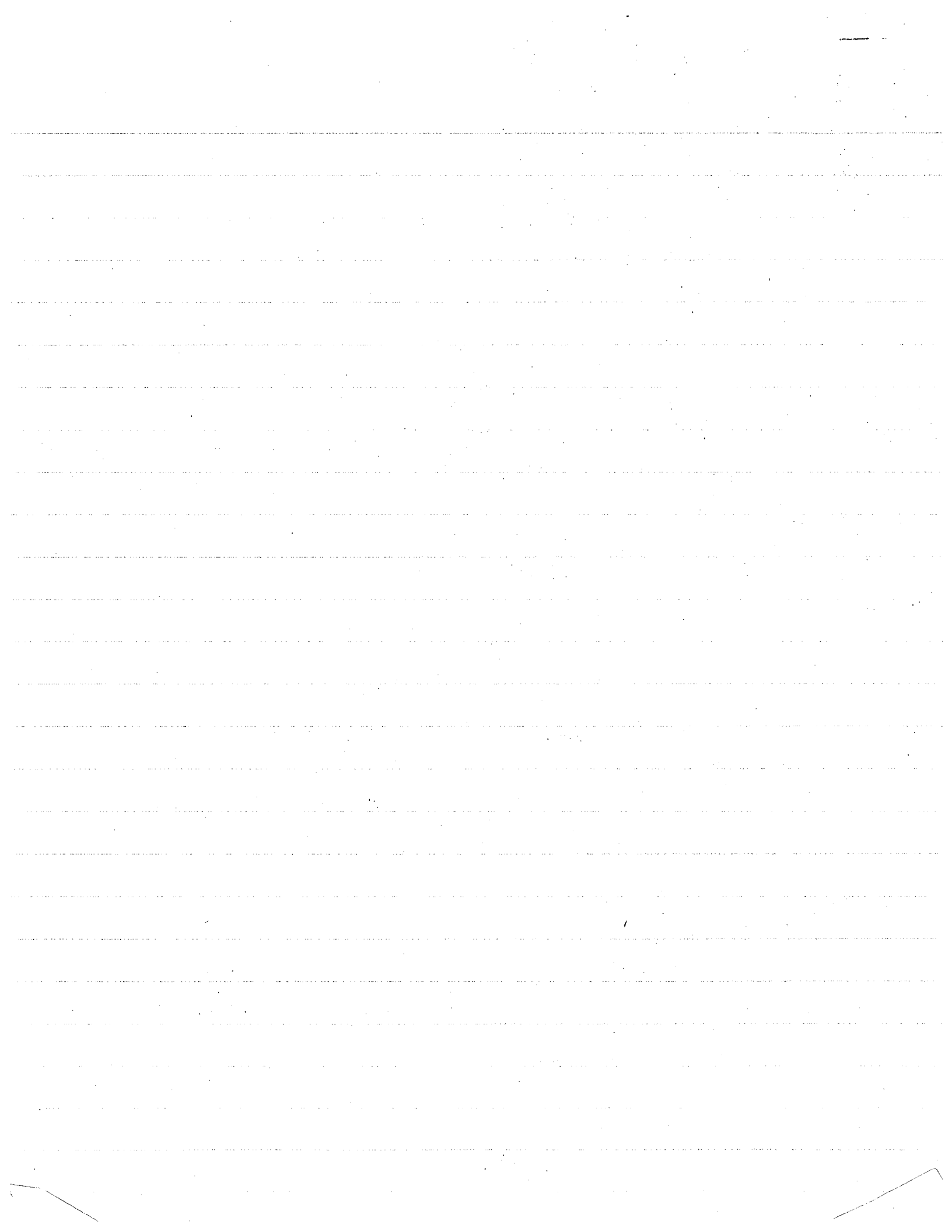
~~HEE~~

~~HEE~~ ~~HEE~~

$$G 1 = (A I 2 - A I 3) * Y(2) * Y(3) + A J 1 * [Y(3) * B 2 - Y(2) * B 3] * Y(4)$$

$$G 2 = (A I 3 - A I 1) * Y(3) * Y(1) + A J 1 * [Y(1) * B 3 - Y(3) * B 1] * Y(4)$$

$$G 3 = (A I 1 - A I 2) * Y(1) * Y(2)$$



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3.7 Dynamical equations for a simple gyrost

In Fig. 3.7.1, G designates a simple gyrost consisting of a rigid body A and an axisymmetric rotor B whose axis and mass center, B^* , are fixed in A . $\underline{a}_1, \underline{a}_2, \underline{a}_3$ form a dextal set of mutually perpendicular unit vectors fixed in A , each parallel to a central principal axis of inertia of G . The axis of B is parallel to $\underline{a}_3$ and G has central principal moments of inertia I_1, I_2, I_3 while B has an axial moment of inertia J .

After defining ${}^A\omega^B$, ω_i , β_i , and M_i as

$${}^A\omega^B \triangleq \underline{\omega}^B \cdot \underline{\beta} \quad (1)$$

$$\omega_i \triangleq \underline{\omega}^A \cdot \underline{a}_i \quad (i=1,2,3) \quad (2)$$

and

$$\beta_i \triangleq \underline{\beta} \cdot \underline{a}_i \quad (i=1,2,3) \quad (3)$$

$$M_i \triangleq \underline{M} \cdot \underline{a}_i \quad (i=1,2,3) \quad (4)$$

where ${}^A\omega^B$ is the angular velocity of B in A , $\underline{\omega}^A$ is the angular velocity of A in a Newtonian reference frame N , and $\underline{M}$ is the moment about the mass center G^* of G of all forces acting on G , one can write the following set of coupled differential equations governing ${}^A\omega^B$ and ω_i ($i=1,2,3$):

$$(I_3 - J) \dot{{}^A\omega^B} = (I_1 - I_2) \omega_1 \omega_2 - M_3 + (I_3/J) \underline{a}_3 \cdot \underline{M}^{A/B} \quad (4)$$

where $\underline{M}^{A/B}$ is the moment about B^* of all forces exerted by A on B , and

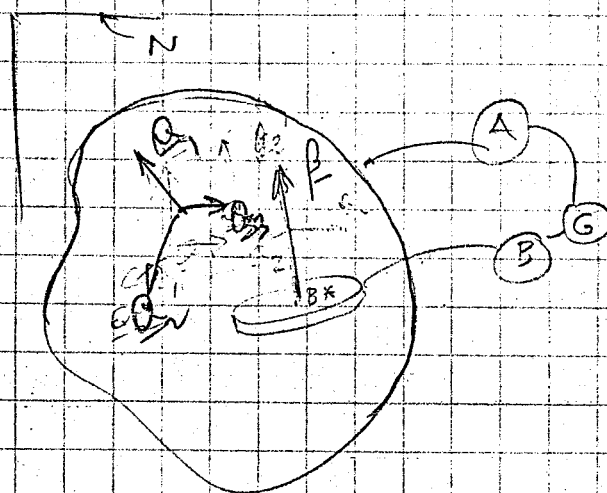


Fig. 3.7.1

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$$\begin{aligned}
 {}^{A \cdot B} \dot{\omega} \left[1 - \sqrt{\left(\frac{\beta_1^2}{I_1} + \frac{\beta_2^2}{I_2} + \frac{\beta_3^2}{I_3} \right)} \right] &= \frac{\beta \cdot \overset{T}{M}^{A/B}}{J} + \\
 &+ \frac{\beta_1}{I_1} \left[\overset{I_2 - I_3}{(I_3 - I_2)} \omega_2 \omega_3 + {}^{A \cdot B} \dot{\omega} (\beta_2 \omega_3 - \beta_3 \omega_2) - M_1 \right] + \\
 &+ \frac{\beta_2}{I_2} \left[\overset{I_3 - I_1}{(I_1 - I_3)} \omega_3 \omega_1 + {}^{A \cdot B} \dot{\omega} (\beta_3 \omega_1 - \beta_1 \omega_3) - M_2 \right] + \\
 &+ \frac{\beta_3}{I_3} \left[\overset{I_1 - I_2}{(I_2 - I_1)} \omega_1 \omega_2 + {}^{A \cdot B} \dot{\omega} (\beta_1 \omega_2 - \beta_2 \omega_1) - M_3 \right] \quad (5)
 \end{aligned}$$

where $M^{A/B}$ is the moment about B^* of all forces exerted by A on B; and

$$I_1 \dot{\omega}_1 = (I_2 - I_3) \omega_2 \omega_3 - J [{}^{A \cdot B} \dot{\omega} \beta_1 - {}^{A \cdot B} \dot{\omega} (\beta_2 \omega_3 - \beta_3 \omega_2)] + M_1 \quad (6)$$

$$I_2 \dot{\omega}_2 = (I_3 - I_1) \omega_3 \omega_1 - J [{}^{A \cdot B} \dot{\omega} \beta_2 - {}^{A \cdot B} \dot{\omega} (\beta_3 \omega_1 - \beta_1 \omega_3)] + M_2 \quad (7)$$

$$I_3 \dot{\omega}_3 = (I_1 - I_2) \omega_1 \omega_2 - J [{}^{A \cdot B} \dot{\omega} \beta_3 - {}^{A \cdot B} \dot{\omega} (\beta_1 \omega_2 - \beta_2 \omega_1)] + M_3 \quad (8)$$

If the axis of B is parallel to a central principal axis of inertia of G, the complexity of these equations is reduced considerably; and further simplifications can be made if B is completely free to rotate relative to A or if B is made to rotate relative to A with a constant angular speed, say by means of a motor each of whose parts belongs either to A or to B. Under these circumstances, and with $\beta = \alpha_3$, the differential equations governing ω_1 , ω_2 , and ω_3 can be written

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$$I_1 \dot{\omega}_1 = (I_2 - \bar{I}_3) \omega_2 \omega_3 - \sqrt{\sigma} \omega_2 + M_1 \quad (9)$$

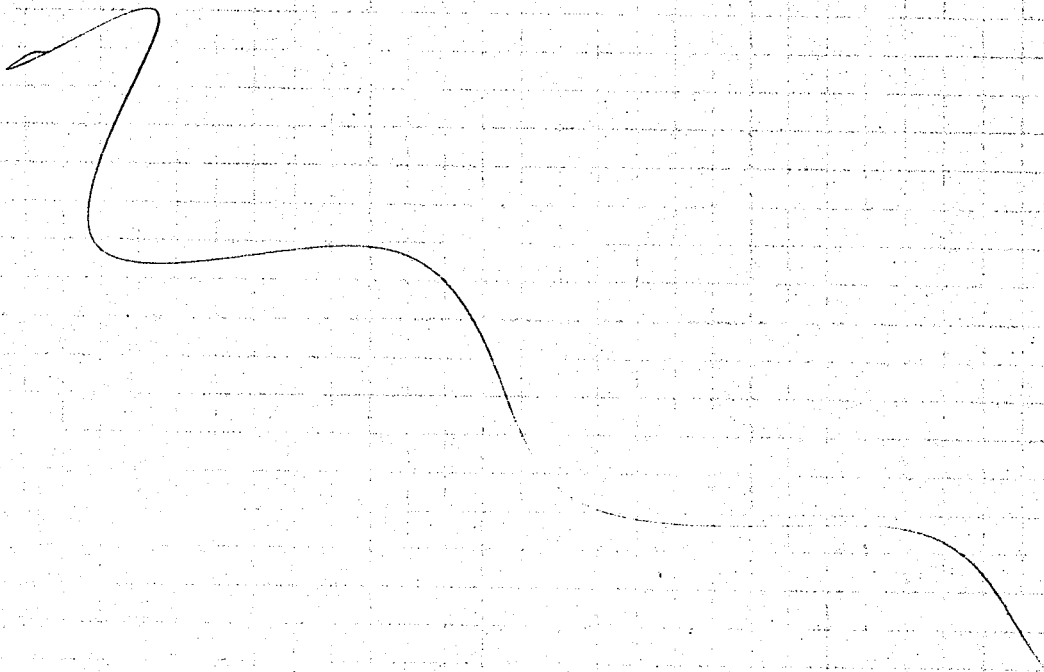
$$I_2 \dot{\omega}_2 = (\bar{I}_3 - I_1) \omega_3 \omega_1 + \sqrt{\sigma} \omega_1 + M_2 \quad (10)$$

$$\bar{I}_3 \dot{\omega}_3 = (I_1 - I_2) \omega_1 \omega_2 + M_3 \quad (11)$$

where $\bar{I}_3$ and σ are given in Table 3.7.1, with $\overset{A \wedge B}{\omega}$ and $\hat{\omega}_3$ denoting the initial values of $\overset{A \wedge B}{\omega}$ and ω_3 , respectively.

Rotor motion	$\bar{I}_3$	σ
free	$I_3 - \sqrt{\quad}$	$\hat{\omega}_3 + \overset{A \wedge B}{\omega}$
$\overset{A \wedge B}{\omega} = \overset{A \wedge B}{\omega}$	I_3	$\overset{A \wedge B}{\omega}$

Table 3.7.1



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Derivations: The inertia torque $\underline{T}_B$ of B in N is given by

$$\underline{T}_B = -\underline{\alpha}^B \cdot \underline{I}_B - \underline{\omega}^B \times \underline{I}_B \cdot \underline{\omega}^B \quad (12) \text{ (45)}$$

where $\underline{\alpha}^B$ and $\underline{\omega}^B$ are the angular acceleration and the angular velocity of B in N, and $\underline{I}_B$, the central inertia dyadic of B, can be expressed as

$$\underline{I}_B = K(\underline{\gamma}\underline{\gamma} + \underline{\delta}\underline{\delta}) + J\underline{\beta}\underline{\beta} \quad (12)$$

$$= K\underline{U} + (J-K)\underline{\beta}\underline{\beta} \quad (13)$$

Here K is the transverse central moment of inertia of B, $\underline{\gamma}$ and $\underline{\delta}$ are unit vectors such that $\underline{\beta}$, $\underline{\gamma}$, and $\underline{\delta}$ are mutually perpendicular, and $\underline{U}$ is the unit dyadic. Hence

$$\underline{\alpha}^B \cdot \underline{I}_B = K\underline{\alpha}^B + (J-K)\underline{\alpha}^B \cdot \underline{\beta}\underline{\beta} \quad (14) \text{ (45)}$$

and

$$\underline{\omega}^B \times \underline{I}_B \cdot \underline{\omega}^B = (J-K)\underline{\omega}^B \times \underline{\beta}\underline{\beta} \cdot \underline{\omega}^B \quad (15) \text{ (45)}$$

from which it follows that

$$\underline{T}_B \cdot \underline{\beta} = -J\underline{\alpha}^B \cdot \underline{\beta} = {}^A\omega_{u_4}^B \cdot \underline{\beta} \quad (16) \text{ (45)}$$

Since by D'Alembert's Principle

$$\underline{T}_B \cdot \underline{\beta} + \underline{M}^{A/B} \cdot \underline{\beta} = 0 \quad (17) \text{ (45)}$$

or thus has

$$J\underline{\alpha}^B \cdot \underline{\beta} = \underline{M}^{A/B} \cdot \underline{\beta} \quad (18) \text{ (45)}$$

Now,

$$\underline{\alpha}^B = \frac{d\underline{\omega}^B}{dt} = \frac{d}{dt}(\underline{\omega}^A + \underline{\omega}^B \underline{\beta}) \quad (1.20.1) \quad (1.16.1)$$

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$$= \underbrace{{}^N \underline{\alpha}^A}_{(1.20.1)} + \underbrace{{}^A \dot{\omega}^B \underline{\beta}}_{(1.11.9)} + \underbrace{{}^A \omega^B \underbrace{{}^N \underline{\omega}^A}_{(1.11.9)} \times \underline{\beta}}_{(19) (18)}$$

Consequently,

$$\int \underbrace{({}^N \underline{\alpha}^A \cdot \underline{\beta}}_{(19)} + \underbrace{{}^A \dot{\omega}^B}_{(18)}) = \underline{\beta} \cdot \underline{M}^{A/B} \quad (20) (19)$$

and, since

$$\underbrace{{}^N \underline{\alpha}^A \cdot \underline{\beta}}_{(21)} = \dot{\omega}_1 \underline{\beta}_1 + \dot{\omega}_2 \underline{\beta}_2 + \dot{\omega}_3 \underline{\beta}_3 \quad (21) (20)$$

$$\underbrace{{}^N \underline{\alpha}^A}_{(22)} = \frac{d \underbrace{{}^N \underline{\omega}^A}_{(22)}}{dt} = \frac{d}{dt} \omega_i \underline{a}_i = \dot{\omega}_i \underline{a}_i + \underbrace{\left(\omega_i \frac{d \underline{a}_i}{dt} \right)}_{(2)} = \omega_i \frac{d \underline{a}_i}{dt} + \omega_i \underbrace{{}^P \underline{\omega}^A \times \underline{a}_i}_{(2)} = \omega_i \omega_j \underline{a}_j \times \underline{a}_i = \omega_i \omega_j \epsilon_{ijk} \underline{a}_k = (\omega_i \omega_j - \omega_j \omega_i) \underline{a}_k$$

one can write

$$\int (\dot{\omega}_1 \underline{\beta}_1 + \dot{\omega}_2 \underline{\beta}_2 + \dot{\omega}_3 \underline{\beta}_3 + \underbrace{{}^A \dot{\omega}^B}_{(20)}) = \underline{\beta} \cdot \underline{M}^{A/B} \quad (22) (21)$$

$\underline{T}_R$, the inertia torque of a rigid body R that has the same mass distribution as G, but moves like A, is given by

$$\underline{T}_R = - \underbrace{{}^N \underline{\alpha}^A}_{(3.6.1)} \cdot \underline{I}_G - \underbrace{{}^N \underline{\omega}^A}_{(3.6.2)} \times \underline{I}_G \cdot \underbrace{{}^N \underline{\omega}^A}_{(3.6.5)} \quad (23) (22)$$

Hence, $\underline{T}_G$, the inertia torque of G in N, can be expressed with the aid of Eqs. (3.6.2) and (3.6.5) as

$$\underline{T}_G = - \underbrace{{}^N \underline{\alpha}^A}_{(3.6.1)} \cdot \underline{I}_G - \underbrace{{}^N \underline{\omega}^A}_{(3.6.2)} \times \underline{I}_G \cdot \underbrace{{}^N \underline{\omega}^A}_{(3.6.5)} - \int \underbrace{({}^A \dot{\omega}^B \underline{\beta}}_{(3.6.2)} + \underbrace{{}^A \omega^B \underbrace{{}^N \underline{\omega}^A}_{(3.6.5)} \times \underline{\beta}}_{(3.6.5)})}_{(24)} \quad (24) (23)$$

Carrying out the operations indicated in Eq. (24) after noting that

$$\underline{I}_G = I_1 \underline{a}_1 \underline{a}_1 + I_2 \underline{a}_2 \underline{a}_2 + I_3 \underline{a}_3 \underline{a}_3 \quad (25) (24)$$

and then appealing to D'Alembert's principle to write

$$\underline{T}_G + \underline{M} = 0 \quad (26) (25)$$

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one arrives at Eqs. (6) - (8); and solution of these equations for $\omega_1, \omega_2, \omega_3$, followed by substitution into Eq. (22), produces Eq. (5).

Suppose now that $\beta = \underline{a}_3$, so that [see Eq. (3)]

$$\beta_1 = \beta_2 = 0, \quad \beta_3 = 1 \quad (27)$$

Then

$${}^{A,B} \dot{\omega} \left(1 - \frac{J}{I_3}\right) = \frac{\beta \cdot M^{A/B}}{J} + \frac{I_2 - I_1}{I_3} \omega_1 \omega_2 - \frac{M_3}{I_3} \quad (28)$$

and

$$I_1 \dot{\omega}_1 = (I_2 - I_3) \omega_2 \omega_3 - J {}^{A,B} \dot{\omega} \omega_2 + M_1 \quad (29)$$

$$I_2 \dot{\omega}_2 = (I_3 - I_1) \omega_3 \omega_1 + J {}^{A,B} \dot{\omega} \omega_1 + M_2 \quad (30)$$

$$I_3 \dot{\omega}_3 = (I_1 - I_2) \omega_1 \omega_2 - J {}^{A,B} \dot{\omega} + M_3 \quad (31)$$

If B is completely free to rotate relative to A, that is, if $\beta \cdot M^{A/B} = 0$, then

$${}^{A,B} \dot{\omega} = \frac{I_2 - I_1}{I_3 - J} \omega_1 \omega_2 - \frac{M_3}{I_3 - J} \quad (32)$$

so that

$$I_3 \dot{\omega}_3 = (I_1 - I_2) \omega_1 \omega_2 - \frac{J(I_2 - I_1)}{I_3 - J} \omega_1 \omega_2 + \frac{J M_3}{I_3 - J} + M_3 \quad (33)$$

or, equivalently,

$$(I_3 - J) \dot{\omega}_3 = (I_1 - I_2) \omega_1 \omega_2 + M_3 \quad (34)$$

$$I_3 = I_2 - J$$

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If B is completely free to rotate relative to A , that is, if $\beta, M^{A/B} = 0$, then [see Eqs. (27)]

$$\dot{\omega}_3 + {}^A\dot{\omega}^B = 0 \quad (32)$$

which implies that

$$\dot{\omega}_3 + {}^A\dot{\omega}^B = \hat{\omega}_3 + {}^A\hat{\omega}^B \quad (33)$$

where $\hat{\omega}_3$ and ${}^A\hat{\omega}^B$ are the initial values of ω_3 and ${}^A\omega^B$, respectively. Hence

$$I_1 \dot{\omega}_1 = (I_2 - I_3) \omega_2 \omega_3 - J(\hat{\omega}_3 + {}^A\hat{\omega}^B - \omega_3) \omega_2 + M_1 \quad (29) \quad (33)$$

$$= (I_2 - I_3 + J) \omega_2 \omega_3 - J(\hat{\omega}_3 + {}^A\hat{\omega}^B) + M_1 \quad (34)$$

Similarly, from Eqs. (30) and (33),

$$I_2 \dot{\omega}_2 = (I_3 - J - I_1) \omega_3 \omega_1 + J(\hat{\omega}_3 + {}^A\hat{\omega}^B) + M_2 \quad (35)$$

Furthermore,

$${}^A\dot{\omega}^B = \frac{I_2 - I_1}{I_3 - J} \omega_1 \omega_2 - \frac{M_3}{I_3 - J} \quad (28) \quad (36)$$

and substitution into Eq. (31) yields

$$(I_3 - J) \dot{\omega}_3 = (I_1 - I_2) \omega_1 \omega_2 + M_3 \quad (37)$$

Replacing $I_3 - J$ with $\bar{I}_3$ and $\hat{\omega}_3 + {}^A\hat{\omega}^B$ with σ in Eqs. (34), (35), and (37), one arrives at Eqs. (9), (10), and (11) respectively. On the other hand, if

If B is made to rotate relative to A with the constant angular speed ${}^A\hat{\omega}^B$, then Eqs. (29) - (31)

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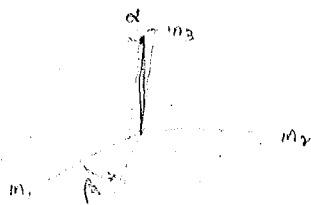
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become Eqs. (9) - (11) when I_3 is replaced with $\bar{I}_3$, $^{A\Delta P}$ is replaced with σ , and $^{A\Delta B}$ is set equal to zero. Thus Eqs. (9) - (11) apply in both cases, provided $\bar{I}_3$ and σ be interpreted in accordance with Table 3.7.1



1. Trends if all else remains the same: as $\mu = \frac{rm}{rm+mb} \downarrow$ angular velocity of body $\downarrow$ as well as that of the rotor. $rm \left(\frac{1-\mu}{\mu} \right) = mb$ for fixed rm as $\mu \downarrow$ $mb \uparrow \therefore$ effect of rotor becomes smaller. Hence the induced $|\omega|$ of body $\downarrow$ as expected.

2. for fixed μ $\frac{mb}{rm} = \frac{1-\mu}{\mu}$. Thus as the rotor mass changes for fixed μ the body mass changes also. Hence we get no effect here. Motion here is still unstable as ω increases as t increases.

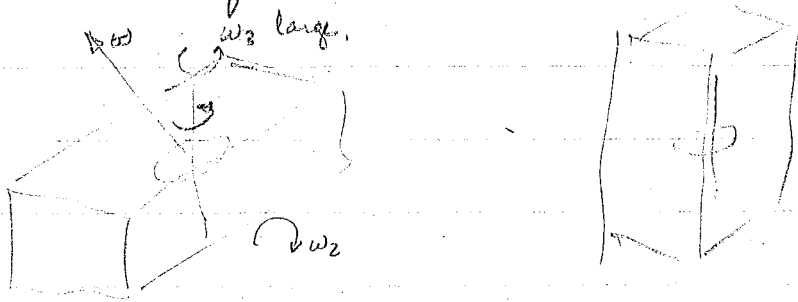
also the value of ω_3 remains negative for longer periods of time. Motion is quite unstable.

3. if we only change the T/J1 ratio and keep it very much below 1 then the value of $|\omega| \downarrow$ as $T/J1 \downarrow$ and the order of magnitudes for $|\omega| \downarrow$ drastically. The ω_3 component decreases then starts to increase. ω_3 component is affected more than others because rotor axis is almost aligned with m_3 axis. For very low values of $T/J1$ body rotates very slowly and $|\omega|$ is very small as expected.

4#5 As α increases the ω 's affected as ω_1 and ω_2 since the rotor axis approaches the m_1 - m_2 plane. This is noticed by the change in sign of ω_1 & ω_2 . As α increases the largest effect is on ω_1 , since large changes in α means that the rotor axis has a major component of itself $\parallel$ to the m_1 axis. We note that the trends in the results show that the axis upon which the major component of the rotor axis lies will always have a positive component of ω associated with it. This is also true with β changes. Here the rotor's axis makes its major projection on the m_3 axis when $\beta = 90^\circ$ hence sign of $\omega_3 = \text{sign of } \omega^B$.

6. If we keep everything else fixed and change the length of the side l_2 only we note that when $l_2 \uparrow$ to 1. Each value of ω_i slows down & speeds up in cycles with max. of speed up for each cycle increasing with time. ω_2 component changes direction cyclicly. When $l_2 > 1$ ω_1 slows down & speeds up with larger periods.

as C gets larger that the motion is more stable & much less in magnitude than that of b getting larger. Reason. since C is generally in direction of axis is takes much less to move body B . In case where B is much larger, B is $\perp$ generally to axis of rotor $\therefore$ must increase ω in order to move body to stay in equilibrium



Instabilities die out quicker in case where C is larger

d. as $d \downarrow$, ω_3 decreases & effect of rotor decreases.

(if direction of axis is kept fixed)

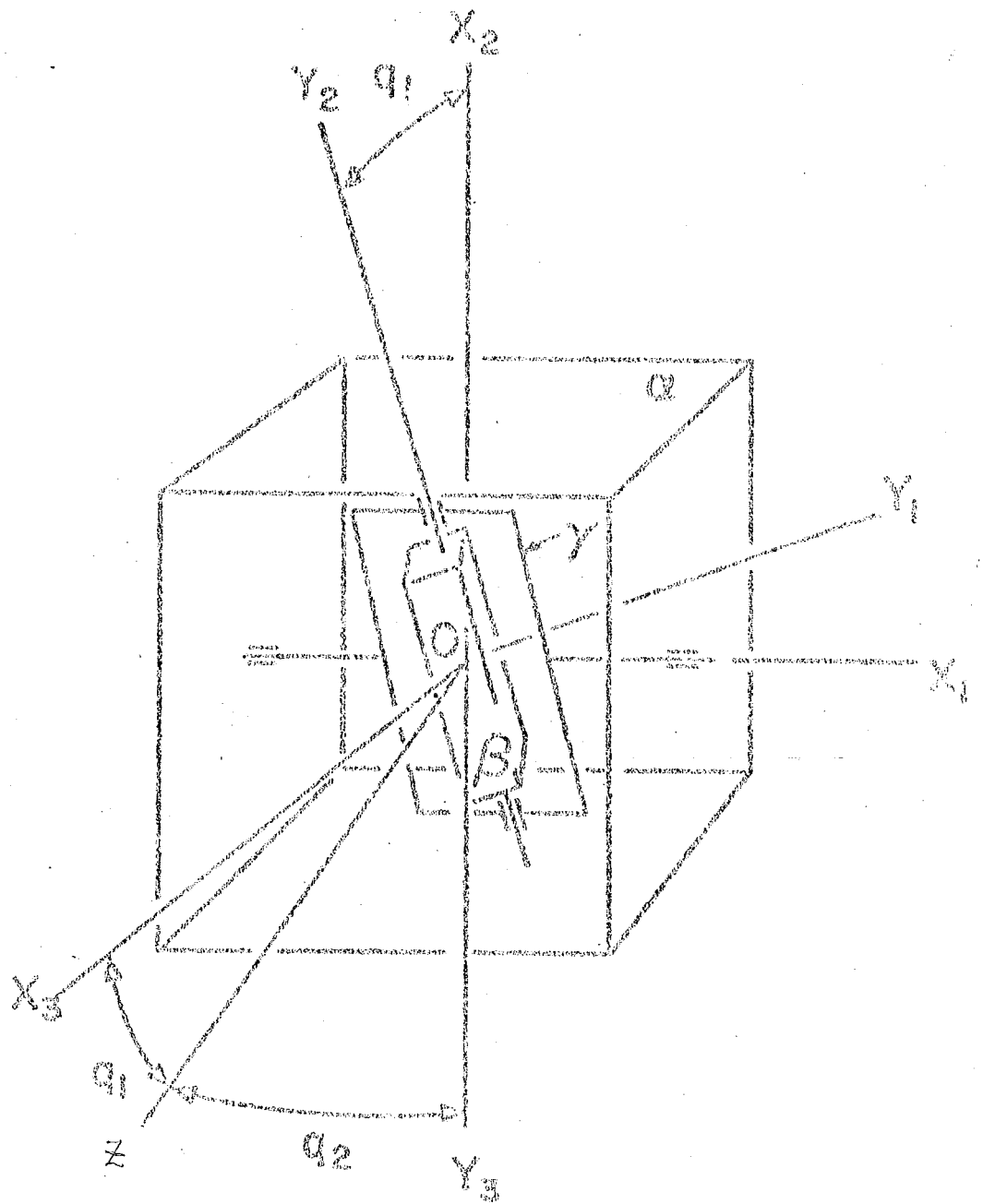
Thus for a stable system, make d small and elongate C_x over the other 2.

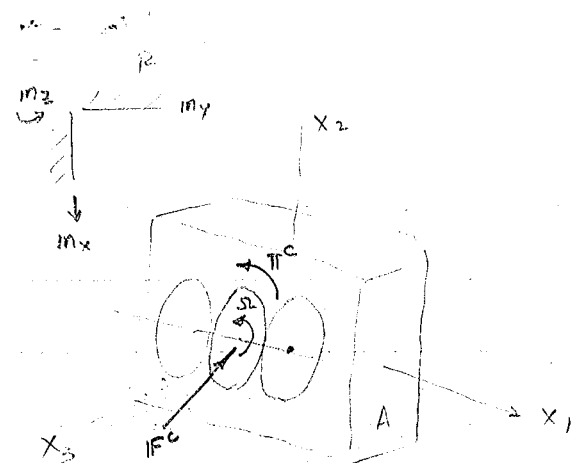
decrease μ and T/J_1

The attached sketch shows a system S composed of two rigid bodies, α and β , which are connected to each other by means of a light global γ . Point O is the mass center of α as well as β ; X_1, X_2, X_3 are principal axes of α ; Y_1, Y_2, Y_3 are principal axes of β ; and γ can rotate relative to α and β only about X_1 and Y_2 , respectively, so that the relative orientation of α and β depends solely on the angles q_1 and q_2 . The central principal moments of inertia of α and β have the values A_1, A_2, A_3 and B_1, B_2, B_3 , respectively.

When $q_1 = q_2 = 0$, the central principal axes of S coincide with those of α and β . Letting Z be the central principal axis of S that coincides with X_3 and Y_3 when $q_1 = q_2 = 0$, and taking $A_1 = 4$, $A_2 = 2$, $A_3 = 3$, $B_1 = 1.5$, $B_2 = 3 \text{ kg cm}^2$, find a value of B_3 such that the moment of inertia of S about Z has a (local) minimum value when $q_1 = q_2 = 0$.

Note: $f(x,y)$ has a local minimum at $x = y = 0$ if $f_x = f_y = 0$, $f_{xx} > 0$, and $f_{xx}f_{yy} - (f_{xy})^2 > 0$, where subscripts denote partial differentiations and all derivatives are evaluated at $x = y = 0$.





this is a 7 degree of freedom system.

6 to describe motion of body A in R

1 to describe motion of body C in A

motions of B1 and B2 are known since their motion depend on motion of C

Assume that $\pi^C = T_1 m_1 + T_2 m_2 + T_3 m_3$

$$F^C = F_1 m_1 + F_2 m_2 + F_3 m_3$$

$${}^R \omega^A = \omega_1 m_1 + \omega_2 m_2 + \omega_3 m_3 \\ = u_4 m_1 + u_5 m_2 + u_6 m_3$$

Assume that C* and A* are same and rotors have radii R

$$u_1 m_1 + u_2 m_2 + u_3 m_3 = V^{A*} = v_1 m_1 + v_2 m_2 + v_3 m_3 = V^{C*}$$

$$u_1 m_1 + u_2 m_2 + u_3 m_3 = V^{C*} = v_1 m_1 + v_2 m_2 + v_3 m_3$$

$$V^{C*} + 2r[u_6 m_2 - u_5 m_3] = V^{B_1*} = V^{C*} + 2r[\omega_3 m_2 - \omega_2 m_3]$$

$$V^{C*} + 2r[-u_6 m_2 + u_5 m_3] = V^{B_2*} = V^{C*} + 2r[-\omega_3 m_2 + \omega_2 m_3]$$

$${}^A \omega^C = \dot{\pi}^C = u_7 m_3$$

$${}^A \omega^C = \dot{\pi}^C = u_7 m_3 \\ = u_7 m_3$$

$${}^A \omega^{B_1} = {}^A \omega^{B_2} = -\dot{\pi}^C = -u_7 m_3 \\ = -u_7 m_3$$

$$V^{B_1*} = V^{C*} + 2r \dot{\pi}^C$$

$$V^{B_1*} = V^{C*} + 2r \frac{d\pi^C}{dt}$$

$$\frac{d\pi^C}{dt} = \frac{d\pi^C}{dt} + \omega^A \times \pi^C$$

$$= V^{C*} + 2r \omega^A \times \pi^C = V^{C*} + 2r[-\omega_2 m_3 + \omega_3 m_2]$$

$$V^{B_2*} = V^{C*} - 2r \omega^A \times \pi^C = V^{C*} + 2r[\omega_2 m_3 - \omega_3 m_2]$$

Now let the generalized coordinates of entire system be $v_i (i=1,3)$, $\omega_i (i=1,2,3)$ and π

now u_1, u_2, u_3 equations of motion will decouple from rest.

$$r=1,2,3 \quad F_r = V_{u_r}^A \cdot F^A + V_{u_r}^C \cdot F^C + V_{u_r}^{B_1} \cdot F^{B_1} + V_{u_r}^{B_2} \cdot F^{B_2} + \omega_{u_r}^A \cdot \pi^A + \omega_{u_r}^C \cdot \pi^C + \omega_{u_r}^{B_1} \cdot \pi^{B_1} + \omega_{u_r}^{B_2} \cdot \pi^{B_2}$$

since only body & contact forces acting are gravitational forces $F_B^{B_1} = F_B^{B_2} = F_B^C = mg m_x$ $F_B^A = Mg m_x$

the contact forces between the rotors contribute nothing

$$F^A = Mg m_x - \sum p_i \quad F^{B_1} = F^{B_2} = mg m_x \quad F^C = mg m_x + \sum_i m_i$$

$$\pi^C = \pi^A \quad \pi^{B_1} = \pi^{B_2} = 0$$

$$u_i = v_i \quad (i=1,2,3)$$

$$u_i = \omega_{i-3} \quad (i=4,5,6)$$

$$u_i = \pi \quad i=7$$

$$F_1 = m_1 \cdot F^A + m_1 \cdot F^C + m_1 \cdot F^{B_1} + m_1 \cdot F^{B_2} + 0 + 0 + 0 + 0$$

since $\omega^{(A,C,B_1,B_2)} \neq f(u_1, u_2, u_3)$

$$F_2 = m_2 \cdot F^A + m_2 \cdot F^C + m_2 \cdot F^{B_1} + m_2 \cdot F^{B_2} + 0 + 0 + 0 + 0$$

$$F_3 = m_3 \cdot F^A + m_3 \cdot F^C + m_3 \cdot F^{B_1} + m_3 \cdot F^{B_2} + 0 + 0 + 0 + 0$$

$$F_4 = 0 + 0 + 0 + 0 + m_1 \cdot \pi^A + m_1 \cdot \pi^C + m_1 \cdot \pi^{B_1} + m_1 \cdot \pi^{B_2}$$

since $\omega^{(A,C,B_1,B_2)} \neq f(u_1, u_2, u_3)$

$$F_5 = 0 + 0 + 2r m_3 \cdot F^{B_1} + 2r m_3 \cdot F^{B_2} + m_2 \cdot \pi^A + m_2 \cdot \pi^C + m_2 \cdot \pi^{B_1} + m_2 \cdot \pi^{B_2}$$

$$F_6 = 0 + 0 + 2r m_2 \cdot F^{B_1} - 2r m_2 \cdot F^{B_2} + m_3 \cdot \pi^A + m_3 \cdot \pi^C + m_3 \cdot \pi^{B_1} + m_3 \cdot \pi^{B_2}$$



$$F_7 = 0 + 0 + 0 + 0 + 0 + m_3 \cdot \pi^c - m_3 \cdot \pi^{B_1} - m_3 \cdot \pi^{B_2} \quad \text{since } \omega_{u_7}^{(A, B_1, B_2)} \neq f_n \text{ of } u_7$$

Question : if motor exerts a force & a torque on C does it exert a reverse force & reverse torque to

$$A? \text{ Assume yes } \therefore \pi^A = -\pi^C \text{ and } IF^A = IF_B^A - IF_T^C \quad IF^C = IF_B^C + IF_T^C$$

$$\therefore F_1 = m_1 \cdot [IF_B^A + IF_B^C + 2IF^{B_1}] = m_1 \cdot m_X [(M+3m)g]$$

$$F_2 = m_2 \cdot [IF_B^A + IF_B^C + 2IF^{B_1}] = m_2 \cdot m_X [(M+3m)g]$$

$$F_3 = m_3 \cdot [IF_B^A + IF_B^C + 2IF^{B_1}] = m_3 \cdot m_X [(M+3m)g]$$

$$F_4 = 0$$

$$F_5 = 0$$

$$F_6 = 0$$

$$F_7 = T_3$$

$$\text{To obtain } F_r^* = \omega_{u_r}^{A^*} \cdot IF^{A^*} + \omega_{u_r}^{C^*} \cdot IF^{C^*} + \omega_{u_r}^{B_1^*} \cdot IF^{B_1^*} + \omega_{u_r}^{B_2^*} \cdot IF^{B_2^*} + \omega_{u_r}^{A^*} \cdot \pi^{A^*} + \omega_{u_r}^{C^*} \cdot \pi^{C^*} + \omega_{u_r}^{B_1^*} \cdot \pi^{B_1^*} + \omega_{u_r}^{B_2^*} \cdot \pi^{B_2^*}$$

$$F_1^* = m_1 \cdot (-M a_1^{A^*}) + m_1 \cdot (-m a_1^{C^*}) + m_1 \cdot (-m a_1^{B_1^*}) + m_1 \cdot (-m a_1^{B_2^*}) + 0 + 0 + 0 + 0 \quad \text{since } \omega_{u_1}^{(A, C, B_1, B_2)} \neq f_n(u_1)$$

$$m_1 \cdot a_1^{A^*} = m_1 \frac{d^2 u_1}{dt^2} = m_1 \frac{d}{dt} \left(\frac{d u_1}{dt} \right) = m_1 \dot{u}_1 + m_1 [u_4 u_3 - u_6 u_2] \times [u_1 m_1 + u_2 m_2 + u_3 m_3]$$

$$m_1 \cdot a_1^{C^*} = m_1 \cdot \frac{d^2 u_1}{dt^2} = m_1 \cdot \frac{d}{dt} \left(\frac{d u_1}{dt} \right) = m_1 \dot{u}_1 + u_5 u_3 - u_6 u_2$$

$$m_1 \cdot a_1^{B_1^*} = m_1 \cdot \frac{d^2 u_1}{dt^2} = m_1 \cdot \frac{d}{dt} \left(\frac{d u_1}{dt} \right) = m_1 \dot{u}_1 - 2r [u_5^2 + u_6^2] + [u_5 u_3 - u_6 u_2]$$

$$m_1 \cdot a_1^{B_2^*} = m_1 \cdot \frac{d^2 u_1}{dt^2} = m_1 \cdot \frac{d}{dt} \left(\frac{d u_1}{dt} \right) = m_1 \dot{u}_1 + 2r [u_5^2 + u_6^2] + [u_5 u_3 - u_6 u_2]$$

$$F_1^* = -(M+3m) \dot{u}_1 - (M+m)(u_5 u_3 - u_6 u_2) = -(M+3m) [\dot{u}_1 + u_5 u_3 - u_6 u_2] \quad \checkmark$$

$$F_2^* = m_2 \cdot (-M a_1^{A^*} - m a_1^{C^*} - m a_1^{B_1^*} - m a_1^{B_2^*}) \quad \text{since } \omega_{u_2}^{(A, C, B_1, B_2)} \neq f_n(u_2)$$

$$= -(M+3m) \dot{u}_2 - (M+m)(u_6 u_1 - u_4 u_3) = -(M+3m) [\dot{u}_2 + u_6 u_1 - u_4 u_3] \quad \checkmark$$

$$F_3^* = m_3 (-M a_1^{A^*} - m a_1^{C^*} - m a_1^{B_1^*} - m a_1^{B_2^*})$$

$$F_3^* = -(M+3m) \dot{u}_3 - (M+m)(u_4 u_2 - u_5 u_1) = -(M+3m) [\dot{u}_3 + u_4 u_2 - u_5 u_1]$$

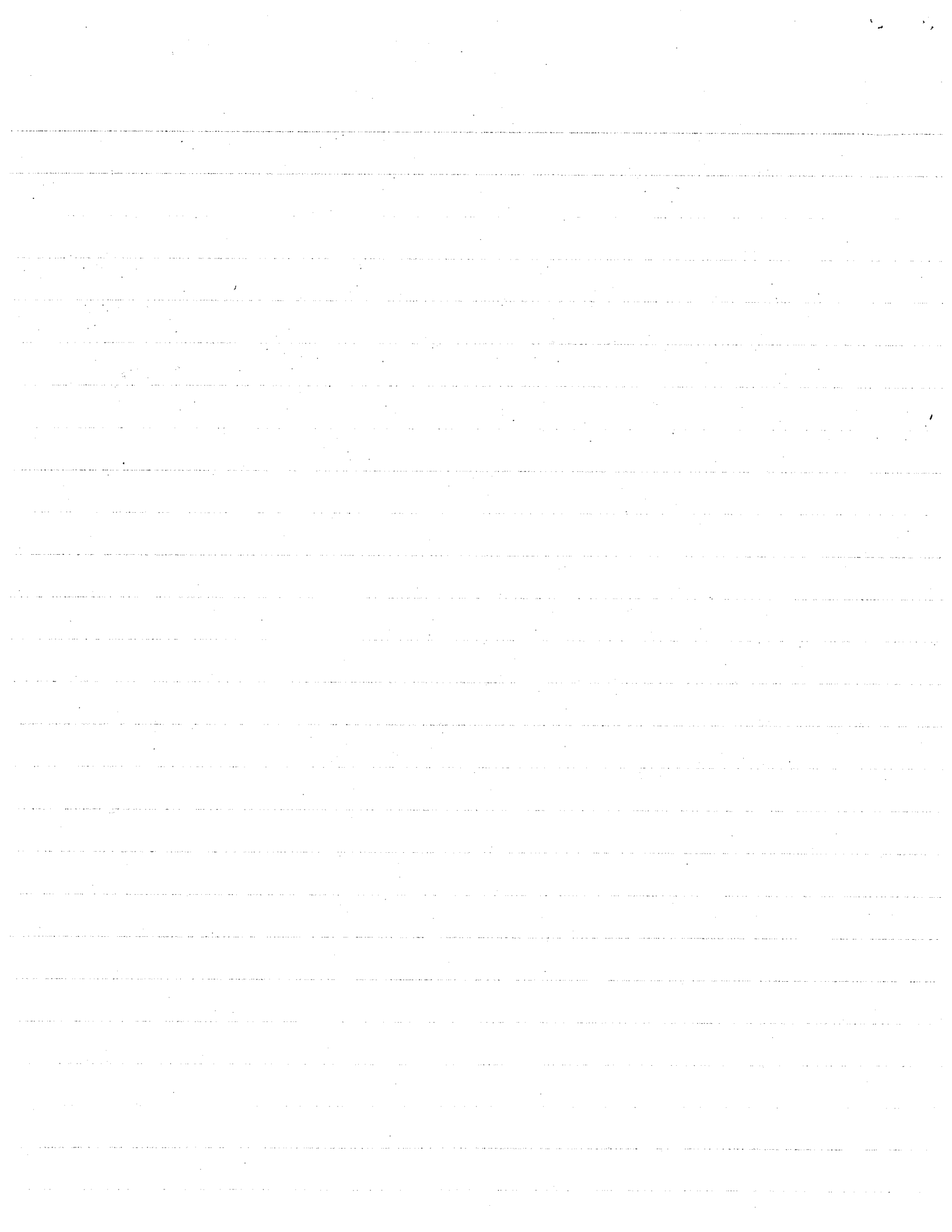
$$F_4^* = 0 + 0 + 0 + 0 + m_1 \cdot \pi^{A^*} + m_1 \cdot \pi^{C^*} + m_1 \cdot \pi^{B_1^*} + m_1 \cdot \pi^{B_2^*} \quad \text{since } \omega_{u_4}^{(A, B_1, C, B_2)} \neq f(u_4)$$

$$F_4^* = m_1 \cdot [\pi^{A^*} + \pi^{C^*} + \pi^{B_1^*} + \pi^{B_2^*}]$$

$$F_5^* = 0 + 0 - 2r m_3 [-m a_1^{B_1^*}] + 2r m_3 [-m a_1^{B_2^*}] + m_2 \cdot [\pi^{A^*} + \pi^{C^*} + \pi^{B_1^*} + \pi^{B_2^*}]$$

$$= +2r m [m_3 (a_1^{B_1^*} - a_1^{B_2^*})] + m_2 \cdot [\pi^{A^*} + \pi^{C^*} + \pi^{B_1^*} + \pi^{B_2^*}]$$

$$F_5^* = 2r m [-4r \dot{u}_5 + 4r u_4 u_6] + m_2 \cdot [\pi^{A^*} + \pi^{C^*} + \pi^{B_1^*} + \pi^{B_2^*}]$$



$$a_1^{B_1} - a_2^{B_2} = 4r \frac{d\omega^A}{dt} \times m_1 + 4r \left[\left(\frac{R}{\omega^A} m_1 \right) \frac{R}{\omega^A} - \omega^A m_1 \right] \quad \text{CHECK THIS}$$

$$F_6^* = 0 + 0 + 2r m_2 [-m a_1^{B_1}] - 2r m_2 [-m a_1^{B_2}] + m_3 \cdot [\pi^{*A} + \pi^{*B} + \pi^{*C} + \pi^{*D}]$$

$$= -2rm [a_1^{B_1} - a_1^{B_2}] + m_3 \cdot [$$

$$F_6^* = -2rm [4r u_6 + 4r u_4 u_5] + m_3 \cdot [\pi^{*A} + \pi^{*B} + \pi^{*C} + \pi^{*D}]$$

$$F_7^* = 0 + 0 + 0 + 0 + 0 + m_3 \cdot \pi^{*C} - m_3 \cdot \pi^{*B_1} - m_3 \cdot \pi^{*B_2} \quad \text{derive } W^{(B_1, B_2, B_3)} \neq f(u_7) \text{ or } \omega^A \neq f(u_7)$$

Since C, B_1, B_2 have simple angular velocities in reference frame A let us rewrite

$$\pi^{*C} = -\frac{R}{dt} \mathbb{H} \dot{\omega}^C = -\frac{R}{dt} \mathbb{H} \dot{\omega}^C - \omega^A \times \mathbb{H} \dot{\omega}^C \neq \pi^{*C} + \mathbb{H} \dot{\omega}^C \times \omega^A = \pi^{*C} + (\mathbb{I} \dot{\omega}^C \cdot \omega^A) \times \omega^A$$

$$m_3 \cdot \pi^{*C} = m_3 \cdot \pi^{*C} + m_3 \cdot (\mathbb{I} \dot{\omega}^C \cdot \omega^A) \times \omega^A \quad ; \quad m_3 \cdot (\mathbb{I} \dot{\omega}^C \cdot \omega^A) \times \omega^A = m_3 \cdot \mathbb{I} \dot{\omega}^C \cdot \omega^A \times \omega^A = -J \dot{u}_7$$

$$= -J \dot{\Omega} + m_3 \cdot (\mathbb{I} \dot{\omega}^C \cdot \omega^A) \times \omega^A + m_3 \cdot (\mathbb{I} \dot{\omega}^C \cdot \omega^A) \times \omega^A \quad \mathbb{I} \dot{\omega}^C = \mathbb{I}^{B/B^*} \dot{\omega}^C = \mathbb{I}^{B/B^*}$$

$$= -J \dot{\Omega} + 0 + 0 \quad m_3 \cdot J \dot{\Omega} m_3 \times \omega^A$$

$$\pi^{*B_1} = \pi^{*B/B_1} + (\mathbb{I}^{B/B_1} \dot{\omega}^B) \times \omega^A = \pi^{*B/B_1} + (\mathbb{I}^{B/B_1} \cdot \omega^A) \times \omega^A + (\mathbb{I}^{B/B_1} \cdot \omega^B) \times \omega^A$$

$$m_3 \cdot \pi^{*B/B_1} = J \dot{\Omega} + 0 + 0 \quad -J \dot{\Omega} m_3$$

$$m_3 \cdot \pi^{*B/B_2} = J \dot{\Omega} \quad \text{by same token.}$$

$$\therefore \boxed{F_7^* = -3 J \dot{\Omega}}$$

$$\mathbb{I}^{C/C^*} \dot{\omega}^C = [J m_3 m_3 + K(m_1 m_1 + m_2 m_2)] \cdot [u_6 m_1 + u_5 m_2 + (u_6 + u_7)]$$

$$= [J(u_6 + u_7) m_3 + K(u_4 m_1 + u_5 m_2)] \cdot [u_6 m_1 + u_5 m_2 + u_7]$$

$$= J(u_6 + u_7) u_4 m_2 - J(u_6 + u_7) u_5 m_1 + K[u_5 u_4 m_3 + u_4 u_5 m_3 - u_4 u_6 m_2]$$

$$\pi^{*C} = -\frac{R}{dt} \mathbb{H} \dot{\omega}^C = -\frac{R}{dt} (\mathbb{I} \dot{\omega}^C \cdot \omega^A) = -\frac{R}{dt} (\mathbb{I} \dot{\omega}^C \cdot \omega^A) - \frac{R}{dt} (\mathbb{I} \cdot \omega^A)$$

$$= (\mathbb{I} \dot{\omega}^C \cdot \omega^A) \times \omega^A - \mathbb{I} \dot{\omega}^C \cdot \omega^A \times \omega^A + \mathbb{H} \dot{\omega}^C \times \omega^A$$

$$m_3 \cdot \pi^{*C} = -J m_3 \cdot \dot{\omega}^C + m_3 \cdot \pi^{*C} = -J \dot{u}_6 - J \dot{u}_7 = -J(\dot{u}_6 + \dot{u}_7)$$

$$m_3 \cdot \pi^{*B_1} = -J(\dot{u}_6 - \dot{u}_7) = m_3 \cdot \pi^{*B_2} \quad m_3 \cdot \pi^{*B_1} = (-\mathbb{I}^{B/B_1} \dot{\omega}^B \cdot \omega^A) \times \omega^A = -J \dot{u}_6 + J \dot{u}_7$$

$$F_7^* = -J(\dot{u}_6 + \dot{u}_7) + 2J(\dot{u}_6 - \dot{u}_7) = -3J \dot{u}_7 + J \dot{u}_6 \quad \boxed{F_7^* = -3J \dot{u}_7 + J \dot{u}_6}$$

$$V^{A*} = V^{C*} = u_1 m_1 + u_2 m_2 + u_3 m_3$$

$$R \omega^A = u_4 m_1 + u_5 m_2 + u_6 m_3$$

$$V^{B_1*} = V^{C*} + \frac{d}{dt} 2r m_1 = V^{C*} + 2r \frac{R}{\omega} \times m_1 = V^{C*} + 2r (u_6 m_2 - u_5 m_3) ; V^{B_2*} = V^{C*} - 2r \frac{R}{\omega} \times m_3 = V^{C*} - 2r (u_6 m_2 - u_5 m_3)$$

$$a^{B_1*} = \frac{dV^{B_1*}}{dt} = \frac{dV^{C*}}{dt} + \frac{R}{\omega} \times \frac{dV^{C*}}{dt} = \dot{u}_1 m_1 + \dot{u}_2 m_2 + \dot{u}_3 m_3 + 2r (u_6 \dot{m}_2 - u_5 \dot{m}_3) + \frac{R}{\omega} \times V^{C*} + \frac{R}{\omega} \times 2r \left(\frac{R}{\omega} \times m_1 \right) + 2r \left[\frac{(\omega^A m_1)}{u_4} \omega^A - \omega^{A^2} m_1 \right]$$

$$a^{B_2*} = \dot{u}_1 m_1 + \dot{u}_2 m_2 + \dot{u}_3 m_3 - 2r (u_6 \dot{m}_2 - u_5 \dot{m}_3) + \frac{R}{\omega} \times V^{C*} - 2r \frac{R}{\omega} \times \left(\frac{R}{\omega} \times m_1 \right)$$

$$a^{A*} = a^{C*} = \dot{u}_1 m_1 + \dot{u}_2 m_2 + \dot{u}_3 m_3 + \left[\frac{R}{\omega} \times V^{C*} - \left[(u_5 u_3 - u_6 u_2) m_1 + (u_6 u_1 - u_4 u_3) m_2 + (u_4 u_2 - u_5 u_1) m_3 \right] \right]$$

Now $\frac{d}{dt} [m_1 a^{A*}] + m \left[m_1 \cdot a^{C*} + m_1 \cdot a^{B_1*} + m_1 \cdot a^{B_2*} \right]$

$$= -M \dot{u}_1 - M (u_5 u_3 - u_6 u_2) - m \left[\dot{u}_1 + (u_5 u_3 - u_6 u_2) \right] - m \left[\dot{u}_1 + (u_5 u_3 - u_6 u_2) + 2r (u_4^2 - u_4^2 u_5^2 - u_6^2) \right]$$

$$- m \left[\dot{u}_1 + (u_5 u_3 - u_6 u_2) - 2r (u_4^2 - u_4^2 u_5^2 - u_6^2) \right]$$

$$= - (M + 3m) (\dot{u}_1 + u_5 u_3 - u_6 u_2) - m \left[2r (-u_5^2 - u_6^2) - 2r (-u_5^2 - u_6^2) \right]$$

$$= -M [\dot{u}_2 + (u_6 u_1 - u_4 u_3)] - m [\dot{u}_2 + (u_6 u_1 - u_4 u_3) + \dot{u}_2 + 2r \dot{u}_2 + (u_6 u_1 - u_4 u_3) + 2r u_4 u_5 + \dot{u}_2 - 2r \dot{u}_6 + (u_6 u_1 - u_4 u_3) - 2r u_4 u_5] = - (M + 3m) [\dot{u}_2 + (u_6 u_1 - u_4 u_3)]$$

$$- 2r m_3 \cdot \omega^{B_1*} + 2r m_3 \cdot \omega^{B_2*} = - 2r m_3 \cdot (-m a^{B_1*}) + 2r m_3 \cdot (-m a^{B_2*})$$

$$= 2r m [\dot{u}_3 + 2r \dot{u}_5 + (u_4 u_2 - u_5 u_1) + 2r u_4 u_6] - 2r m [\dot{u}_3 + 2r \dot{u}_5 + (u_4 u_2 - u_5 u_1) - 2r u_4 u_6] = 2r m [-4r \dot{u}_5 + 4r u_4 u_6]$$

$$- 2r m [m_2 \cdot a^{B_1*}] + 2r m [m_2 \cdot a^{B_2*}] = - 2r m [\dot{u}_2 + 2r \dot{u}_6 + (u_6 u_1 - u_4 u_3) + 2r u_4 u_5] + 2r m [\dot{u}_2 - 2r \dot{u}_6 + (u_6 u_1 - u_4 u_3) - 2r u_4 u_5] = - 2r m [4r \dot{u}_6 + 4r u_4 u_5]$$

Now K_1, K_2, K_3 are the moments of inertia of cylinder about x_1, x_2, x_3

$$\Pi^A + \Pi^{B_1} + \Pi^C + \Pi^{B_2} = - \frac{d}{dt} \left[\Pi^A \omega^A + \Pi^C \omega^C + \Pi^{B_1} \omega^{B_1} + \Pi^{B_2} \omega^{B_2} \right]$$

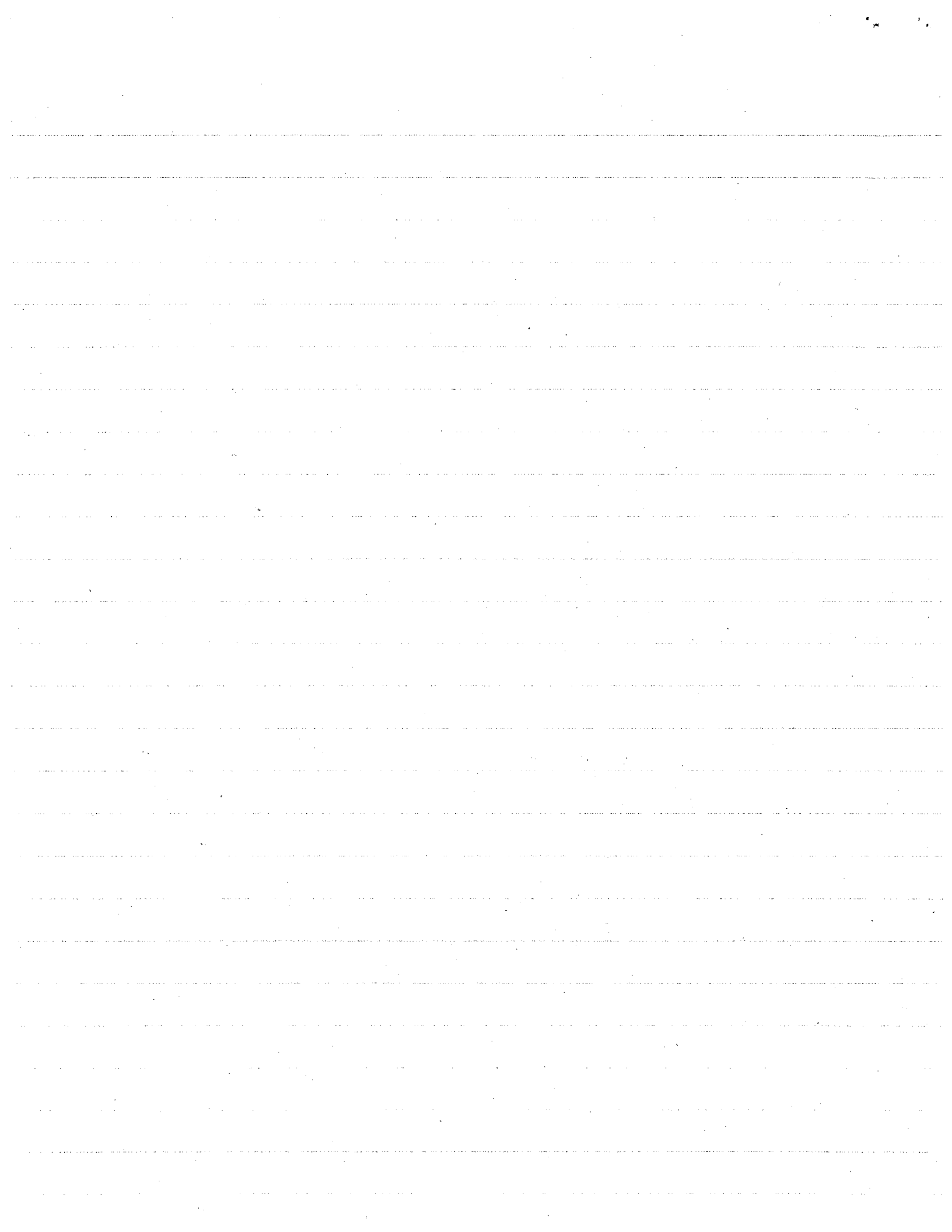
$$= - \frac{d}{dt} \left[\Pi^A \omega^A + \Pi^C \omega^C + \Pi^{B_1} \omega^{B_1} + \Pi^{B_2} \omega^{B_2} \right]$$

$$\left[(\Pi^A \omega^A + \Pi^C \omega^C + \Pi^{B_1} \omega^{B_1} + \Pi^{B_2} \omega^{B_2}) \omega^A + \Pi^C \omega^C + \Pi^{B_1} \omega^{B_1} + \Pi^{B_2} \omega^{B_2} \right]$$

$$\text{Now } \Pi^{B/A*} = \Pi^{B/B_1} + \Pi^{B_1/A*}$$

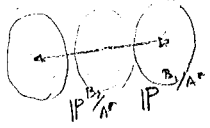
$$\text{Now } \frac{A}{\omega} = - \frac{A}{\omega} \omega^B = - \frac{A}{\omega} \omega^B \quad \& \quad \Pi^C \omega^C = \Pi^{B/B_2} \omega^{B_2} = \Pi^{B/B_2} \omega^{B_2}$$

$$\therefore (\Pi^A \omega^A + \Pi^C \omega^C + \Pi^{B_1/A*} \omega^{A*} + \Pi^{B_2/A*} \omega^{A*} + \Pi^{B_2/A*} \omega^{A*}) = \Pi^{(A+C+B_1+B_2)/A*} - \Pi^{B_1/A*} - \Pi^{B_2/A*}$$



$$I^C \omega^C + 2I^{B_1} \omega^C - \omega^C$$

$$\pi^A + \pi^B + \pi^C + \pi^D = -\frac{d}{dt} \left\{ \begin{aligned} & I^{D/D^*} \omega^A - (I^{B_1/A^*} + I^{B_2/A^*}) \cdot \omega^A - I^{C/A^*} \omega^C \\ & R \pi^D - \pi^C + \frac{d}{dt} (\theta r^2 m [m_2 m_2 + m_3 m_3] \cdot \omega^A) + \omega^A \times \frac{A}{|A|} \frac{C}{C} \end{aligned} \right\}$$



$$\begin{aligned} I^{B_1/A^*} &= m(P^2 U - |P|^2) & P^{B_1/A^*} &= 2rm_1 & |P|^2 &= 4r^2 \\ I^{B_2/A^*} &= m(4r^2 U - 4r^2 m_1 m_1) = m \cdot 4r^2 (m_2 m_2 + m_3 m_3) \\ I^{B_2/A^*} &= m(4r^2 U - 4r^2 m_1 m_3) & P^{B_2/A^*} &= -2rm_1 & |P|^2 &= 4r^2 \end{aligned}$$

only non-zero terms are $m_2 m_2$ and $m_3 m_3$

$$= R \pi^D - \pi^C + \theta r^2 m \frac{d}{dt} [u_5 m_2 + u_6 m_3] = R \pi^D - \pi^C + \theta r^2 m \left\{ (\dot{u}_5 m_2 + \dot{u}_6 m_3) + \omega^A \times \frac{A}{|A|} \frac{C}{C} \right\}$$

$$\text{now } \omega = \omega^A - u_4 m_1$$

$$\omega^A \times \omega = (\omega^A \times \omega^A) - \omega^A \times u_4 m_1 = -(u_4 m_1 + u_5 m_2 + u_6 m_3) \times u_4 m_1$$

$$= +u_5 u_4 m_3 - u_6 u_4 m_2$$

$$\therefore \pi^A + \pi^B + \pi^C + \pi^D = R \pi^D - \pi^C + \theta r^2 m \left[(\dot{u}_5 + u_6 u_4) m_2 + (\dot{u}_6 + u_5 u_4) m_3 \right] + \omega^A \times \frac{A}{|A|} \frac{C}{C}$$

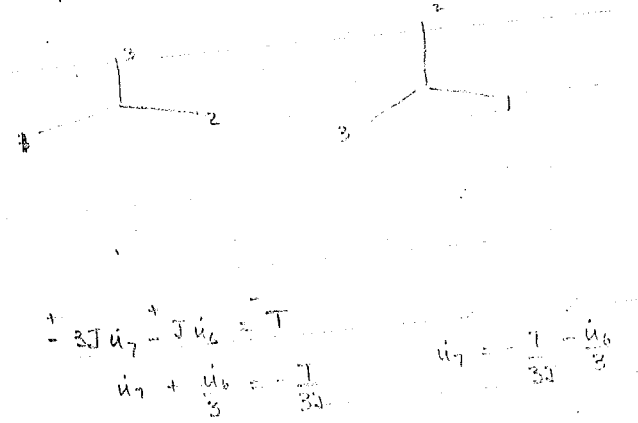
$$\begin{aligned} F_4^* &= m_1 \cdot \left[\begin{aligned} & m_1 \cdot R \pi^D - m_1 \cdot \pi^C + m_1 \cdot \omega^A \times \frac{A}{|A|} \frac{C}{C} \\ & F_5^* = -8r^2 m [\dot{u}_5 + u_4 u_6] + m_2 \cdot R \pi^D - m_2 \cdot \pi^C + 8r^2 m [\dot{u}_5 - u_6 u_4] \\ & F_6^* = -8r^2 m [\dot{u}_6 + u_4 u_5] + m_3 \cdot R \pi^D - m_3 \cdot \pi^C + 8r^2 m [\dot{u}_6 + u_5 u_4] \end{aligned} \right] \\ &= m_2 \cdot \left(R \pi^D - \pi^C \right) + m_2 \cdot \omega^A \times \frac{A}{|A|} \frac{C}{C} \\ &= m_3 \cdot \left(R \pi^D - \pi^C \right) + m_3 \cdot \omega^A \times \frac{A}{|A|} \frac{C}{C} \end{aligned}$$

$$\begin{aligned} R \pi^D &= (I^{D/D^*} \omega^A) \times \omega^A - I^{D/D^*} \omega^A \\ \text{now } R \pi^D &= \frac{d}{dt} \left(\frac{R \omega^A}{\omega^A} \right) = \frac{d}{dt} \left(\frac{R \omega^A}{\omega^A} \right) = \dot{u}_4 m_1 + \dot{u}_5 m_2 + \dot{u}_6 m_3 \\ I^{D/D^*} \omega^A &= K_1 u_4 m_1 + K_2 u_5 m_2 + K_3 u_6 m_3 \end{aligned}$$

$$\begin{aligned} \text{Now } R \pi^D &= (I^{D/D^*} \omega^A) \times \omega^A = \left[(K_2 - K_3) u_5 u_6 - K_1 \dot{u}_4 \right] m_1 + \left[(K_3 - K_1) u_4 u_6 - K_2 \dot{u}_5 \right] m_2 + \\ & \quad \left[(K_1 - K_2) u_4 u_5 - K_3 \dot{u}_6 \right] m_3 \end{aligned}$$

$$A \pi^C = \left(\frac{I^{C/A^*} \omega^C}{u_7 m_3} \right) \times \omega^C - I^{C/A^*} \omega^C = -J \dot{u}_7 m_3 \parallel \frac{R \omega^A \times \frac{A}{|A|} \frac{C}{C}}{\omega^A \times \frac{A}{|A|} \frac{C}{C}} = \frac{R \omega^A \times \frac{A}{|A|} \frac{C}{C}}{\omega^A \times \frac{A}{|A|} \frac{C}{C}} [J u_7 m_3] = J u_7 (u_5 m_1 -$$

$$\begin{aligned} F_4^* &= (K_2 - K_3) u_5 u_6 - K_1 \dot{u}_4 + J u_7 u_5 \\ F_5^* &= (K_3 - K_1) u_4 u_6 - K_2 \dot{u}_5 - J u_7 u_4 \\ F_6^* &= (K_1 - K_2) u_4 u_5 - K_3 \dot{u}_6 + J u_7 u_3 \\ F_7^* &= -3J \dot{u}_7 + J \dot{u}_6 \end{aligned}$$





$$\dot{u}_4 = \frac{J}{K_1} u_7 u_5 + \frac{K_2 - K_3}{K_1} u_5 u_6 = \left[\frac{J}{K_1} \Omega \omega_2 + \frac{K_2 - K_3}{K_1} \omega_2 \omega_3 = \dot{\omega}_1 \right]$$

$$\dot{u}_5 = -\frac{J u_7 u_4}{K_2} + \frac{K_3 - K_1}{K_2} u_4 u_6 = \left[-\frac{J}{K_2} \Omega \omega_1 + \frac{K_3 - K_1}{K_2} \omega_1 \omega_3 = \dot{\omega}_2 \right]$$

$$\dot{u}_6 = \frac{J \dot{u}_7}{3K_3} + \frac{K_1 - K_2}{3K_3} u_4 u_5 \Rightarrow \frac{J \dot{u}_6}{3K_3} + \frac{J_3}{3K_3} + \frac{K_1 - K_2}{3K_3} u_4 u_5 \Rightarrow \dot{u}_6 = \left[\frac{J_3}{3K_3} + \frac{K_1 - K_2}{3K_3} u_4 u_5 \right] / \left(1 - \frac{J}{3K_3} \right)$$

$$\dot{u}_3 = \left[\frac{J_3}{3K_3} + \frac{K_1 - K_2}{3K_3} \omega_1 \omega_2 \right] / \left(1 - \frac{J}{3K_3} \right)$$

$$\dot{u}_7 = \frac{\dot{u}_6}{3} + \frac{J_3}{3J} = \frac{J \dot{u}_7}{3K_3} + \frac{J_3}{3J} + \frac{K_1 - K_2}{3K_3} u_4 u_5 \Rightarrow \frac{J \dot{u}_7}{K_3} = \frac{J \dot{u}_6}{3K_3} + \frac{J_3}{3K_3} + \frac{K_1 - K_2}{3K_3} u_4 u_5$$

$$\dot{u}_7 = \left[\frac{J_3}{3J} + \frac{K_1 - K_2}{3K_3} + \frac{K_1 - K_2}{3K_3} u_4 u_5 \right] / \left(1 - \frac{J}{3K_3} \right)$$

$$\dot{\Omega} = \left[\frac{J_3}{3J} + \frac{K_1 - K_2}{3K_3} + \frac{K_1 - K_2}{3K_3} \omega_1 \omega_2 \right] / \left(1 - \frac{J}{3K_3} \right)$$

$$\dot{u}_6 = \left[\frac{J_3}{3K_3} + \frac{K_1 - K_2}{3K_3} \frac{J u_4 u_5}{3K_3} \right] / \left(1 - \frac{J}{3K_3} \right) + \frac{K_1 - K_2}{K_3} u_4 u_5 \left(1 - \frac{J}{3K_3} \right) / \left(1 - \frac{J}{3K_3} \right)$$

$$\dot{u}_6 = \left[\frac{J_3}{3K_3} + \frac{K_1 - K_2}{K_3} u_4 u_5 \right] / \left(1 - \frac{J}{3K_3} \right)$$

$$\omega_2 \omega_3 (I_2 - I_3) = J \dot{\omega}^B = I_1 \dot{\omega}_1$$

$$\omega_3 \omega_1 (I_3 - I_1) = J \dot{\omega}^B \omega_3 = I_2 \dot{\omega}_2$$

$$\omega_1 \omega_2 (I_1 - I_2) + J \dot{\omega}^B \omega_1 = I_3 \dot{\omega}_3$$

$$\left(\frac{T}{J} - \frac{F_1}{I_1} \right) / \left(1 - \frac{J}{I_1} \right) = \dot{\omega}^B = \left(\frac{T}{J} - \frac{\omega_2 \omega_3 (I_2 - I_3)}{I_1} \right) / \left(1 - \frac{J}{I_1} \right)$$

$$F_1 = \omega_2 \omega_3 (I_2 - I_3)$$

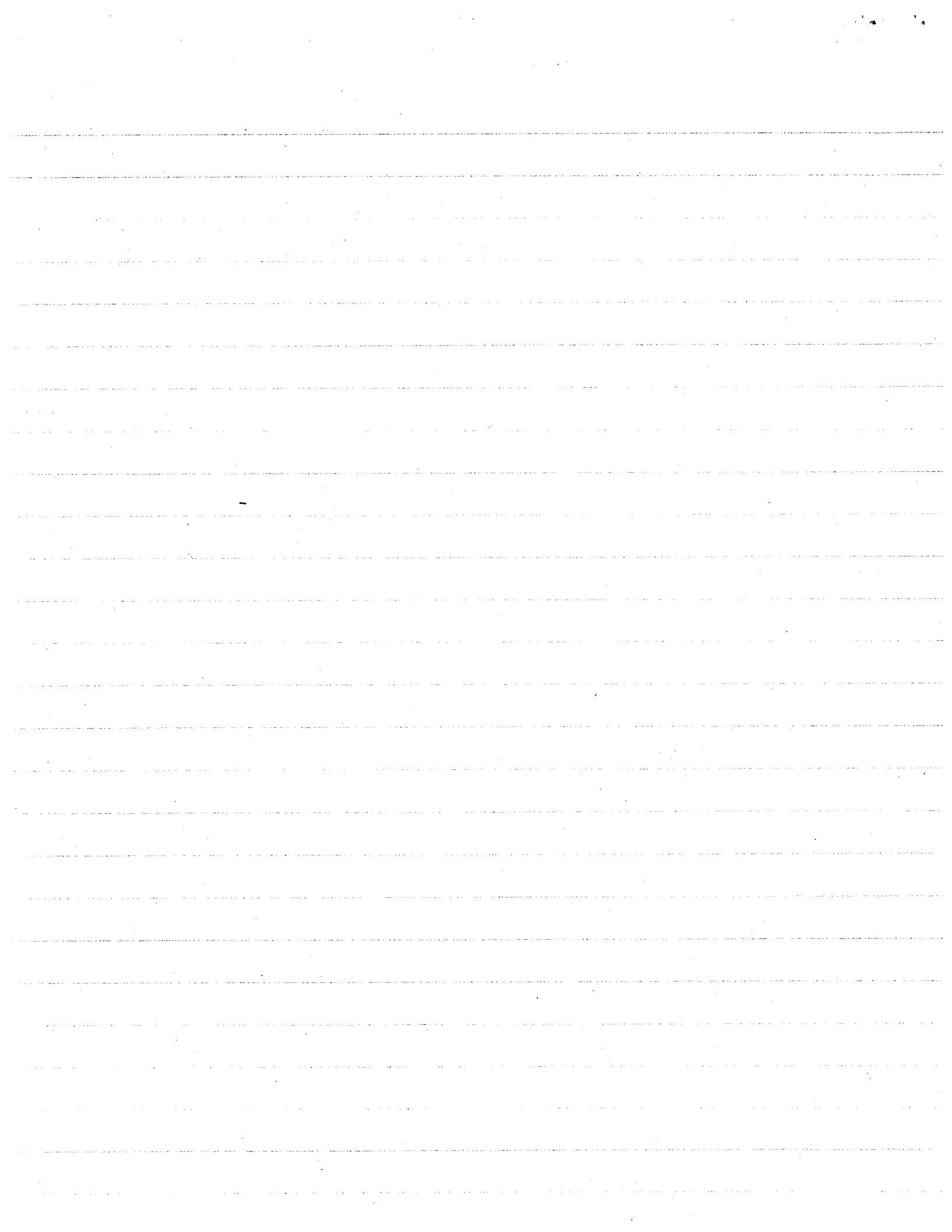
$$+ \omega_1 \omega_3 (I_2 - I_3) + \left(-\frac{T}{J} + \frac{\omega_2 \omega_3 (I_2 - I_3) J}{I_1} \right) / \left(1 - \frac{J}{I_1} \right) = -J \dot{\omega}^B + \omega_2 \omega_3 (I_2 - I_3)$$

$$= I_1 \dot{\omega}_1 + \omega_2 \omega_3 (I_2 - I_3) - \omega_2 \omega_3 (I_2 - I_3) \frac{J}{I_1} - \frac{T}{I_1} + \frac{\omega_2 \omega_3 (I_2 - I_3) J}{I_1}$$

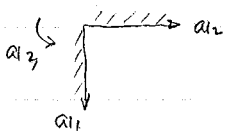
$$1 - \frac{J}{I_1}$$

$$\dot{\omega}_1 = \frac{\omega_2 \omega_3 (I_2 - I_3)}{I_1} - \frac{T}{I_1}$$

$$1 - \frac{J}{I_1}$$



From problem 2a we obtain the following if ${}^A_B \omega = \dot{q}_1 I_1 + \dot{q}_2 I_2 + \dot{q}_3 I_3 = \dot{q}_1 a_1 + \dot{q}_2 a_2 + \dot{q}_3 a_3$

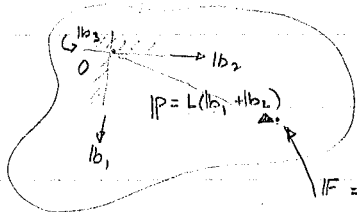


where initially a_i & b_i correspond before the successive rotations

$$\begin{array}{c|ccc} & a_1 & a_2 & a_3 \\ \hline i_1 & 1 & 0 & 0 \\ i_2 & 0 & c_1 & s_1 \\ i_3 & 0 & -s_1 & c_1 \end{array}$$

$$\begin{array}{c|ccc} & i_1 & i_2 & i_3 \\ \hline j_1 & c_2 & 0 & -s_2 \\ j_2 & 0 & 1 & 0 \\ j_3 & s_2 & 0 & c_2 \end{array}$$

$$\begin{array}{c|ccc} & j_1 & j_2 & j_3 \\ \hline l_1 & c_3 & s_3 & 0 \\ l_2 & -s_3 & c_3 & 0 \\ l_3 & 0 & 0 & 1 \end{array}$$



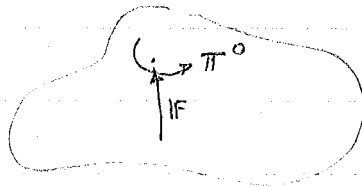
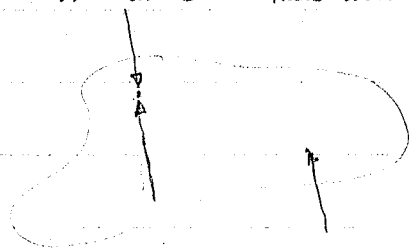
find a potential function for $\underline{IF}$. To find a potential fn.

for $\underline{IF}$ we must first find $\underline{IF}$'s contribution to the generalized

active forces

Since O is fixed let us find an equivalent Moment & force system about point O

$\therefore$ at O there will be a force $\underline{IF}$ and a couple $= \underline{\Pi} = \underline{P} \times \underline{IF}$



$$\begin{aligned} \underline{\Pi} &= L(l_1 + l_2) \times F_1 l_1 + F_2 l_2 + F_3 l_3 \\ &= L(F_2 l_3 - F_3 l_2 - F_1 l_3 + F_3 l_1) \\ &= L[(F_2 - F_1)l_3 + F_3 l_1 - F_3 l_2] \end{aligned}$$

$$\underline{\Pi} = KL^2 [c_2(s_3 + c_3)l_3 + s_2 l_1 + s_2 l_2]$$

Now $\underline{F}_r'$ (being the contribution of $\underline{IF}$ to the generalized active forces) $= \nabla \dot{q}_r \cdot \underline{IF}^0 + {}^A_B \omega \dot{q}_r \cdot \underline{\Pi}^0$

but since O is fixed $\nabla^0 = 0$ $\therefore \underline{F}_r' = {}^A_B \omega \dot{q}_r \cdot \underline{\Pi}^0$ and $\underline{F}_r' = -\frac{\partial P}{\partial q_r}$

$${}^A_B \omega \dot{q}_1 = a_1, \quad {}^A_B \omega \dot{q}_2 = i_2, \quad {}^A_B \omega \dot{q}_3 = l_3$$

$$\begin{aligned} \therefore F_1' &= a_1 \cdot \underline{\Pi}^0 = i_1 \cdot \underline{\Pi}^0 = (c_2 j_1 + s_2 j_3) \cdot \underline{\Pi}^0 = (c_2 [c_3 l_1 - s_3 l_2] + s_2 l_3) \cdot \underline{\Pi}^0 \\ &= KL^2 [(c_2 c_3) l_1 - c_2 s_3 l_2 + s_2 l_3] \cdot [-s_2 l_1 + s_2 l_2 + c_2 (s_3 + c_3) l_3] \end{aligned}$$

$$= KL^2 [-c_2 c_3 s_2 - c_2 s_3 s_2 + s_2 c_2 s_3 + s_2 c_2 c_3] = 0 = -\frac{\partial P}{\partial q_1} \quad (a)$$

$$\begin{aligned} F_2' &= i_2 \cdot \underline{\Pi}^0 = j_2 \cdot \underline{\Pi}^0 = (s_3 l_1 + c_3 l_2) \cdot KL^2 [-s_2 l_1 + s_2 l_2 + c_2 (s_3 + c_3) l_3] \\ &= KL^2 [-s_3 s_2 + c_3 s_2] = -\frac{\partial P}{\partial q_2} \quad (b) \end{aligned}$$

$$F_3' = l_3 \cdot \underline{\Pi}^0 = l_3 \cdot KL^2 [-s_2 l_1 + s_2 l_2 + c_2 (s_3 + c_3) l_3] = KL^2 [c_2 s_3 + c_2 c_3] = -\frac{\partial P}{\partial q_3} \quad (c)$$

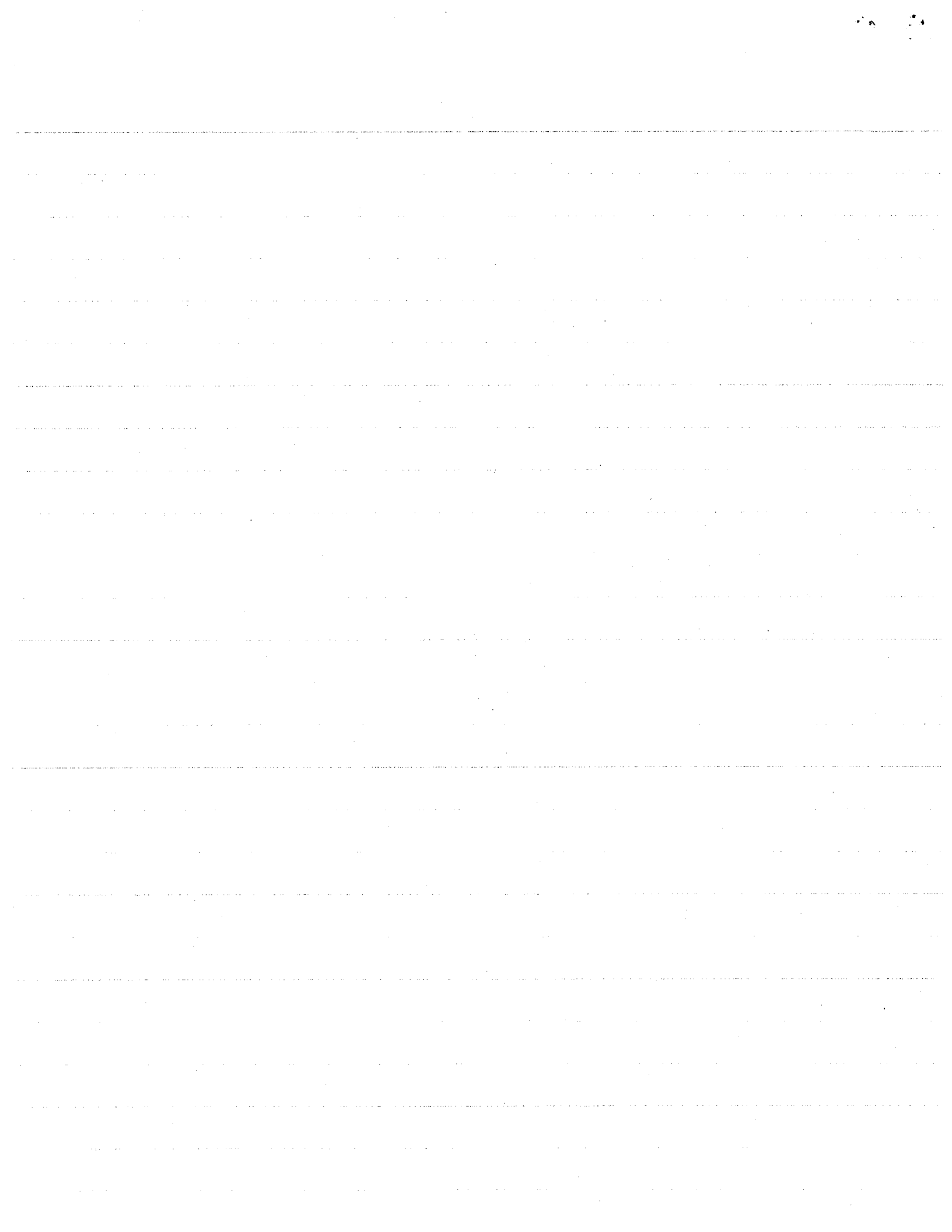
$\therefore$ integrating (a) $P = f(q_1, q_2, q_3, t)$

$$(b) -P = KL^2 (s_3 c_2 + c_3 c_2) + f(q_2, t) \Rightarrow P = KL^2 (c_3 - s_3) c_2 + f(q_2, t)$$

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$$-\frac{\partial P}{\partial q_3} = Kl^2 (c_3 c_2 + s_3 c_2) - \frac{\partial f}{\partial q_3} = Kl^2 (c_3 c_2 + c_2 s_3) \Rightarrow \frac{\partial f}{\partial q_3} = 0 \quad \text{and} \quad f = f(t) \text{ only}$$

$$\therefore P = Kl^2 c_2 (c_3 - s_3) + f(t) \quad \text{for IF}$$



Excellent

A

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#23
LEVY

A rigid body A carries three identical rotors B_1 , B_2 , and C on parallel axes fixed in A , as shown in the attached sketch. By means of a motor attached to A , C is made to rotate relative to A at a rate $\dot{\theta}$ (not necessarily constant), and B_1 and B_2 are driven by C at the same rate, but with opposite sense.

X_1 , X_2 , X_3 (see sketch) are principal axes of inertia of A for the mass center of A , and the moments of inertia of the entire system about these axes have the values K_1 , K_2 , K_3 , respectively. The mass center of each of the three rotors lies on X_1 , and each rotor has moment of inertia J about its axis of symmetry.

Let $\underline{n}_i$ be a unit vector pointing in the direction of the positive X_i axis ($i = 1, 2, 3$); replace the system of forces exerted on C by A (by means of the motor) with a couple of torque

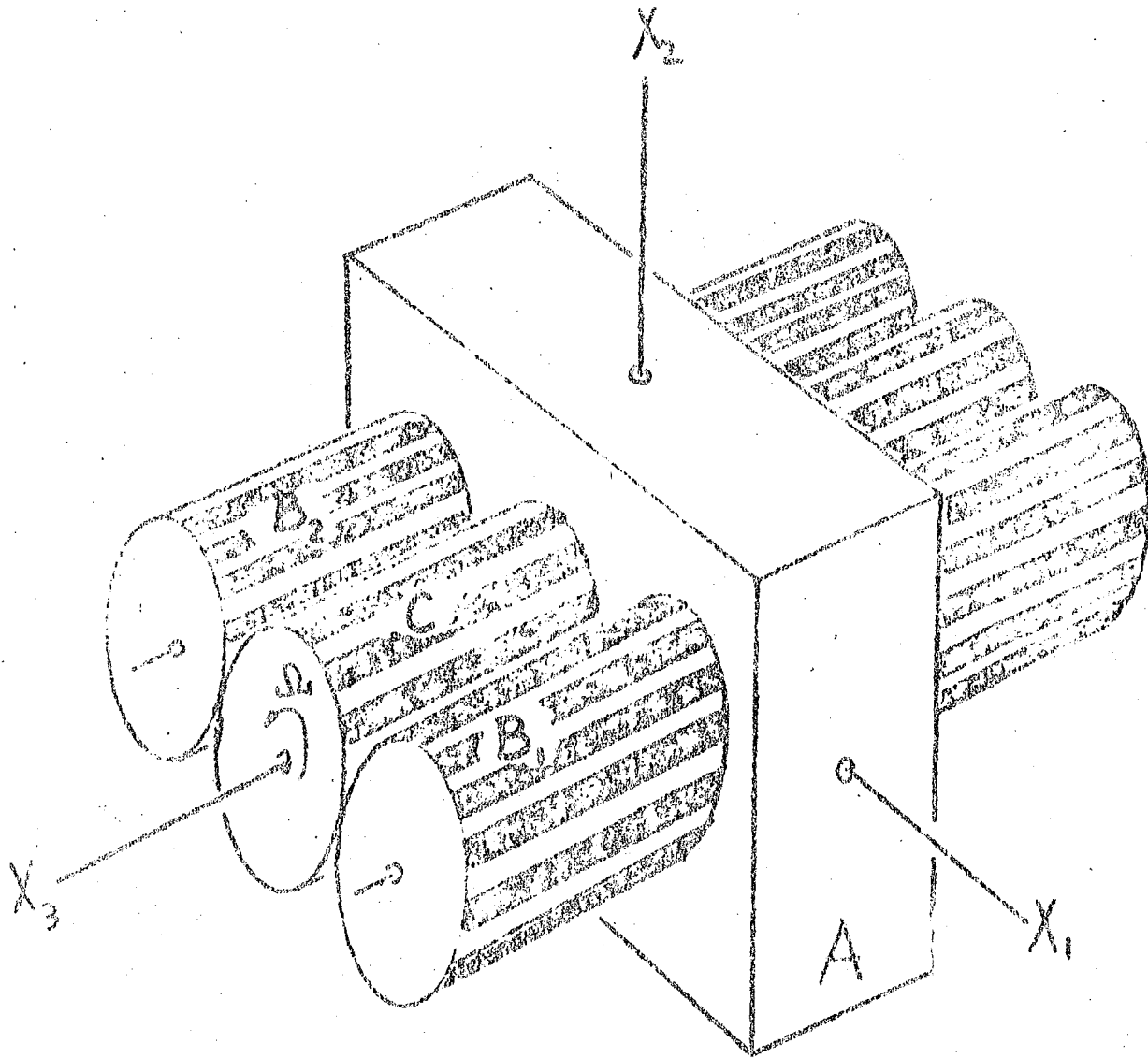
$$T_1 \underline{n}_1 + T_2 \underline{n}_2 + T_3 \underline{n}_3$$

together with a force applied at the mass center of C ; and assume that the only body and contact forces acting on the system are gravitational forces.

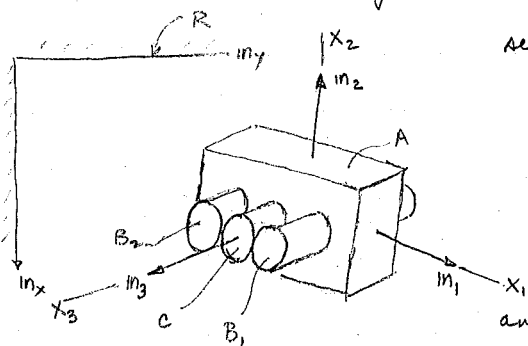
The differential equations of motion of this system can be expressed as

$$\dot{\underline{Q}} = \underline{F}; \quad \ddot{\underline{a}}_i = \underline{G}_i \quad (i = 1, 2, 3)$$

where $\underline{F}$ and $\underline{G}_i$ are functions of θ , ω_1 , ω_2 , ω_3 , J , K_1 , K_2 , K_3 , I_1 , I_2 , I_3 . Determine $\underline{F}$ and $\underline{G}_i$ ($i=1,2,3$).



We assume a reference frame R (with mutually perpendicular vectors m_x, m_y, m_z) which is fixed and in which the system in question moves. The system in question has 7 degrees of freedom, 6 to describe the motion of body A and 1 to describe the motion of body C relative to body A (similarly this same degree of freedom describes the motion of bodies B_1 and B_2 relative to A). Since a motor causes a



set of forces to act on body C and we replace them by a torque π^C and a resultant force IF_T^C acting at C^* . Similarly by the laws of action and reaction a torque $-\pi^C$ and a resultant force $-IF_T^C$ acting at A^* exists. Since the only body and contact forces involved are $Mg m_x$ on body A and

$mg m_x$ on each of the bodies C, B_1, B_2 . Because the translational motion can be decoupled from the rotational motion, we need only look at the rotational equations to obtain our results. However I will set up the general results.

If we assume $(\cdot)^*$ represent the values of the center of mass of the body and

$$V^{A*} = V_1 m_1 + V_2 m_2 + V_3 m_3 \quad (1)$$

then we can define the following generalized coordinates

$$u_i = \begin{cases} V_i & (i=1,2,3) \\ \omega_{i-3} & (i=4,5,6) \\ \Omega & i=7 \end{cases}$$

where

$${}^R \omega^A \cdot m_i = \omega_i \quad (2)$$

$${}^A \omega^C = -{}^A \omega^{B_1} = -{}^A \omega^{B_2} = -\Omega m_3 \quad (3)$$

also

$$IF^A = -IF_T^C + Mg m_x \quad (4)$$

$$IF^C = mg m_x + IF_T^C \quad (5)$$

$$IF^{B_1} = mg m_x \quad (6)$$

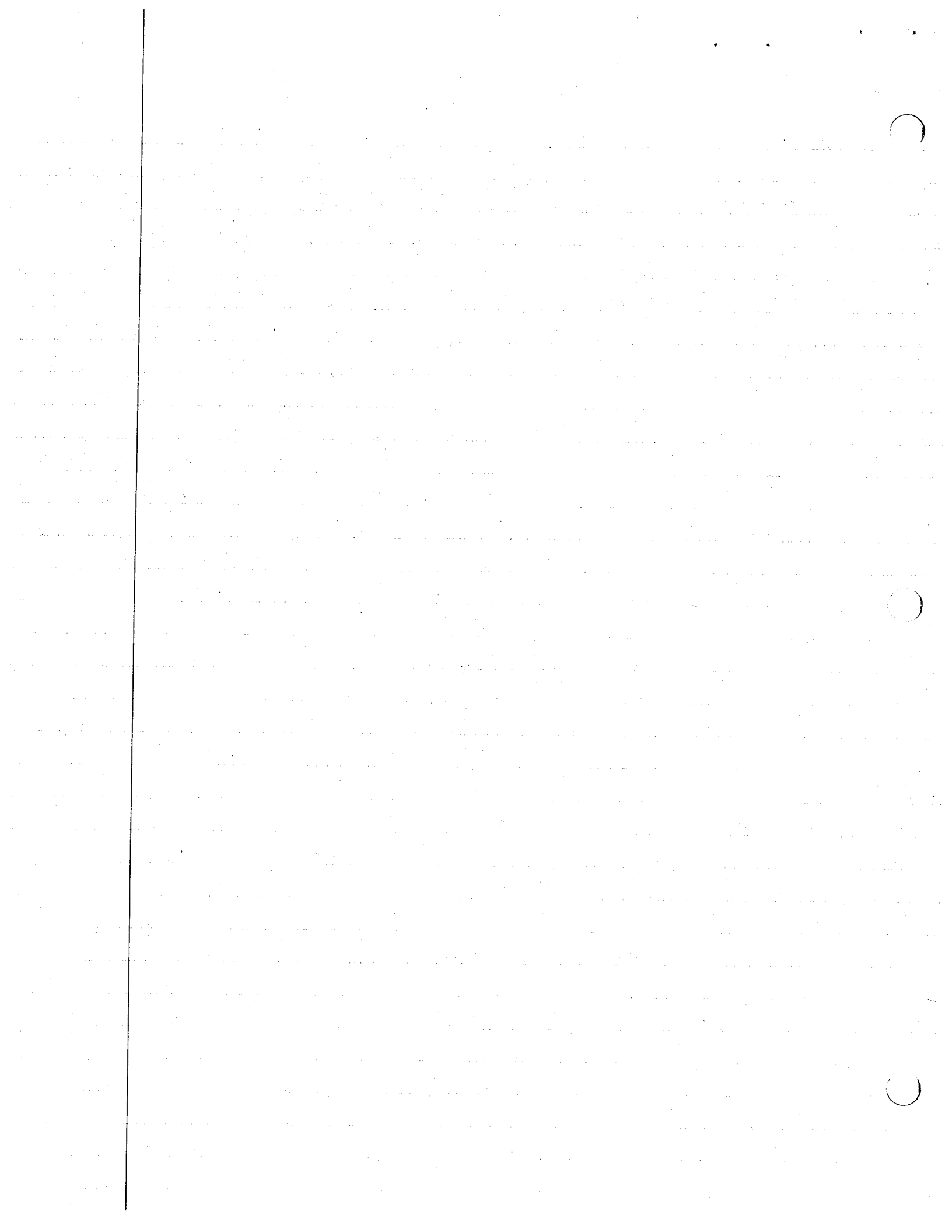
$$IF^{B_2} = mg m_x \quad (7)$$

Similarly

$$\pi^C = T_1 m_1 + T_2 m_2 + T_3 m_3 \quad (8)$$

$$\pi^A = -\pi^C \quad (9)$$

$$\pi^{B_1} = \pi^{B_2} = 0 \quad (10)$$



Now we can define

$$V^{A*} = V^{C*} = u_1 m_1 + u_2 m_2 + u_3 m_3 \quad (11)$$

$$V^{B_1*} = V^{C*} + l \vec{\omega}^A \times m_1 = V^{C*} + l [u_6 m_2 - u_5 m_3] \quad (12)$$

$$V^{B_2*} = V^{C*} - l \vec{\omega}^A \times m_1 = V^{C*} - l [-u_6 m_2 + u_5 m_3] \quad (13)$$

where l is the magnitude of the distance between A^* and B_1^* and between A^* and B_2^* . We now define the generalized active force F_r by

$$F_r = V_{u_r}^{A*} \cdot IF^{A*} + V_{u_r}^{C*} \cdot IF^{C*} + V_{u_r}^{B_1*} \cdot IF^{B_1*} + V_{u_r}^{B_2*} \cdot IF^{B_2*} + \omega_{u_r}^A \cdot \pi^A + \omega_{u_r}^C \cdot \pi^C + \omega_{u_r}^{B_1*} \cdot \pi^{B_1*} + \omega_{u_r}^{B_2*} \cdot \pi^{B_2*} \quad \text{for } r = 1, \dots, 7 \quad (14)$$

We note certain items: all the ω 's are not functions of u_1, u_2, u_3 ; all the V 's are not functions of u_4 and u_7 and V^{A*}, V^{C*} are not functions of u_5 and u_6 . Using these and (1-4) we obtain

$$F_1 = m_1 \cdot m_x [(M+3m)g] \quad (15)$$

$$F_2 = m_2 \cdot m_x [(M+3m)g] \quad (16)$$

$$F_3 = m_3 \cdot m_x [(M+3m)g] \quad (17)$$

$$F_4 = F_5 = F_6 = 0 \quad (18-20)$$

$$F_7 = T_3 \quad (21)$$

To obtain the generalized inertia forces F_r^*

$$F_r^* = V_{u_r}^{A*} \cdot IF^{A*} + V_{u_r}^{C*} \cdot IF^{C*} + V_{u_r}^{B_1*} \cdot IF^{B_1*} + V_{u_r}^{B_2*} \cdot IF^{B_2*} + \omega_{u_r}^A \cdot \pi^A + \omega_{u_r}^C \cdot \pi^C + \omega_{u_r}^{B_1*} \cdot \pi^{B_1*} + \omega_{u_r}^{B_2*} \cdot \pi^{B_2*} \quad (22)$$

$$\begin{aligned} \text{where } IF^{A*} &= -M a_1^{A*} & \pi^A &= -\frac{d}{dt} \left[\frac{R}{H} \frac{A}{A^*} \right] \\ IF^{C*} &= -m a_1^{C*} & \pi^C &= -\frac{d}{dt} \left[\frac{R}{H} \frac{C}{C^*} \right] \\ IF^{B_1*} &= -m a_1^{B_1*} & \pi^{B_1*} &= -\frac{d}{dt} \left[\frac{R}{H} \frac{B_1}{B_1^*} \right] \\ IF^{B_2*} &= -m a_1^{B_2*} & \pi^{B_2*} &= -\frac{d}{dt} \left[\frac{R}{H} \frac{B_2}{B_2^*} \right] \end{aligned} \quad (23-30)$$

Now

$$a_1^{A*} = A \frac{dV^{C*}}{dt} + \vec{\omega}^A \times V^{C*} = a_1^{C*} \quad (31, 32)$$

$$a_1^{B_1*} = a_1^{C*} + l \frac{d\vec{\omega}^A}{dt} \times m_1 + l [(\vec{\omega}^A \cdot m_1) \vec{\omega}^A - \vec{\omega}^{A^2} m_1] \quad (33)$$

$$a_1^{B_2*} = a_1^{C*} - l \frac{d\vec{\omega}^A}{dt} \times m_1 - l [(\vec{\omega}^A \cdot m_1) \vec{\omega}^A - \vec{\omega}^{A^2} m_1] \quad (34)$$

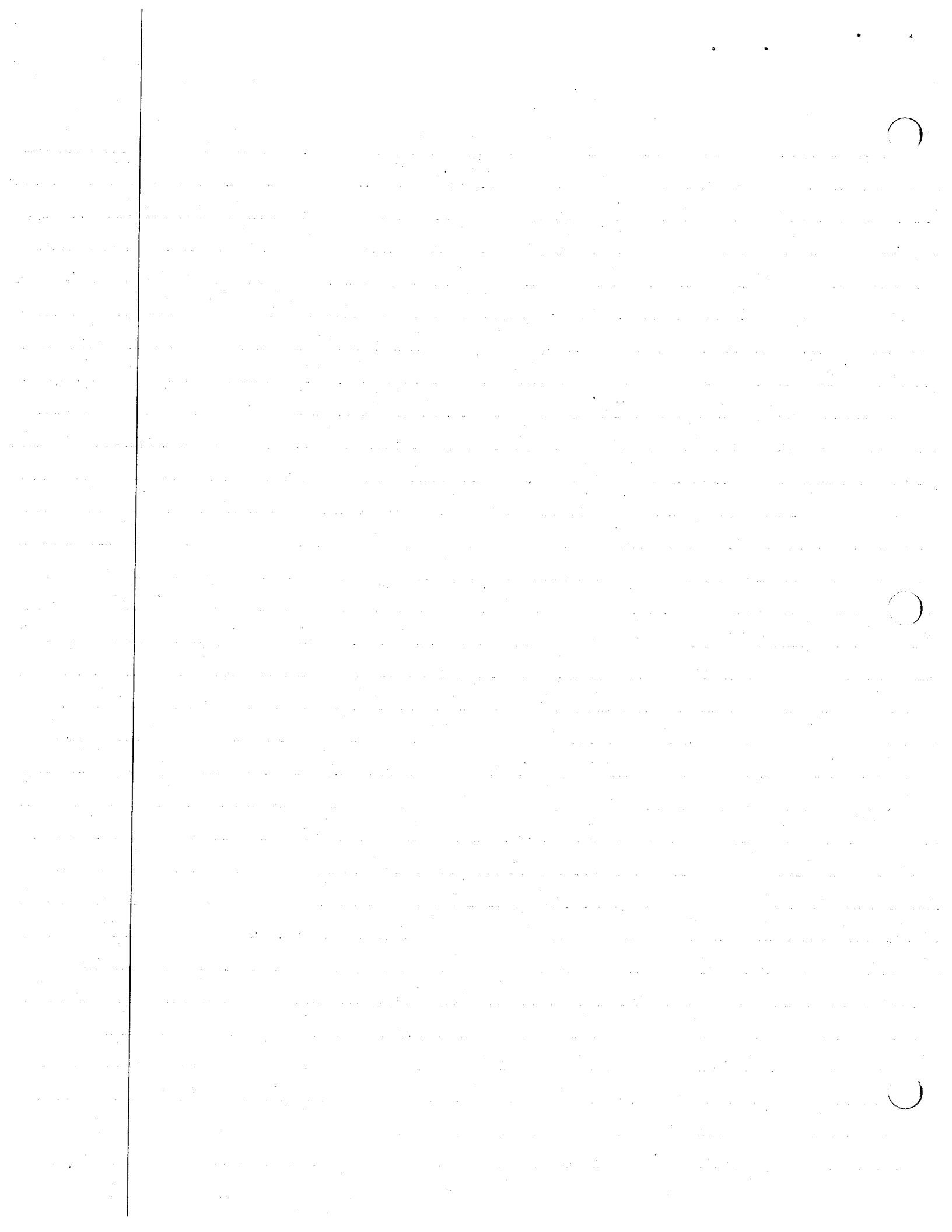
Using these results we find

$$F_1^* = -(M+3m)[\ddot{u}_1 + u_5 u_3 - u_6 u_2] \quad (35)$$

$$F_2^* = -(M+3m)[\ddot{u}_2 + u_6 u_1 - u_4 u_3] \quad (36)$$

$$F_3^* = -(M+3m)[\ddot{u}_3 + u_4 u_2 - u_5 u_1] \quad (37)$$

$$F_4^* = m_1 \cdot (\pi^A + \pi^C + \pi^{B_1*} + \pi^{B_2*}) \quad (38)$$



$$F_5^* = -2l^2 m [\dot{u}_5 - u_4 u_6] + m_2 \cdot [\pi^{*A} + \pi^{*C} + \pi^{*B_1} + \pi^{*B_2}] \quad (39)$$

$$F_6^* = -2l^2 m [\dot{u}_6 + u_4 u_5] + m_3 \cdot [\pi^{*A} + \pi^{*B_1} + \pi^{*B_2}] \quad (40)$$

$$F_7^* = m_3 \cdot [\pi^{*C} - \pi^{*B_1} - \pi^{*B_2}] \quad (41)$$

Now

$$\pi^{*C} = -\frac{R}{dt} |H|^{C/C^*} = -\mathbb{I}^{C/C^*} \cdot \frac{R}{dt} \dot{A} + \pi^{*C} = -\mathbb{I}^{C/C^*} (\frac{R}{dt} \dot{A} + \dot{A}^C) \quad (42)$$

$$\pi^{*B_1^*} = -\frac{R}{dt} |H|^{B_1/B_1^*} = -\mathbb{I}^{C/C^*} \cdot \frac{R}{dt} \dot{A} + \pi^{*B_1^*} \quad \text{Since } \mathbb{I}^{C/C^*} = \mathbb{I}^{B_1/B_1^*} = \mathbb{I}^{B_2/B_2^*} \quad (43)$$

$$\pi^{*B_2^*} = -\frac{R}{dt} |H|^{B_2/B_2^*} = -\mathbb{I}^{C/C^*} \cdot \frac{R}{dt} \dot{A} + \pi^{*B_2^*} = -\mathbb{I}^{C/C^*} (\frac{R}{dt} \dot{A} + \dot{A}^B) = \pi^{*B_1^*} \quad (44)$$

Using these in (41) leads to

$$F_7^* = -J(\dot{u}_6 + \dot{u}_7) + 2J(\dot{u}_6 - \dot{u}_7) = -3J\dot{u}_7 + J\dot{u}_6 \quad (45)$$

Now

$$\begin{aligned} \pi^{*A} + \pi^{*C} + \pi^{*B_1^*} + \pi^{*B_2^*} &= -\frac{R}{dt} \left[|H|^{A/A^*} + |H|^{C/C^*} + |H|^{B_1/B_1^*} + |H|^{B_2/B_2^*} \right] \\ &= -\frac{R}{dt} \left[\left(\mathbb{I}^{A/A^*} + \mathbb{I}^{C/C^*} + \mathbb{I}^{B_1/B_1^*} + \mathbb{I}^{B_2/B_2^*} \right) \cdot \frac{R}{dt} \dot{A} + \mathbb{I}^{C/C^*} \cdot \dot{A}^C + \mathbb{I}^{B_1/B_1^*} \cdot \dot{A}^{B_1} + \mathbb{I}^{B_2/B_2^*} \cdot \dot{A}^{B_2} \right] \end{aligned} \quad (46)$$

$$\text{Using } \mathbb{I}^{B_1/B_1^*} = \mathbb{I}^{B_1/B_1^*} + \mathbb{I}^{B_1^*/A^*} \quad \text{and} \quad \mathbb{I}^{B_2/B_2^*} = \mathbb{I}^{B_2/B_2^*} + \mathbb{I}^{B_2^*/A^*} \quad (47 \& 48)$$

then (46) becomes:

$$\begin{aligned} &= -\frac{R}{dt} \left[\mathbb{I}^{(A+C+B_1+B_2)/A^*} \cdot \frac{R}{dt} \dot{A} - \left(\mathbb{I}^{B_1^*/A^*} + \mathbb{I}^{B_2^*/A^*} \right) \cdot \frac{R}{dt} \dot{A} - \mathbb{I}^{C/C^*} \cdot \dot{A}^C \right] \\ &= \frac{R}{dt} \pi^D - \frac{A}{dt} \pi^C + 2l^2 m \frac{R}{dt} \left[(m_2 m_2 + m_3 m_3) \cdot \frac{R}{dt} \dot{A} \right] + \frac{R}{dt} \dot{A} \times |H|^{C/C^*} \end{aligned} \quad (49)$$

This is obtained since

$$\mathbb{I}^{B_1^*/A^*} = m l^2 (\mathbb{V} - m_1 m_1) = m l^2 (m_2 m_2 + m_3 m_3) = \mathbb{I}^{B_2^*/A^*} \quad \text{where } \mathbb{V} \text{ is the unit dyadic (50)}$$

If we note that $\frac{R}{dt} [(m_2 m_2 + m_3 m_3) \cdot \frac{R}{dt} \dot{A}] = (\dot{u}_5 - u_4 u_6) m_2 + (\dot{u}_6 + u_4 u_5) m_3$ then

$$\pi^{*A} + \pi^{*C} + \pi^{*B_1^*} + \pi^{*B_2^*} = \frac{R}{dt} \pi^D - \frac{A}{dt} \pi^C + 2l^2 m \{ (\dot{u}_5 - u_4 u_6) m_2 + (\dot{u}_6 + u_4 u_5) m_3 \} + \frac{R}{dt} \dot{A} \times |H|^{C/C^*} \quad (51)$$

Now D is a fictitious rigid body whose mass distribution is the same as $A+B_1+C+B_2$ and whose motion is that of body A . (ie $\frac{R}{dt} \pi^D = (\mathbb{I}^{D/D^*} \cdot \frac{R}{dt} \dot{A}) \times \frac{R}{dt} \dot{A} - \mathbb{I}^{D/D^*} \cdot \frac{R}{dt} \dot{A}$) (52)

$$\text{Using (51) in (38-40) and noting that } \mathbb{I}^{D/D^*} = K_1 m_1 m_1 + K_2 m_2 m_2 + K_3 m_3 m_3 \quad (53)$$

$$F_4^* = (K_2 - K_3) u_5 u_6 - K_1 \dot{u}_4 + J u_7 u_5 \quad (54)$$

$$F_5^* = (K_3 - K_1) u_4 u_6 - K_2 \dot{u}_5 - J u_7 u_4 \quad (55)$$

$$F_6^* = (K_1 - K_2) u_4 u_5 - K_3 \dot{u}_6 + J \dot{u}_7 \quad (56)$$

These are obtained by observing that $\frac{A}{dt} \pi^C = -J \dot{u}_7 m_3$ and $\frac{R}{dt} \dot{A} \times |H|^{C/C^*} = J u_7 (u_5 m_1 - u_6 m_2)$ (57, 58)

Now using the equations of motion $F_r + F_r^* = 0$ for $r=4, \dots, 7$ i.e. eqns (8-21) & (45), (54), we obtain

$$K_1 \dot{u}_4 = (K_2 - K_3) u_5 u_6 + J u_7 u_5 \quad (59)$$

$$K_2 \dot{u}_5 = (K_3 - K_1) u_4 u_6 - J u_7 u_4 \quad (60)$$

$$K_3 \dot{u}_6 = J \dot{u}_7 + (K_1 - K_2) u_4 u_5 \quad (61)$$

$$3J \dot{u}_7 = J \dot{u}_7 + T_2 \quad (62)$$

$$K_3 \dot{u}_6 = \frac{J \dot{u}_6}{3} + \frac{T_3}{3} + (K_1 - K_2) u_4 u_5 \Rightarrow \dot{u}_6 = \left(\frac{T_3}{3K_3} + \frac{K_1 - K_2}{K_3} u_4 u_5 \right) / \left(1 - \frac{J}{3K_3} \right) \quad (63)$$

$$\dot{u}_7 = \frac{\dot{u}_6}{3} + \frac{T_3}{3J} = \left(\frac{T_3}{3J} + \frac{K_1 - K_2}{3K_3} u_4 u_5 \right) / \left(1 - \frac{J}{3K_3} \right) \quad (64)$$

$$\dot{u}_4 = \left(\frac{K_2 - K_3}{K_1} u_5 u_6 + \frac{J}{K_1} u_7 u_5 \right) \quad (65)$$

$$\dot{u}_5 = \left(\frac{K_3 - K_1}{K_2} u_4 u_6 - \frac{J}{K_2} u_7 u_4 \right) \quad (66)$$

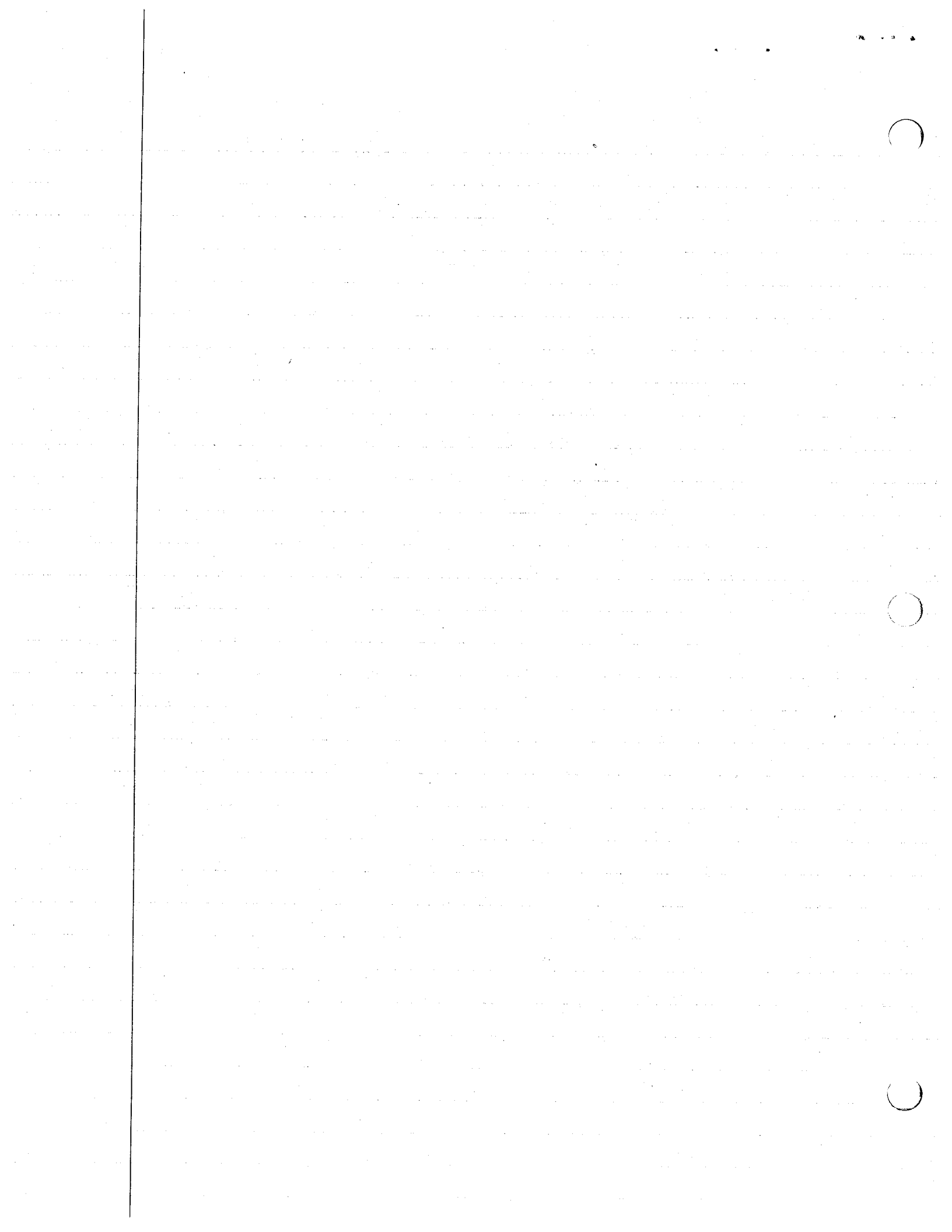
Using the definitions for the u_i 's to obtain

$$\dot{\omega}_1 = \left[\frac{K_2 - K_3}{K_1} \omega_3 + \frac{J}{K_1} \Omega \right] \omega_2 \quad (67)$$

$$\dot{\omega}_2 = \left[\frac{K_3 - K_1}{K_2} \omega_3 - \frac{J}{K_2} \Omega \right] \omega_1 \quad (68)$$

$$\dot{\omega}_3 = \left[\frac{T_3}{3K_3} + \frac{K_1 - K_2}{K_3} \omega_1 \omega_2 \right] / \left(1 - \frac{J}{3K_3} \right) \quad (69)$$

$$\dot{\Omega} = \left[\frac{T_3}{3J} + \frac{K_1 - K_2}{3K_3} \omega_1 \omega_2 \right] / \left(1 - \frac{J}{3K_3} \right) \quad (70)$$



Very nice

800310

A point O of a rigid body B is fixed. Mutually perpendicular axes B_1, B_2, B_3 are fixed in B and intersect at O . Initially, B_1, B_2, B_3 are aligned with fixed axes A_1, A_2, A_3 , respectively, and successive rotations of B of amounts q_1, q_2, q_3 are then performed about B_1, B_2, B_3 , respectively, to bring B into a general position.

A force $\underline{F}$ is applied to B at a point P . $\underline{F}$ depends on the orientation of B as follows:

$$\underline{F} = kL[(1 - c_2 c_3)\underline{b}_1 + (1 + c_2 s_3)\underline{b}_2 - s_2 \underline{b}_3]$$

where k and L are constants, $\underline{b}_1$ is a unit vector parallel to B_1 , and $s_j = \sin q_j$, $c_j = \cos q_j$. The position vector $\underline{p}$ of P relative to O is given by

$$\underline{p} = L(\underline{b}_1 + \underline{b}_2)$$

Find a potential function for $\underline{F}$.

From problem 2a: If we denote fixed reference frame A having axes A_1, A_2, A_3 with unit vectors a_1, a_2, a_3 along these axes respectively; and the body B having axes B_1, B_2, B_3 which are fixed in the body and mutually perpendicular with unit vectors b_1, b_2, b_3 along these axes respectively, then by introducing intermediate reference frames I_1 and I_2 with unit vectors $(\hat{i}_1, \hat{i}_2, \hat{i}_3)$ and $(\hat{j}_1, \hat{j}_2, \hat{j}_3)$ which are fixed in each reference frame and mutually perpendicular we may write

$$\overset{A}{\omega} \overset{B}{=} \overset{A}{\omega} \overset{I_1}{=} + \overset{I_1}{\omega} \overset{I_2}{=} + \overset{I_2}{\omega} \overset{B}{=} = \dot{q}_1 a_1 + \dot{q}_2 \hat{i}_2 + \dot{q}_3 b_3 \quad (1)$$

where $a_1 = c_2 c_3 b_1 - c_2 s_3 b_2 + s_2 b_3 \quad (2)$

$$\hat{i}_2 = s_3 b_1 + c_3 b_2 \quad (3)$$

In order to find a potential function for IF we must first form the contribution of IF to the generalized active forces F_r' and then use

$$F_r' = - \frac{\partial P}{\partial q_r} \quad r=1,2,3 \quad (4)$$

It should be noted that because point O is fixed $\dot{W}^0 = 0 \quad (5)$

and it would be advantageous to replace IF acting at P by an equivalent force at O and moment about point O. This leads to

$$IF^0 = KL [(1 - c_2 c_3) b_1 + (1 + c_2 s_3) b_2 - s_2 b_3] \quad (6)$$

$$\text{and } \Pi^0 = p \times IF^0 = L (b_1 + b_2) \times IF^0 = KL^2 [-s_2 b_1 + s_2 b_2 + c_2 (s_3 + c_3) b_3] \quad (7)$$

To form F_r'

$$F_r' = \overset{A}{\omega} \overset{0}{q_r} \cdot IF^0 + \overset{A}{\omega} \overset{B}{q_r} \cdot \Pi^0 = \overset{A}{\omega} \overset{B}{q_r} \cdot \Pi^0 \quad r=1,2,3 \quad (8)$$

$$\text{Thus } F_1' = a_1 \cdot \Pi^0 = 0 = - \frac{\partial P}{\partial q_1} \quad (9)$$

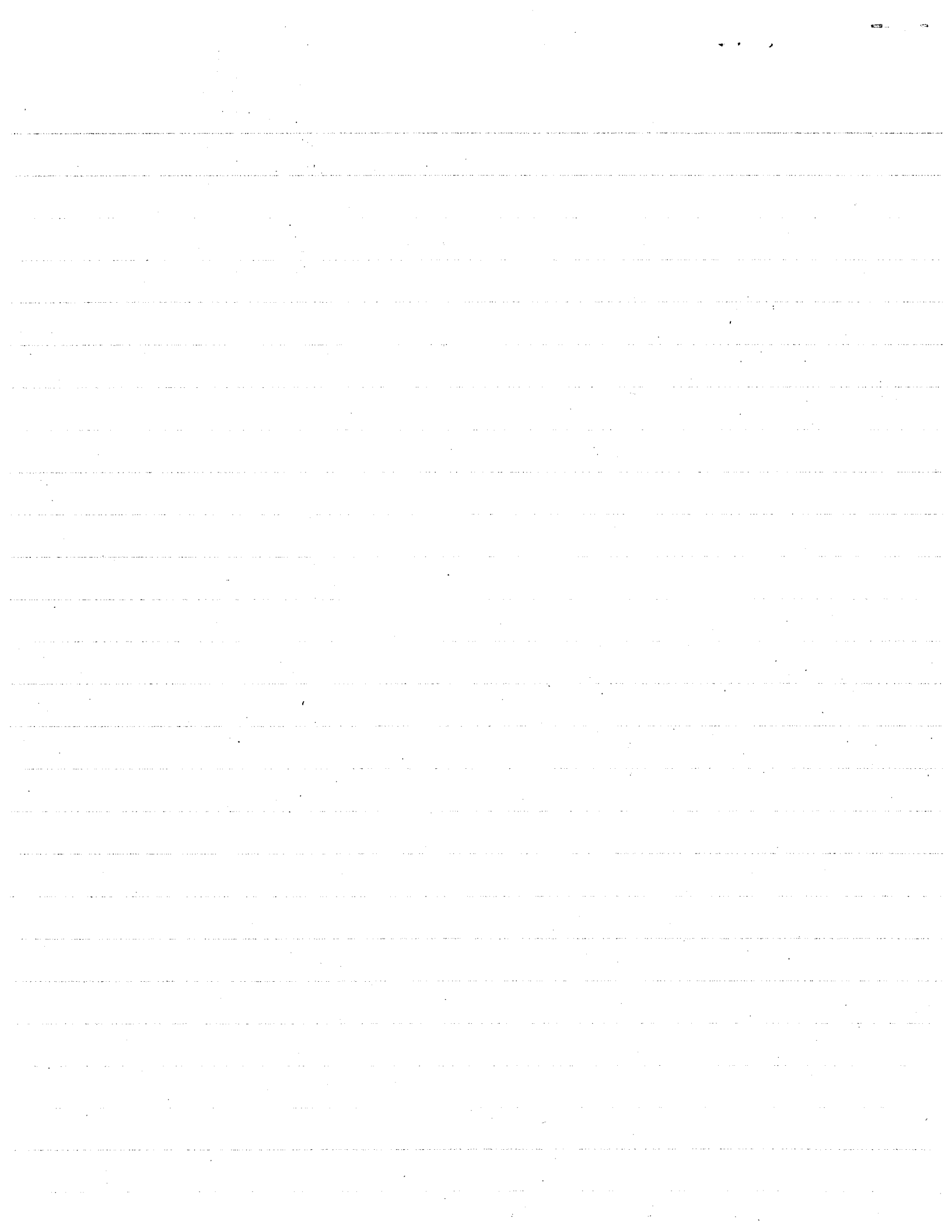
$$F_2' = \hat{i}_2 \cdot \Pi^0 = KL^2 [s_2 (c_3 - s_3)] = - \frac{\partial P}{\partial q_2} \quad (10)$$

$$F_3' = b_3 \cdot \Pi^0 = KL^2 [c_2 (c_3 + s_3)] = - \frac{\partial P}{\partial q_3} \quad (11)$$

$$\text{from (9): } P = f(q_2, q_3, t) \text{ only} \quad (12)$$

$$\text{from (10): } -P = KL^2 [-c_2 (c_3 - s_3)] - g(q_3, t) \quad (13)$$

$$\text{from (13): } - \frac{\partial P}{\partial q_3} = KL^2 [c_2 (s_3 + c_3)] - \frac{\partial g}{\partial q_3} = KL^2 [c_2 (c_3 + s_3)] \Rightarrow \frac{\partial g}{\partial q_3} = 0 \text{ and } g = g(t) \text{ only!}$$



therefore using the results of (12), (13), (14) we obtain

$$P = K L^2 [c_2(c_3 - s_3)] + q(t) \quad \text{for force IF} \quad (15)$$





DYNAMICS PROJECT:

STUDY

OF A

SIMPLE GYROSTAT

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ME 231C

Winter 1980

Prof. Kane



Study of a Simple Gyrostat

The study of a simple gyrostat G consisting of an axisymmetric rotor B whose axis and mass center B^* are fixed in a rectangular body A is performed with the additional condition that the mass centers of both bodies are to coincide (see Figure 1). A set of unit vectors a_1, a_2, a_3 form a right-handed mutually perpendicular triad fixed in A , each vector parallel to a central principal axis of inertia of G . The axis of B is parallel to a unit vector β and the rotor-rectangular body system is such that the central principal moments of inertia of G are equal to those of the rectangular body. These central principal moments of inertia are denoted by I_1, I_2, I_3 and B has a polar moment of inertia about β equal to J .

If we define ${}^A\omega^B$, ω_i and β_i as

$${}^A\omega^B \triangleq {}^R\omega^B \cdot \beta \quad (1)$$

$$\omega_i \triangleq {}^R\omega^A \cdot a_i \quad (i=1,2,3) \quad (2)$$

$$\beta_i \triangleq \beta \cdot a_i \quad (i=1,2,3) \quad (3)$$

with ${}^A\omega^B$ being the angular velocity of B in A and ${}^R\omega^A$ being the angular velocity of A in a "fixed" reference frame R , we may then write the following set of differential equations that govern ${}^A\omega^B$ and ω_i ($i=1,2,3$):

$${}^A\omega^B = (T/J - H)/C_1, \quad (4)$$

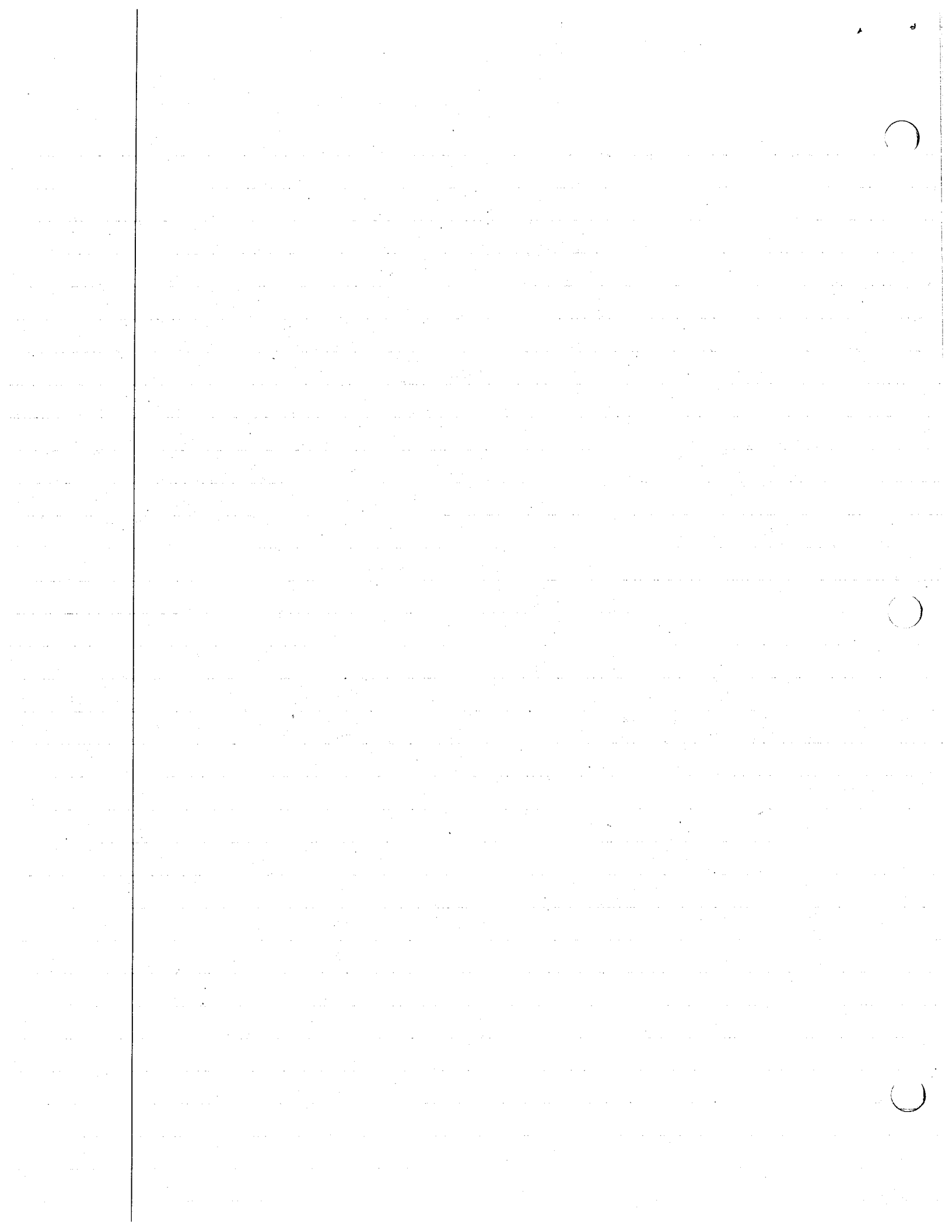
$$\dot{\omega}_1 = [F_1 - \beta_1 J (T/J - H)]/I_1, \quad (5)$$

$$\dot{\omega}_2 = [F_2 - \beta_2 J (T/J - H)]/I_2, \quad (6)$$

$$\text{and } \dot{\omega}_3 = [F_3 - \beta_3 J (T/J - H)]/I_3. \quad (7)$$

In the preceding equations C_1 , F_i ($i=1,2,3$), H and T are defined as follows:

$$C_1 \triangleq 1 - J [\beta_1^2/I_1 + \beta_2^2/I_2 + \beta_3^2/I_3] \quad (8)$$



$$H \triangleq \beta_1 F_1 / I_1 + \beta_2 F_2 / I_2 + \beta_3 F_3 / I_3 \quad (9a)$$

$$F_1 \triangleq (I_2 - I_3) \omega_2 \omega_3 + J \dot{\omega}^B [\omega_3 \beta_2 - \omega_2 \beta_3] \quad (9b)$$

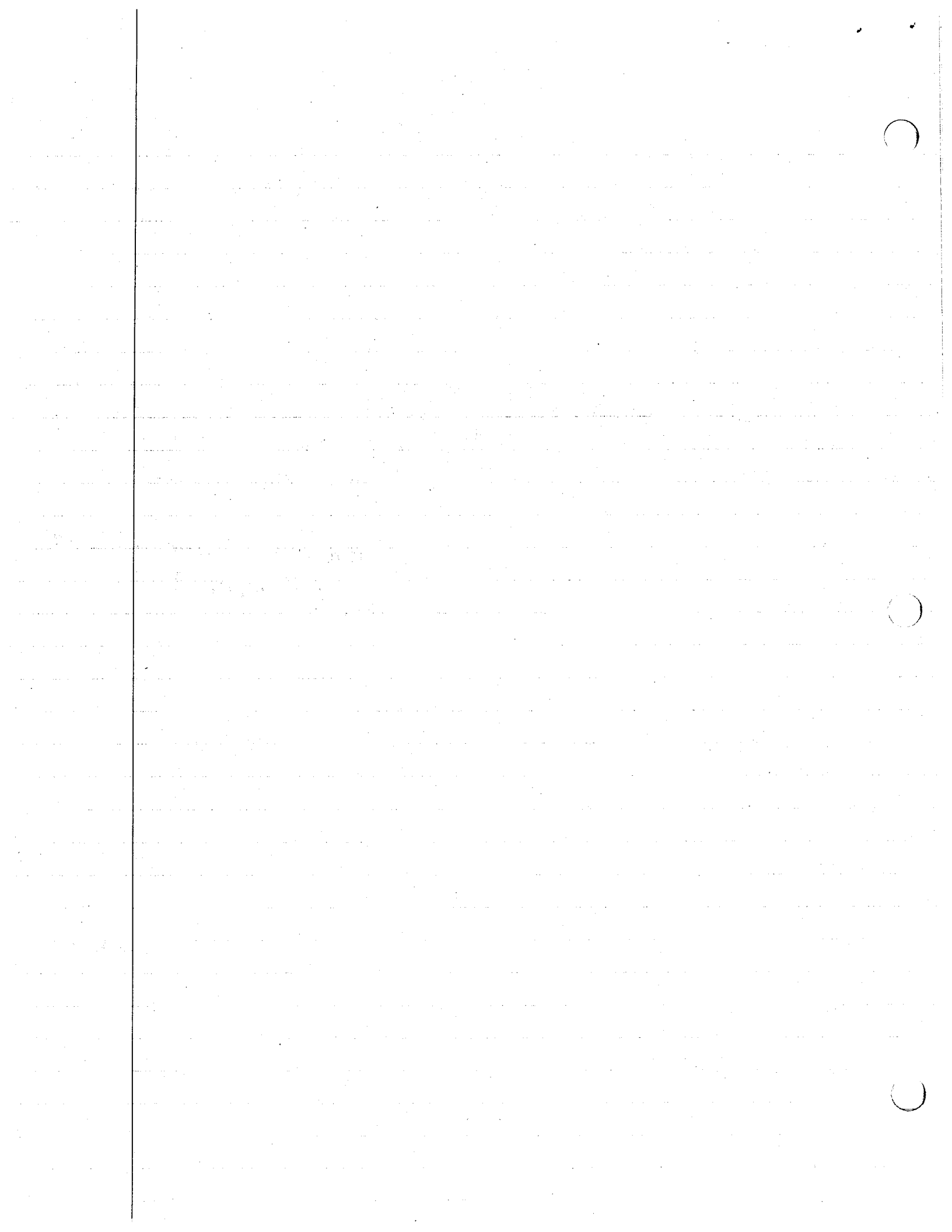
$$F_2 \triangleq (I_3 - I_1) \omega_3 \omega_1 + J \dot{\omega}^B [\omega_1 \beta_3 - \omega_3 \beta_1] \quad (9c)$$

$$F_3 \triangleq (I_1 - I_2) \omega_1 \omega_2 + J \dot{\omega}^B [\omega_2 \beta_1 - \omega_1 \beta_2] \quad (9d)$$

and T is the induced torque on B by A when B is free to rotate relative to A. [?]

With the equations (4-7) the significant parameters that affect the stability of motion of body A may be studied based upon the induced torque T. We note that β is defined in the derivation section.

What is "based upon ... T"?
Body A? Motion?



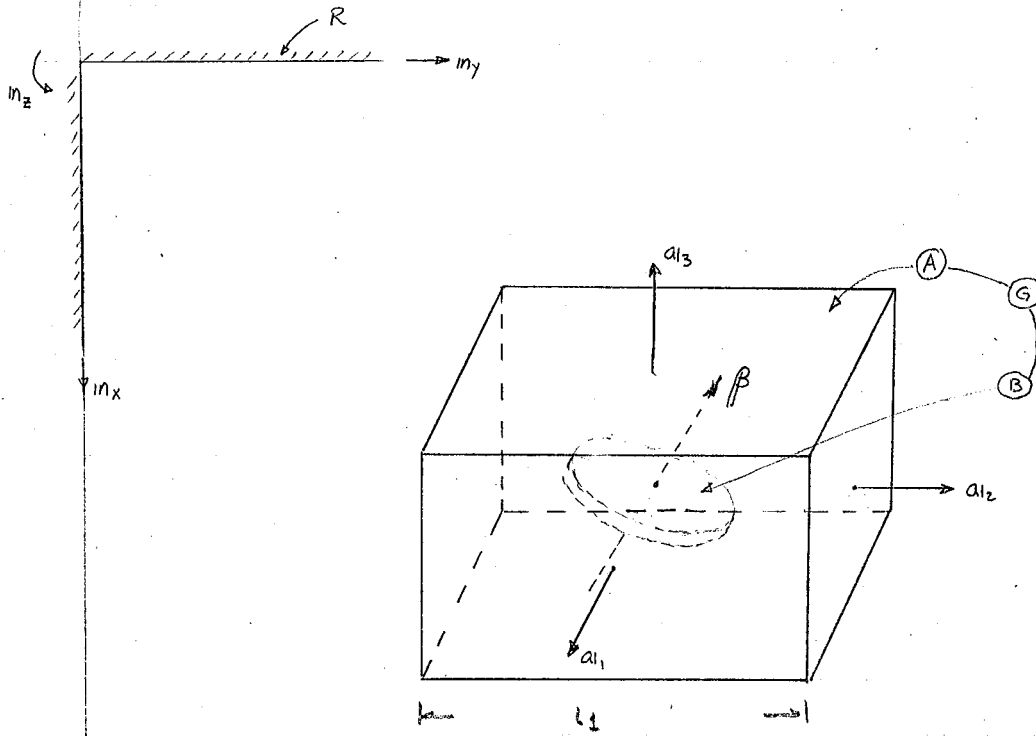
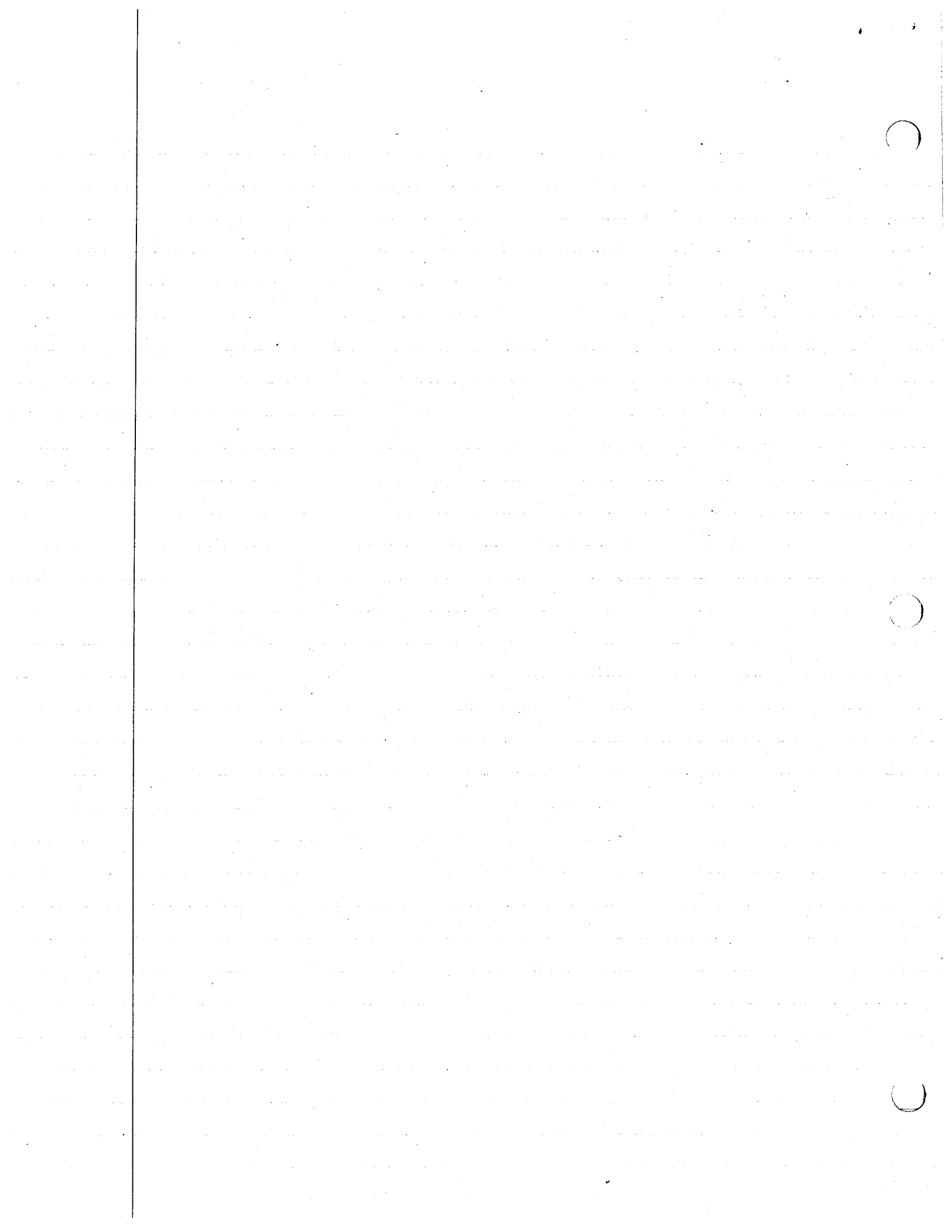


FIG. 1.

SIMPLE GYROSTAT SYSTEM



Where is the program?

2. Results

An initial value problem was solved for starting values of ω_i ($i=1,2,3$) and $\dot{\omega}^B$ equal to zero by varying the following parameters one at a time:

$\mu = \frac{m_b}{m_b + m_a}$, the ratio of the rotor mass to that of the rotor + body mass;
 ~~μ does not appear in the equations of motion. How can changing μ affect the motion?~~
 T/J , the ratio of induced torque to rotor polar moment of inertia;

α, β the cant angle of the rotor axis; how defined?

$b = L_2/L_1$, $c = L_3/L_1$, where L_1, L_2, L_3 are the rectangular body's lengths;

$d = r/2L_1$ where r is the rotor radius; and

L_1 being the length of the side of the rectangular body whose normal is a ,

Figures (2-6) present the results for the variables indicated on the figures for $|{}^R\omega^A|$ versus T (time) and $\dot{\omega}^B$ versus T for times up to 40 seconds.
(same symbol as for torque!)

Figures (2a,b) indicate the variation of μ with the remaining variables being unchanged. We note the linear character of the plots and that increase in μ causes corresponding increases in both $|{}^R\omega^A|$ and $\dot{\omega}^B$. We also observe that $\dot{\omega}^B$ is very much larger than $|{}^R\omega^A|$ but that the magnitude of the changes in $|{}^R\omega^A|$ between $\mu=0.03$ and $\mu=0.1$ is larger than the magnitude of $\dot{\omega}^B$. The trend of the plots can be explained because the dominance of the rotor becomes significant.

We also find that by keeping μ fixed and varying the rotor mass m_b there is no change in the plots and the results for, let's say $\mu=0.03$, are characterized by the $\mu=0.03$ plots in Figures 2a and 2b. We see this when we note that $m_b/m_a = (1-\mu)/\mu$ and that for fixed μ , changes in m_b require changes in m_a such that the ratio of moments of inertia to rotor polar moment of inertia is unchanged.

Figure 3 indicates the variations of T/J , all else being fixed; again we note the linear character of the plots and that $\hat{\omega}^B$ is still larger than $|{}^R\omega^A|$. As T/J increases tenfold we note that $\hat{\omega}^B$ and $|{}^R\omega^A|$ does so, too. This is so because the dominant term in our differential equations is the T/J term.

Figure 4 provides a plot for variations in the cant angles α and β . We note that the same results occur no matter what the variation in α or no matter what the variations of β . The plots are such that each angle is varied separately while the other is kept fixed. This is due to the cube nature of body A. From the numerical results obtained when only varying α (not plotted) show that for $\alpha = 90^\circ$ the ω_3 component is zero and that both ω_1 and ω_2 double when α changes from 30° to 90° . Similarly when β is the only parameter varied, from the numerical results (not plotted) the ω_1 component is zero for $\beta = 90^\circ$, ω_2 doubles when β changes from 30° to 90° , but ω_3 remains unchanged. These phenomena can be explained due to the fact that the components of the rotor axis ($\beta_1, \beta_2, \beta_3$) are close to being zero or one and therefore suppress terms in the differential equations. If the starting values of α and β were not 5° and 10° respectively, these effects would not be as apparent.

Figure 5 is a plot of the variations in both b and c . Again we note the linear character of all these plots and that as either b or c increases, $|{}^R\omega^A|$ decreases; this is due to conservation of momentum. As b , for instance, increases then correspondingly I_2 and I_3 increase. Since initially all the values of ω_i and $\hat{\omega}^B$ were zero, the initial momentum was zero. Because of the increase in $\hat{\omega}^B$ when the rotor is spun up, then the pertinent momentum equations

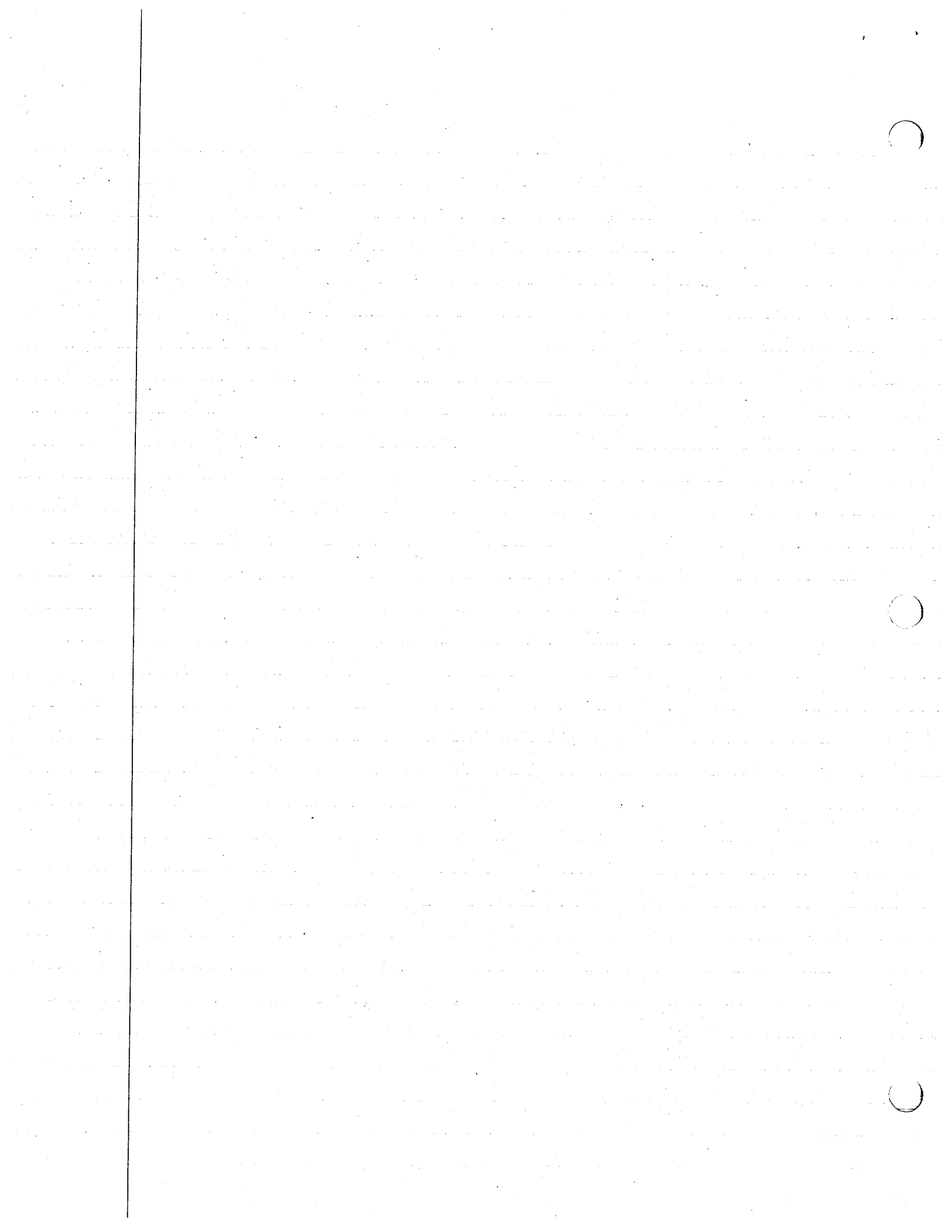
$$I_2 \omega_2 + \beta_2 J \hat{\omega}^B = 0$$

$$I_3 \omega_3 + \beta_3 J \hat{\omega}^B = 0$$

show that for fixed $I_2, I_3, \beta_2, \beta_3$ and J , that ω_2 and ω_3 must increase. Also that for fixed β_3, J and $\hat{\omega}^B$, an increase in I_2 or I_3 (ie when b increases) requires a corresponding decrease in ω_2 and ω_3 .

Since $|{}^R\omega^A| = \sum_{i=1}^3 (\omega_i \omega_i)^{1/2}$ and ω_i being close to the same value (numerical results not shown) then $|{}^R\omega^A|$ must decrease.

Figure 6 provides us with the plots of the variations in d , the rotor diameter non dimensionalization factor. We note again the linear character of the plots and as in all cases that $|{}^R\omega^A|$ is less than its corresponding $\hat{\omega}^B$ plot.



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Here, as in the case of the μ variations, as d increases $|{}^R\omega^A|$ also increases. This is due to the rotor size being "felt" by the rectangular body (ie $J \propto d^2$).

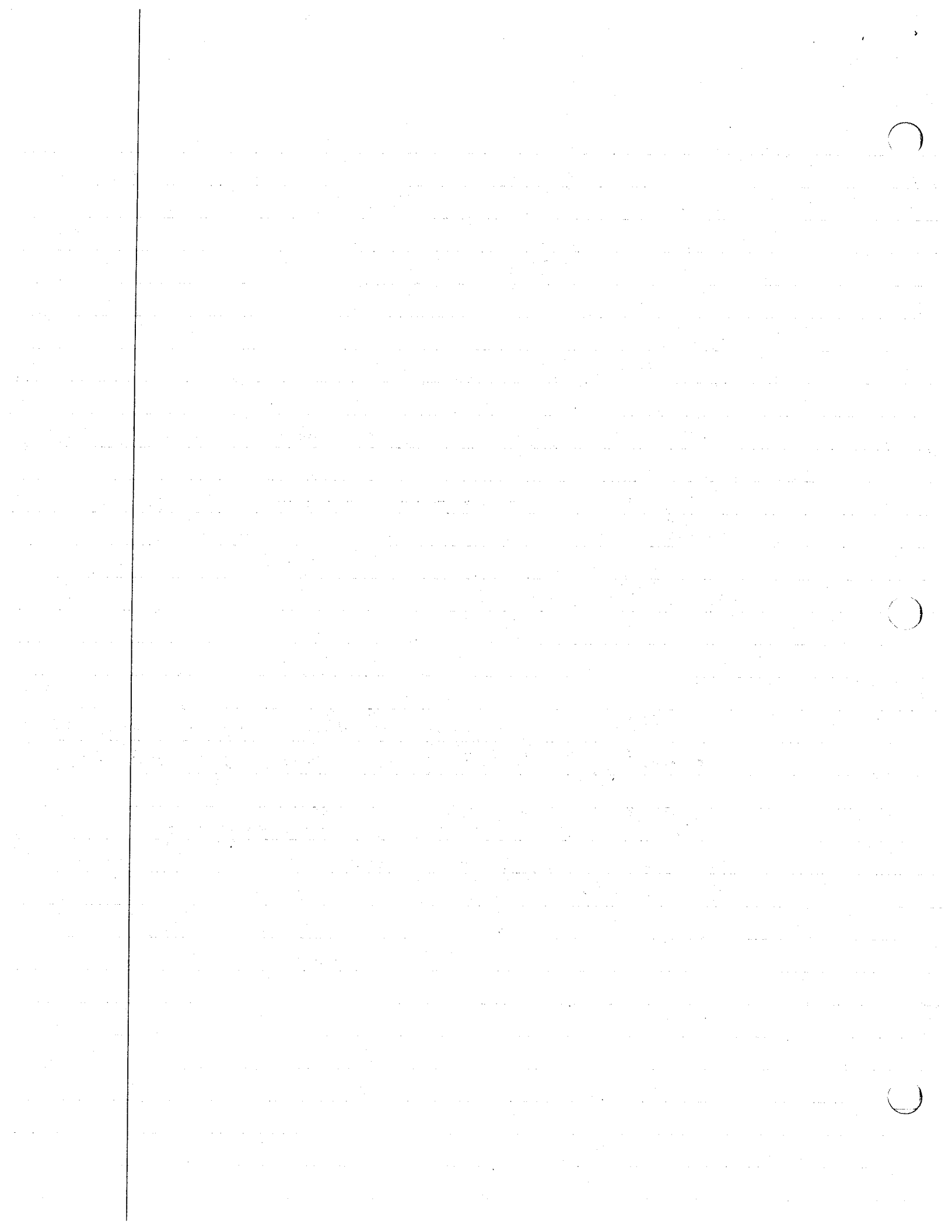
It is interesting to note that even though there are large variations in all the plots of $|{}^R\omega^A|$, the variations of ${}^A\omega^B$ is still close even at time $T = 40$ sec. Of course we expect this to change over longer periods of time.

Based on the results obtained we may then recommend the following in order to keep the rectangular body's motion stable:

1. Keep μ as small as possible yet large enough for the rotor to do its job;
2. Keep T/J small or cycle the rotor so that T varies sinusoidally over a short period;
3. If the cant angle of the rotor axis is to lie along one of the central principle axes, increase the length of sides "perpendicular" to the rotor axis (ie if axis is parallel to L_3 , increase L_1 and L_2);
4. Keep the rotor diameter as small as possible.

What does any of this have to do with stability? What do even mean by stability?

So far as I can see, the dynamics aspect is o.k. The use you made of the simulation is questionable.



FIGURES (2a, b)

$M_b = 10 \text{ KG}$

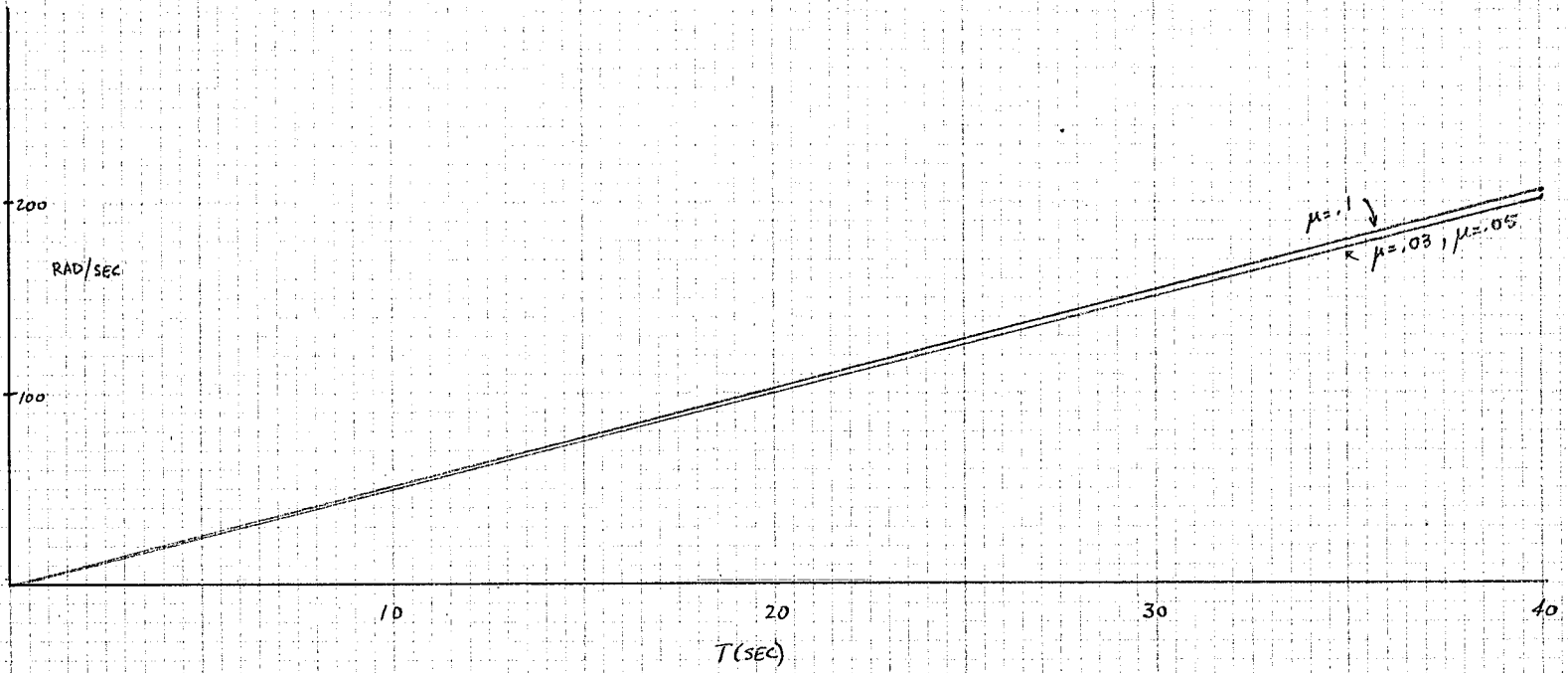
$T/J = 5 \text{ RAD/SEC}^2$

$\alpha = 5^\circ; \beta = 10^\circ$

$b = 1.0; c = 1.0; d = 0.5$

$L = 10 \text{ M}$

A_{WB} vs. T



$|R_{WA}|$ vs. T

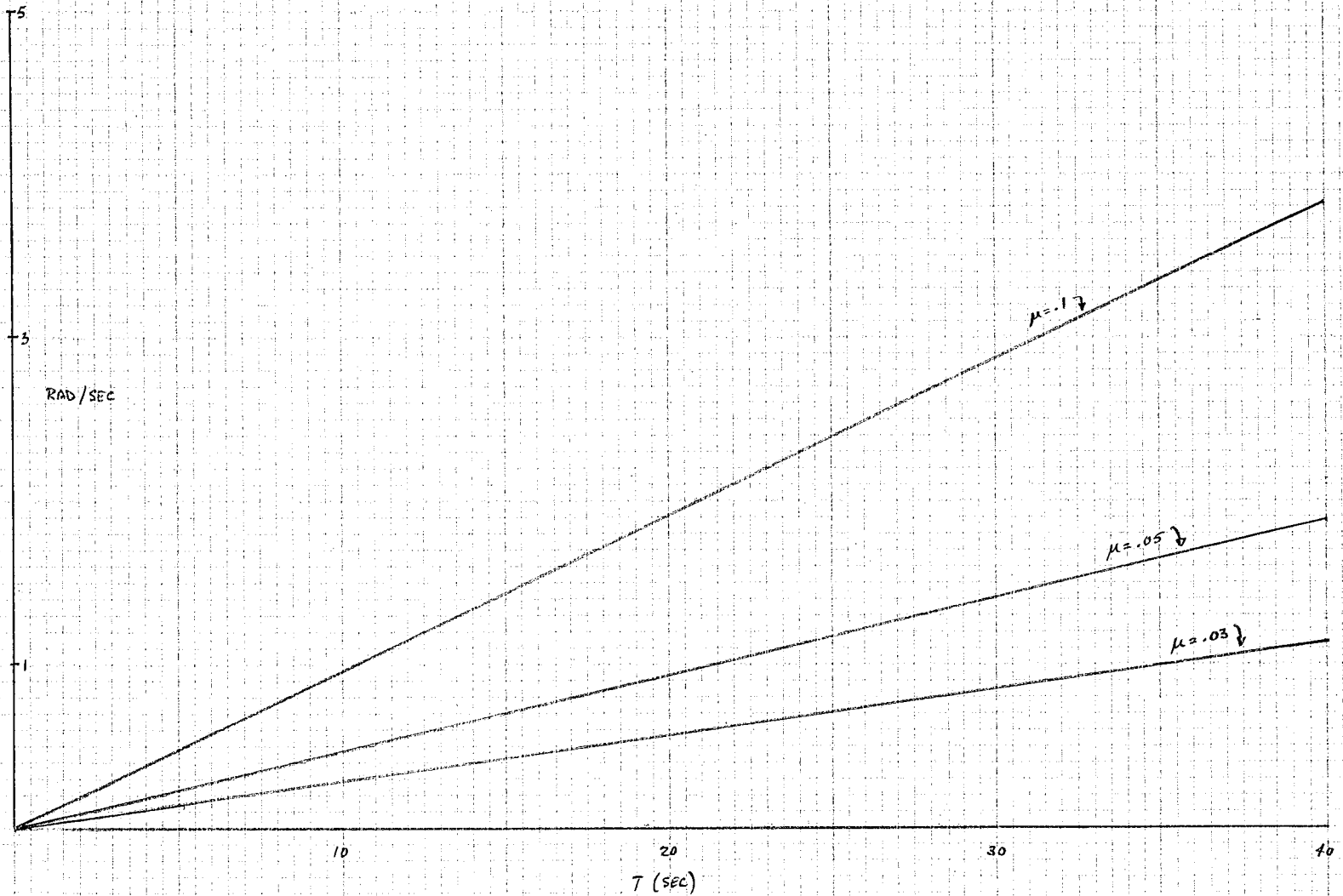
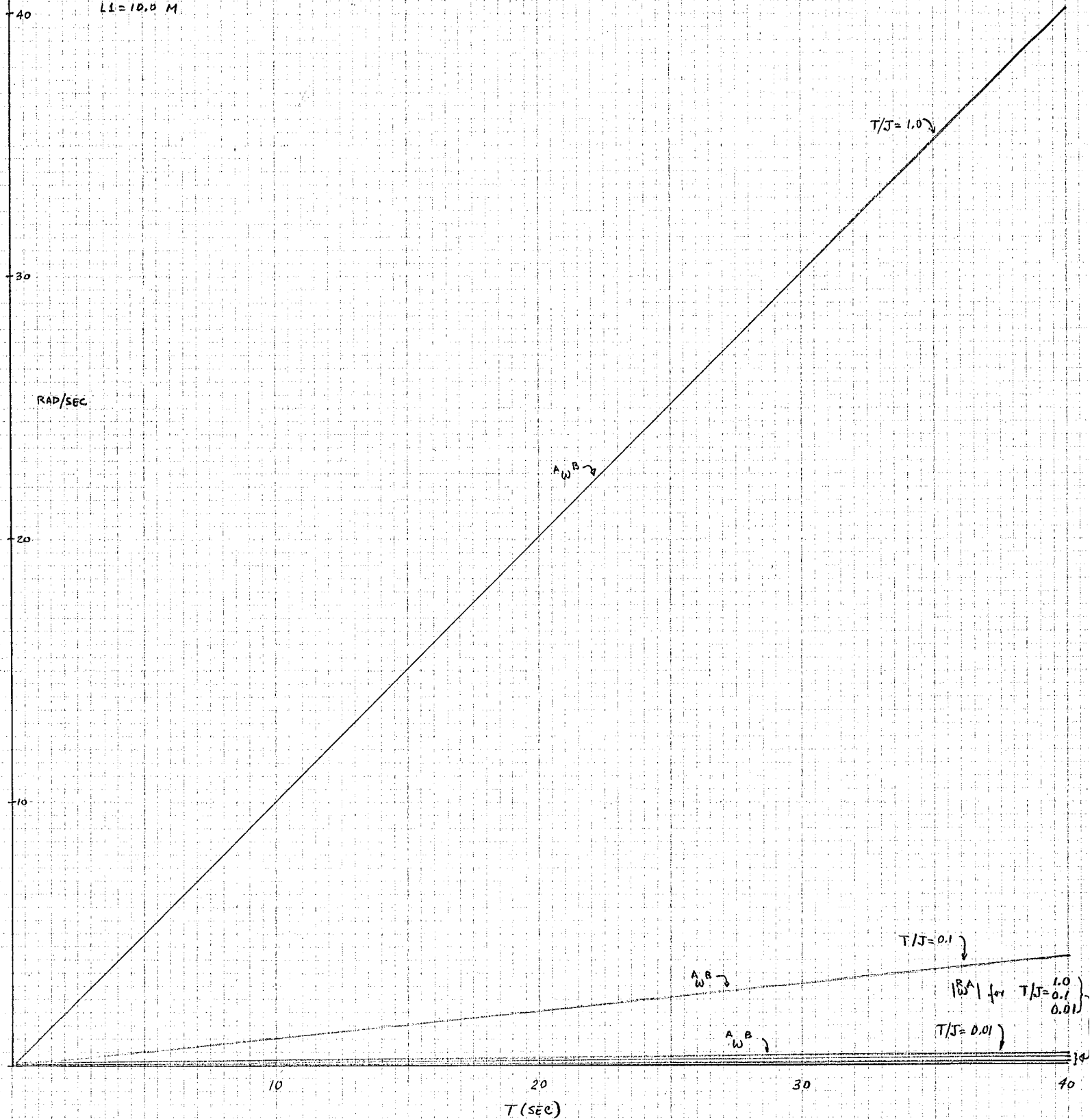


FIGURE 3
 ω^B and $|\omega^A|$ vs. T

$\mu = 0.8$
 $m_b = 10 \text{ KG}$
 $\alpha = 5^\circ, \beta = 10^\circ$
 $b = c = 1.0, d = 0.5$
 $L = 10.0 \text{ M}$



ω^B vs. T

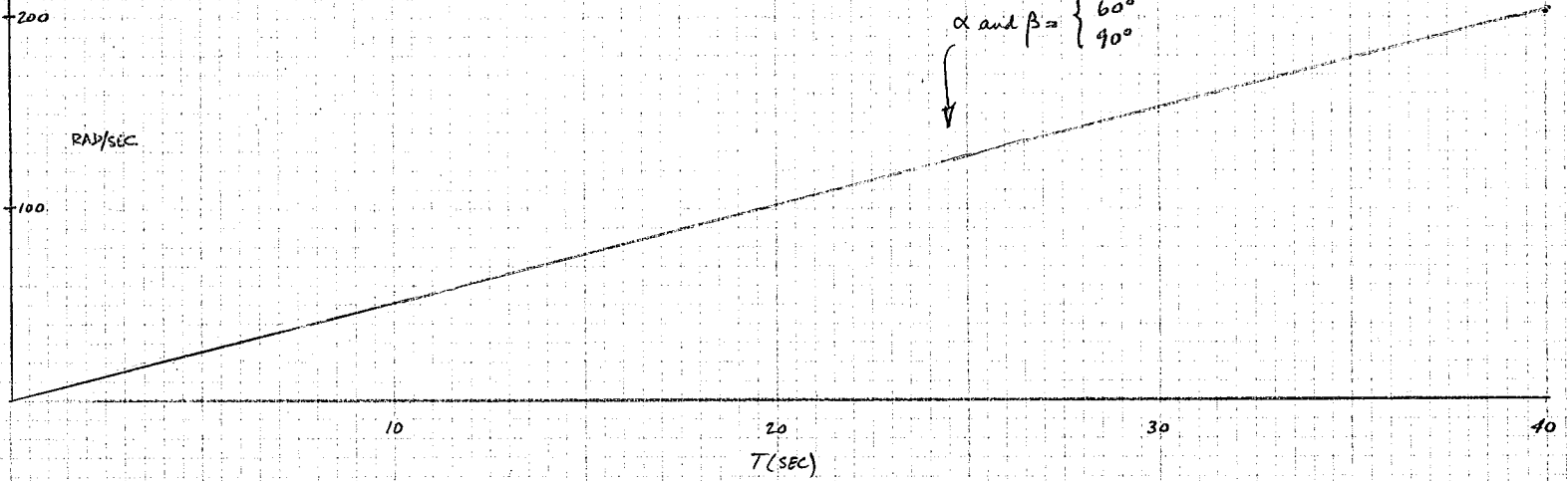
$\mu = .03$

$m_b = 10 \text{ KG}$

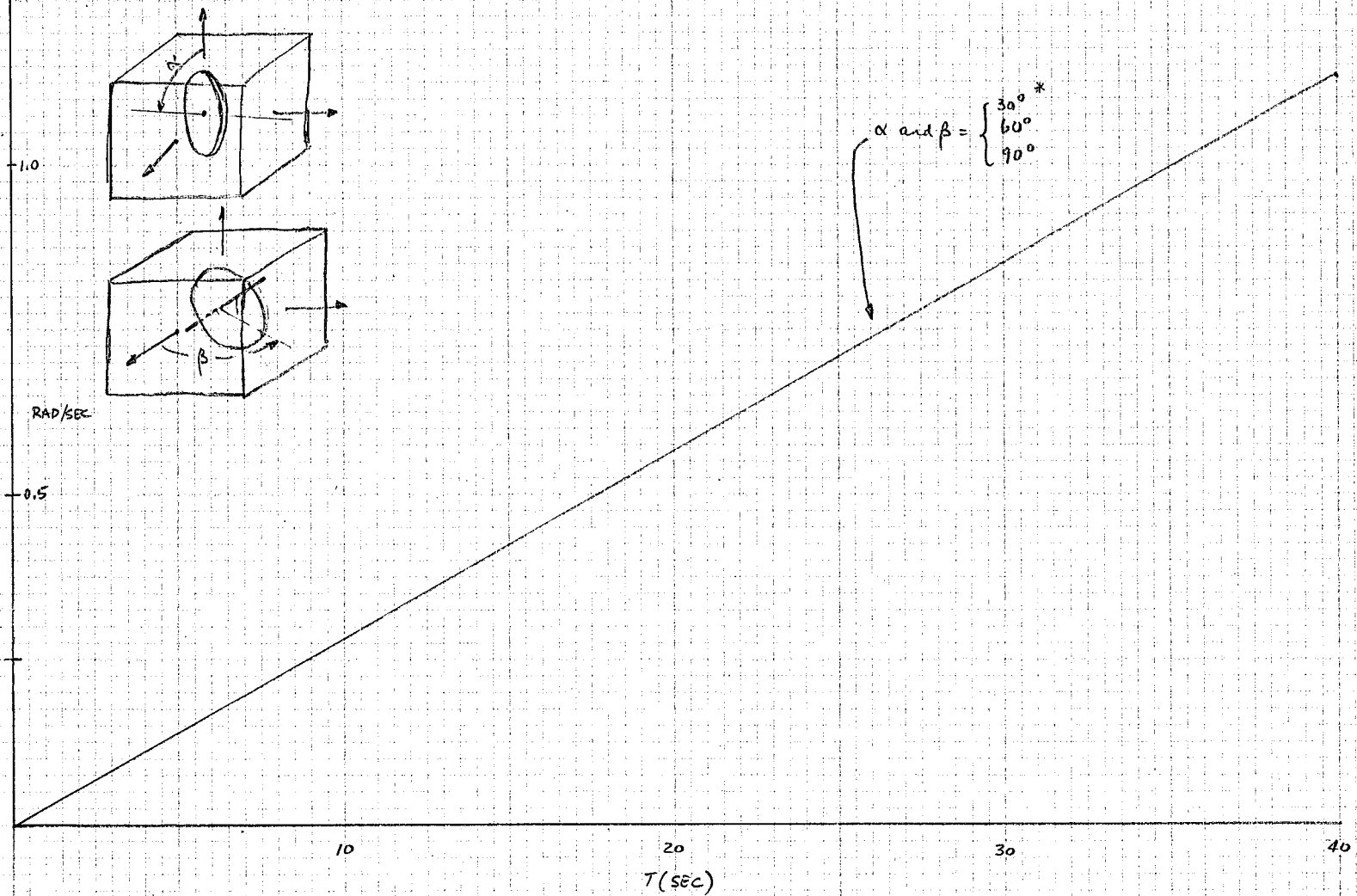
$T/J = 5 \text{ RAD/SEC}^2$

$b=c=1.0, d=0.5$

$LL = 10 \text{ M}$



$|\omega^A|$ vs. T

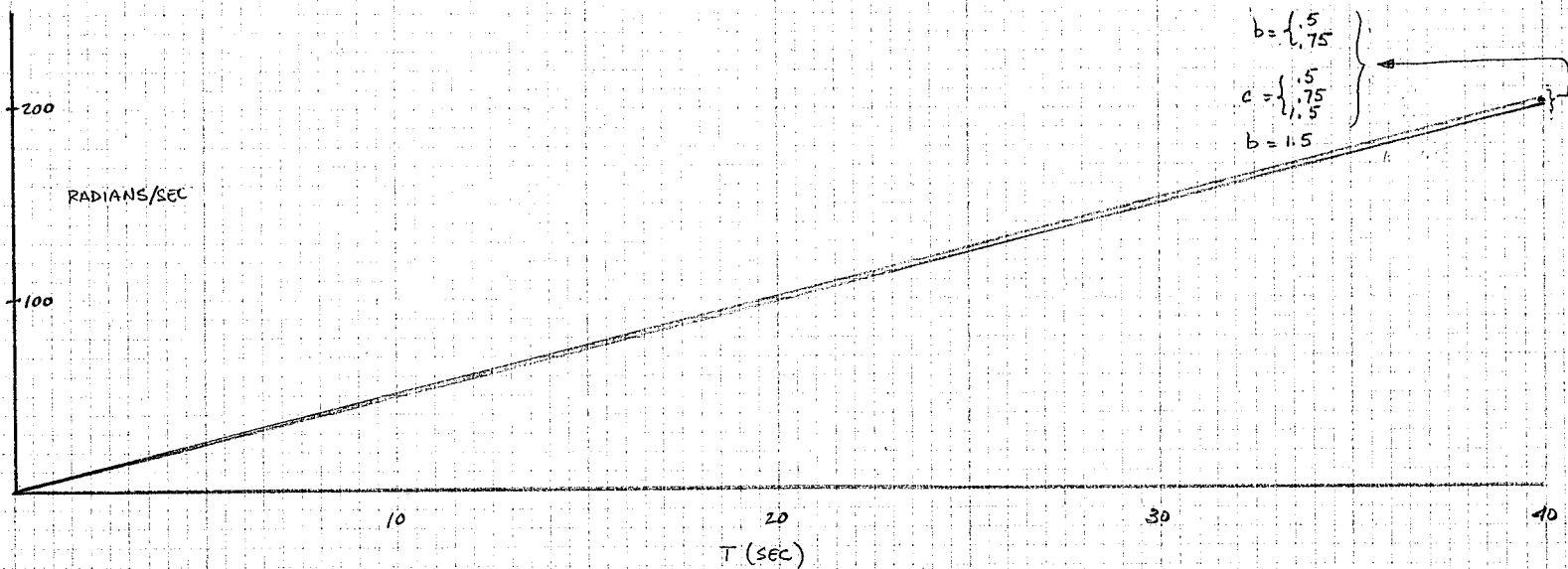


FIGURES (Aa, b)

* WHEN α IS VARIED β IS KEPT FIXED AT 10°
WHEN β IS VARIED α IS KEPT FIXED AT 5°

FIGURES (5a,b)

$\dot{\omega}^B$ vs T



$\mu = .03$

$m_b = 10 \text{ KG}$

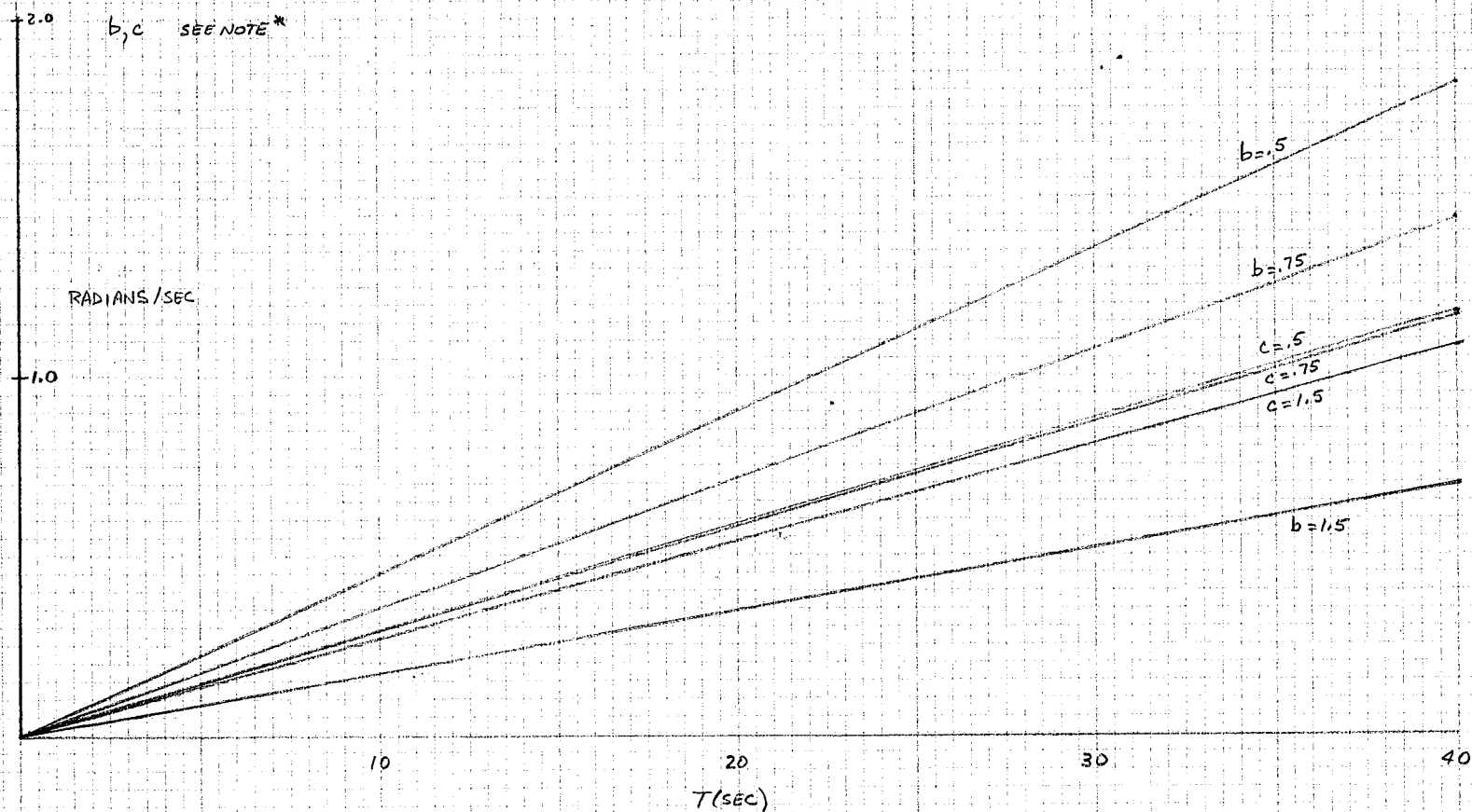
$T/J = 5 \text{ RADE/SEC}^2$

$\alpha = 5^\circ, \beta = 10^\circ$

$d = 0.5$

b, c SEE NOTE*

$|\dot{\omega}^A|$ vs T

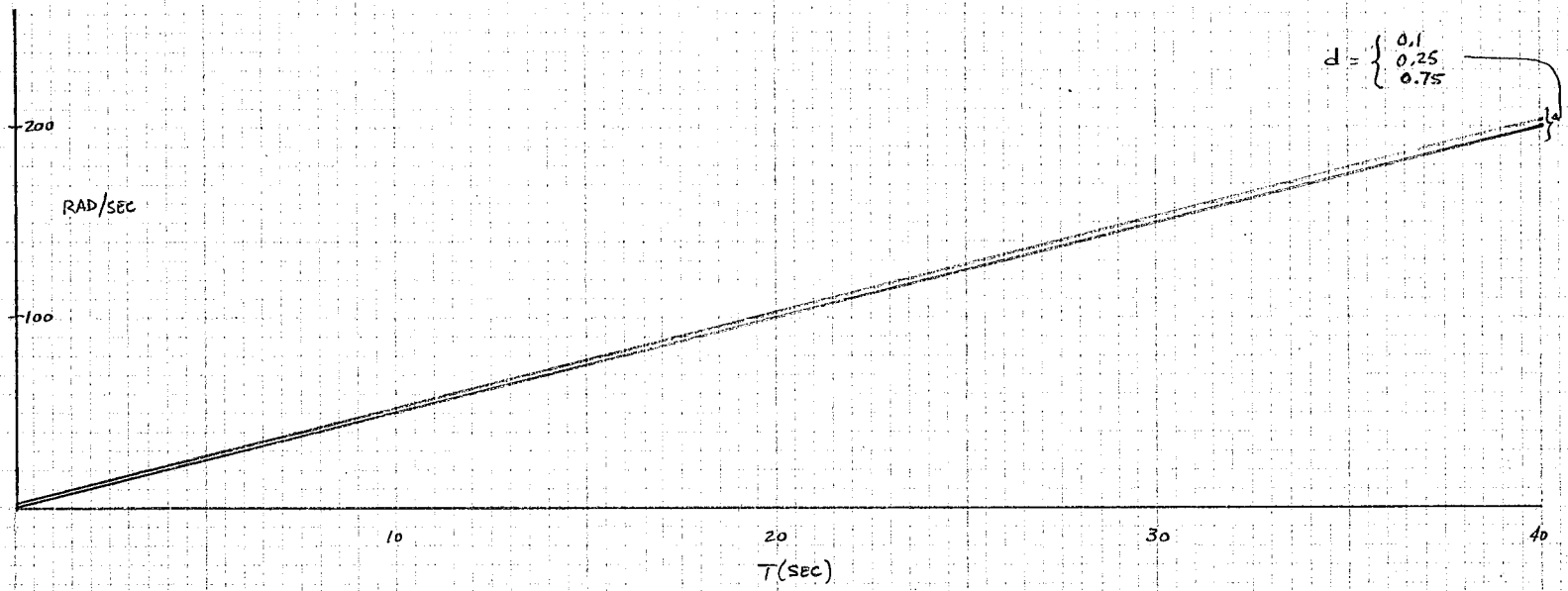


FOR VARIATIONS IN b , $c = 1.0$ ALWAYS

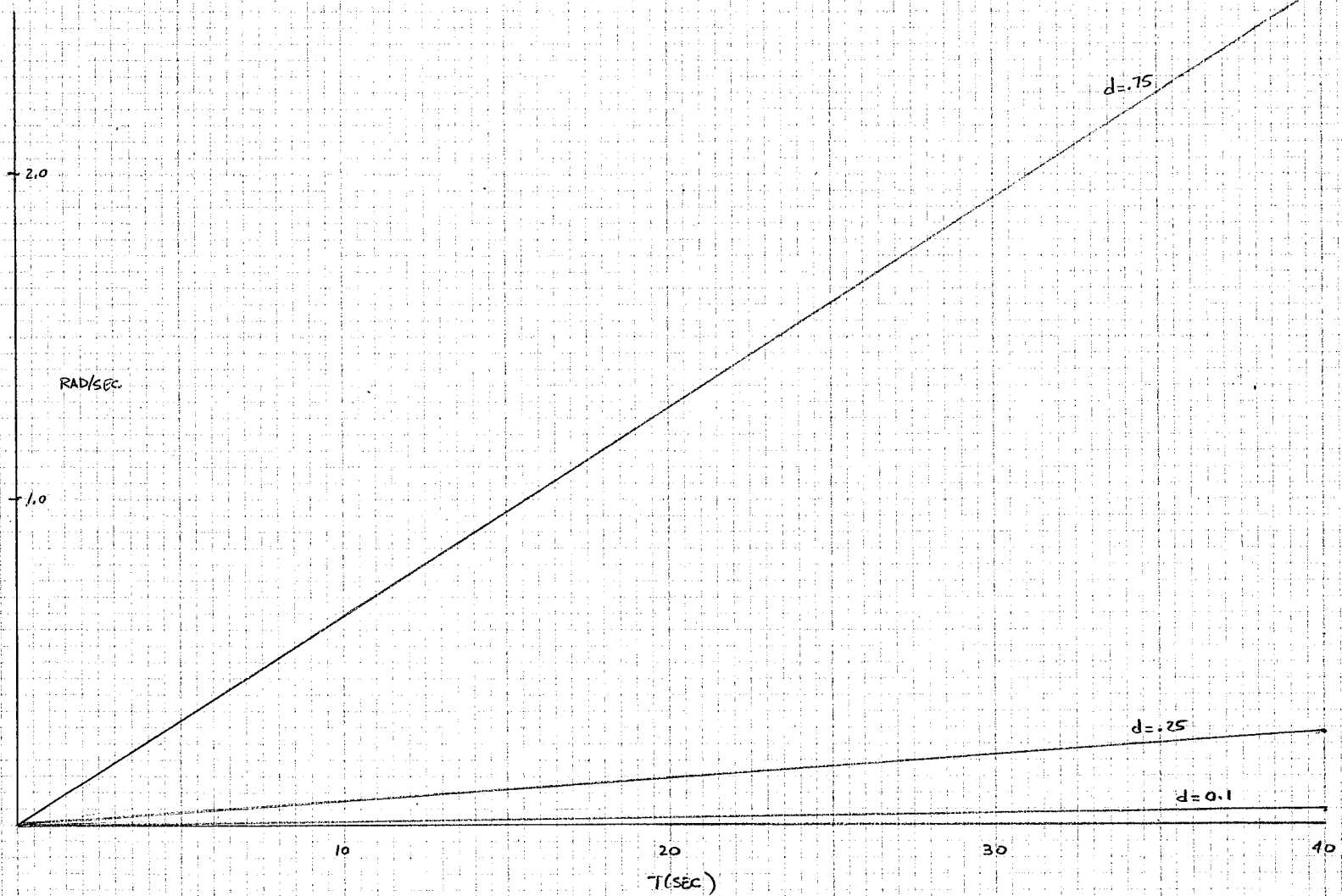
FOR VARIATIONS IN c , $b = 1.0$ ALWAYS

FIGURES (6a, b)

ω^B vs. T



$|R\omega^A|$ vs. T



3. Derivation

In deriving equations (4-7) we note that our system has seven degrees of freedom, namely: (x, y, z) the coordinates of the center of mass of G (also of A and B); ω_i ($i=1,2,3$) and $\hat{\omega}^B$ as defined previously. We then use Lagrange's Form of D'Alembert's Principle

$$F_r + F_r^* = 0 \quad r=1, \dots, 7 \quad (1)$$

For x, y, z we may define $(x, y, z) = (q_1, q_2, q_3)$ coordinates and

$$F_r = V_{\dot{q}_r}^A \cdot IF^A + V_{\dot{q}_r}^B \cdot IF^B + \omega_{\dot{q}_r}^A \cdot \Pi^A + \omega_{\dot{q}_r}^B \cdot \Pi^B \quad (2)$$

where $V_{\dot{q}_r}^A$ and $V_{\dot{q}_r}^B$ are the partial velocities of the points of application of resultant of all applied forces IF^A and IF^B on bodies A and B in R ; $\omega_{\dot{q}_r}^A$ and $\omega_{\dot{q}_r}^B$ are the partial angular velocities of the points of application of the resultant of all applied torques Π^A and Π^B on bodies A and B in R .

We may also define

$$F_r^* = V_{\dot{q}_r}^{A*} \cdot IF^{*A} + V_{\dot{q}_r}^{B*} \cdot IF^{*B} + \omega_{\dot{q}_r}^{A*} \cdot \Pi^{*A} + \omega_{\dot{q}_r}^{B*} \cdot \Pi^{*B} \quad (3)$$

where $V_{\dot{q}_r}^{A*}$ and $V_{\dot{q}_r}^{B*}$ are the partial velocities of the centers of mass of bodies A and B in R ; $IF^{*A} = -m_A \hat{a}^{*A}$ and $IF^{*B} = -m_B \hat{a}^{*B}$, m_A and m_B are the masses of bodies A and B and $\hat{a}^{*}$ is the acceleration of the center of mass; Π^{*A} and Π^{*B} are the inertia torques of bodies A and B .

For ω_i ($i=1,2,3$) and $\hat{\omega}^B$ we may define $(\omega_1, \omega_2, \omega_3, \hat{\omega}^B) = (u_1, u_2, u_3, u_4)$ generalized coordinates;

$$\text{thus } F_r = V_{u_r}^A \cdot IF^A + V_{u_r}^B \cdot IF^B + \omega_{u_r}^A \cdot \Pi^A + \omega_{u_r}^B \cdot \Pi^B \quad (4)$$

$$\text{and } F_r^* = V_{u_r}^{A*} \cdot IF^{*A} + V_{u_r}^{B*} \cdot IF^{*B} + \omega_{u_r}^{A*} \cdot \Pi^{*A} + \omega_{u_r}^{B*} \cdot \Pi^{*B} \quad (5)$$

where $()_{u_r}$ are the partial rates with respect to the generalized coordinates.

$$\text{Since } V^A = V^B = V^{*A} = V^{*B} = \dot{x}m_x + \dot{y}m_y + \dot{z}m_z \quad \text{and} \quad (6)$$

$$\hat{a}^{*A} = \hat{a}^{*B} = \ddot{x}m_x + \ddot{y}m_y + \ddot{z}m_z \quad \text{and because} \quad (7)$$

$${}^R\omega^A = \omega_1 a_1 + \omega_2 a_2 + \omega_3 a_3 \quad \text{and} \quad {}^R\omega^B = {}^R\omega^A + \hat{\omega}^B = {}^R\omega^A + \hat{\omega}^B \beta \quad (8,9)$$

we may then write, using $\Pi^A = -T\beta$ and $\Pi^B = T\beta$, $IF^A = m_A g m_x$, $IF^B = m_B g m_x$

$$F_1 = Mg$$

$$F_5 = 0$$

$$F_2^* = -M\ddot{y}$$

$$F_2 = 0$$

$$F_6 = 0$$

$$F_3^* = -M\ddot{z}$$

$$(10-18)$$

$$F_3 = 0$$

$$F_7 = T$$

$$\text{with } M = m_A + m_B$$

$$F_4 = 0$$

$$F_1^* = -M\ddot{x}$$



Now to obtain $F_4 - F_7$ we may rewrite (6) as

$$\begin{aligned} F_r^* &= W_{ur}^{*A} \cdot (I_r^{*A} + I_r^{*B}) + \omega_{ur}^A \cdot (\pi^{*A} + \pi^{*B}) + \omega_{ur}^{AB} \cdot \pi^{*B} \\ &= \omega_{ur}^A \cdot (\pi^{*A} + \pi^{*B}) + \omega_{ur}^{AB} \cdot \pi^{*B} \end{aligned} \quad (19)$$

Since $W_{ur}^{*A} = 0$

$$\begin{aligned} \text{Now } \pi^{*A} + \pi^{*B} &= -\frac{d}{dt} (I^{*A} + I^{*B}) = -\frac{d}{dt} (I^{A/B} \cdot \omega^A + I^{B/B} \cdot \omega^B) \\ &= -\frac{d}{dt} (I^{A/B} \cdot \omega^A + I^{B/B} \cdot \omega^B) \\ &= (I^{G/B} \cdot \omega^A) \times \omega^A - I^{G/B} \cdot \alpha^A + (I^{B/B} \cdot \omega^B) \times \omega^B - I^{B/B} \cdot \alpha^B + (I^{B/B} \cdot \omega^B) \times \omega^B \end{aligned} \quad (20)$$

Using (8,9)

$$\alpha^A = \dot{\omega}_1 a_1 + \dot{\omega}_2 a_2 + \dot{\omega}_3 a_3 \quad \alpha^B = \dot{\omega}^B \beta \quad (21, 22)$$

Remembering that $I^{G/B} = I_1 a_1 a_1 + I_2 a_2 a_2 + I_3 a_3 a_3$ and $I^{B/B} = J \beta \beta + \dots$ (23, 24)

and using these in (19) along with (20) and (8,9) we find

$$F_4^* = [\omega_2 \omega_3 (I_2 - I_3) - I_1 \dot{\omega}_1 - J \dot{\omega}^B \beta_1 + J \dot{\omega}^B (\beta_2 \omega_3 - \beta_3 \omega_2)] \quad (25)$$

$$F_5^* = [\omega_3 \omega_1 (I_3 - I_1) - I_2 \dot{\omega}_2 - J \dot{\omega}^B \beta_2 + J \dot{\omega}^B (\beta_3 \omega_1 - \beta_1 \omega_3)] \quad (26)$$

$$F_6^* = [\omega_1 \omega_2 (I_1 - I_2) - I_3 \dot{\omega}_3 - J \dot{\omega}^B \beta_3 + J \dot{\omega}^B (\beta_1 \omega_2 - \beta_2 \omega_1)] \quad (27)$$

$$F_7^* = -J [\dot{\omega}_1 \beta_1 + \dot{\omega}_2 \beta_2 + \dot{\omega}_3 \beta_3 + \dot{\omega}^B] \quad (28)$$

Using (10-18, 25-28) and (1) we obtain after rearranging

$$\ddot{x} = g \quad (29)$$

$$\ddot{y} = 0 \quad (30)$$

$$\ddot{z} = 0 \quad (31)$$

and the equations shown in section 1.

β is defined by

$$\beta = \cos \alpha a_3 + \sin \alpha \sin \beta a_2 + \sin \alpha \cos \beta a_1 \quad (31)$$



For a set S of particles $P_1, \dots, P_N$, a quantity G , known as a Gibbs function, is defined as

$$G \triangleq \frac{1}{2} \sum_{i=1}^N m_i \underline{a}_i^2$$

where m_i and $\underline{a}_i$ are, respectively, the mass and the acceleration of P_i . Regarding a rigid body B as a set S of a large number of particles, express G as a function of the acceleration $\underline{a}$ of the mass center of B , the angular velocity $\underline{\omega}$ of B , the angular acceleration $\underline{\alpha}$ of B , the mass m of B , and the central inertia dyadic $\underline{I}$ of B .

$$G \triangleq \frac{1}{2} \sum_{i=1}^N m_i (a_i \cdot a_i)$$

$$\text{let } p_i = p + r_i$$

$$v_i = v^* + \omega^B \times r_i$$

$$a_i = a^* + \alpha \times r_i + \omega^B \times (\omega^B \times r_i)$$

$$a_i \cdot a_i = a^* \cdot a^* + 2a^* \cdot \alpha \times r_i + 2a^* \cdot \omega^B \times (\omega^B \times r_i) + (\alpha \times r_i)^2 + 2(\alpha \times r_i) \cdot [\omega^B \times (\omega^B \times r_i)] + [\omega^B \times (\omega^B \times r_i)]^2$$

$$\text{now } \sum_{i=1}^N m_i (a_i \cdot a_i) = \left(\sum_{i=1}^N m_i a^* \cdot a^* = m a^{*2} \right) + \left(\sum m_i a^* \cdot \alpha \times r_i = \sum m_i r_i \cdot (a^* \times \alpha) = 0 \right)$$

$$+ 2 \left(\sum m_i a^* \cdot \omega^B \times (\omega^B \times r_i) = a^* \cdot \omega^B \times (\omega^B \times \sum m_i r_i) = 0 \right)$$

$$+ \left(\sum m_i (\alpha \times r_i) \cdot (\alpha \times r_i) = \alpha^2 m_a \sum m_i r_i \times (m_a \times r_i) = \alpha^2 m_a \cdot \mathbb{I}_a^{B/B^*} \right)$$

$$+ 2 \left(\sum m_i (\alpha \times r_i) \cdot [\omega^B \times (\omega^B \times r_i)] = \alpha \omega^2 \sum m_i (m_a \times r_i) \cdot [m_o \times (m_o \times r_i)] = \right.$$

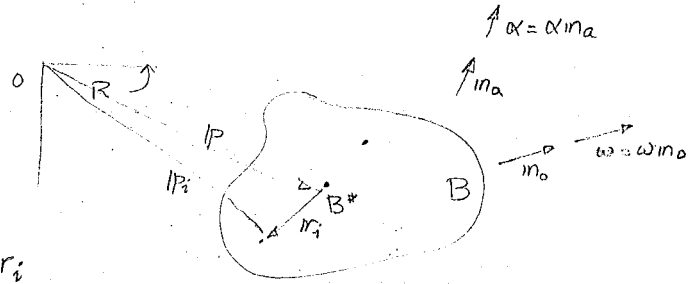
$$\alpha \omega^2 \sum m_i [r_i \times (m_o \times (m_o \times r_i))] \cdot m_a = \alpha \omega^2 m_a \cdot m_o \times \left[\sum m_i r_i \times (m_o \times r_i) \right] =$$

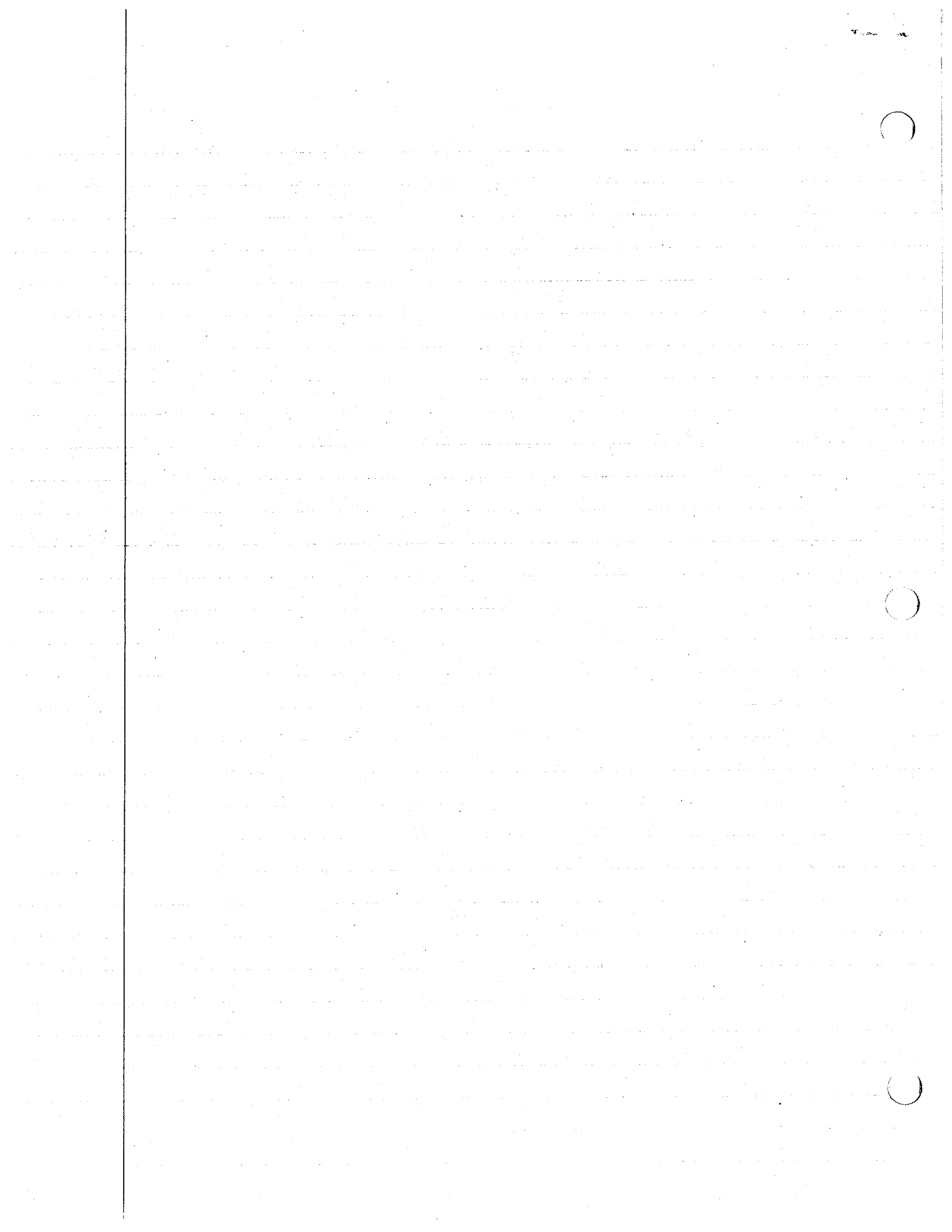
$$\omega \cdot \left[\alpha \cdot \left\{ \omega^B \times \mathbb{I}_o^{B/B^*} \right\} \right] + \left(\sum m_i [\omega^B \times (\omega^B \times r_i)]^2 = \sum m_i \left\{ \omega^B \cdot [\omega \times r_i]^2 - [\omega \cdot (\omega \times r_i)]^2 \right\} \right)$$

$$= \sum m_i \omega^B \cdot [r_i \times (\omega \times r_i)] \cdot \omega = (\omega^B)^2 \omega \cdot \sum m_i r_i \times (\omega \times r_i) = (\omega^B)^2 \omega \cdot \omega \cdot \mathbb{I}_o^{B/B^*}$$

$$\text{thus } G \triangleq \frac{1}{2} \left[m a^2 + \alpha \cdot \mathbb{I} \cdot \alpha + 2 \alpha \cdot \left\{ \omega^B \times \left[\omega \cdot \mathbb{I}^{B/B^*} \right] \right\} + \omega^2 (\omega \cdot \mathbb{I}^{B/B^*} \cdot \omega) \right]$$

$$G \triangleq \frac{1}{2} \left[m a^2 + \alpha \cdot \mathbb{I} \cdot \alpha + 2 \omega \cdot \left[(\mathbb{I}^{B/B^*} \cdot \omega) \times \alpha \right] + \omega^2 \omega \cdot \mathbb{I}^{B/B^*} \cdot \omega \right]$$





6a. let P & Q in $3(j)$ has equal weights W find F_1^* assoc. with θ_1

$$F_{\theta_1}^* = \sum_{i=1}^N \tilde{W}_{\theta_1}^{P_i} \cdot F_i^*$$

If we assume that the joints are smooth then



from previous $\tilde{W}_{\theta_1}^{P_i} = L \int_1$ $\tilde{W}_{\theta_1}^Q = L \int_2 \frac{\sin(\theta_3 - \theta_1)}{\sin(\theta_3 - \theta_2)} = L \frac{\sin(\theta_3 - \theta_1)}{\sin(\theta_3 - \theta_2)} [\cos \theta_2 m_y - \sin \theta_2 m_x]$

also $W^P = \dot{\theta}_1 L j_1$ and $W^Q = \dot{\theta}_1 L j_1 + \dot{\theta}_3 [-3L \sin \theta_3 m_x + 3L \cos \theta_3 m_y] = L \int_2 \frac{\sin(\theta_3 - \theta_1)}{\sin(\theta_3 - \theta_2)} \dot{\theta}_1$

$a_1^P = \frac{dW^P}{dt} = \ddot{\theta}_1 L j_1 + \dot{\theta}_1 L [-\sin \theta_1 \dot{\theta}_1 m_y - \cos \theta_1 \dot{\theta}_1 m_x] = \ddot{\theta}_1 L j_1 - \dot{\theta}_1^2 L \hat{1}_1$

thus $[\tilde{W}_{\theta_1}^P \cdot a_1^P] = L^2 \ddot{\theta}_1$

$a_1^Q = \frac{dW^Q}{dt} = \frac{d}{dt} (2 \dot{\theta}_2 L \int_2) = 2 \ddot{\theta}_2 L \int_2 + 2 \dot{\theta}_2 L \frac{d \int_2}{dt} = 2 \ddot{\theta}_2 L \int_2 - 2 \dot{\theta}_2^2 L \hat{1}_2$

$\tilde{W}_{\theta_1}^Q \cdot a_1^Q = L \int_2 \frac{\sin(\theta_3 - \theta_1)}{\sin(\theta_3 - \theta_2)} \cdot 2 \ddot{\theta}_2 L \int_2 = 2 \ddot{\theta}_2 L^2 \frac{\sin(\theta_3 - \theta_1)}{\sin(\theta_3 - \theta_2)}$

thus $F_{\theta_1}^* = -m \left[L^2 \ddot{\theta}_1 + 2 L^2 \ddot{\theta}_2 \frac{\sin(\theta_3 - \theta_1)}{\sin(\theta_3 - \theta_2)} \right] = -\frac{WL^2}{g} \left[\ddot{\theta}_1 + 2 \ddot{\theta}_2 \frac{\sin(\theta_3 - \theta_1)}{\sin(\theta_3 - \theta_2)} \right]$

6b. Example on pg 107 $\mathbb{I}_2^{P/O}$ is not $\parallel m_2$

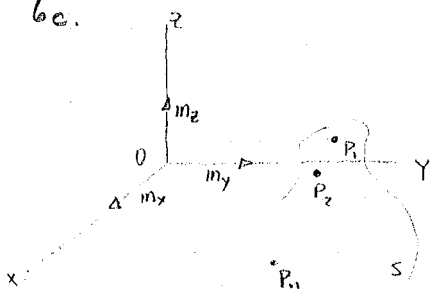
also in most cases $|p_i| = a_i m_a + b_i m_b$ when $a_i = m_a \cdot |p_i|$ $b_i = m_b \cdot |p_i|$ $m_a \cdot m_b = 1$

$\therefore m_a \times |p_i| = b_i m_a \times m_b$ let $m_a \times m_b = m_c$

now $\mathbb{I}_a = \sum_{i=1}^n m_i [a_i m_a + b_i m_b] \times b_i m_c$
 $= \sum_{i=1}^n m_i a_i b_i (-m_b) + \sum_{i=1}^n m_i b_i^2 m_a$

in most case $\sum_{i=1}^n m_i a_i b_i \neq 0 \therefore \mathbb{I}_a$ will not be parallel to m_a

6c.



$$\mathbb{I}_x = \sum m_i |p_i| \times (m_x \times |p_i|)$$

Let $|p_i| = x_i m_x + y_i m_y + z_i m_z$

$m_x \times |p_i| = y_i m_z - z_i m_y$

$\mathbb{I}_x = \sum m_i (x_i m_x + y_i m_y + z_i m_z) \times (y_i m_z - z_i m_y)$
 $= \sum m_i [-x_i y_i m_y - x_i z_i m_z + y_i^2 m_x + z_i^2 m_x]$

$\mathbb{I}_x = \mathbb{I}_x \cdot m_x = \sum m_i (y_i^2 + z_i^2)$



$$I_y = \sum m_i p_i \times (m_y \times p_i)$$

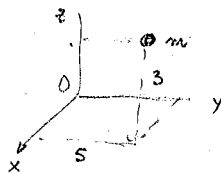
$$m_y \times p_i = -x_i m_z + z_i m_x$$

$$p_i \times (-x_i m_z + z_i m_x) = +x_i^2 m_y - y_i x_i m_x - y_i z_i m_z + z_i^2 m_y$$

$$I_{yz} = I_y \cdot m_z = -\sum m_i y_i z_i$$

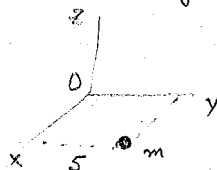
- RHS or LHS doesn't matter since you're taking the double cross product. This produces two minus signs which cancel each other.
- For non mutually $\perp$ coordinate system we will find that results are not as simple as those shown, because we will get components that involve the sines of the angle between these vectors. In a $\perp$ system the sines are ≈ 0 or 1 .
- The minus sign comes in from the definition of $I_{yz} = \Pi_y \cdot m_z$

6(d)



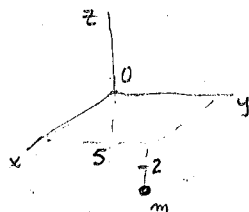
1 particle

$$I_{yz} = -m \cdot 15 = -15m < 0$$

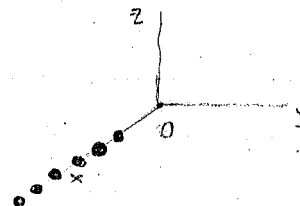


1 particle

$$I_{yz} = -m \cdot 5 \cdot 0 = 0$$



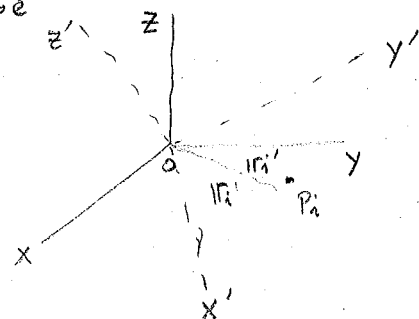
$$I_{yz} = -m(5)(-2) = 10m > 0$$



$$I_x = \sum_{i=1}^6 m_i (z_i^2 + y_i^2) = 0$$

$$\therefore k=0 \quad M = \sum_{i=1}^6 m_i$$

6e



$$I_x = A \quad I_y = B \quad I_z = C$$

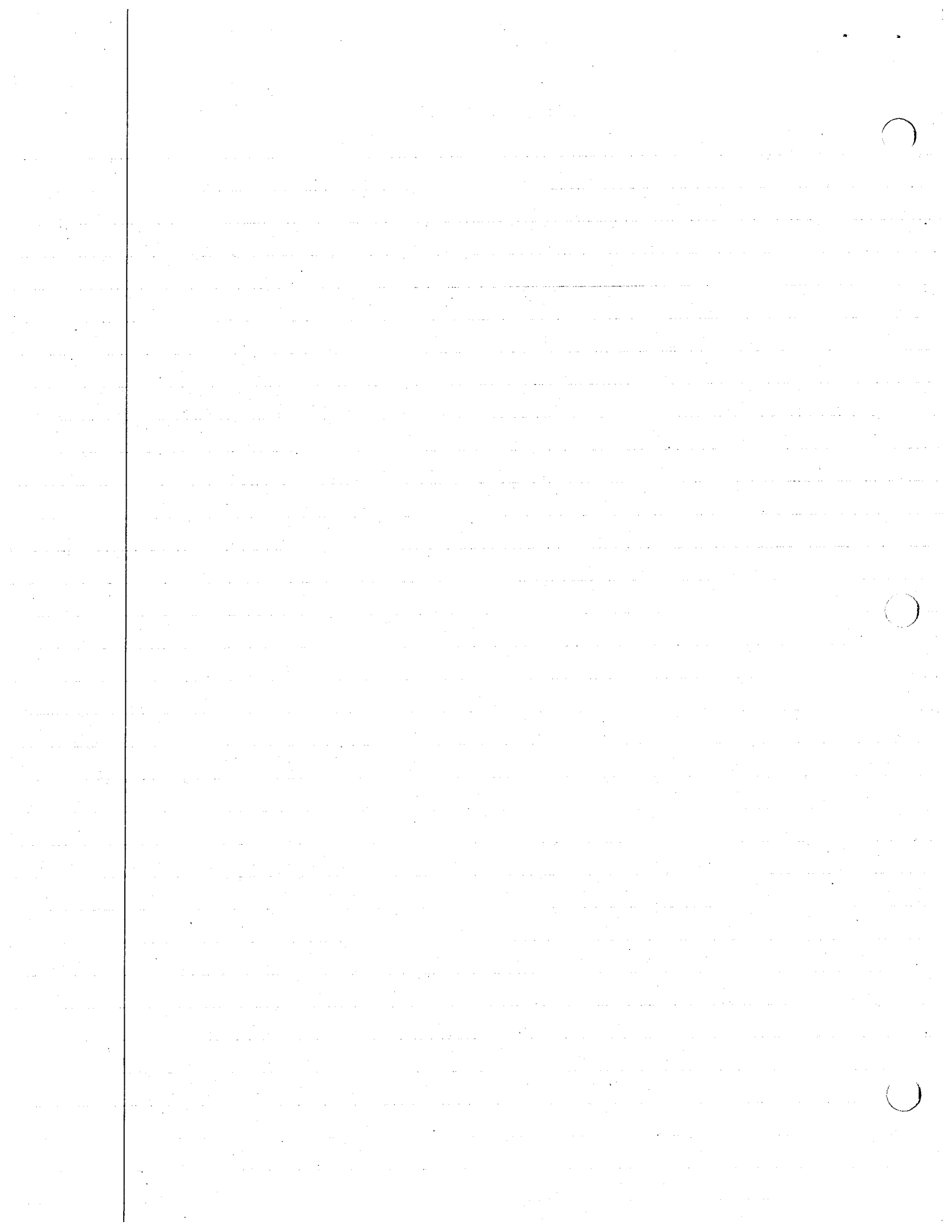
$$I_{x'} = A' \quad I_{y'} = B' \quad I_{z'} = C'$$

$$\text{now } I_x + I_y + I_z = \sum_{i=1}^N 2m_i (x_i^2 + y_i^2 + z_i^2) = \sum 2m_i r_i \cdot r_i$$

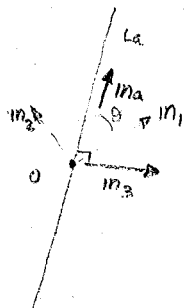
but $x_i^2 + y_i^2 + z_i^2 =$ the distance squared from O to the point P_i . Distances are the same under transformations

$$\text{hence } I_{x'} + I_{y'} + I_{z'} = \sum 2m_i r_i' \cdot r_i' = \sum 2m_i r_i \cdot r_i$$

$$\text{thus } I_x + I_y + I_z = I_{x'} + I_{y'} + I_{z'}$$



6f.



Since m_3 is $\perp$ to La then La must lie in the plane

of $m_1 \times m_2$ hence $m_a = (m_a \cdot m_1) m_1 + (m_a \cdot m_2) m_2$

$$\Pi = \sum_{j=1}^3 \sum_{k=1}^3 m_j I_{jk} m_k$$

$$m_a \cdot \Pi \cdot m_a = I_a = \sum_{j=1}^3 \sum_{k=1}^3 m_a \cdot m_j I_{jk} m_k \cdot m_a$$

$$m_a \cdot m_3 = 0$$

$$m_a = \begin{vmatrix} m_1 & m_2 \\ \cos \theta & \sin \theta \end{vmatrix}$$

now $I_a = I_1 \cos^2 \theta + I_2 \sin^2 \theta + I_{12} \sin 2\theta$

$$\frac{\partial I_a}{\partial \theta} = 0 \Rightarrow -I_1 \cdot 2 \cos \theta \sin \theta + 2 I_2 \sin \theta \cos \theta + 2 I_{12} \cos 2\theta = 0$$

$$-(-I_2 + I_1) \sin 2\theta + 2 I_{12} \cos 2\theta = 0 \text{ or } \tan 2\theta = \frac{2 I_{12}}{I_1 - I_2}$$

$$\triangle \begin{matrix} 2\theta \\ I_1 - I_2 \end{matrix}$$

$$I_a = I_1 \left(\frac{\cos 2\theta + 1}{2} \right) + I_2 \left(\frac{1 - \cos 2\theta}{2} \right) + I_{12} \sin 2\theta$$

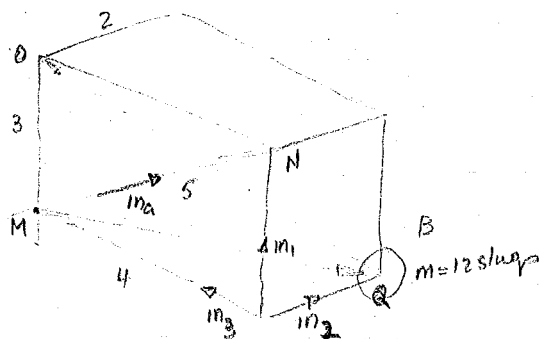
$$= \frac{I_1 + I_2}{2} + \frac{I_1 - I_2}{2} \cdot \frac{I_1 - I_2}{\sqrt{(I_1 - I_2)^2 + 4 I_{12}^2}} + \frac{2 I_{12}^2}{\sqrt{(I_1 - I_2)^2 + 4 I_{12}^2}}$$

$$\frac{(I_1 - I_2)^2}{2\sqrt{(I_1 - I_2)^2 + 4 I_{12}^2}} + \frac{4 I_{12}^2}{2\sqrt{(I_1 - I_2)^2 + 4 I_{12}^2}} = \left(\frac{\pm [\quad]^{1/2}}{2\sqrt{(I_1 - I_2)^2 + 4 I_{12}^2}} \right)^2$$

$$\sqrt{\quad} = \sqrt{(I_1 - I_2)^2 + 4 I_{12}^2}$$

$$I_a = \frac{I_1 + I_2}{2} \pm \left[\left(\frac{I_1 - I_2}{2} \right)^2 + I_{12}^2 \right]^{1/2} \quad \begin{matrix} + \text{ gives max} \\ - \text{ gives min} \end{matrix}$$

6g.



$I_{jk}^{B/O}$	1	2	3
1	260	72	-144
2	72	325	96
3	-144	96	169

find $I_a^{B/M} = m_a \cdot \Pi \cdot m_a$

$$m_a = (3m_1 - 4m_2) \frac{1}{5}$$

$$\Pi^{B/O} = \sum_{j=1}^3 \sum_{k=1}^3 m_j I_{jk}^{B/O} m_k$$

$$m_a \cdot m_1 = \frac{3}{5} \quad m_a \cdot m_2 = 0 \quad m_a \cdot m_3 = -\frac{4}{5}$$

$$I_a^{B/O} = \frac{3}{5} \left[I_{11} \cdot \frac{3}{5} + I_{12} \cdot 0 + I_{13} \cdot \left(-\frac{4}{5}\right) \right] + 0 \left[\quad \right] - \frac{4}{5} \left[I_{31} \cdot \frac{3}{5} + I_{33} \left(-\frac{4}{5}\right) \right]$$

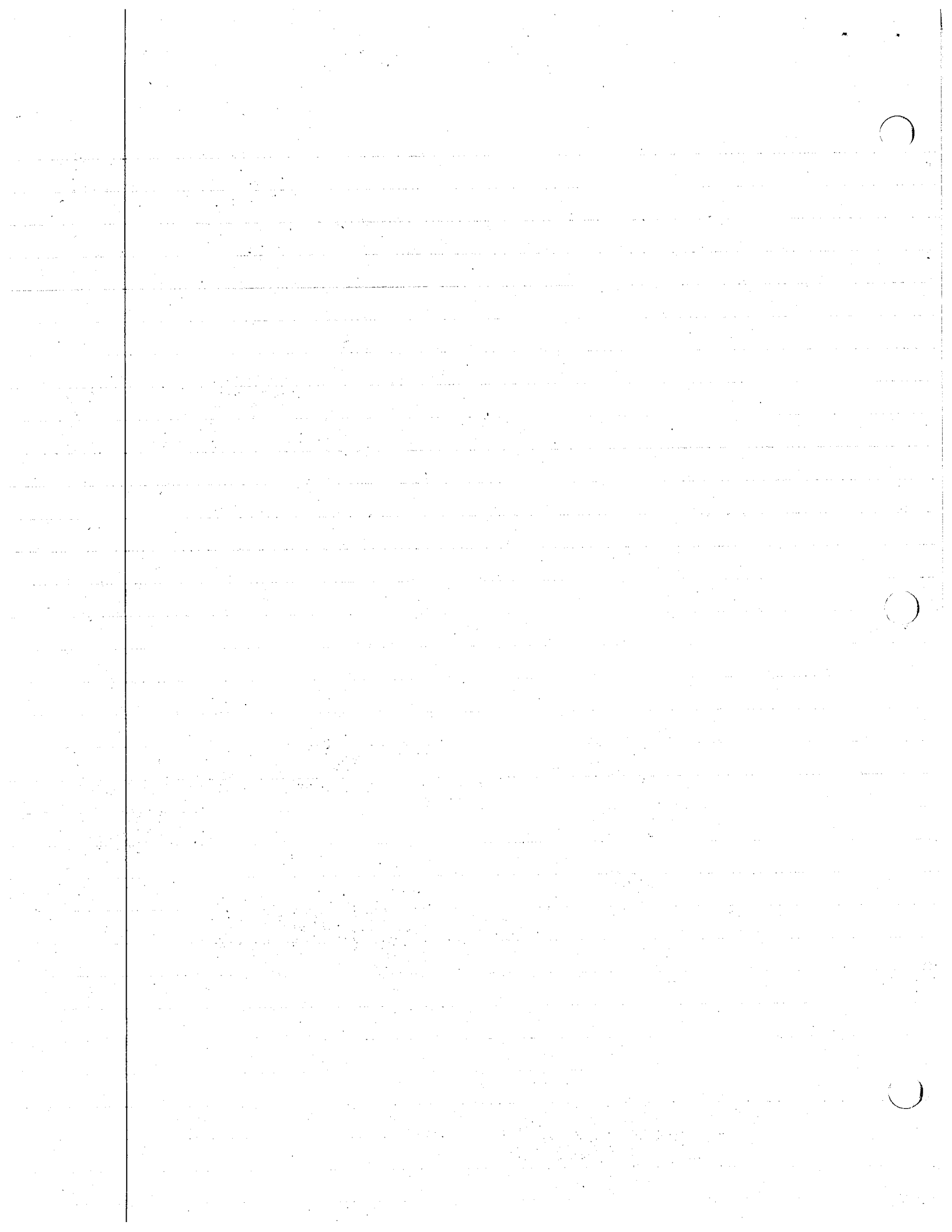
$$= \frac{9}{25} I_{11} - \frac{12}{25} I_{13} - \frac{12}{25} I_{31} + \frac{16}{25} I_{33}$$

$$I_a^{B/O} = \frac{1}{25} [9 \cdot 260 + 24 \cdot 144 + 169 \cdot 16] = \frac{8500}{25}$$

$$I_a^{B/M} = I_a^{B/Q} + I_a^{Q/M}$$

$$I_a^{Q/M} = I_a^{Q/M} m_a = m_a \cdot \left[\rho^{Q/M} \times (m_a \times \rho^{Q/M}) \right] m$$

$$I_a^{B/Q} = I_a^{B/O} + I_a^{O/Q} \quad ; \quad I_a^{O/Q} = I_a^{O/Q} m_a = m_a \cdot m \left[\rho^{O/Q} \times (m_a \times \rho^{O/Q}) \right]$$



$$\begin{vmatrix} m_1 & m_2 & m_3 \\ 3/5 & 0 & -4/5 \\ 8 & -2 & 4 \end{vmatrix}$$

$$p^{o/a} = -2m_2 + 3m_1 + 4m_3$$

$$- p^{o/a} \times \left[+\frac{8}{5}m_1 + \frac{24}{5}m_2 + \frac{6}{5}m_3 \right]$$

$$m_a \times p^{o/a} = -\frac{6}{5}m_3 - \frac{12}{5}m_2 - \frac{8}{5}m_1 - \frac{12}{5}m_3 = \left[+\frac{6}{5}m_3 + \frac{24}{5}m_2 + \frac{8}{5}m_1 \right]$$

$$\begin{vmatrix} m_1 & m_2 & m_3 \\ +3 & +2 & +4 \\ +8/5 & +24/5 & +6/5 \end{vmatrix} = +\frac{12}{5}m_1 - \frac{32}{5}m_2 - \frac{72}{5}m_3 - \frac{16}{5}m_3 + \frac{12}{5}m_2 + \frac{48}{5}m_1$$

$$= \frac{108}{5}m_1 - \frac{14}{5}m_2 - \frac{88}{5}m_3$$

$$\therefore m_a \cdot [p^{o/a} \times (m_a \times p^{o/a})] = \frac{3m_1 - 4m_3}{5} \cdot \left(\frac{108}{5}m_1 - \frac{14}{5}m_2 - \frac{88}{5}m_3 \right)$$

$$\frac{324}{25} + \frac{352}{25} = \frac{676}{25}$$

$$\therefore I_a^{o/a} = m \cdot \frac{676}{25}$$

$$I_a^{a/o} = -\frac{676}{25}m$$

$$p^{a/H} = (2m_2 - 4m_3)$$

$$m_a \times p^{a/H} = \begin{vmatrix} m_1 & m_2 & m_3 \\ 3/5 & 0 & -4/5 \\ 0 & 2 & -4 \end{vmatrix} = \frac{6}{5}m_3 + \frac{12}{5}m_2 + \frac{8}{5}m_1$$

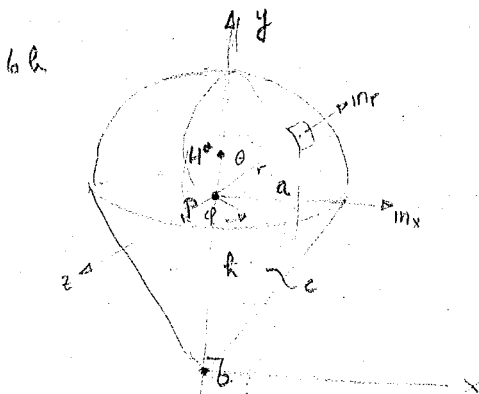
$$p^{a/H} \times [m_a \times p^{a/H}] = \begin{vmatrix} m_1 & m_2 & m_3 \\ 0 & 2 & -4 \\ +8/5 + 12/5 & +4/5 & +6/5 \end{vmatrix} = \frac{12}{5}m_1 - \frac{32}{5}m_2 - \frac{16}{5}m_3 + \frac{48}{5}m_1 = \frac{60}{5}m_1 - \frac{32}{5}m_2 - \frac{16}{5}m_3$$

$$m_a \cdot [p^{a/H} \times (m_a \times p^{a/H})] = \frac{(3m_1 - 4m_3)}{5} \cdot \left(\frac{60}{5}m_1 - \frac{32}{5}m_2 - \frac{16}{5}m_3 \right) = \frac{180 + 64}{25} = \frac{244}{25}$$

$$I_a^{a/H} = m \cdot \frac{244}{25}$$

$$I_a^{B/H} = I_a^{B/o} - I_a^{Q/o} + I_a^{B/Q} = \frac{8500 + m(244 - 676)}{25}$$

$$I_a^{B/H} = \frac{8500}{25} + \left(-\frac{676}{25} + \frac{244}{25} \right) m = 336$$



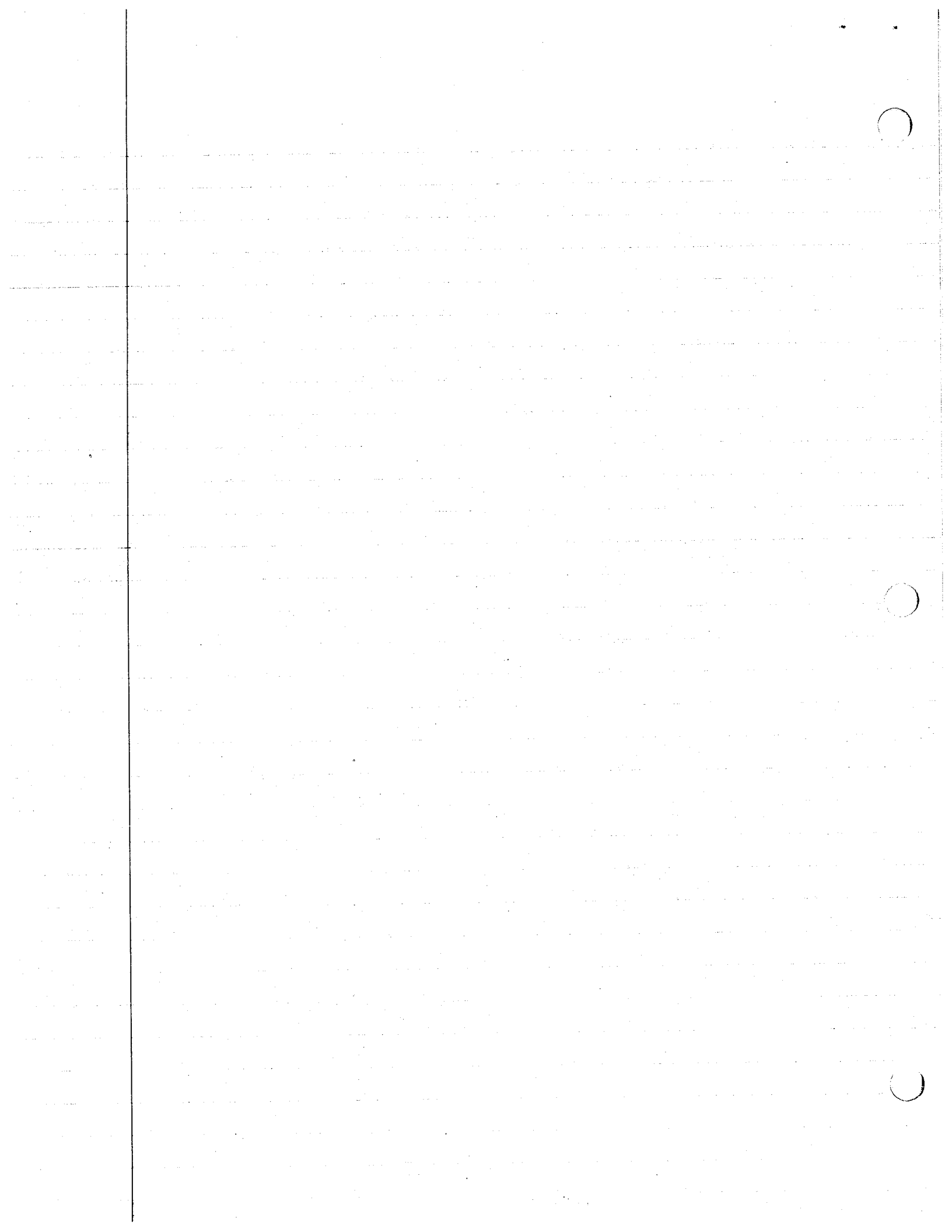
$$I_x^{c/o} = \frac{3}{5}m \left(\frac{1}{4}a^2 + h^2 \right) \quad \text{if } r=h=a$$

$$= \frac{3}{5}m \left(\frac{5}{4}r^2 \right) = \frac{3}{4}mr^2$$

$$I_x^{H/o} = I_x^{H/H^*} + I_x^{H^*/o}$$

$$= \frac{83}{320} \cdot 2mR^2 + 2m \cdot \left(\frac{3}{8}R \right)^2 = \frac{86}{20}mR^2$$

$$I_x^{S/o} = I_x^{H/o} + I_x^{c/o} = \left(\frac{15}{20} + \frac{86}{20} \right) mR^2 = \frac{101}{20}mR^2$$

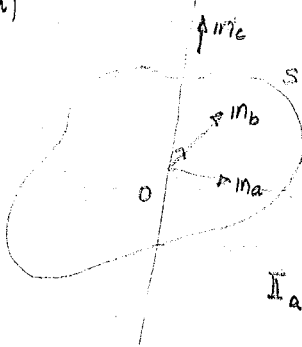


$$\therefore I_x^{H/H^*} = -I_x^{H^*/P} + I_x^{H/P} = -\frac{16mR^2}{9\pi^2} + \frac{4mR^2}{5}$$

$$I_x^{H/A} = I_x^{H/H^*} + I_x^{H^*/A} = -\frac{16mR^2}{9\pi^2} + \frac{4mR^2}{5} + 2m \left[\frac{4R}{3\pi} + R \right]^2 = \left(\frac{8R^2}{3\pi} + R^2 \right) 2m + \frac{4mR^2}{5}$$

$$I_x = \frac{15}{20} mR^2 + \frac{16mR^2}{20} + \frac{40R^2}{20}$$

6i)



$$I_c \cdot m_c = I_c = \sum m_i [p_i \times (m_c \times p_i)]$$

$$\text{where } p_i = a_i m_a + b_i m_b$$

$$m_c \times p_i = a_i m_b + b_i m_a$$

$$(a_i m_a + b_i m_b) \times (a_i m_b + b_i m_a) = a_i a_i m_c + b_i b_i m_c = (a_i^2 + b_i^2) m_c$$

$$I_c \cdot m_c = I_c = \sum m_i (a_i^2 + b_i^2)$$

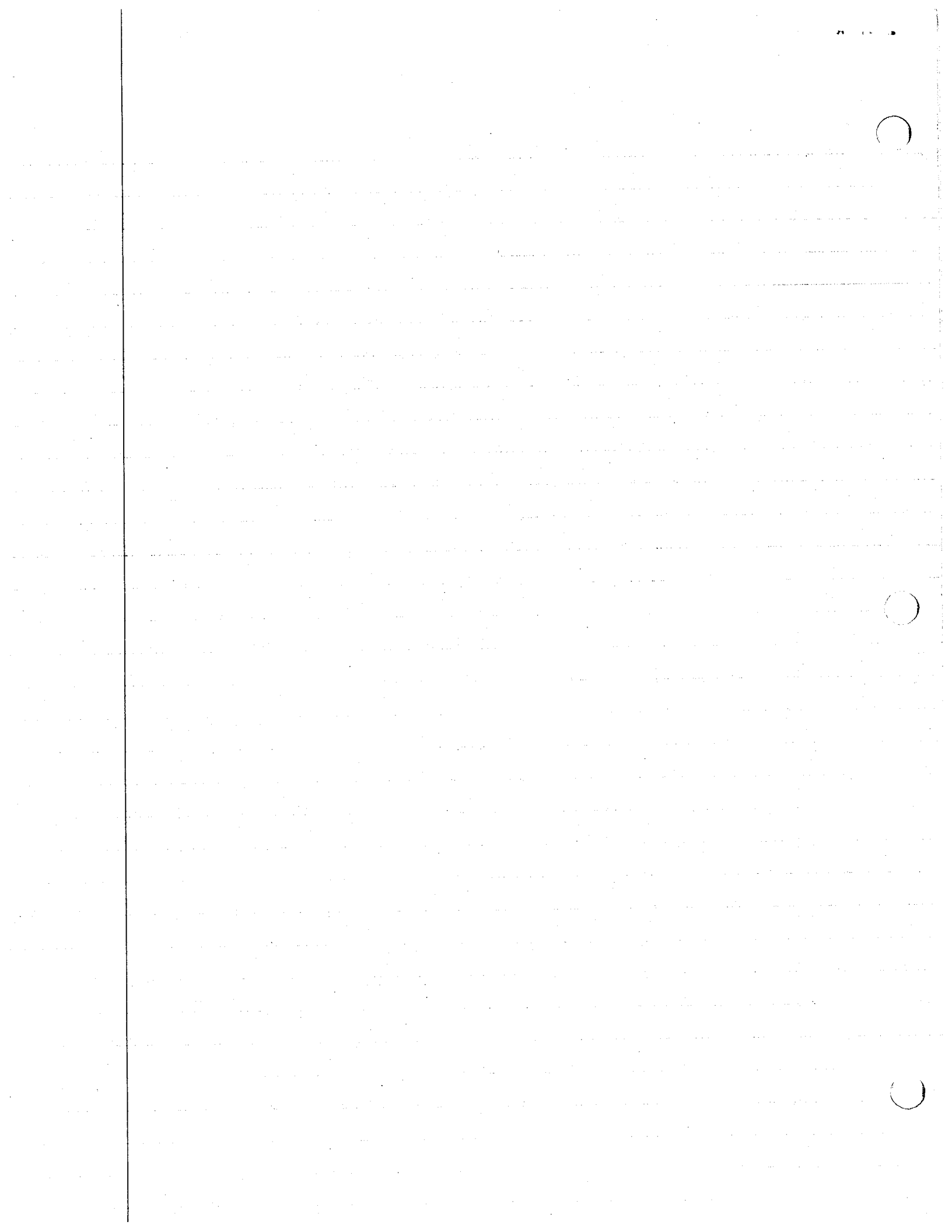
$$I_a \cdot m_a = I_a = \sum m_i [p_i \times (m_a \times p_i) \cdot m_a]$$

$$m_a \times p_i = b_i m_c$$

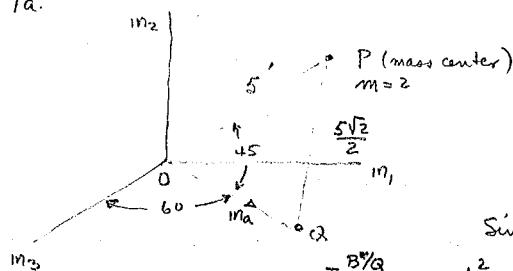
$$p_i \times (m_a \times p_i) = -a_i b_i m_b + b_i^2 m_a$$

$$I_a = \sum m_i b_i^2 ; I_b = \sum m_i a_i^2 \text{ similarly}$$

$$\therefore I_c = I_a + I_b$$



7a.



$$I = 20 \quad I_2 = 30 \quad I_3 = 40$$

find $I_a^{B/Q}$

$$m_a = \frac{1}{2} m_3 + \frac{\sqrt{3}}{2} m_1$$

$$\text{Since we want } I_a^{B/Q} = I_a^{B/B^*} + I_a^{B^*/Q}$$

$$I_a^{B^*/Q} = m \cdot l_{PQ}^2 = 2 \cdot \left(\frac{5\sqrt{2}}{2} \right)^2 = 2 \cdot \frac{25}{2} = 25 \text{ slug} \cdot \text{ft}^2$$

$$I_a^{B/B^*} = m_a \cdot \bar{I}^{B/B^*} \cdot m_a$$

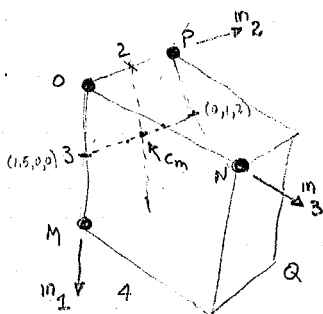
$$\text{where } \bar{I}^{B/B^*} = I_1 m_1 m_1 + I_2 m_2 m_2 + I_3 m_3 m_3 \quad ; \quad \text{since } m_a \text{ has no } m_2 \text{ component}$$

$$\text{then } \bar{I}_a^{B/B^*} = (I_1 m_1) \frac{\sqrt{3}}{2} + (I_3 m_3) \cdot \frac{1}{2} \quad I_a = I_1 \left(\frac{\sqrt{3}}{2} \right)^2 + I_3 \left(\frac{1}{2} \right)^2 = I_1 \cdot \frac{3}{4} + I_3 \cdot \frac{1}{4}$$

$$= 20 \cdot \frac{3}{4} + 40 \cdot \frac{1}{4} = 25$$

$$I_a^{B/Q} = I_a^{B/B^*} + I_a^{B^*/Q} = 25 + 25 = 50 \text{ slug} \cdot \text{ft}^2$$

7b.



To find min radius of gyration we need to find the min moment of inertia, which will be the minimum principal moment of inertia. Thus if we call it I_{min} then $k = \sqrt{\frac{I_{min}}{4m}}$. Now the min principal moment of inertia will be at the mass center. By construction

we can find the mass center. To this end, define a set of coordinates and let us find the moments of inertia about the given directions m_1, m_2, m_3 for point O for the system.

$$\text{Using (problem 6c)} \quad I_1^{S/O} = \sum m_i (x_2^2 + x_3^2) \quad I_2^{S/O} = \sum m_i [(x_3)^2 + (x_1)^2]$$

$$I_3^{S/O} = \sum m_i [(x_1)^2 + (x_2)^2] \quad I_{12}^{S/O} = -\sum m_i (x_1)_i (x_2)_i \quad I_{23}^{S/O} = -\sum m_i (x_2)_i (x_3)_i$$

$$\text{and } I_{31}^{S/O} = -\sum m_i (x_3)_i (x_1)_i$$

$$P_1 \text{ located at } (0, 0, 0) \quad P_2 (3, 0, 0) \quad P_3 (0, 2, 0) \quad P_4 (0, 0, 4) \quad . \text{ Since all have same mass } m$$

$$\text{thus } \sum (x_2)_i^2 = 4 \quad \sum (x_1)_i^2 = 9 \quad \sum (x_3)_i^2 = 16 \quad \sum (x_1 x_2)_i = 0 \quad \sum [(x_2 x_3)_i] = 0$$

$$I_1^{S/O} = m \cdot 20 \quad I_2^{S/O} = m \cdot 25 \quad I_3^{S/O} = m \cdot 13 \quad \sum (x_3 x_1)_i = 0$$

$I_{jk}^{S/O}$	1	2	3
1	20m	0	0
2	0	25m	0
3	0	0	13m

$$\rightarrow \text{cm} = (3/4, 1/2, 1) \text{ obtained by } \bar{x}_i = \frac{\sum (x_i m_i)}{\sum m_i}$$

$$\rightarrow I_{jk}^{S/S^*} = I_{jk}^{S/O} - I_{jk}^{S^*/O} \text{ w/ } I_{jk}^{S^*/O} = 4m \left\{ p^2 \delta_{jk} - (m_j \cdot p)(p \cdot m_k) \right\}$$

$$(p^2)^{S/O} = \frac{9}{16} + \frac{4}{16} + \frac{16}{16} = \frac{29}{16}$$

thus

I_{jk}^{S/S^*}	1	2	3
1	5m	$-3/2 m$	$-3m$
2	$-3/2 m$	$25/4 m$	$-2m$
3	$-3m$	$-2m$	$13/4 m$

NOTE:

$$I_{jk}^{S/S^*} = 4m \left[p^2 \delta_{jk} - (m_j \times p)(p \times m_k) \right]$$

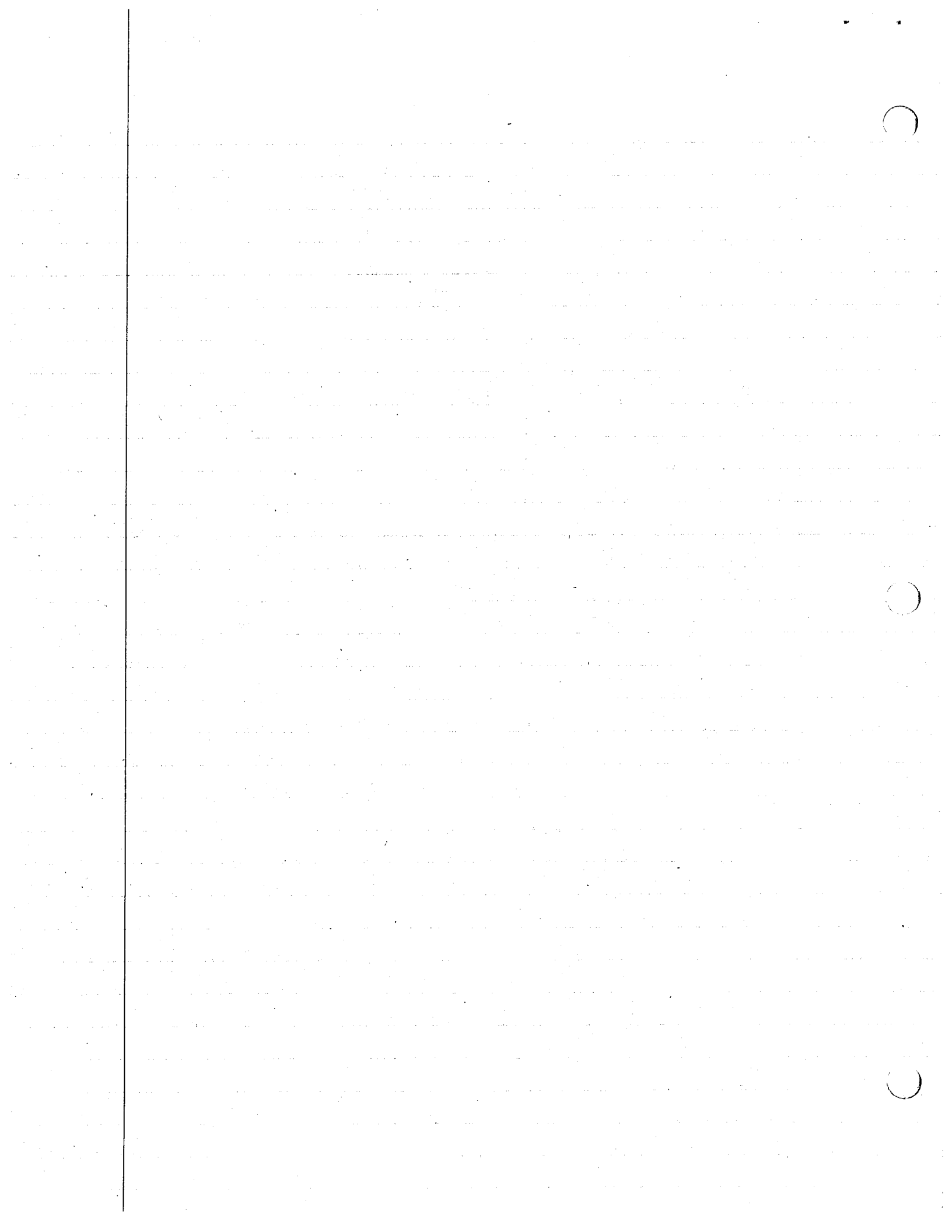
$$4m \left[p^2 \delta_{jk} - (m_j \times p)(p \times m_k) \right]$$

$$\text{b/w } p \times (m_j \times p) = (p \cdot p) m_j - (p \cdot m_j) p$$

$$\text{dot w/ } m_k = (p \cdot p) \delta_{jk} - (p \cdot m_j)(p \cdot m_k)$$

$$\text{thus } I_{jk}^{S/S^*}$$

	1	2	3
1	15m	1.5m	3m
2	1.5m	18.75m	2m
3	3m	2m	9.75m



Aside: we note that when $j=k$

$$I_j^{s/o} = (p^2 - (p \cdot m_j)^2) \cdot 4m \text{ but } (p \cdot m_j) \text{ is the proj onto the } m_j \text{ axis} \\ = 4m[p^2 - p^2 \cos^2(p, m_j)] = [p^2 \sin^2(p, m_j)] 4m = (4m) l^2 \text{ where } l = \perp \text{ distance}$$

from s^* to m_j axis.

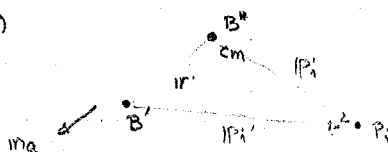
thus to minimize $I_{j/s^*} \Rightarrow \det_{m^3} \begin{bmatrix} 15 - \tilde{I} & 1.5 & 3 \\ 1.5 & 18.75 - \tilde{I} & 2 \\ 3 & 2 & 9.75 - \tilde{I} \end{bmatrix} = 0$ where $\tilde{I} = I/m$

thus $(15 - \tilde{I})(18.75 - \tilde{I})(9.75 - \tilde{I}) + 18 - 9(18.75 - \tilde{I}) - 2.25(9.75 - \tilde{I}) - 4(15 - \tilde{I}) = 0$ Solution to this

for min I must be $< 9.75m$. $I_{\min} = 8.243m$ is obtained after solution

now $k_{\min} = \sqrt{\frac{I_{\min}}{4m}} = \sqrt{2.061} = 1.436 \text{ ft}$

7c. (a)



$$I_a^{B/B^*} = \sum m_i p_i \times (m_a \times p_i) \quad \text{for a principal axis } I_a = \lambda m_a$$

thus $I_a^{B/B^*} \times m_a = 0$

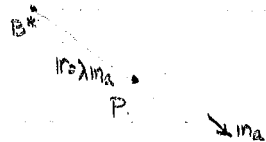
$$I_a^{B/B'} = I_a^{B/B^*} + I_a^{B'/B^*} \quad \text{if } p^{B'/B^*} = \hat{\lambda} m_a$$

$$= I_a^{B/B^*} + M(\hat{\lambda} m_a \times (m_a \times \hat{\lambda} m_a))$$

thus $I_a^{B/B'} = I_a^{B/B^*}$ but $I_a^{B'/B^*} \times m_a = 0$ $I_a^{B/B^*} \times m_a = 0$

hence principal axis for B^* is a principal axis for any point on the axis.

7c(ii)



now $I_a^{B/P} \times m_a = 0$ since m_a is a principal axis for P .

but it also passes through B^* hence we can form

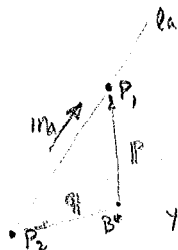
$$I_a^{B/B^*} \text{ yet } I_a^{B/B^*} = I_a^{B/P} + I_a^{P/B^*} = I_a^{B/P} - I_a^{B^*/P}$$

now $I_a^{B/B^*} \times m_a = (I_a^{B/P} \times m_a = 0 \text{ by hypothesis}) - I_a^{B^*/P} \times m_a$

but $I_a^{B^*/P} \times m_a = M[\lambda m_a \times (m_a \times \lambda m_a)] \times m_a = 0 \Rightarrow I_a^{B/B^*} \times m_a = 0$ which is only true if

$$I_a^{B/B^*} = \mu m_a \Rightarrow \text{the axis is a principal axis for } B^*$$

7c(iii)



if $I_a^{B/P_1} \times m_a = 0$ & $I_a^{B/P_2} \times m_a = 0 \Rightarrow I_a^{B/B^*} \times m_a = 0$

$$I_a^{B/P_1} = I_a^{B/B^*} + I_a^{B^*/P_1} \quad \text{and} \quad -I_a^{B^*/P_1} \times m_a = I_a^{B^*/P_1} \times m_a$$

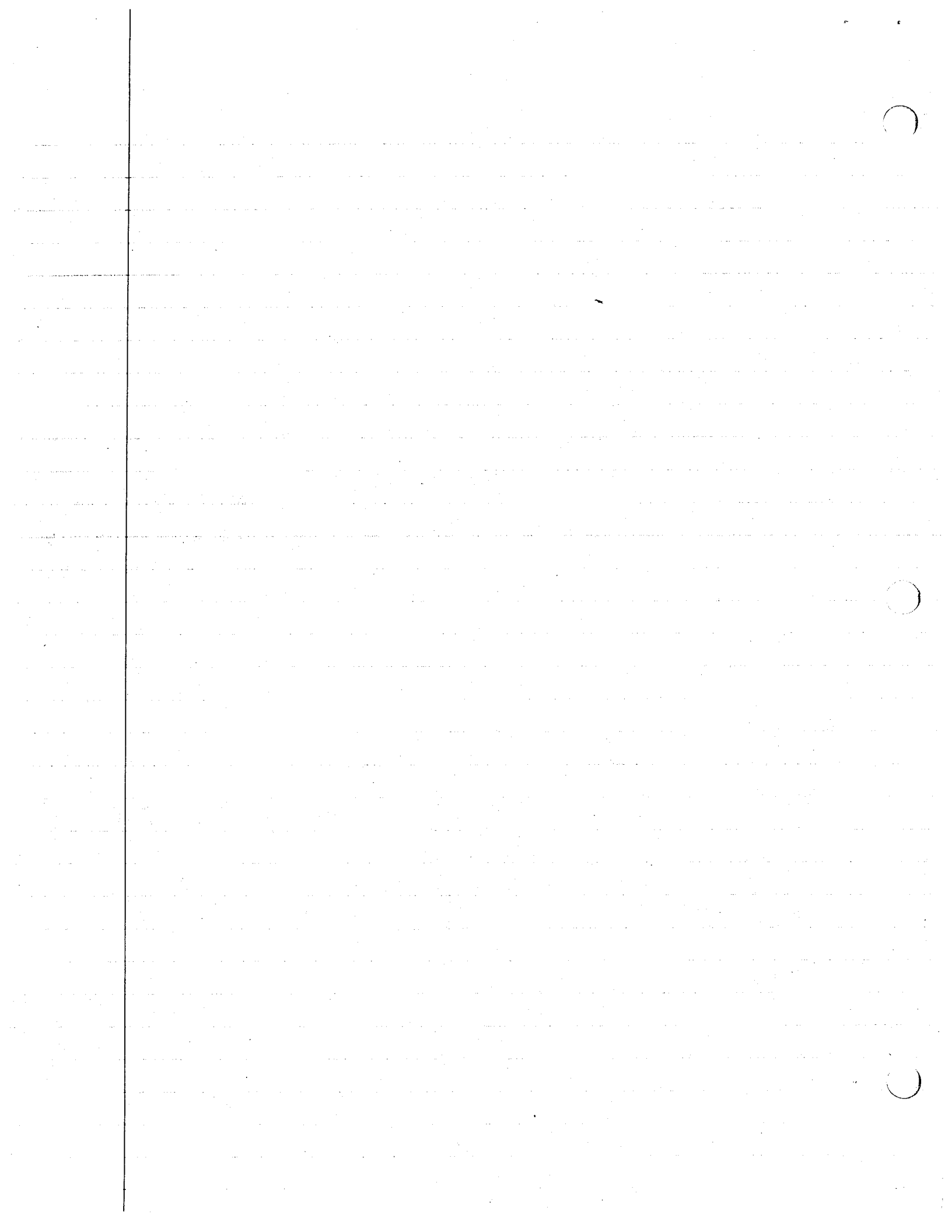
$$I_a^{B/P_2} = I_a^{B/B^*} + I_a^{B^*/P_2} \quad \text{and} \quad -I_a^{B^*/P_2} \times m_a = I_a^{B^*/P_2} \times m_a$$

then $I_a^{B/P_1} - I_a^{B/P_2} = I_a^{B^*/P_1} - I_a^{B^*/P_2}$ and

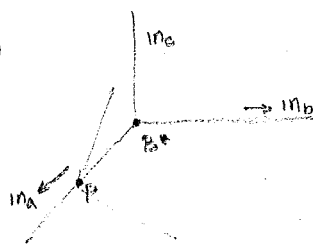
$$I_a^{B^*/P_1} \times m_a = I_a^{B^*/P_2} \times m_a \quad \text{but since } P_2, P_1 \text{ are arbitrary}$$

then $\Rightarrow$ either $p = q$ or $p \neq q$ must be $\parallel$ to $m_a \Rightarrow I_a^{B^*/P_1} \times m_a = 0$ since $p \neq q$ in g

$$\Rightarrow I_a^{B/B^*} \times m_a = 0 \Rightarrow I_a^{B/B^*} \parallel m_a \Rightarrow l_a \text{ is a principal axis for } B^* \text{ by 7c(ii)}$$



7c(iv)



Since P lies on a principal axis of the

mass center then $I_a^{B/P} \times m_a = 0$ thus $I_a^{B/B^*} \cdot m_b = 0$

and $I_a^{B/B^*} \cdot m_c = 0$

but $I_a^{B/P} = I_a^{B/B^*} + I_a^{B^*/P}$

$$I_a^{B/P} \cdot m_b = 0 + I_{ab}^{B^*/P} \quad \text{but } I_{ab}^{B^*/P} = m \cdot 0 = 0 \Rightarrow I_{ab}^{B/P} = 0$$

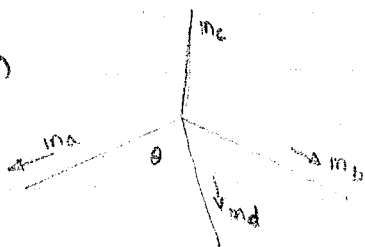
similarly we can show that $I_a^{B/P} \cdot m_c = I_{ac}^{B/P} = 0$

also $I_b^{B/P} = I_b^{B/B^*} + I_b^{B^*/P}$ but $I_b^{B^*/P} = m(\lambda m_a \times (m_b \times \lambda m_a)) = +m\lambda^2 m_b$ where $I^{B^*/P} = \lambda m_a$

hence $I_b^{B/P} \times m_b = I_b^{B/B^*} \times m_b + m\lambda^2 m_b \times m_b = 0 \Rightarrow I_b^{B/P} \parallel m_b$ and hence is a principal axis of $I_b^{B/P}$. We can do this too for $I_c^{B^*/P}$ & show that it is $\parallel$ to m_c . Similarly $I_{ac}^{B^*/P} = I_{bc}^{B^*/P} = 0$

hence the principal axes for any point on a principal axis for the mass center are $\parallel$ to the principal axes for the mass center.

7c(v)



$$\text{if } I_a = I_b \Rightarrow I_d = I_a = I_b$$

$$I = I(m_a m_a + m_b m_b) + I m_c m_c$$

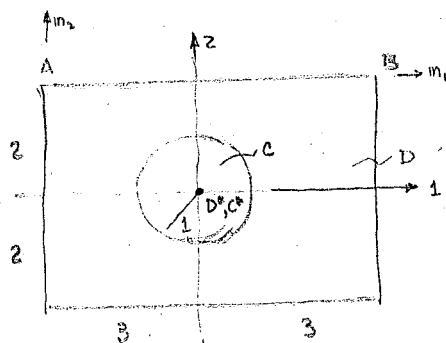
$$m_d \cdot I m_d = m_d I (m_a m_a + m_b m_b) \cdot m_d$$

$$I_d = I(\cos^2 \theta + \sin^2 \theta) = I$$

$$\text{thus } I_d = I_a = I_b$$

Since m_a & m_b are principal axes, $\exists$ a direction $m_c \perp$ to m_a, m_b also principal
 $m_d \cdot m_a = \cos \theta$
 $m_d \cdot m_b = \sin \theta$

7(d)



$$\text{let } S = D - C$$

$$I_{jk}^{D/B^*}$$

	1	2	3
1	$\frac{4m}{3}$	0	0
2	0	$3m$	0
3	0	0	$\frac{13m}{3}$

$$I_{jk}^{C/A}$$

	1	2	3
1	$m/4$	0	0
2	0	$m/4$	0
3	0	0	$m/2$

$$I_{jk}^{S/A}$$

	1	2	3
1	$\frac{13m}{12}$	0	0
2	0	$\frac{11m}{4}$	0
3	0	0	$\frac{23m}{6}$

$$I_{jk}^{S/A} = I_{jk}^{S/S^*} + I_{jk}^{S^*/A}$$

$$I_{jk}^{S^*/A} = m \left[\rho^{S^*/A} \times (m_j \times \rho^{S^*/A}) \right] \cdot m_k = (\rho \cdot \rho) \delta_{jk} - (\rho \cdot m_j)(\rho \cdot m_k)$$

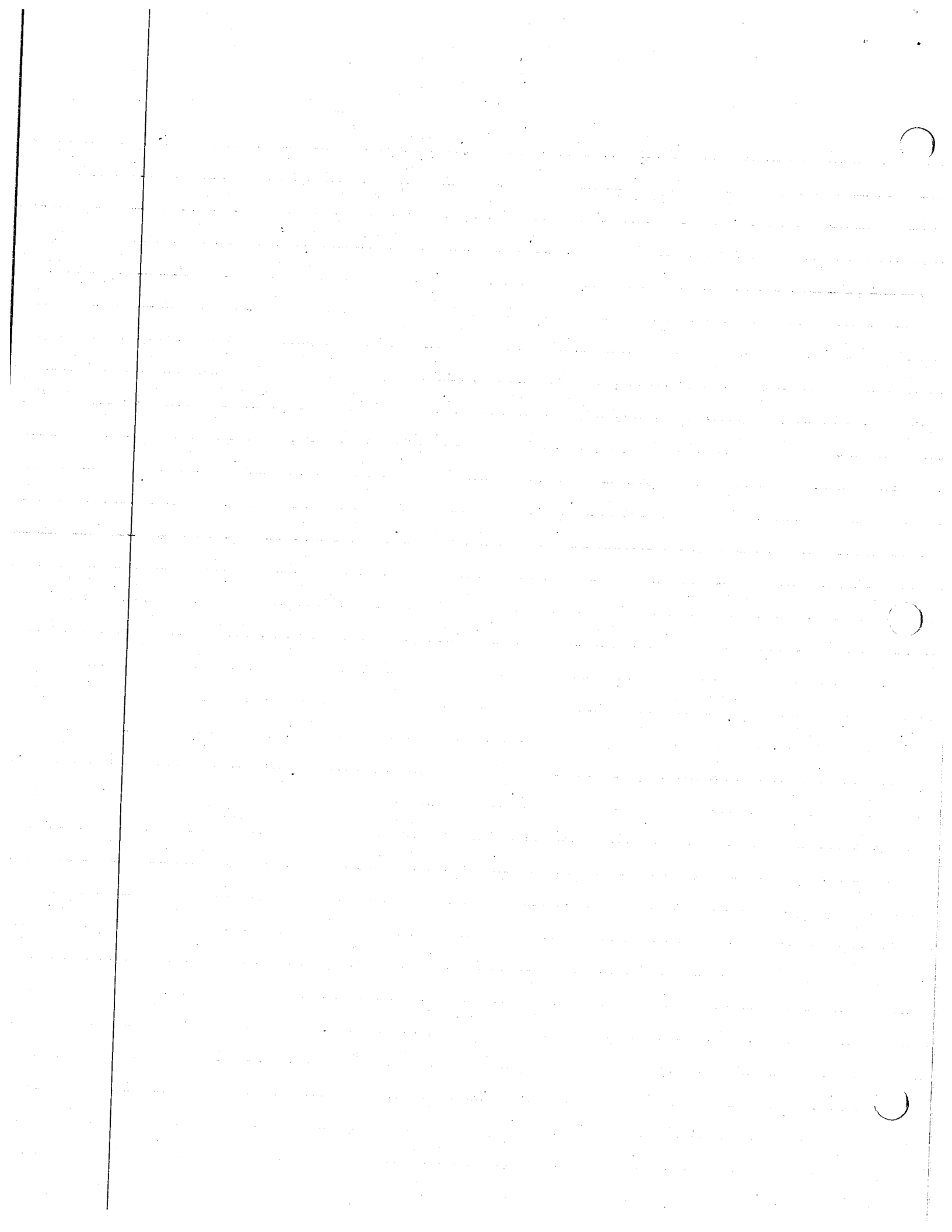
$$\rho = -2m_2 + 3m_1 \quad \rho \cdot \rho = 13$$

$$I_{jk}^{S^*/A}$$

	1	2	3
1	$4m$	$6m$	0
2	$6m$	$9m$	0
3	0	0	$13m$

$$\text{thus } I_{jk}^{S/A}$$

	1	2	3
1	$\frac{61m}{12}$	$6m$	0
2	$6m$	$\frac{47m}{4}$	0
3	0	0	$\frac{101m}{6}$



now form the determinant $(I_{jk} - I\Lambda) = 0$ $\Delta_{ii} = 1$
 $i, j = 0 \quad i \neq j$
 thus $I_3 = \frac{101}{6} m$ and $(\frac{61}{12} m - I)(\frac{47}{4} m - I) - 36 m^2 = 0$ for the other 3.

$$\frac{61 \cdot 47}{48} m^2 - \frac{202}{12} mI + I^2 - 36 m^2 = 0 \quad \text{or} \quad I^2 - 16.83 mI + 23.73 = 0$$

$$I_{1,2} = \frac{16.83 \pm 13.73}{2} \Rightarrow I_1 = 15.28 m \quad I_2 = 1.55 m$$

$$\begin{bmatrix} 5.08 m - I & 6 m & 0 \\ 6 m & 11.75 m - I & 0 \\ 0 & 0 & 16.83 m - I \end{bmatrix} \begin{pmatrix} a \\ b \\ c \end{pmatrix} = 0$$

$$\text{for } I_3 = 16.83 m \Rightarrow \hat{m}_3 = (0, 0, 1)$$

$$\text{for } I_1 = 15.28 m \Rightarrow \hat{m}_1 = (.5, .86, 0)$$

$$\text{for } I_2 = 1.55 m \Rightarrow \hat{m}_2 = (.86, .75, 0)$$

$$\text{for } I_1 \quad -10.2 m \cdot a + 6 m \cdot b = 0$$

$$\text{let } a = 1 \quad b = \frac{10.2}{6} = 1.7$$

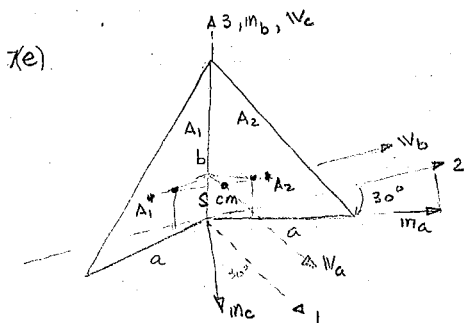
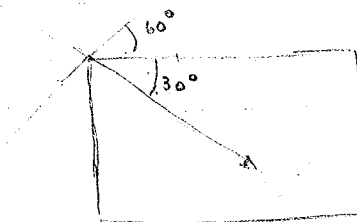
$$\therefore \sqrt{a^2 + b^2} = 1.97; \quad \hat{a} = \frac{1}{1.97} \quad \hat{b} = \frac{1.7}{1.97}$$

$$\text{for } I_2 \quad 3.53 m \cdot a + 6 m \cdot b = 0$$

$$\text{let } a = 1 \quad b = -\frac{3.53}{6} = -.588$$

$$\therefore \sqrt{a^2 + b^2} = 1.16; \quad \hat{a} = \frac{1}{1.16} \quad \hat{b} = \frac{-.588}{1.16}$$

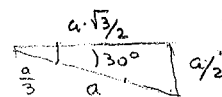
thus the ^{min} angle is 30°



$$\text{cm of } A_2 = (\frac{a}{6}, \frac{a}{2\sqrt{3}}, \frac{1}{3}b)$$

$$\text{cm of } A_1 = (\frac{a}{6}, -\frac{a}{2\sqrt{3}}, \frac{b}{3})$$

$$\text{cm of } S (= A_1 + A_2) \text{ is } (\frac{a}{6}, 0, \frac{1}{3}b)$$



now by geometric construction I expect the principal axes to be the W axes shown



To prove the above statement:

$I_{jk}^{A_1/A_2^*}$	m_a	m_b	m_c
m_a	$\frac{mb^2}{18}$	$\frac{mab}{36}$	0
m_b	$\frac{mab}{36}$	$\frac{ma^2}{18}$	0
m_c	0	0	$m(\frac{a^2+b^2}{18})$

$$\text{now } I_{jk}^{A_2/S^*} = I_{jk}^{A_2/A_2^*} + I_{jk}^{A_2^*/S^*} = I_{jk}^{A_2/A_2^*} - I_{jk}^{S^*/A_2^*}$$

$$I_{jk}^{S^*/A_2^*} = -\frac{a}{2\sqrt{3}} m_2 \quad m_2 = -\frac{1}{2} m_b + \frac{\sqrt{3}}{2} m_a$$

$$I_{jk}^{S^*/A_2^*} = \frac{a}{2\sqrt{3}} (\frac{1}{2} m_b - \frac{\sqrt{3}}{2} m_a) = \frac{a}{4\sqrt{3}} m_b - \frac{a}{4} m_a$$

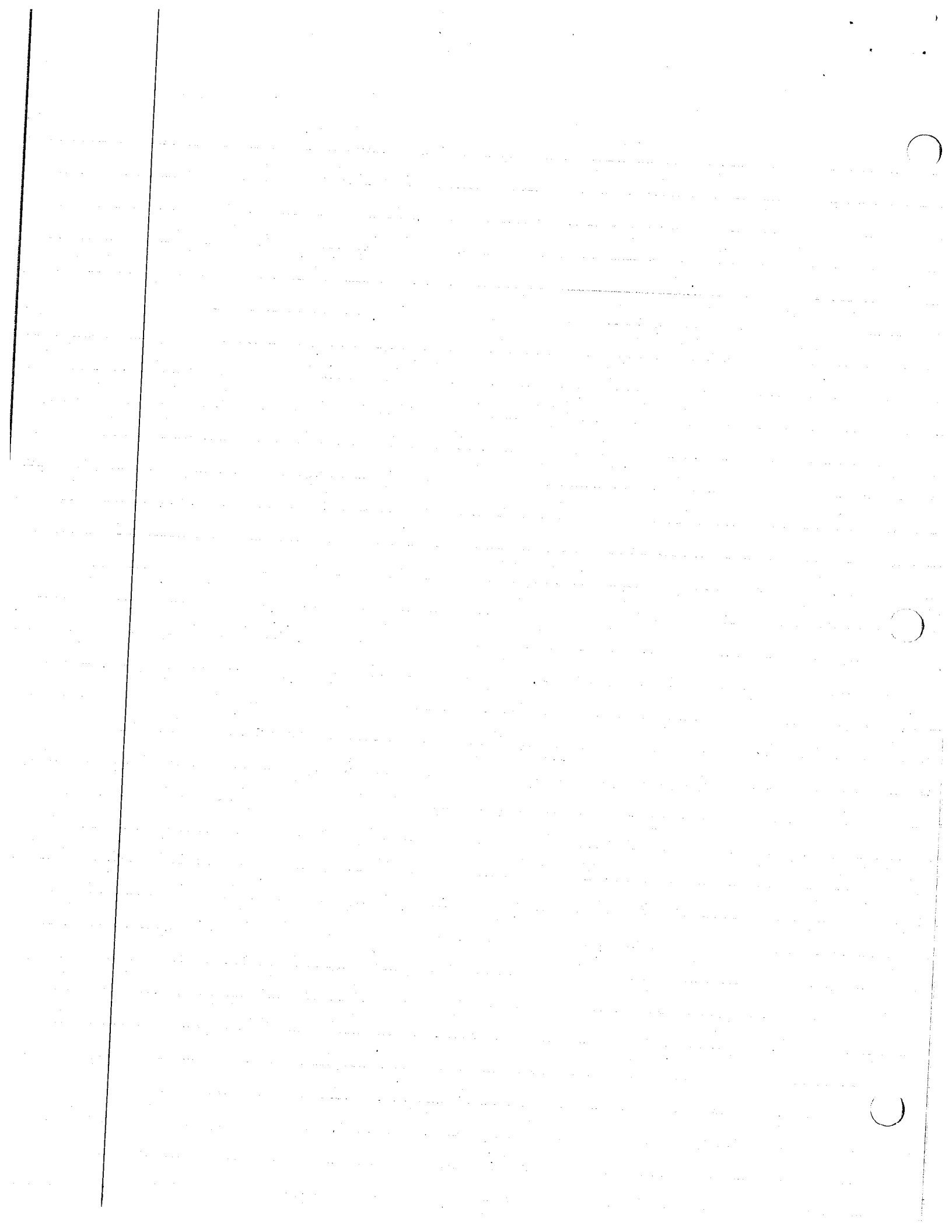
$$\text{now } I_{jk}^{S^*/A_2^*} = m [p \cdot p \delta_{jk} - (p \cdot m_j)(p \cdot m_k)]$$

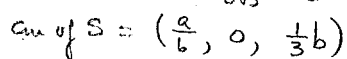
$$p \cdot p = \frac{a^2}{48} + \frac{a^2}{16} = \frac{a^2}{12}$$

$I_{jk}^{A_2/S^*}$	m_a	m_b	m_c
m_a	$m(\frac{b^2-a^2}{18} - \frac{a^2}{48})$	$m(\frac{ab}{36} - \frac{a^2}{16\sqrt{3}})$	0
m_b	$m(\frac{ab}{36} - \frac{a^2}{16\sqrt{3}})$	$m(\frac{a^2}{18} - \frac{a^2}{16})$	0
m_c	0	0	$m(\frac{a^2}{12} - \frac{a^2+b^2}{18})$

thus

$I_{jk}^{S^*/A_2^*}$	m_a	m_b	m_c
m_a	$\frac{ma^2}{48}$	$\frac{ma^2}{16\sqrt{3}}$	0
m_b	$\frac{ma^2}{16\sqrt{3}}$	$\frac{ma^2}{16}$	0
m_c	0	0	$\frac{ma^2}{12}$





now $I_{km}^{A_2/A^*} = \sum_{j=a-k}^c \sum_{i=k}^c a_{ij} I_{jk} b_{ik}$

where $a_{ij} \Rightarrow i=1,2,3$
 $j=a,b,c$

$b_{km} \Rightarrow k=a,b,c$
 $m=1,2,m$

	m_1	m_2	m_3
m_a	$\frac{1}{2}$	$\sqrt{3}/2$	0
m_b	0	0	1
m_c	$\sqrt{3}/2$	$-\frac{1}{2}$	0

b_{km}

$$\begin{pmatrix} \frac{1}{2} & 0 & \sqrt{3}/2 \\ \sqrt{3}/2 & 0 & -1/2 \\ 0 & 1 & 0 \end{pmatrix} \begin{pmatrix} \frac{mb^2}{18} & \frac{mab}{36} & 0 \\ \frac{mab}{36} & \frac{ma^2}{18} & 0 \\ 0 & 0 & m(a^2+b^2) \end{pmatrix} \begin{pmatrix} 1/2 & \sqrt{3}/2 & 0 \\ 0 & 0 & 1 \\ \sqrt{3}/2 & -1/2 & 0 \end{pmatrix} = \begin{pmatrix} \frac{mb^2}{72} & \frac{mab}{72} & \frac{m(a^2+b^2)\sqrt{3}}{18} \\ \frac{mab\sqrt{3}}{72} & \frac{ma^2}{36} & 0 \\ \frac{ma^2}{18} & 0 & 0 \end{pmatrix} = \begin{pmatrix} \frac{mb^2}{72} + \frac{m(a^2+b^2) \cdot 3}{72} & \frac{mab\sqrt{3}}{72} - \frac{m(a^2+b^2)\sqrt{3}}{72} & \frac{mab}{72} \\ \frac{mab\sqrt{3}}{72} - \frac{m(a^2+b^2)\sqrt{3}}{72} & \frac{ma^2}{36} + \frac{m(a^2+b^2)}{72} & \frac{mab\sqrt{3}}{72} \\ \frac{ma^2}{72} & \frac{mab\sqrt{3}}{72} & \frac{ma^2}{18} \end{pmatrix}$$

$$\mu_{\text{eff}} \rightarrow I_{\text{Ln}}^{A_2/S^H} = I_{\text{Ln}}^{A_2/A_2^*} + I_{\text{Ln}}^{A_2^{\text{tr}}/S^H}$$

	m_1	m_2	m_3
m_d	$\frac{1}{2}$	$-\sqrt{3}/2$	0
m_b	0	0	1
m_e	$-\sqrt{3}/2$	$-1/2$	0

blau.

$$= \begin{pmatrix} \frac{mb^2}{36} & \frac{mab}{72} & -\frac{m(a^2+b^2)G}{36} \\ -\frac{mb^2\sqrt{3}}{36} & -\frac{mab\sqrt{3}}{72} & -\frac{m(a^2+b^2)}{36} \\ \frac{mab}{36} & \frac{ma^2}{18} & 0 \end{pmatrix} = \begin{pmatrix} \frac{mb^2}{72} + \frac{m(a^2+b^2)\sqrt{3}}{72} - \frac{mb^2\sqrt{3}}{72} + \frac{m(a^2+b^2)\sqrt{3}}{72} & \frac{mab}{72} \\ -\frac{mb^2\sqrt{3}}{72} + \frac{m(a^2+b^2)\sqrt{3}}{72} & \frac{mb^2}{72} + \frac{m(a^2+b^2)}{72} \\ \frac{mab}{72} & -\frac{mab\sqrt{3}}{72} & \frac{ma^2}{18} \end{pmatrix}$$

Now we must translate this to $I_{lm}^{A_1/S^*} = I_{lm}^{A_1/A_1^*} + I_{lm}^{A_1^*/S^*} =$

$$\begin{aligned} P^{A_2^+ / S^*} &= + \frac{a}{2\sqrt{3}} m_2 \\ P^2 &= \frac{a^2}{12} \end{aligned} \qquad \begin{aligned} P^{A_1^+ / S^*} &= - \frac{a}{2\sqrt{2}} m_2 \\ P^2 &= \frac{a^2}{12} \end{aligned}$$

now $I_{lm}^{A_2/s^*} = m [p^2 \delta_{lm} - (p \cdot m_l)(p \cdot m_m)]$

$$|p \cdot m_2| = \frac{+a}{2\sqrt{3}} \quad |p \cdot m_1| = |p \cdot m_3| = 0$$

thus $I_{cm}^{A_2/S^*}$

m_1	m_2	m_3
$\frac{ma^2}{12}$	0	0
0	0	0
0	0	$\frac{ma^2}{12}$

and the same is true for $I_{cm}^{A_1/S^*}$

thus $I_{cm}^{A_1/S^*}$

$\frac{mb^2}{12} + \frac{3m(a^2+b^2)}{12} + \frac{ma^2}{12}$	$-\frac{mb^2\sqrt{3}}{12}$	$-\frac{\sqrt{3}m(a^2+b^2)}{12}$	$\frac{mab}{12}$
$-\frac{mb^2\sqrt{3}}{12} + \frac{m(a^2+b^2)\sqrt{3}}{12}$	$\frac{3mb^2}{12} + \frac{m(a^2+b^2)}{12}$	$-\frac{mab\sqrt{3}}{12}$	$-\frac{mab\sqrt{3}}{12}$
$\frac{mab}{12}$	$-\frac{mab\sqrt{3}}{12}$	$\frac{ma^2}{18} + \frac{ma^2}{12}$	

and $I_{cm}^{A_2/S^*}$

$\frac{mb^2}{12} + \frac{3m(a^2+b^2)}{12} + \frac{ma^2}{12}$	$\frac{mb^2\sqrt{3}}{12}$	$-\frac{m(a^2+b^2)\sqrt{3}}{12}$	$\frac{mab}{12}$
$\frac{mb^2\sqrt{3}}{12} - \frac{m(a^2+b^2)\sqrt{3}}{12}$	$\frac{3mb^2}{12} + \frac{m(a^2+b^2)}{12}$	$\frac{mab\sqrt{3}}{12}$	$\frac{mab\sqrt{3}}{12}$
$\frac{mab}{12}$	$\frac{mab\sqrt{3}}{12}$	$\frac{ma^2}{18} + \frac{ma^2}{12}$	

Now $S/S^* = I_{cm}^{A_1/S^*} + I_{cm}^{A_2/S^*}$

$\frac{mb^2}{36} + \frac{m(a^2+b^2)}{12} + \frac{ma^2}{6}$	0	$\frac{mab}{36}$
$\frac{ma^2}{4} + \frac{mb^2}{9}$	$\frac{m(a^2+b^2)}{36} + \frac{mb^2}{12}$	0
$\frac{mab}{36}$	$m(\frac{b^2}{9} + \frac{a^2}{36})$	$\frac{ma^2}{9} + \frac{ma^2}{6}$
	0	$\frac{10ma^2}{36}$

for $a=2b$ & $b=1$ $a=2$

$m[1 + \frac{1}{9}] = m\frac{10}{9}b^2$	0	$\frac{m}{18}b^2$
0	$\frac{2m}{9}b^2$	0
$\frac{2m}{18}b^2$	0	$m\frac{10}{9}b^2$

Thus $(\frac{10}{9}m - I)(\frac{2m}{9}I)(\frac{10}{9}m - I) - \frac{m^2}{(18)^2}(\frac{2m}{9} - I) = 0$ $\therefore (\frac{2m}{9} - I)\left\{\left[\frac{10}{9}m - I\right]^2 - \frac{m^2}{324}\right\} = 0$

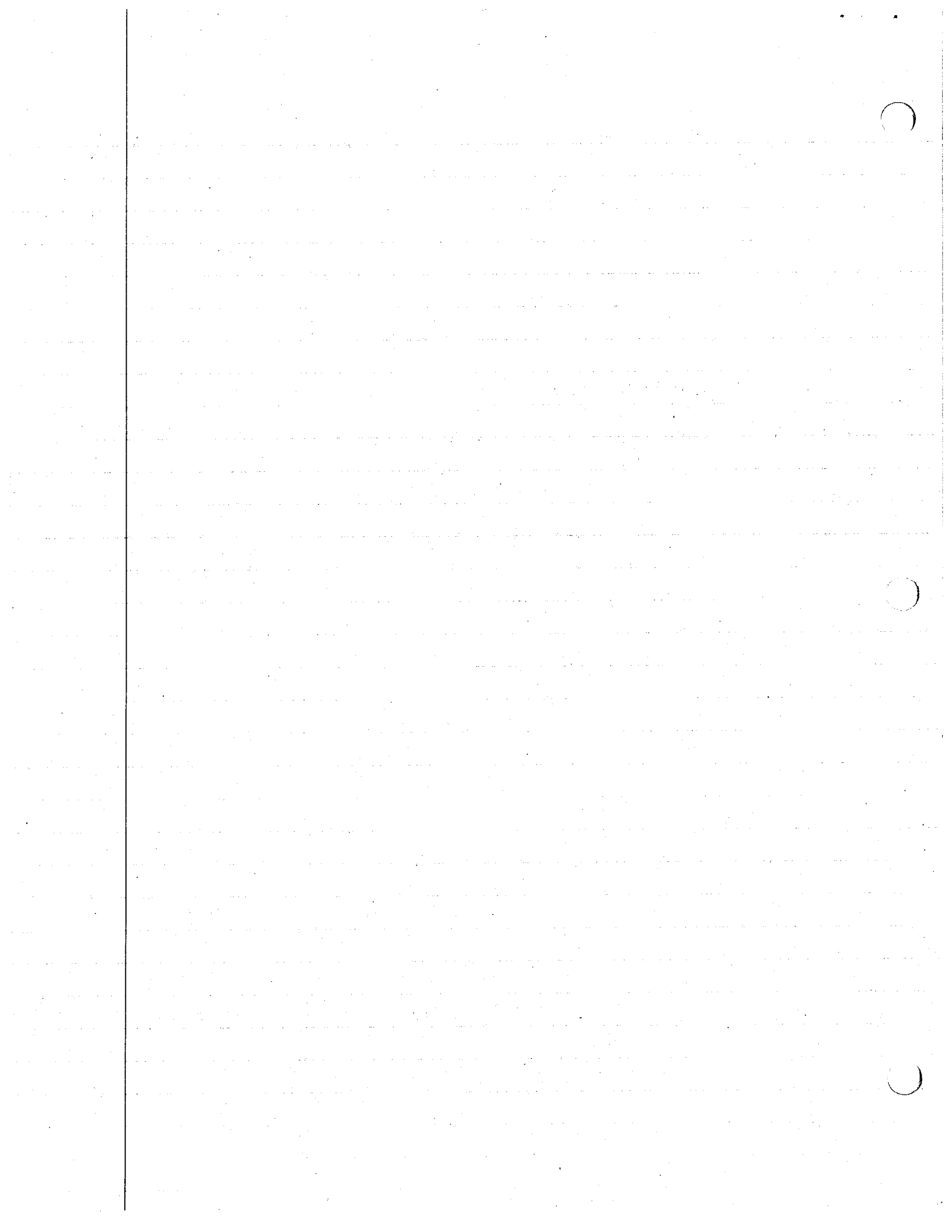
① $I = \frac{2mb^2}{9}$ ② $b^4(\frac{100}{81}m^2 - \frac{20m}{9}I + I^2 - \frac{m^2}{324}) = b^4(\frac{399}{324}m^2 - \frac{20m}{9}I + I^2) = 0$

$I = b^2(\frac{20}{18}m \pm \frac{1}{2}\sqrt{\frac{400m^2}{81} - 4\frac{399m^2}{324}}) = \frac{20}{18}m \pm \frac{1}{2}\sqrt{\frac{m^2}{81}} = \frac{20}{18}mb^2 \pm \frac{mb^2}{18} \Rightarrow \frac{21}{18}mb^2 \text{ or } \frac{19}{18}mb^2$

$I = 2mk^2 = \frac{21}{18}mb^2 \text{ or } \frac{19}{18}mb^2$ $k^2 = \sqrt{\frac{21}{36}b^2} \text{ or } \sqrt{\frac{19}{36}b^2}$

$\frac{2mb^2}{9} = I_{min} = 2mk_{min}^2 \Rightarrow k_{min} = \frac{1}{3}b$

13
18



for $b = 2a$ let $A = 1$ $b = 2$ thus

$$I_{cm}^{S/S^*} = \begin{pmatrix} m \left[\frac{1}{4} + \frac{1}{9} \right] = \frac{25}{36} m a^2 & 0 & \frac{m}{18} a^2 \\ 0 & m \left(\frac{1}{9} + \frac{1}{36} \right) = m \frac{17}{36} a^2 & 0 \\ \frac{m}{18} a^2 & 0 & \frac{10m}{36} a^2 \end{pmatrix}$$

$$\text{then } \left[\left(\frac{25m}{36} - I \right) \left(\frac{17m}{36} - I \right) \left(\frac{10m}{36} - I \right) - \frac{m^2}{324} \left(\frac{17m}{36} - I \right) \right] a^6 = 0$$

$$\therefore I = \frac{17m}{36} a^2$$

$$\Phi = \left[\left(\frac{25m}{36} - I \right) \left(\frac{10m}{36} - I \right) - \frac{m^2}{324} \right] a^4 = 0$$

$$k = \sqrt{\frac{17}{72}} a = \frac{1}{12} \sqrt{\frac{17}{2}} b$$

$$a^4 \left[\frac{250m^2}{36^2} - \frac{35}{36} mI + I^2 - \frac{m^2}{(18)^2} \right] = 0$$

$$a^4 \left[\frac{246m^2}{(36)^2} - \frac{35}{36} mI + I^2 \right] = 0$$

$$I = a^2 \left[\frac{35m}{72} \pm \frac{m}{2} \sqrt{\left(\frac{35}{36} \right)^2 - 4 \cdot \frac{246}{(36)^2}} \right] = \left[\frac{35m}{72} \pm \frac{m}{72} \sqrt{(35)^2 - 4(246)} \right] a^2$$

$$= \left[\frac{35m}{72} \pm \frac{m}{72} \sqrt{241} \right] a^2$$

k

$$= \sqrt{35 \pm \sqrt{241}} \cdot \frac{1}{12} a = \frac{1}{24} [35 \pm (241)^{1/2}]^{1/2} b$$

$$35 \sim 16 \sim (3) \cdot 9$$

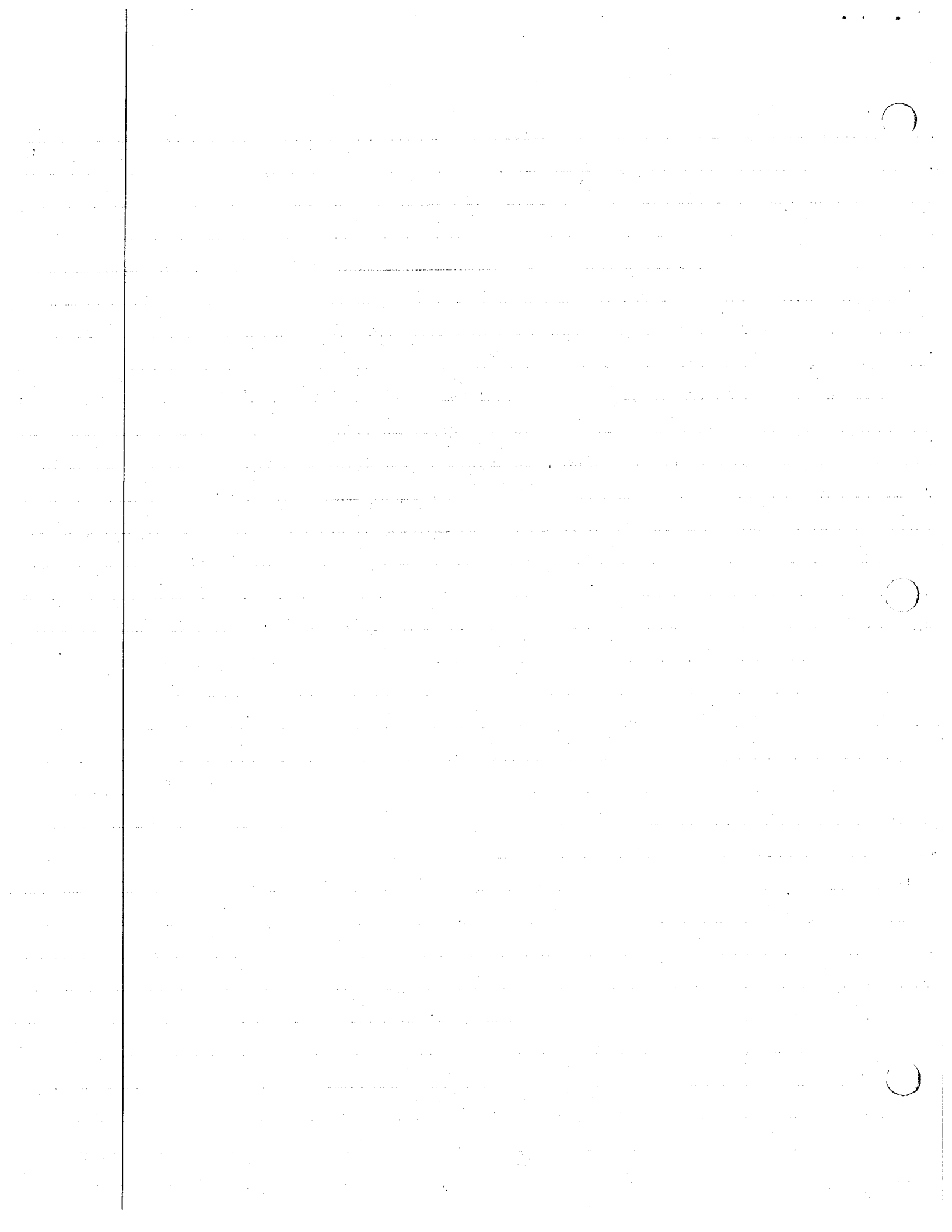
$$\sqrt{18 \sim 19} = \frac{4.5}{24} = .18 \sim .2$$

$$35 + 16 \sim 51 \quad \sqrt{51} \sim 7$$

$$\frac{7}{24} \sim .30$$

$$\frac{1}{12} \sqrt{\frac{17}{2}} = \frac{4.3}{12 \cdot 1.41} = \frac{1}{9} \cdot .25$$

$$\text{thus } k_{\min} = \frac{1}{24} [35 - (241)^{1/2}]^{1/2} b$$



now $m_a = \frac{\sqrt{3}}{2} m_2 + \frac{1}{2} m_1$ $m_b = \frac{\sqrt{3}}{2} m_1 - \frac{1}{2} m_2$ $m_c = m_2$

thus $I_{lm} = \sum_{j=1}^3 \sum_{k=1}^3 a_{lj} I_{jk} a_{km}$

7f $I_{jk}^{B/N} = I_{jk}^{B/Q} + I_{jk}^{A/N}$ $I_{jk}^{B/O} = I_{jk}^{B/Q} + I_{jk}^{A/O}$

$P^{A/N} = -3m_1 + 2m_2$

$P \cdot P = 13$ $I_{jk}^{A/N} = m [13 \delta_{jk} - (P \cdot m_j)(P \cdot m_k)]$

now $I_{jk}^{B/N} = I_{jk}^{B/O} - I_{jk}^{A/O} + I_{jk}^{A/N}$

$P^{A/O} = -3m_1 - 4m_2 + 2m_3$

$P \cdot P = 29$ $I_{jk}^{A/O} = m [29 \delta_{jk} - (P \cdot m_j)(P \cdot m_k)]$

$I_{jk}^{A/N}$	1	2	3
1	48	72	0
2	72	108	0
3	0	0	156

$I_{jk}^{A/O}$	1	2	3
1	240	72	-144
2	72	300	96
3	-144	96	156

$\therefore I_{jk}^{B/N}$

	1	2	3
1	68	72	0
2	72	133	0
3	0	0	169

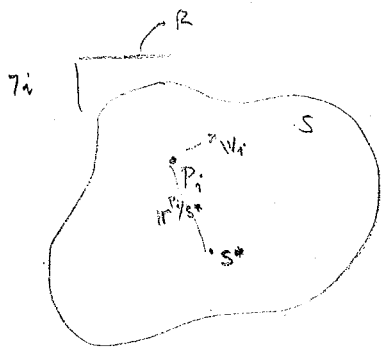
$\Pi = 68 m_1 m_1 + 72 (m_1 m_2 + m_2 m_1) + 133 m_2 m_2 + 169 m_3 m_3$

7g $m = \frac{3}{5} m_1 - \frac{4}{5} m_3$ $I_n = m \cdot \Pi \cdot m = 68 \cdot \frac{9}{25} + 169 \left(\frac{16}{25} \right) = \frac{3316}{25}$ Same result

hence $I_n = m \cdot \Pi \cdot m$

7h. Since $\Pi_a = \Pi \cdot m_a = \sum m_i |p_i| \times (m_a \times |p_i|)$
 $= \sum m_i [(|p_i| \cdot |p_i|) m_a - (m_a \cdot |p_i|) |p_i|] = \sum m_i [|p_i|^2 (m_a \cdot m_1) m_1 + (m_a \cdot m_2) m_2 + (m_a \cdot m_3) m_3] - (m_a \cdot |p_i|) |p_i|]$
 $= \sum m_i \{ m_a [m_1 m_1 + m_2 m_2 + m_3 m_3] |p_i|^2 - m_a (|p_i| |p_i|) \} = m_a \cdot \sum m_i (|p_i|^2 - |p_i| |p_i|)$
 thus since $\Pi \cdot m_a = m_a \cdot \Pi = m_a \cdot \sum m_i (|p_i|^2 - |p_i| |p_i|) \Rightarrow$
 $\Pi = \sum m_i (|p_i| \cdot |p_i| - |p_i| |p_i|)$



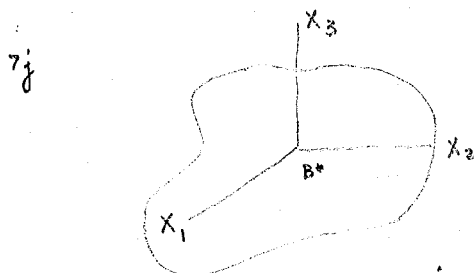


$${}^R H^{S/S^*} = \sum_{i=1}^N m_i r_i \times v_i$$

$$\text{now } v_i = v^{S^*} + \omega \times r_i$$

$${}^R H^{S/S^*} = \sum m_i r_i \times v^{S^*} + \sum m_i r_i \times (\omega \times r_i) \quad \text{Since } S^* \text{ is mass center}$$

$${}^R H^{S/S^*} = \sum m_i r_i \times (\omega \times r_i) = I \cdot \omega = I \omega$$

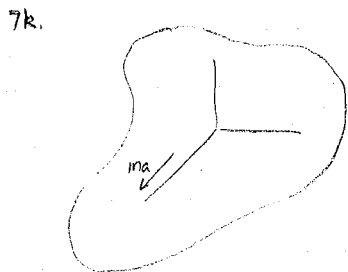


$$\omega = \omega_i n_i \quad \text{where } \omega_i = \omega \cdot n_i$$

$${}^R H^{B/B^*} = I \cdot \omega$$

$$\text{Since } X_i \text{'s are principal axes } I = I_1 n_1 n_1 + I_2 n_2 n_2 + I_3 n_3 n_3$$

$$\therefore I \cdot \omega = I_1 \omega_1 n_1 + I_2 \omega_2 n_2 + I_3 \omega_3 n_3 = {}^R H^{B/B^*}$$



$$\text{Show that if } {}^R H^{B/B^*} \text{ is } \parallel \text{ to } \omega \Rightarrow \omega \text{ is } \parallel \text{ to } I_a$$

$$\text{and if } \omega \text{ is } \parallel \text{ to } I_a \Rightarrow {}^R H^{B/B^*} \text{ is } \parallel \text{ to } \omega$$

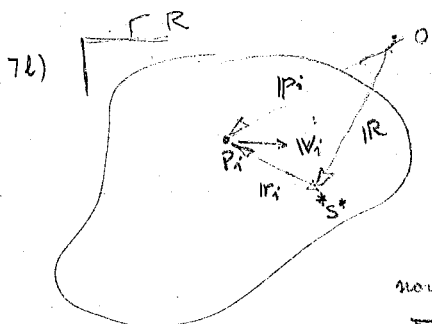
$$\text{Let } \omega = \omega n_a \quad \text{then let } n_a \cdot n_i = a_i$$

$$\text{if } \omega \text{ is } \parallel \text{ to } n_1 \text{ then } I = I_1 n_1 n_1 \Rightarrow I \omega = \omega I_1 n_1 \Rightarrow I H \text{ is } \parallel \text{ to } \omega$$

$$\text{If } {}^R H^{B/B^*} \text{ is } \parallel \text{ to } \omega \text{ then } I H = H n_a = I \cdot \omega = \omega I \cdot n_a = \omega I_a \Rightarrow I_a = \frac{H}{\omega} n_a \Rightarrow$$

$$I_a \parallel n_a \Rightarrow n_a \text{ must be a vector in principal direction } \Rightarrow \omega \text{ is } \parallel$$

to a principal axis



$${}^R H^{S/O} = \sum m_i p_i \times v_i \quad \text{let } p_i = r_i^{S^*} + r_i^{S^*/O}$$

$$= \sum m_i (r_i + R) \times (v^{S^*} + \omega \times r_i)$$

$$= \sum m_i r_i \times v^{S^*} + \sum m_i r_i \times (\omega \times r_i) + \sum m_i R \times v^{S^*} + \sum m_i R \times (\omega \times r_i)$$

$$\text{now } \sum m_i r_i = 0 \quad \& \quad \sum m_i r_i \times (\omega \times r_i) = {}^R H^{S/S^*}$$

$$\sum m_i R \times v^{S^*} + \sum m_i R \times (\omega \times r_i) = m R \times v^{S^*} + \sum m_i \{ (R \cdot r_i) \omega - (\omega \cdot R) r_i \}$$

$$\sum m_i (R \cdot r_i) \omega = [R \cdot (\sum m_i r_i)] \omega = 0; \quad \sum m_i (\omega \cdot R) r_i = (\omega \cdot R) \sum m_i r_i = 0$$

$$\text{Thus } {}^R H^{S/O} = {}^R H^{S/S^*} + m R \times v^{S^*} \quad m = \sum m_i \quad m R \times v^{S^*} = R \times (m v^{S^*})$$

of a



8b) From problem 7i) we showed that ${}^R H^{B/B^*} = {}^R H^{B/B}$

Now $\frac{d}{dt}({}^R H \cdot \omega) = \frac{d}{dt}({}^R H \cdot \omega) + {}^R \omega \times ({}^R H \cdot \omega)$ but what is $\frac{d}{dt}({}^R H \cdot \omega)$

$${}^R H \cdot \omega = {}^R H_{\omega} = \sum m_i r_i \times (\omega \times r_i) \quad \text{thus} \quad \frac{d}{dt}(\quad) = \sum m_i \dot{r}_i \times (\omega \times r_i) + \sum m_i r_i \times \left(\frac{d\omega}{dt} \times r_i\right) + \sum m_i r_i \times (\omega \times \dot{r}_i)$$

but since r_i are fixed wrt body B hence $\dot{r}_i = \frac{d}{dt} r_i = 0$. Thus 2nd & 3rd sums are zero.

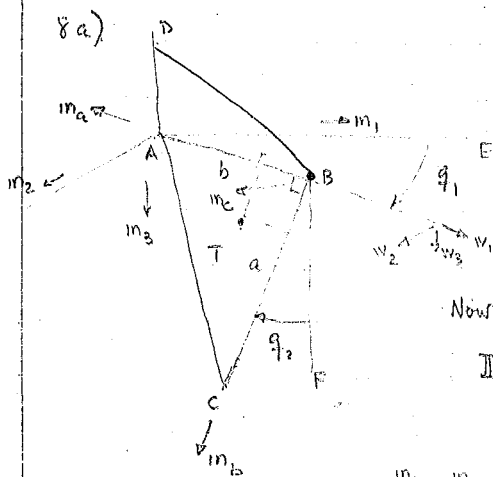
$$\frac{d}{dt}({}^R H \cdot \omega) = {}^R \alpha \times {}^R H \cdot \omega + {}^R \omega \times ({}^R H \cdot \omega) = {}^R \alpha \times {}^R H \cdot \omega \quad \text{since } \omega \times \omega = 0$$

hence,

$$\frac{d}{dt}({}^R H \cdot \omega) = \sum m_i r_i \times ({}^R \alpha \times r_i) = {}^R H \cdot {}^R \alpha \quad \text{thus}$$

$${}^R H^{B/B^*} = {}^R H \cdot {}^R \alpha + \omega \times ({}^R H \cdot \omega) = {}^R H \cdot {}^R \alpha - ({}^R H \cdot \omega) \times \omega = -\Pi^*$$

hence $\Pi^* = -{}^R H^{B/B^*}$ as required



$${}^R \omega^T = \dot{q}_1 m_3 + \dot{q}_2 m_a$$

$$F_2^* = W_{q_2}^* \cdot F^* + \omega \cdot \Pi^*$$

$$\text{where } F^* = -m a \ddot{q}_1 \quad \text{and } \Pi^* = ({}^R H \cdot \omega) \times \omega - {}^R H \cdot {}^R \alpha$$

and W^* is the velocity of the mass center

Now for the plate

$$I = \frac{m a^2}{18} m_a m_a + \frac{m a b}{36} (m_a m_b + m_b m_a) + \frac{m b^2}{18} m_b m_b + \frac{m (a^2 + b^2)}{18} m_c m_c$$

	m_1	m_2	m_3
v_1	c_1	s_1	0
v_2	$-s_1$	c_1	0
v_3	0	0	1

	v_1	v_2	v_3
m_a	-1	0	0
m_b	0	s_2	c_2
m_c	0	c_2	$-s_2$

$$m_3 = m_3 = c_2 m_b - s_2 m_c$$

$${}^R \omega^T = \dot{q}_1 (c_2 m_b - s_2 m_c) + \dot{q}_2 m_a$$

$${}^R \omega^T = m_a$$

$${}^R \alpha^T = \ddot{q}_1 + \omega \times \omega = \frac{d}{dt} \omega$$

$$I \omega = I \cdot \omega = \frac{m a^2}{18} \dot{q}_2 m_a + \frac{m a b}{36} \dot{q}_2 m_b + \frac{m a b}{36} \dot{q}_1 c_2 m_a + \frac{m b^2}{18} \dot{q}_1 c_2 m_b + m \frac{(a^2 + b^2)}{18} (-\dot{q}_1 s_2 m_c)$$

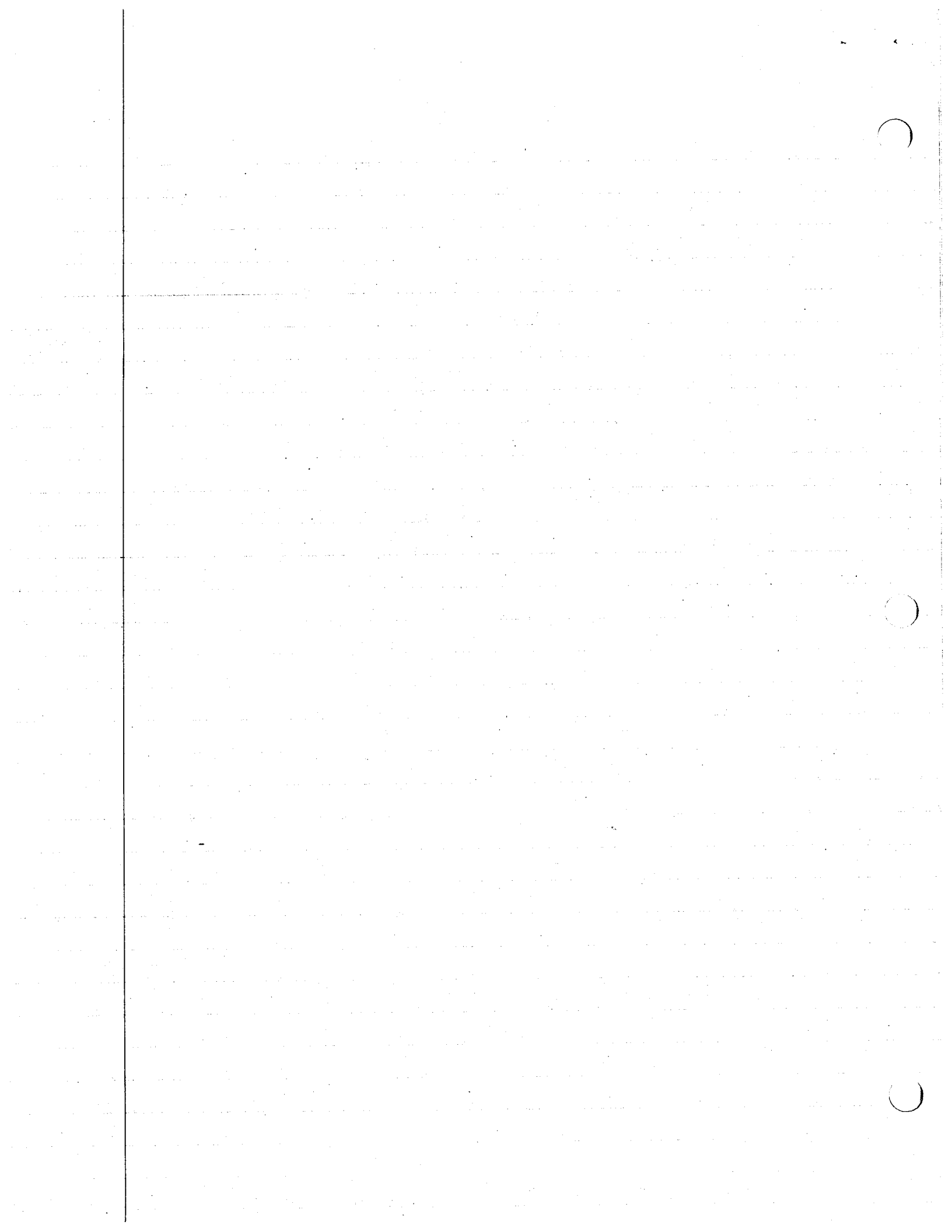
$$= \left(\frac{m a^2}{18} \dot{q}_2 + \frac{m a b}{36} \dot{q}_1 c_2 \right) m_a + \left(\frac{m a b}{36} \dot{q}_2 + \frac{m b^2}{18} \dot{q}_1 c_2 \right) m_b + m \frac{(a^2 + b^2)}{18} \dot{q}_1 s_2 m_c$$

$$I \omega \times \omega = -\dot{q}_2 \left[\frac{m a b}{36} \dot{q}_2 + \frac{m b^2}{18} \dot{q}_1 c_2 \right] m_c - m \frac{(a^2 + b^2)}{18} \dot{q}_1 \dot{q}_2 s_2 m_b + \dot{q}_1 c_2 \left(\frac{m a^2}{18} \dot{q}_2 + \frac{m a b}{36} \dot{q}_1 c_2 \right) m_a +$$

$$\dot{q}_1 s_2 \left(\frac{m a^2}{18} \dot{q}_2 + \frac{m a b}{36} \dot{q}_1 c_2 \right) m_b - \dot{q}_1 s_2 \left(\frac{m a b}{36} \dot{q}_2 + \frac{m b^2}{18} \dot{q}_1 c_2 \right) m_a + \dot{q}_1 c_2 \left[m \frac{(a^2 + b^2)}{18} \dot{q}_1 s_2 \right] m_a$$

$${}^R \alpha^T = \frac{d}{dt} \omega = \ddot{q}_1 (c_2 m_b - s_2 m_c) + \ddot{q}_2 (-s_2 \dot{q}_1 m_b - c_2 \dot{q}_2 m_c) + \ddot{q}_2 m_a = \ddot{q}_2 m_a + (\ddot{q}_1 c_2 - \dot{q}_1 \dot{q}_2 s_2) m_b - (\ddot{q}_1 s_2 + \dot{q}_1 \dot{q}_2 c_2) m_c$$

$$\rightarrow I \cdot \alpha = \frac{m a^2}{18} \ddot{q}_2 m_a + \frac{m a b}{36} \ddot{q}_2 m_b + \frac{m a b}{36} (\ddot{q}_1 c_2 - \dot{q}_1 \dot{q}_2 s_2) m_a + \frac{m b^2}{18} (\ddot{q}_1 c_2 - \dot{q}_1 \dot{q}_2 s_2) m_b + m \frac{(a^2 + b^2)}{18} (\ddot{q}_1 s_2 + \dot{q}_1 \dot{q}_2 c_2) m_c$$



$$\therefore \omega \dot{q}_2 \cdot \Pi^* = m_a \cdot \Pi^* = -\dot{q}_1 s_2 \left(\frac{mab}{36} \dot{q}_2 + \frac{mb^2}{18} \dot{q}_1 c_2 \right) + \dot{q}_1 c_2 \left[\frac{m(a^2+b^2)}{18} \dot{q}_1 s_2 \right] - \frac{ma^2}{18} \ddot{q}_2 - \frac{mab}{36} (\ddot{q}_1 c_2 - \dot{q}_1 \dot{q}_2 s_2)$$

$$\text{now } \Pi^{T*} = -\frac{2}{3} b m_a + \frac{a}{3} m_b$$

$$\boxed{R_{V \dot{q}_2}^{T*} = \frac{a m_c}{3}}$$

$$\begin{aligned} {}^R V^{T*} &= {}^R V^{T*} + \omega \times \Pi^{T*} = \left[\dot{q}_1 (c_2 m_b - s_2 m_c) + \dot{q}_2 m_a \right] \left(-\frac{2b}{3} m_a + \frac{a}{3} m_b \right) \\ &= \frac{2b}{3} \dot{q}_1 c_2 m_c + \dot{q}_1 s_2 \cdot \frac{2b}{3} m_b + \dot{q}_1 s_2 a m_a + \dot{q}_2 \frac{a}{3} m_c \\ &= \dot{q}_1 s_2 a m_a + \dot{q}_1 s_2 \cdot \frac{2b}{3} m_b + \left(\frac{2b}{3} \dot{q}_1 c_2 + \dot{q}_2 \frac{a}{3} \right) m_c \end{aligned}$$

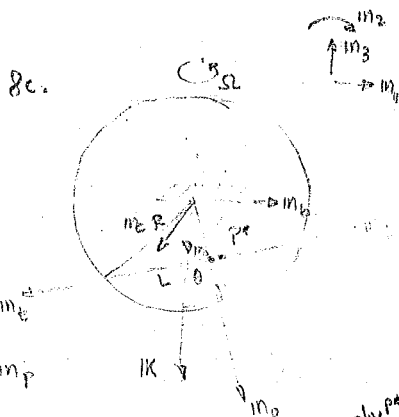
$${}^R a^{T*} = {}^T a^{T*} + \omega \times V^{T*}$$

$${}^T a^{T*} = \frac{2b}{3} m_c (\ddot{q}_1 c_2 + \dot{q}_1 s_2 \dot{q}_2) + \frac{2b}{3} m_b (\ddot{q}_1 s_2 + \dot{q}_1 \dot{q}_2 c_2) + (\ddot{q}_1 s_2 a + \dot{q}_1 \dot{q}_2 c_2 a) \frac{m_a}{3} + \ddot{q}_2 \frac{a}{3} m_c$$

$$\begin{aligned} \omega \times V &= \dot{q}_2 \dot{q}_1 s_2 \frac{2b}{3} m_c - \dot{q}_2 \left(\frac{2b}{3} \dot{q}_1 c_2 + \dot{q}_2 \frac{a}{3} \right) m_b - \dot{q}_1 c_2 a (\dot{q}_1 s_2 \frac{m_c}{3}) + \dot{q}_1 c_2 \left(\frac{2b}{3} \dot{q}_1 c_2 + \dot{q}_2 \frac{a}{3} \right) m_a \\ &\quad - \dot{q}_1 s_2 a (\dot{q}_1 s_2 \frac{m_b}{3}) + \dot{q}_1 s_2 \left(\dot{q}_1 s_2 \cdot \frac{2b}{3} \right) m_a \end{aligned}$$

$$\text{Thus } W_{\dot{q}_2} \cdot a^{T*} = \frac{2ba}{9} (\ddot{q}_1 c_2 - \dot{q}_1 s_2 \dot{q}_2) + \ddot{q}_2 \frac{a^2}{9} + \dot{q}_2 \dot{q}_1 s_2 \frac{2ba}{9} - \dot{q}_1 \dot{q}_1 c_2 s_2 \frac{a^2}{9}$$

$$\begin{aligned} \text{hence } F_2^* &= -\frac{2bma}{9} (\ddot{q}_1 c_2 + \dot{q}_1 \dot{q}_2 s_2) - \frac{m}{9} \ddot{q}_2 a^2 - m \dot{q}_2 \dot{q}_1 s_2 \cdot \frac{2ba}{9} + m \dot{q}_1^2 c_2 s_2 a^2 \\ &= -\dot{q}_1 \dot{q}_2 \frac{mab}{36} s_2 - \dot{q}_1^2 \frac{mb^2}{18} s_2 c_2 + \dot{q}_1^2 c_2 s_2 \frac{m(a^2+b^2)}{18} - \frac{ma^2}{18} \ddot{q}_2 - \frac{mab}{36} \ddot{q}_1 c_2 + \frac{mab}{36} \dot{q}_1 \dot{q}_2 s_2 \\ &= \ddot{q}_1 \left[-\frac{2bma}{9} c_2 - \frac{mab}{36} c_2 \right] + \ddot{q}_2 \left[-\frac{ma^2}{9} - \frac{ma^2}{18} \right] + \dot{q}_1^2 \left[\frac{ma^2}{9} c_2 s_2 - \frac{mb^2}{18} s_2 c_2 + \frac{m(a^2+b^2)}{18} c_2 s_2 \right] \\ &\quad + \dot{q}_1 \dot{q}_2 \left[\frac{2mba}{9} s_2 - \frac{2mba}{9} s_2 - \frac{mabs_2}{36} + \frac{mabs_2}{36} \right] \\ &= \ddot{q}_1 \left[-\frac{1}{4} m b a c_2 \right] + \ddot{q}_2 \left[-\frac{ma^2}{6} \right] + \dot{q}_1^2 \left[\frac{ma^2}{6} c_2 s_2 \right] \\ &= \frac{ma}{12} \left[2a (c_2 s_2 \dot{q}_1^2 - \ddot{q}_2) - 3 \ddot{q}_1 b c_2 \right] \end{aligned}$$



$$m_b \times m_c = m_p$$

m_c, m_b, m_a are fixed in plane of ring

$${}^R \omega^{R \rightarrow B} = -\Omega K$$

$${}^R \omega^{R \rightarrow B} = \dot{\theta} m_c - \Omega K \quad r = (R^2 - L^2)^{1/2} m_a$$

$$F_\theta^* = W_\theta \cdot F^* + \omega_\theta \cdot \Pi^*$$

$$\text{where } F^* = -m a^*$$

$$\text{from 5a we found } W^{P*} = (R^2 - L^2)^{1/2} [\dot{\theta} \cos \theta m_b - \dot{\theta} \sin \theta K - \Omega s m_c]$$

$$\text{Thus } W_\theta^* = (R^2 - L^2)^{1/2} [\cos \theta m_b - \sin \theta K] = (R^2 - L^2)^{1/2} [\cos \theta m_b - \dot{\theta} m_a]$$

$$\text{now } a^* = \frac{dW^{P*}}{dt} = \frac{dW^{P*}}{dt} + \omega \times W^{P*} \quad {}^R \omega^{R \rightarrow B} = -\Omega K$$

$$\frac{dW^{P*}}{dt} = (R^2 - L^2)^{1/2} [\ddot{\theta} \cos \theta m_b - \ddot{\theta} \sin \theta m_b - \ddot{\theta} \sin \theta K - \dot{\theta}^2 K - \Omega \dot{\theta} m_c]$$

$$\omega \times W = (R^2 - L^2)^{1/2} [-\dot{\theta} \cos \theta m_b - \dot{\theta} \sin \theta m_b - \dot{\theta} \sin \theta K - \Omega \dot{\theta} m_c - \Omega \dot{\theta} m_b]$$

$$a^* = (R^2 - L^2)^{1/2} [(-\ddot{\theta} \cos \theta - \dot{\theta}^2 \cos \theta) K + (-\ddot{\theta} \sin \theta - \dot{\theta}^2 \sin \theta) m_b - 2\Omega \dot{\theta} m_c]$$

thus $\omega_{\dot{\theta}} \cdot IF^* = +m(R^2 - L^2) [-s(\dot{\theta}^2 + \ddot{\theta}s) + (\dot{\theta}^2 s + \Omega_s^2 - \ddot{\theta}c)c] = m(R^2 - L^2) [\Omega_s^2 sc - \ddot{\theta}]$
 $\omega_{\dot{\theta}} = m_c$

Now

	m_a	m_b	m_c
m_a	0	0	0
m_b	0	$\frac{mL^2}{3}$	0
m_c	0	0	$\frac{mL^2}{3}$

$$I = \frac{mL^2}{3} (m_a m_a + m_c m_c) \quad m_a = \cos\theta m_k + \sin\theta m_b$$

$$m_a m_a = \cos^2\theta m_k m_k + \sin^2\theta (m_b m_b + m_k m_k) + 2\sin\theta\cos\theta m_b m_k$$

$$I_{xx}(t) = \frac{mL^2}{3} (c^2 m_k m_k + sc [m_b m_b + m_k m_k] + s^2 m_b m_b + m_c m_c) \cdot (\dot{\theta} m_c - \Omega m_k)$$

$$(I \cdot \omega) \times \omega = \frac{mL^2}{3} [\dot{\theta} m_c \cdot c^2 \Omega m_k - \Omega sc m_b] \times [\dot{\theta} m_c - \Omega m_k] = \frac{mL^2}{3} [-\dot{\theta} \Omega m_b + c^2 \Omega \dot{\theta} m_b - \Omega sc \dot{\theta} m_a - \Omega^2 sc m_c]$$

$$\alpha^R = \frac{d\omega^R}{dt} = \frac{d}{dt} \left(\frac{R}{L} \omega^R \right) + \omega^R \times \omega^R = \ddot{\theta} m_c + (-\Omega m_k) \times (\dot{\theta} m_c - \Omega m_k) = \ddot{\theta} m_c + \Omega \dot{\theta} m_b$$

since m_b, m_a, m_c are fixed in ring

now $I \cdot \alpha = \frac{mL^2}{3} \{ c^2 \Omega m_k + sc [m_b m_b + m_k m_k] + s^2 m_b m_b + m_c m_c \} \cdot (\ddot{\theta} m_c + \Omega \dot{\theta} m_b)$
 $= \frac{mL^2}{3} \{ \ddot{\theta} m_c + \Omega \dot{\theta} sc m_k + \Omega \dot{\theta} s^2 m_b \}$

$$\Pi^* = (I \cdot \omega) \times \omega - I \cdot \alpha = \frac{mL^2}{3} \{ (-2\Omega sc \dot{\theta}) m_a + (-\dot{\theta} \Omega m_b + c^2 \Omega \dot{\theta} - \Omega \dot{\theta} s^2) m_b + (-\Omega^2 sc - \ddot{\theta}) m_c \}$$

now $\omega_{\dot{\theta}} \cdot \Pi^* = -(\Omega^2 sc + \ddot{\theta}) \frac{mL^2}{3}$

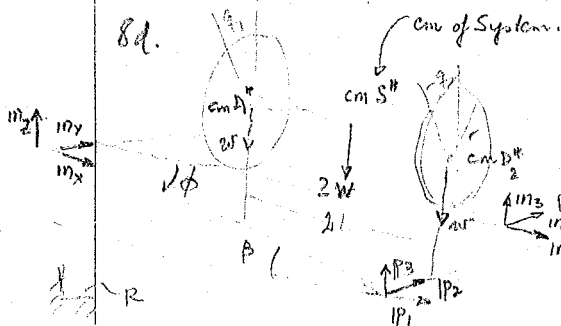
Hence

$$F_{\dot{\theta}}^* = W_{\dot{\theta}} IF^* + \omega_{\dot{\theta}} \cdot \Pi^* = \frac{3m(R^2 - L^2)}{3} [\Omega^2 sc - \ddot{\theta}] - \frac{mL^2}{3} (\Omega^2 sc + \ddot{\theta})$$

$$= -\frac{W}{3g} \left[\frac{3R^2 - 3L^2 + L^2}{(3R^2 - 2L^2)} \ddot{\theta} + \Omega^2 sc \frac{(3L^2 - 3R^2 + L^2)}{(4L^2 - 3R^2)} \right]$$

$$= -\frac{W}{3g} \left[(3R^2 - 2L^2) \ddot{\theta} + \Omega^2 sc (4L^2 - 3R^2) \right] \quad \text{where } \Omega = \dot{\theta}$$

8d.



$$R' W^{D_2^*} = r \dot{q}_2 (\sin\phi \cos\beta p_1 - \sin\phi \sin\beta p_3 - \cos\phi p_2) = r \dot{q}_2 (\sin\phi m_x - \cos\phi m_y)$$

$$W^{D_1^*} = r \dot{q}_1 (\sin\phi m_x - \cos\phi m_y) = r \dot{q}_1 (\sin\phi \cos\beta p_1 - \sin\phi \sin\beta p_3 - \cos\phi p_2)$$

$$W^{S^*} = \frac{r}{2} (\dot{q}_1 + \dot{q}_2) (\sin\phi m_x - \cos\phi m_y) = -\frac{r}{2} (\dot{q}_1 + \dot{q}_2) m_z$$

$$R' W^{D_2^*} = \frac{r}{2L} [\dot{q}_1 - \dot{q}_2] m_3 \quad R' W^S = \dot{\phi} m_z = \dot{\phi} m_3 = \frac{r}{2L} [\dot{q}_1 - \dot{q}_2] m_3$$

$$R' W^{D_1^*} = \dot{\phi} m_2 + \dot{q}_1 m_1 = \frac{r}{2L} [\dot{q}_1 - \dot{q}_2] m_3 + \dot{q}_1 m_1$$

now $(F_r^*)_B = W_{\dot{q}_r}^* \cdot IF^* + \omega_{\dot{q}_r} \cdot \Pi^*$ where $IF^* = -m a_1^*$

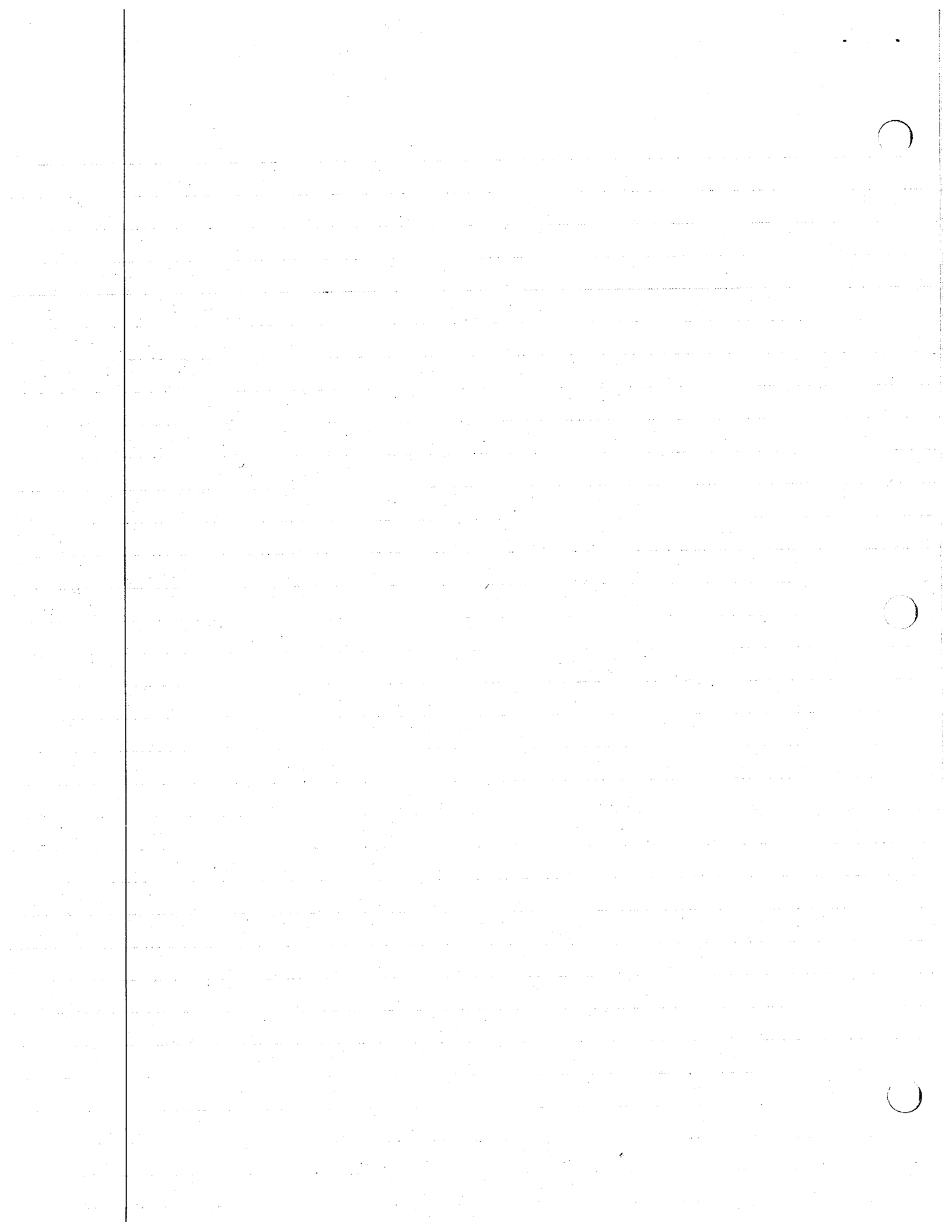
now $W_{\dot{q}_1}^{D_2^*} = 0 \quad W_{\dot{q}_2}^{D_2^*} = r (\sin\phi \cos\beta p_1 - \sin\phi \sin\beta p_3 - \cos\phi p_2)$

$$W_{\dot{q}_1}^{D_1^*} = r (\sin\phi \cos\beta p_1 - \sin\phi \sin\beta p_3 - \cos\phi p_2) \quad W_{\dot{q}_2}^{D_1^*} = 0$$

$$W_{\dot{q}_1}^{S^*} = \frac{r}{2} (\sin\phi \cos\beta p_1 - \sin\phi \sin\beta p_3 - \cos\phi p_2) = W_{\dot{q}_2}^{S^*}$$

$$a_1^{D_2^*} = r \dot{q}_2 (\sin\phi \cos\beta p_1 - \sin\phi \sin\beta p_3 - \cos\phi p_2) + r \dot{q}_1 \dot{\phi} (\cos\phi \cos\beta p_1 - \cos\phi \sin\beta p_3 + \sin\phi p_2)$$

$$a_1^{D_1^*} = -\ddot{\phi} r$$



$$R'_{a1} D_2^* = -r \ddot{q}_2 m_2 - r \dot{q}_2 (-\dot{\phi} m_1 + \dot{q}_2 m_3)$$

$$R'_{a1} D_1^* = -r \ddot{q}_1 m_2 - r \dot{q}_1 (-\dot{\phi} m_1 + \dot{q}_1 m_3)$$

$$R'_{a1} S^* = -\frac{r}{2} (\ddot{q}_1 + \ddot{q}_2) m_2 - \frac{r}{2} (\dot{q}_1 + \dot{q}_2) (-\dot{\phi} m_1)$$

$$II^S = \frac{2ML^2}{3} (m_1 m_2 + m_3 m_3) = \frac{2WL^2}{3g} (m_2 m_2 + m_3 m_3)$$

$$IV^{D_2^*} = -r \dot{q}_2 m_2$$

$$IV^{D_1^*} = -r \dot{q}_1 m_2$$

$$IV^{S^*} = -\frac{r}{2} (\dot{q}_1 + \dot{q}_2) m_2$$

$$\ddot{\phi} = \frac{r}{2L} [\ddot{q}_1 - \ddot{q}_2]$$

$$II^{D_2^*} = II^{D_1^*} = \frac{Wr^2}{4g} [2m_1 m_1 + m_2 m_2 + m_3 m_3]$$

$$R'_{\omega} D_2^* = \dot{\phi} m_3 + \dot{q}_2 m_1 \quad R'_{\omega} D_1^* = \dot{\phi} m_3 + \dot{q}_1 m_1$$

$$R'_{\omega} S^* = \dot{\phi} m_3; \quad R'_{\alpha} D_2^* = \dot{\phi} m_3 + \ddot{q}_2 m_1 + \dot{q}_2 \dot{\phi} m_2$$

$$R'_{\alpha} D_1^* = \dot{\phi} m_3 + \ddot{q}_1 m_1 + \dot{q}_1 \dot{\phi} m_2; \quad R'_{\alpha} S^* = \dot{\phi} m_3$$

$$(II^{D_1^*} \cdot \omega) = \frac{Wr^2}{4g} [2\dot{q}_1 m_1 + \dot{\phi} m_3]$$

$$(II^{D_1^*} \cdot \omega) \times \omega = \frac{Wr^2}{4g} [2\dot{q}_1 \dot{\phi} m_2 + \dot{\phi} \dot{q}_1 m_2] = -\frac{Wr^2}{4g} \dot{q}_1 \dot{\phi} m_2$$

$$II^{D_2^*} \cdot \omega = \frac{Wr^2}{4g} [2\dot{q}_2 m_1]$$

$$(II^{D_2^*} \cdot \omega) \times \omega = -\frac{Wr^2}{4g} [\dot{q}_2 \dot{\phi} m_2]$$

$$II^{D_1^*} \cdot \alpha = \frac{Wr^2}{4g} [2\ddot{q}_1 m_1 + \ddot{q}_1 \dot{\phi} m_2 + \dot{\phi} m_3]$$

$$II^{D_2^*} \cdot \alpha = \frac{Wr^2}{4g} [2\ddot{q}_2 m_1 + \ddot{q}_2 \dot{\phi} m_2 + \dot{\phi} m_3]$$

$$R'_{\dot{q}_1} D_1^* = \left(\frac{r}{2L} m_3 + m_1 \right)$$

$$\omega_{\dot{q}_2}^{D_1^*} = -\frac{r}{2L} m_3$$

$$R'_{\dot{q}_1} D_2^* = \frac{r}{2L} m_3$$

$$\omega_{\dot{q}_2}^{D_2^*} = -\frac{r}{2L} m_3 + m_1$$

$$R'_{\dot{q}_1} S^* = \frac{r}{2L} m_3$$

$$\omega_{\dot{q}_2}^{S^*} = \frac{r}{2L} m_3$$

$$(II^S \cdot \omega) = \frac{2WL^2}{3g} (\dot{\phi} m_3); \quad (II^S \cdot \omega) \times \omega = 0 \quad II^S \cdot \alpha = \frac{2WL^2}{3g} \ddot{\phi} m_3$$

$$\text{Thus } \omega_{\dot{q}_1}^{D_1^*} \cdot \Pi^{D_1^*} = \frac{r}{2L} \left(-\ddot{\phi} \frac{Wr^2}{4g} \right) - \frac{Wr^2}{4g} (2\ddot{q}_1)$$

$$\omega_{\dot{q}_2}^{D_1^*} \cdot \Pi^{D_1^*} = -\frac{r}{2L} \left[-\frac{Wr^2}{4g} \ddot{\phi} \right]$$

$$\omega_{\dot{q}_1}^{D_2^*} \cdot \Pi^{D_2^*} = +\frac{r}{2L} \left[-\frac{Wr^2}{4g} \ddot{\phi} \right]$$

$$\omega_{\dot{q}_2}^{D_2^*} \cdot \Pi^{D_2^*} = -\frac{r}{2L} \left[-\frac{Wr^2}{4g} \ddot{\phi} \right] - \frac{Wr^2}{4g} 2\ddot{q}_2$$

$$\omega_{\dot{q}_1}^S \cdot \Pi^S = \frac{r}{2L} \cdot \frac{2WL^2}{3g} \ddot{\phi}$$

$$\omega_{\dot{q}_2}^S \cdot \Pi^S = +\frac{r}{2L} \cdot \frac{2WL^2}{3g} \ddot{\phi}$$

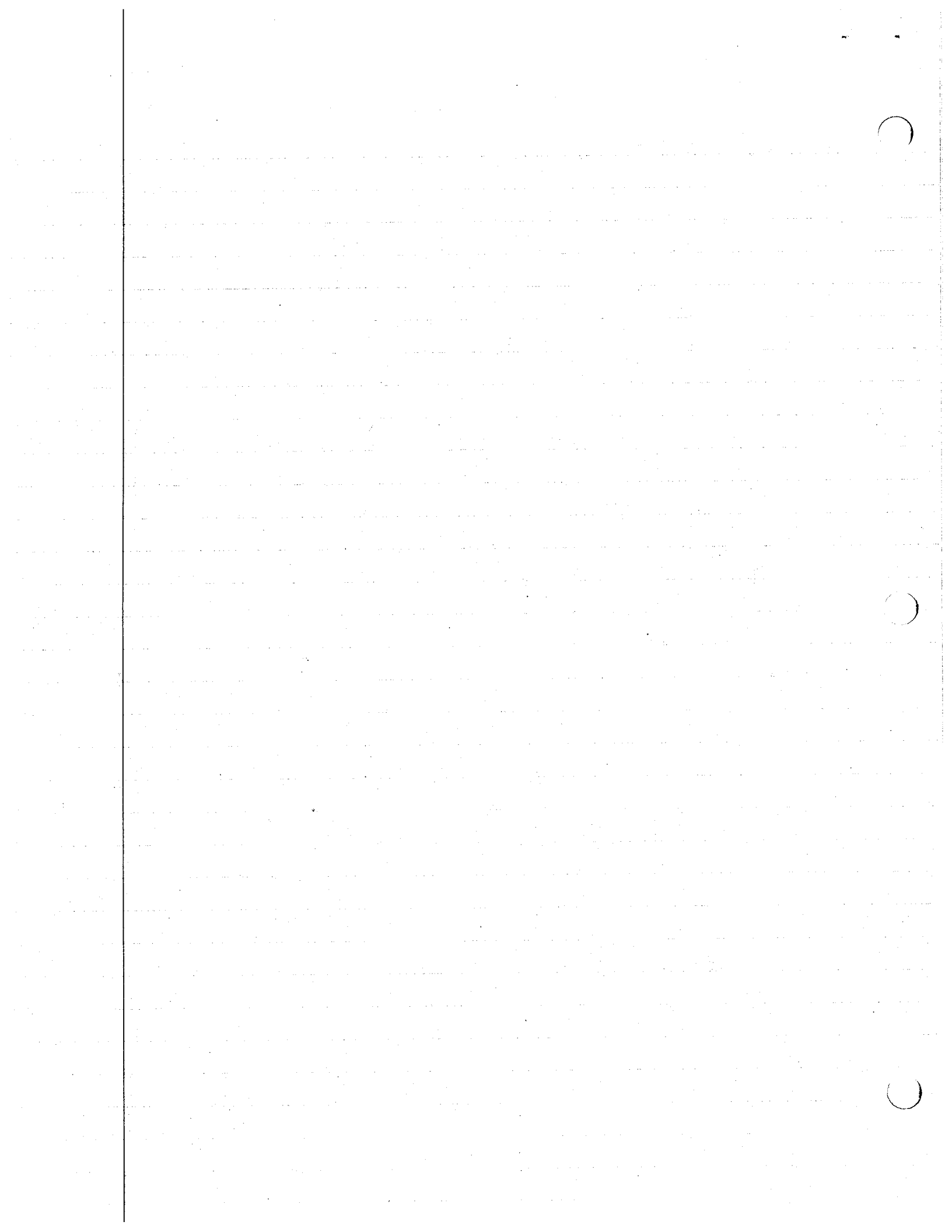
$$\text{Now } IV_{\dot{q}_1}^{D_2^*} = 0 \quad IV_{\dot{q}_2}^{D_2^*} = -r m_2 \quad IV_{\dot{q}_1}^{D_1^*} = -r m_2 \quad IV_{\dot{q}_2}^{D_1^*} = 0 \quad IV_{\dot{q}_1}^{S^*} = -\frac{r}{2} m_2 \quad IV_{\dot{q}_2}^{S^*} = -\frac{r}{2} m_2$$

$$-M IV_{\dot{q}_1}^{D_2^*} \cdot a1^{D_2^*} = 0; \quad -M IV_{\dot{q}_2}^{D_2^*} \cdot a1^{D_2^*} = -Mr^2 \ddot{q}_2; \quad -M IV_{\dot{q}_1}^{D_1^*} \cdot a1^{D_1^*} = -Mr^2 \ddot{q}_1; \quad -M IV_{\dot{q}_2}^{D_1^*} \cdot a1^{D_1^*} = 0;$$

$$-M IV_{\dot{q}_1}^{S^*} \cdot a1^S = -\frac{Mr^2}{4} (\ddot{q}_1 + \ddot{q}_2); \quad -M IV_{\dot{q}_2}^{S^*} \cdot a1^S = -\frac{Mr^2}{4} (\ddot{q}_1 + \ddot{q}_2)$$

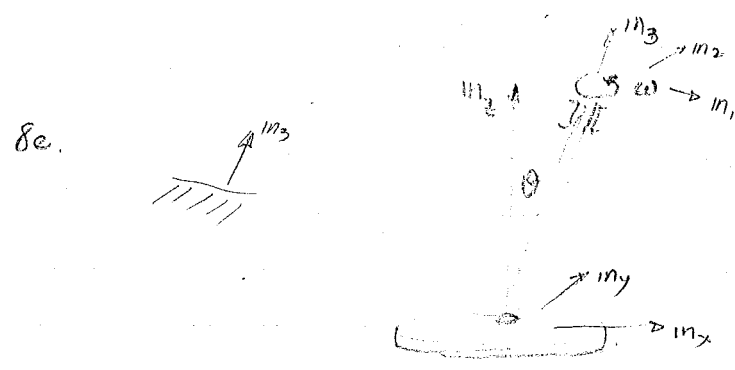
Now

$$F_1^* = \left[-\frac{2Wr^2}{4g} (\ddot{q}_1 + \ddot{q}_2) - \frac{Mr^2}{g} \ddot{q}_1 + \frac{r}{2L} \cdot \frac{2WL^2}{3g} \cdot \frac{r}{2L} (\ddot{q}_1 - \ddot{q}_2) + \frac{r}{2L} \left[\frac{Wr^2}{4g} \cdot \frac{r}{2L} (\ddot{q}_1 - \ddot{q}_2) \right] - \frac{Wr^2}{4g} 2\ddot{q}_1 \right. \\ \left. - \frac{r}{2L} \frac{Wr^2}{4g} \cdot \frac{r}{2L} (\ddot{q}_1 - \ddot{q}_2) \right] = -\frac{3Wr^2}{2g} \ddot{q}_1 - \frac{Wr^4}{8gL^2} (\ddot{q}_1 - \ddot{q}_2) + \frac{Wr^2}{6g} (\ddot{q}_1 - \ddot{q}_2) - \frac{Wr^2}{2g} (\ddot{q}_1 + \ddot{q}_2) \\ = -\ddot{q}_1 \left(+\frac{3Wr^2}{2g} + \frac{Wr^4}{8gL^2} + \frac{2Wr^2}{3g} \right) - \ddot{q}_2 \left(\frac{4Wr^2}{3g} + \frac{Wr^4}{8gL^2} \right)$$



$$F_2^* = \left[-\frac{2Wr^2}{4g} (\ddot{q}_1 + \ddot{q}_2) - \frac{Wr^2}{g} \ddot{q}_2 + \frac{r^2}{24L^2} \cdot \frac{2WL^2}{3g} (\ddot{q}_1 - \ddot{q}_2) + \frac{r^4}{4L^2} \frac{W}{4g} (\ddot{q}_1 - \ddot{q}_2) + \frac{r^4}{4L^2} \frac{W}{4g} (\ddot{q}_1 - \ddot{q}_2) - \frac{Wr^2}{2g} \ddot{q}_2 \right] = -\frac{3Wr^2}{2g} \ddot{q}_2 + \frac{r^4 W}{8L^2 g} (\ddot{q}_1 - \ddot{q}_2) + \frac{Wr^2}{4g} \left(\frac{1}{3} \ddot{q}_1 - \frac{1}{3} \ddot{q}_2 \right)$$

$$= -\ddot{q}_1 \left(-\frac{r^4 W}{8L^2 g} + \frac{1}{3} \frac{Wr^2}{g} \right) + \ddot{q}_2 \left(\frac{r^4 W}{8L^2 g} + \frac{3r^2 W}{2g} + \frac{2}{3} \frac{Wr^2}{g} \right)$$



8e.

$$R_{\text{Rod}} = \omega m_3$$

$$R_{\text{Disk}} = R_{\text{Rod}} + R_{\text{Disk}} = \omega + \omega = 2\omega$$

$$= \omega \cos \theta m_2 + \omega \sin \theta m_x$$

what is Π^* disk $m_2 = \cos \theta m_1$

$$\Pi = \frac{mR^2}{2} m_2 m_2 + \frac{mR^2}{4} (m_x m_x + m_y m_y)$$

$$\Pi \cdot \omega = \frac{mR^2}{2} \omega \cos \theta m_2 + \frac{mR^2}{4} \omega \sin \theta m_x$$

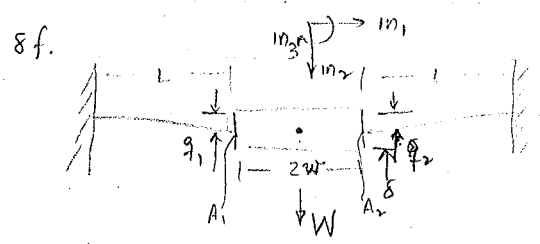
$$(\Pi \cdot \omega) \times \omega = \frac{mR^2}{2} \omega^2 \sin \theta \cos \theta m_y - \frac{mR^2}{4} \omega^2 \sin \theta \cos \theta m_y$$

$$= \frac{mR^2 \omega^2 \sin \theta \cos \theta m_y}{4} = \frac{mR^2 \omega^2 \sin 2\theta m_y}{8}$$

$$\alpha^{\text{Rod}} = \omega m_3 \text{ but } \omega = \text{const} \therefore \alpha = 0 \Rightarrow \Pi \cdot \alpha = 0$$

$$\Pi^* = (\Pi \cdot \omega) \times \omega - \Pi \cdot \alpha = \frac{mR^2 \omega^2 \sin 2\theta m_y}{8}$$

D'Alembert Principal $\Pi + \Pi^* = 0 \Rightarrow \Pi^* = -\Pi = \text{torque applied by bearings}$



Assume $\delta \ll 2w$ thus block is like a thin rod.

We had found $\omega^B = (-\dot{q}_1 + \dot{q}_2)/2w m_3$ and that $v^{A1} = \dot{q}_1 m_2$ $v^{A2} = \dot{q}_2 m_2$

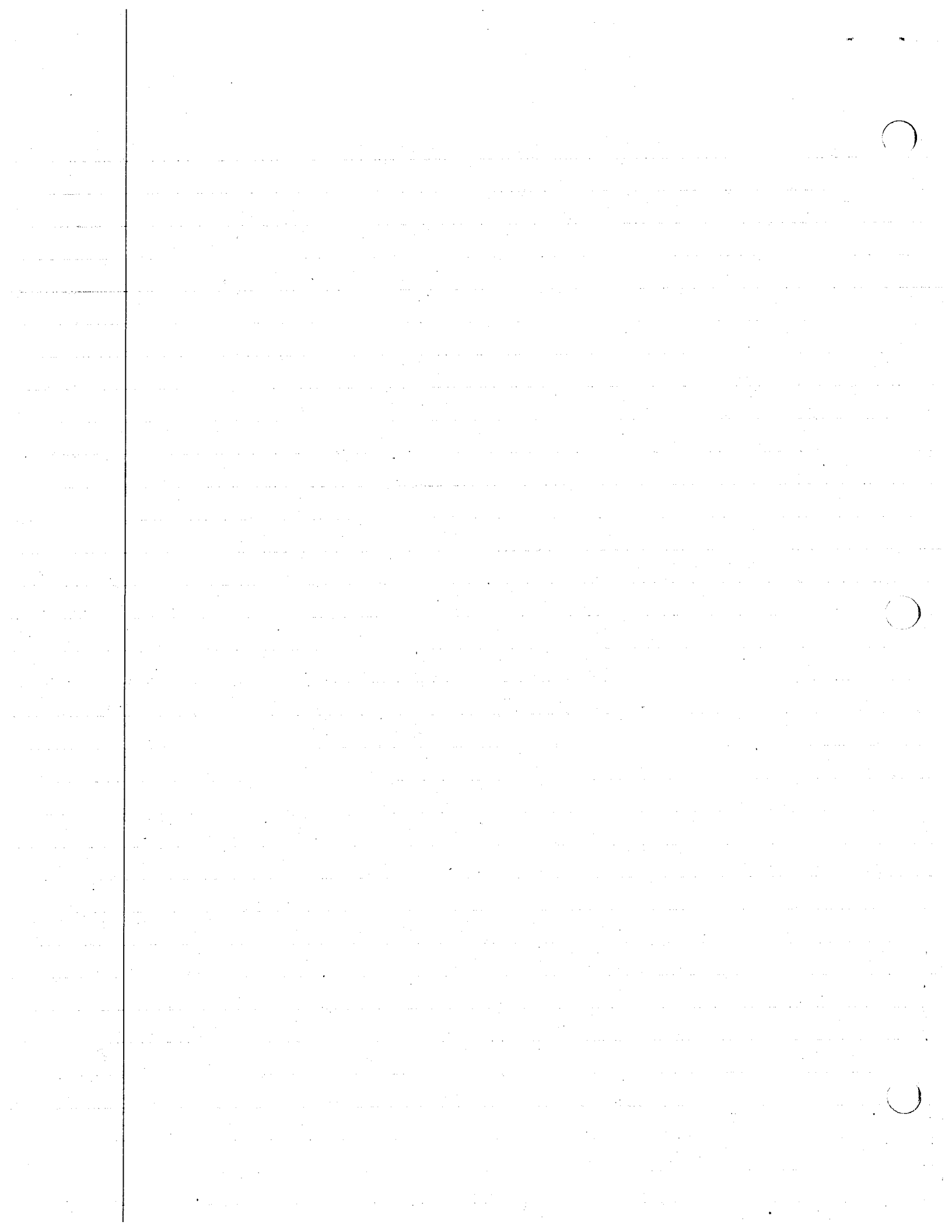
$$\omega = \dot{\theta} m_3 \quad \omega \dot{q}_1 = -\frac{1}{2w} m_3 \quad \omega \dot{q}_2 = \frac{1}{2w} m_3$$

$$v^{B1} = v^{A1} + \omega \times r m_1 = \frac{\dot{q}_2 + \dot{q}_1}{2} m_2$$

$$v^{B2} = \frac{1}{2} m_2 \quad v^{B1} = \frac{m_2}{2}$$

$$a^{B1} = \ddot{q}_2 + \ddot{q}_1 m_2$$

$$\therefore -m v_{\dot{q}_1} \cdot a^{B1} = -\frac{m}{4} (\ddot{q}_2 + \ddot{q}_1) \quad -m v_{\dot{q}_2} \cdot a^{B2} = -\frac{m}{4} (\ddot{q}_2 + \ddot{q}_1)$$



$$I^{B*} = \frac{m l^2}{12} (m_2 m_2 + m_3 m_3) \quad m = \frac{W}{g} \quad l = 2w$$

$$= \frac{W}{3g} w^2 (m_2 m_2 + m_3 m_3)$$

$$I \cdot \omega = \frac{W w^2}{3g} (\ddot{q}_2 - \ddot{q}_1) m_3 \quad (I \cdot \omega) \times \omega = 0$$

$$\alpha = \frac{\ddot{q}_2 - \ddot{q}_1}{2w} m_3$$

$$I \cdot \alpha = \frac{W}{3g} w^2 \frac{\ddot{q}_2 - \ddot{q}_1}{2w} m_3$$

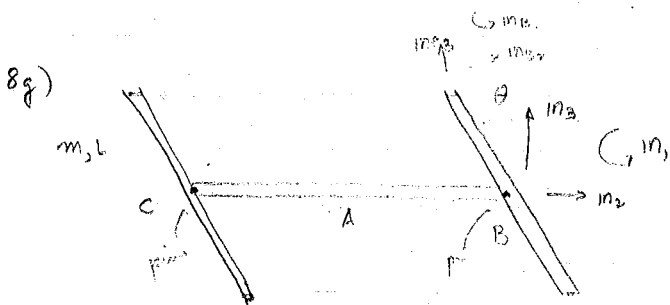
$$\pi^* = -\frac{W}{6g} w (\ddot{q}_2 - \ddot{q}_1) m_3$$

$$\omega_{\dot{q}_1} \cdot \pi^* = \frac{W}{12g} (\ddot{q}_2 - \ddot{q}_1) \quad \omega_{\dot{q}_2} \cdot \pi^* = -\frac{W}{12g} (\ddot{q}_2 - \ddot{q}_1)$$

$$F_1^* = -\frac{W}{4g} (\ddot{q}_2 + \ddot{q}_1) + \frac{W}{12g} (\ddot{q}_2 - \ddot{q}_1) = -\ddot{q}_1 \left(\frac{1}{3} \frac{W}{g} \right) - \ddot{q}_2 \left(\frac{1}{6} \frac{W}{g} \right)$$

$$= -\frac{W}{6g} [2\ddot{q}_1 + \ddot{q}_2]$$

$$F_2^* = -\frac{W}{4g} (\ddot{q}_2 + \ddot{q}_1) - \frac{W}{12g} (\ddot{q}_2 - \ddot{q}_1) = -\ddot{q}_1 \left(\frac{1}{6} \frac{W}{g} \right) - \ddot{q}_2 \left(\frac{1}{3} \frac{W}{g} \right) = -\frac{1}{6} \frac{W}{g} [\ddot{q}_1 + 2\ddot{q}_2]$$



$$\text{let } u_i = \begin{cases} \omega \cdot m_i & i=1,2,3 \\ \dot{\theta} & i=4 \\ W \cdot m_{i-4} & i=5,6,7 \end{cases}$$

$$\text{where } W^A, \omega^A$$

drop all 2nd & higher degrees of θ
& 2nd & higher degrees of $\dot{\theta}$, $\ddot{\theta}$ etc

find F_1^*, F_2^*, F_7^*

$$\text{now } W^A = (W \cdot m_{i-4}) m_{i-4} = u_5 m_1 + u_6 m_2 + u_7 m_3 \quad \left(W_{u_1}^A = 0 \quad W_{u_4}^A = 0 \quad W_{u_7}^A = m_3 \right)$$

$$\omega^A = u_1 m_1 + u_2 m_2 + u_3 m_3 \quad \left| \begin{matrix} \omega_{u_7}^A = 0 & \omega_{u_1}^A = m_1 \\ \omega_{u_4}^A = 0 \end{matrix} \right| \quad \left(\begin{matrix} l & \gamma & \frac{m l^2}{12} \\ & & \end{matrix} \right)$$

$$I^{A/A^*} = \frac{m l^2}{12} (m_3 m_3 + m_1 m_1)$$

for m_1 fixed C A mly

$$\alpha^A = \dot{u}_5 m_1 + \dot{u}_6 m_2 + \dot{u}_7 m_3 + (u_5 u_2 + u_6 u_1) m_3 + (u_5 u_3 - u_7 u_1) m_2 + (-u_6 u_3 + u_7 u_2) m_1$$

$$-m W_{u_7}^A \cdot \alpha^A = -m m_3 \cdot \alpha^* = -m \ddot{u}_7 = m (u_6 \dot{u}_1 - u_5 \dot{u}_2)$$

$$\pi^A = (I \cdot \omega) \times \omega = I \cdot \alpha$$

$$\alpha^A = \dot{u}_1 m_1 + \dot{u}_2 m_2 + \dot{u}_3 m_3 + u_1 (u_2 m_3 + u_3 m_2) + u_2 (u_1 m_3 - u_3 m_1) + u_3 (-u_1 m_2 + u_2 m_1)$$

$$(I \cdot \omega) = \frac{m l^2}{12} (u_3 m_3 + u_1 m_1) \quad (I \cdot \omega) \times \omega = \frac{m l^2}{12} (u_3 u_1 m_2 - u_3 u_2 m_1 + u_1 u_2 m_3 - u_1 u_3 m_2)$$

$$I \cdot \alpha^A = \frac{m l^2}{12} [(\dot{u}_1 - u_2 u_3 + u_3 u_2) m_1 + (\dot{u}_3 - u_1 u_2 + u_2 u_1) m_3]$$

$$\therefore \pi^A = \frac{m l^2}{12} [(-u_3 u_2 - \dot{u}_1) m_1 + (u_1 u_2 - \dot{u}_3) m_3]$$

$$W_{u_7}^A \cdot \pi^A = 0$$

$$\omega^B = \dot{\theta} m_1 = u_4 m_1 \quad \omega^B = u_4 m_1 + \omega^A$$

$$\omega_{u_7}^B = 0 \quad \omega_{u_4}^B = m_1 \quad \omega_{u_1}^B = m_1$$

$$W^B = W^A + \omega^A \times \frac{l}{2} m_2 = W^A + [(u_4 + u_1) m_1 + u_2 m_2 + u_3 m_3] \times \frac{l}{2} m_2 = u_5 m_1 + (u_6 - u_3 \frac{l}{2}) m_2 + (u_7 + u_4 \frac{l}{2}) m_3$$

$$\alpha^B = \dot{u}_5 m_1 + (\dot{u}_6 - \dot{u}_3 \frac{l}{2}) m_2 + (\dot{u}_7 + (\dot{u}_4 + \dot{u}_1) \frac{l}{2}) m_3 + u_5 (-u_2 m_3 + u_3 m_2) + (u_6 - u_3 \frac{l}{2}) (u_1 m_3 + u_2 m_1)$$

$$+ (u_7 + (u_1 + u_4) \frac{l}{2}) (-u_1 m_2 + u_2 m_1)$$

$$W_{u_7}^B = m_3; W_{u_4}^B = \frac{l}{2} m_3; W_{u_1}^B = \frac{l}{2} m_2$$

$${}^A \omega^C = \dot{\theta} \hat{n}_1 = u_4 \hat{n}_1 \quad \omega^C = \omega^A + u_4 \hat{n}_1$$

$$\left[\begin{array}{ccc} \omega_{u_7}^C = 0 & \omega_{u_4}^C = \hat{n}_1 & \omega_{u_1}^C = \hat{n}_1 \end{array} \right]$$

$$\omega^B = (u_4 + u_1) \hat{n}_1 + u_2 \hat{n}_2 + u_3 \hat{n}_3$$

$$W^C = W^A + \omega^A \times \frac{l}{2} \hat{n}_2 = u_5 \hat{n}_2 + (u_6 + u_3 l/2) \hat{n}_2 + (u_7 + (u_1 + u_4) l/2) \hat{n}_3$$

$$\left[\begin{array}{ccc} W_{u_7}^C = \hat{n}_3 & W_{u_4}^C = -\frac{l}{2} \hat{n}_3 & W_{u_1}^C = -\frac{l}{2} \hat{n}_3 \end{array} \right]$$

$$a_1^C = \ddot{u}_5 \hat{n}_2 + (\ddot{u}_6 + \ddot{u}_3 l/2) \hat{n}_2 + [\ddot{u}_7 - (\ddot{u}_1 + \ddot{u}_4) l/2] \hat{n}_3$$

$$+ [u_7 - (u_1 + u_4) l/2] (\omega^B \times \hat{n}_3); \quad \Pi^{B/A} = \frac{m l^2}{12} (m_{B_2} m_{B_2} + m_{B_3} m_{B_3})$$

$$+ u_5 (\omega^B \times \hat{n}_1) + (u_6 + u_3 l/2) (\omega^B \times \hat{n}_2) +$$

	$\hat{n}_1$	$\hat{n}_2$	$\hat{n}_3$
m_{B_1}	1	0	0
m_{B_2}	0	$c\theta$	$+s\theta$
m_{B_3}	0	$-s\theta$	$c\theta$

$$= \frac{m l^2}{12} [(c\theta m_2 + s\theta m_3)(c\theta m_2 + s\theta m_3) + m_1 m_1]$$

$$= \frac{m l^2}{12} [c\theta^2 m_2 m_2 + c\theta s\theta (m_2 m_3 + m_3 m_2) + s\theta^2 m_3 m_3 + m_1 m_1]$$

$$\omega^B = (u_4 + u_1) \hat{n}_1 + u_2 \hat{n}_2 + u_3 \hat{n}_3$$

$$(\Pi^{B/A} \cdot \omega^B) = [(u_4 + u_1) \hat{n}_1 + u_2 (c\theta^2 \hat{n}_2 + c\theta s\theta \hat{n}_3) + u_3 (s\theta^2 \hat{n}_2 + c\theta s\theta \hat{n}_3)] \frac{m l^2}{12}$$

$$(\Pi \cdot \omega^B) \times \omega^B = \left[(u_4 + u_1) u_2 m_3 - (u_6 + u_1) u_3 m_2 + u_2 c\theta^2 (u_4 + u_1) m_3 + u_2 u_3 (c\theta^2 \hat{n}_1 + u_2^2 \hat{n}_1) c\theta s\theta m_2 \right. \\ \left. + u_2 c\theta s\theta u_2 m_1 + u_3 s\theta^2 (u_4 + u_1) m_2 - u_2 u_3 s\theta^2 m_1 - u_3 c\theta s\theta (u_4 + u_1) m_3 + u_3^2 c\theta s\theta m_1 \right] \frac{m l^2}{12}$$

$$= [(u_4 + u_1) u_2 s\theta^2 m_3 - u_3 (u_4 + u_1) c\theta^2 m_2 + u_2 u_3 c_{2\theta} m_1 + u_2^2 (u_4 + u_1) c\theta s\theta m_2 \\ - u_2^2 c\theta s\theta m_1 - u_3 (u_4 + u_1) c\theta s\theta m_3 + u_3^2 c\theta s\theta m_1] \frac{m l^2}{12}$$

$$\omega_{u_7}^B = 0 \quad \omega_{u_7}^B \cdot \Pi^B = \omega_{u_7}^B \cdot [(\Pi \cdot \omega) \times \omega^B + \Pi \cdot \alpha^B] = 0$$

$$\text{Since bar C has same } \Pi^{C/A} \text{ as } \Pi^{B/A} \quad \omega_{u_7}^C = \omega_{u_7}^B = 0$$

$$\text{Then } (\Pi \cdot \omega^C) \times \omega^C = \Pi \cdot \alpha^C = \Pi^{C/A} = \Pi^{B/A} \quad \text{hence } \omega_{u_7}^C \cdot \Pi^C = 0 \text{ also}$$

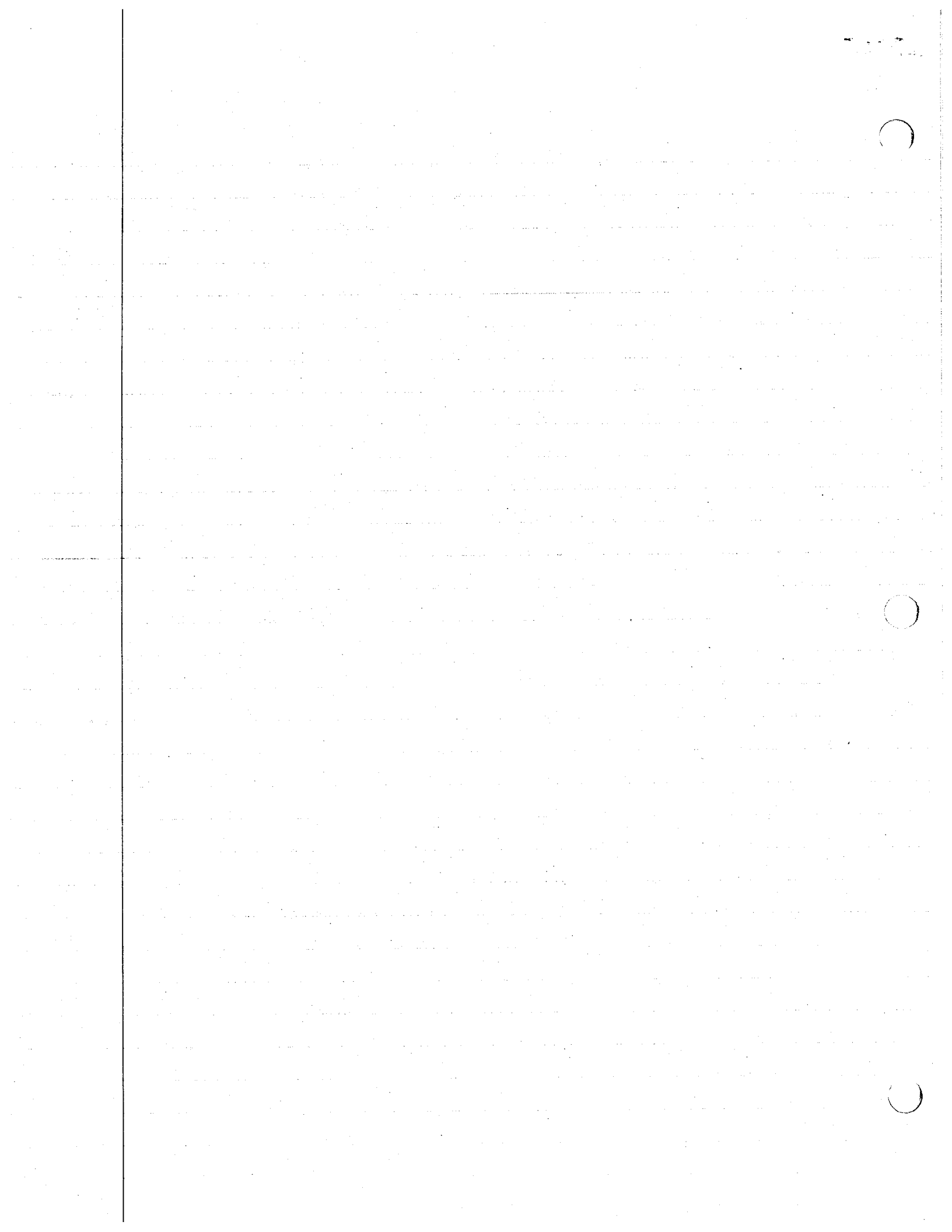
Thus along contribute

$$\text{to: } F_7^* = -m W_{u_7}^A \cdot a_1^A - m W_{u_7}^B \cdot a_1^B - m W_{u_7}^C \cdot a_1^C \\ = -m \ddot{u}_7 - m (u_6 u_1 - u_5 u_2) - m (\ddot{u}_7 + \ddot{u}_4 + \ddot{u}_1) l/2 - m (u_1 u_6 - u_2 u_5) \\ - m (\ddot{u}_7 - (\ddot{u}_1 + \ddot{u}_4) l/2) - m (u_1 u_6 - u_2 u_5) = -3m [\ddot{u}_7 + (u_6 u_1 - u_5 u_2)]$$

$$\text{to } F_4^* = + \omega_{u_4}^C \cdot \Pi^C + W_{u_4}^C \cdot F^C + \omega_{u_4}^B \cdot \Pi^B + W_{u_4}^B \cdot F^{*B} \quad \text{since } W_{u_4}^A = \omega_{u_4}^A = 0$$

$$\text{to } F_1^* = (\omega_{u_1}^B \cdot \Pi^{*B} + W_{u_1}^B \cdot F^{*B}) + (\omega_{u_1}^C \cdot \Pi^{*C} + W_{u_1}^C \cdot F^{*C}) + \omega_{u_1}^A \cdot \Pi^{*A}$$

$$\text{in obtaining } \alpha^A \text{ \& } \alpha^B \quad \text{we must use } \dot{\hat{n}}_1 = \omega \times \hat{n}_1 \quad \dot{\hat{n}}_2 = \omega \times \hat{n}_2 \quad \dot{\hat{n}}_3 = \omega \times \hat{n}_3$$



Set #9

9a.

$$F_1 = \frac{G m_1 m_2}{r^2} \hat{r}, \quad F_2 = -\frac{G m_1 m_2}{r^2} \hat{r}, \quad \text{if } \vec{r} = \vec{r}_1 + \vec{r}_2(t)$$

$$V^P = u_{ij} m_i, \quad V^B = u_{2i} m_i$$

$$\therefore V_{u_{11}}^P = m_1, \quad V_{u_{11}}^B = m_1 \text{ since } \vec{r} \text{ is not a fn of } u_{11}$$

$$\therefore F_i' = V_{u_{ii}}^P \cdot F_i + V_{u_{ii}}^B \cdot F_2 = 0 \quad i=1,2,3; \quad F_4' = V_{u_{44}}^P \cdot F_2 = -\frac{G m_1 m_2}{r^2} m_1 = -\frac{G m_1 m_2}{r^2}$$

$$\text{but } F_4' = -\frac{G m_1 m_2}{r^2} = -\frac{\partial P}{\partial r} \quad \therefore \frac{G m_1 m_2}{r^2} = \frac{\partial P}{\partial r} \Rightarrow P = -\frac{G m_1 m_2}{r} + f(u_1, u_2, u_3, t)$$

$$\text{but } \frac{\partial P}{\partial u_1} = 0, \quad \frac{\partial P}{\partial u_2} = 0, \quad \frac{\partial P}{\partial u_3} = 0 \Rightarrow f(u_1, u_2, u_3, t) = w(t) \quad \therefore P = -\frac{G m_1 m_2}{r} + w(t)$$

9b.

$$\omega^B = \omega^I + \omega^B, \quad \omega^I = \Omega m_z + \omega^B \quad \text{let } \omega^B = \omega_{ij} n_i \text{ where } n_i \text{'s are } \parallel \text{ to principal axes of } B$$

$$F = -F n_a \quad \text{and } T = \frac{3GM}{R^3} m_a \times I_a$$

$$V^B = \omega^B \times R n_a = R (\omega^B \times m_a) \quad V_{q_r}^B = \omega_{q_r}^B \times m_a$$

$$\text{now } F_r = V_{q_r}^B \cdot F + \omega_{q_r}^B \cdot T = -F m_a (\omega_{q_r}^B \times n_a) + \omega_{q_r}^B \cdot \frac{3GM}{R^3} m_a \times I_a$$

$$= \frac{3GM}{R^3} \omega_{q_r}^B \cdot (m_a \times I_a) = \frac{3GM}{R^3} I_a \cdot (\omega_{q_r}^B \times m_a)$$

$$\text{but } \frac{\partial m_a}{\partial q_r} = \frac{\partial m_a}{\partial q_r} + \omega_{q_r}^B \times m_a \quad \text{and } \frac{\partial m_a}{\partial q_r} = 0 \text{ since } m_a \text{ is not a fn of } q_r \text{ in } \mathbb{R}$$

$$\therefore F_r = \frac{3GM}{R^3} I_a \cdot \left(-\frac{\partial m_a}{\partial q_r} \right) = -\frac{3GM}{R^3} I_a \frac{\partial m_a}{\partial q_r}$$

$$\text{if } m_a = a_i m_i \text{ then } I_a = I_i a_i m_i + \dots \quad \text{then } I_a \cdot \frac{\partial m_a}{\partial q_r} = I_i (a_i \frac{\partial a_i}{\partial q_r}) = I_i \frac{1}{2} \frac{\partial a_i^2}{\partial q_r}$$

$$\therefore F_r = -\frac{3GM}{2R^3} \frac{\partial (a_i^2 I_i)}{\partial q_r} \quad \text{but } a_i^2 I_i = I = m_a \cdot I_a \cdot m_a$$

$$F_r = -\frac{\partial}{\partial q_r} \left(\frac{3GM}{2R^3} I \right) \quad \therefore P = \frac{3GM I}{2R^3} + w(t)$$

9c.

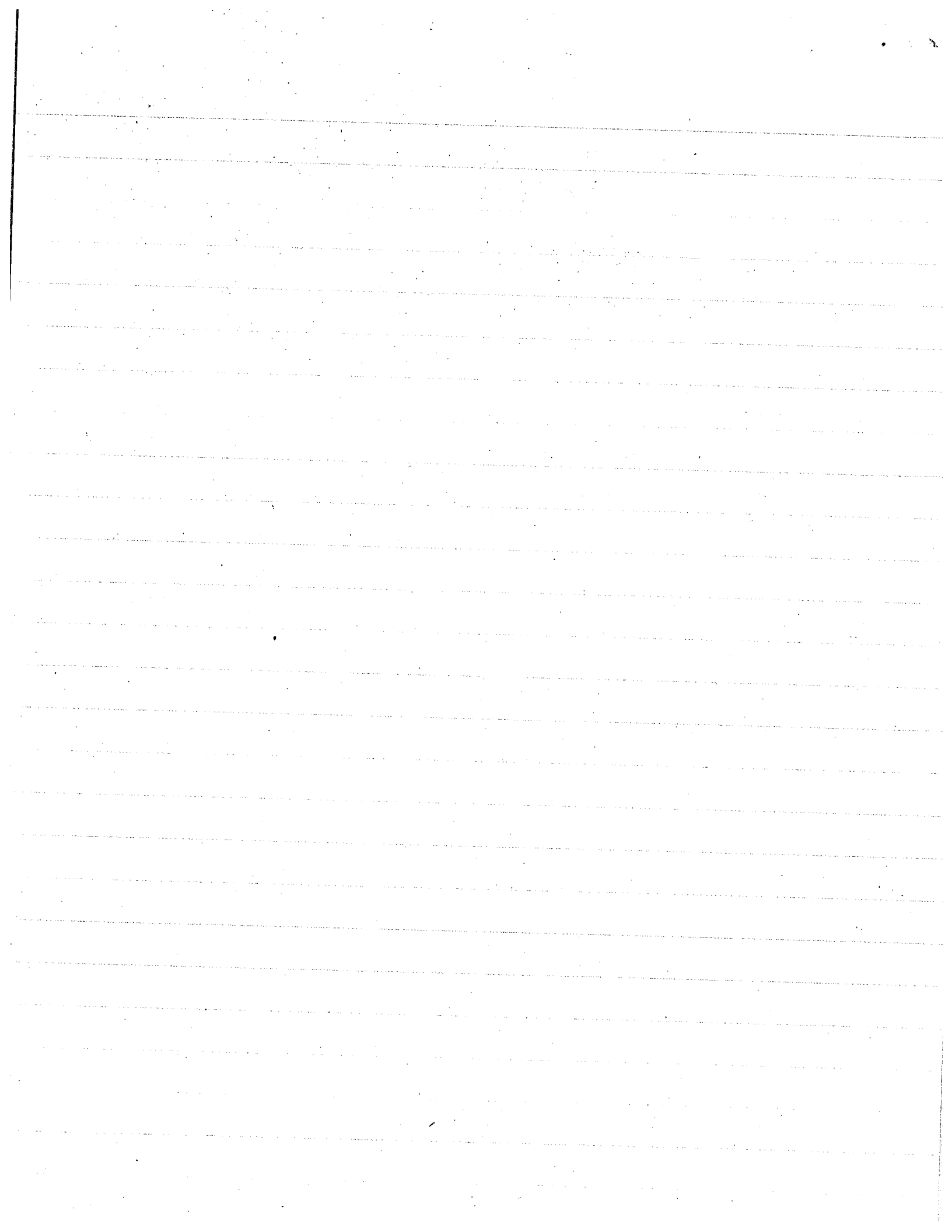
$$W = \rho g \quad \text{displaced volume is } \frac{A y}{\cos \theta} \quad \therefore \text{by Archimedes' principle}$$

$$\text{the buoyant force} = \text{displaced volume} \times \text{weight density}$$

$$F = \frac{A y}{\cos \theta} \cdot \rho g \text{ acting upward at mass center}$$

$$\text{now } V^c = V^A + \theta m_3 \times m_a \quad \text{of submerged portion } V_{\theta}^c = m_3 \times m_a + V_{\theta}^A \quad V_{\theta}^A = 0$$

$$\therefore (F_{\theta})_{\text{buoyant force}} = \frac{A y \rho}{\cos \theta} m_3 \times m_a \cdot m_1$$



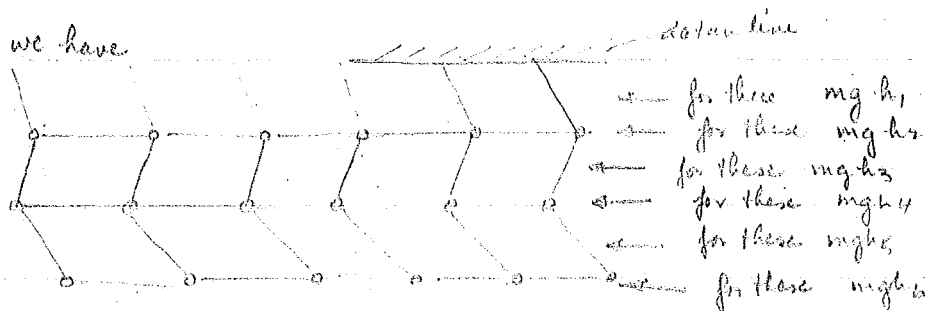
but $m_a = m_1 \cos \theta + m_2 \sin \theta$ $m_3 \times m_a = \cos \theta m_2 - m_1 \sin \theta$ and $m_3 \times m_2 \cdot m_1 = -\sin \theta$

$\therefore F_\theta = -r A y w \tan \theta$ but $r = \frac{y}{2 \cos \theta}$ $\therefore F_\theta = -\frac{A y^2 w \sin \theta}{2 \cos^2 \theta}$

but $F_\theta = -\frac{\partial P}{\partial \theta} = -\frac{A y^2 w \sin \theta}{2 \cos^2 \theta}$ $P = \frac{A y^2 w}{2 \cos \theta} + f(t)$

or $P = \frac{A y^2 w \sec \theta}{2} + f(t)$

9d Now we have



$P = \sum m g h_i$

$h_1 = \frac{l}{2} \cos \theta_1$ $h_2 = -l \sin \theta_1$ $h_3 = -l (\sin \theta_1 + \frac{1}{2} \sin \theta_2)$ $h_4 = -l (\cos \theta_1 + \cos \theta_2)$

$h_5 = -l (\cos \theta_1 + \cos \theta_2 + \frac{1}{2} \cos \theta_3)$ $h_6 = -l (\cos \theta_1 + \cos \theta_2 + \cos \theta_3)$

$\therefore 6 m g h_1 + 5 m g h_2 + 6 m g h_3 + 5 m g h_4 + 6 m g h_5 + 5 m g h_6$

$= l m g [3 \cos \theta_1 + 5 \cos \theta_1 + 6 \cos \theta_1 + 3 \cos \theta_1 + 5 \cos \theta_1 + 5 \cos \theta_2 + 6 \cos \theta_1 + 6 \cos \theta_2 + 3 \cos \theta_3 + 5 \cos \theta_1 + 5 \cos \theta_2 + 5 \cos \theta_3]$

$P = -W l [30 \cos \theta_1 + 19 \cos \theta_2 + 8 \cos \theta_3]$

9e Since we found in 5(a) $F_\theta = -W(R^2 - L^2)^{1/2} \sin \theta = -\frac{\partial P}{\partial \theta}$

$\therefore \frac{\partial P}{\partial \theta} = W(R^2 - L^2)^{1/2} \sin \theta$ $\therefore P = -W(R^2 - L^2)^{1/2} \cos \theta + f(t)$

9f) From 5(d) we found

$F_1 = a_1 q_1 + b_1 q_2 + \frac{W}{2} = -\frac{\partial P}{\partial q_1}$

$F_2 = b_1 q_1 + a_1 q_2 + \frac{W}{2} = -\frac{\partial P}{\partial q_2}$

$\therefore P = -a_1 q_1^2/2 + b_1 q_2 q_1 + \frac{W}{2} q_1 + f(q_2, t)$

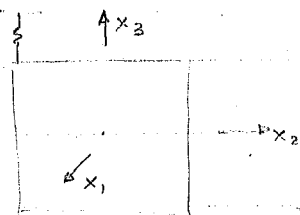
$\frac{\partial P}{\partial q_2} = b_1 q_1 + \frac{\partial f}{\partial q_2} = b_1 q_1 + a_1 q_2 + \frac{W}{2}$

$\frac{\partial f}{\partial q_2} = a_1 q_2 + \frac{W}{2}$ $\therefore f = a_1 q_2^2/2 + \frac{W}{2} q_2 + g(t)$

$\therefore P = -a_1 q_1^2/2 + b_1 q_2 q_1 + \frac{W}{2} (q_1 + q_2) + a_1 q_2^2/2 + g(t)$

$= \left(\frac{q_1^2 - q_2^2}{2} \right) \left[\frac{12 E I}{L^3} \left(1 + \frac{L}{2w} + \frac{1}{6} \frac{L^2}{w^2} \right) \right] + \frac{6 E I}{L^2 w} \left(1 + \frac{L}{3w} \right) q_2 q_1 + \frac{W}{2} (q_1 + q_2) + g(t)$

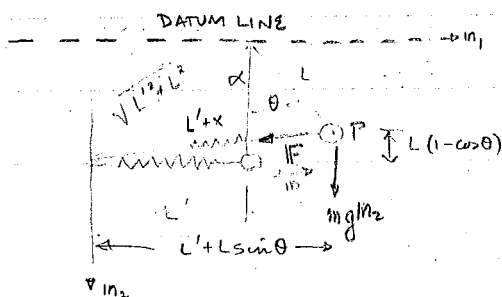
9(g) look at one corner



Movement of the corner of the cube: By looking the movement of the cube corner as the cube ^{center} undergoes displacement to a new point, The cube corner undergoes an extension in the $-x_3$ direction. Now if the cube is rotated about the x_1 axis will give rise to a displacement $a\theta_1$. Now if the cube is rotated about the x_2 direction gives rise to an $a\theta_2$ displacement. Rotation about the x_3 axis gives rise to displacements in the x_1 & x_2 direction thus the overall displacement of the spring is $-x_3 + a\theta_1 + a\theta_2 = x$

Now P for a spring is $\frac{1}{2}kx^2 \therefore P_1 = \frac{1}{2}ka^2(\theta_1 + \theta_2 - x_3/a)^2$. Similarly we can do the same for the other springs to get P_2 & P_3 . Thus $P = \sum P_i$. What we've done here is good for small displacements is displacements for terms of 0 or 1st degree in θ_i and x_i .

9(h)



$$r^P = L \cos \theta m_2 + L \sin \theta m_1$$

$$w^P = (-L \sin \theta m_2 + L \cos \theta m_1) \dot{\theta}$$

$$\text{now } (L' + x)^2 = L^2 + (L'^2 + L^2) - 2L\sqrt{L'^2 + L^2} \cos(\theta + \alpha)$$

$$L'^2 + 2L'x + x^2 = 2L^2 + L'^2 - 2L\sqrt{L'^2 + L^2} \cos(\theta + \alpha)$$

$$= 2L^2 - 2L\sqrt{L'^2 + L^2} \left[\cos \theta \frac{L}{\sqrt{L'^2 + L^2}} - \sin \theta \frac{L'}{\sqrt{L'^2 + L^2}} \right]$$

$$F = -kx \text{ in } m = \frac{(L' + L \sin \theta) m_1 + L(1 - \cos \theta) m_2}{\sqrt{(L' + L \sin \theta)^2 + L^2(1 - \cos \theta)^2}}$$

$$2L'x + x^2 + L'^2 = 2L^2 + 2L^2 \cos \theta + 2LL' \sin \theta + L'^2$$

$$F_{\theta} = -kx \left\{ \frac{L \cos \theta (L' + L \sin \theta) + L^2 \sin \theta (1 - \cos \theta)}{\sqrt{(L' + L \sin \theta)^2 + L^2(1 - \cos \theta)^2}} \right\} - mg L \sin \theta$$

$$x^2 + 2L'x - 2L^2(1 - \cos \theta) - 2LL' \sin \theta = 0$$

the pot. fn due to gravity is $P = mgh$ $h = -L \cos \theta \therefore P = -mgL \cos \theta$ from the datum line

the potential fn due to the spring $= \frac{1}{2}kx^2$

$$\text{Now } (x + L')^2 = 2L^2(1 - \cos \theta) + L'(L' + 2L \sin \theta)$$

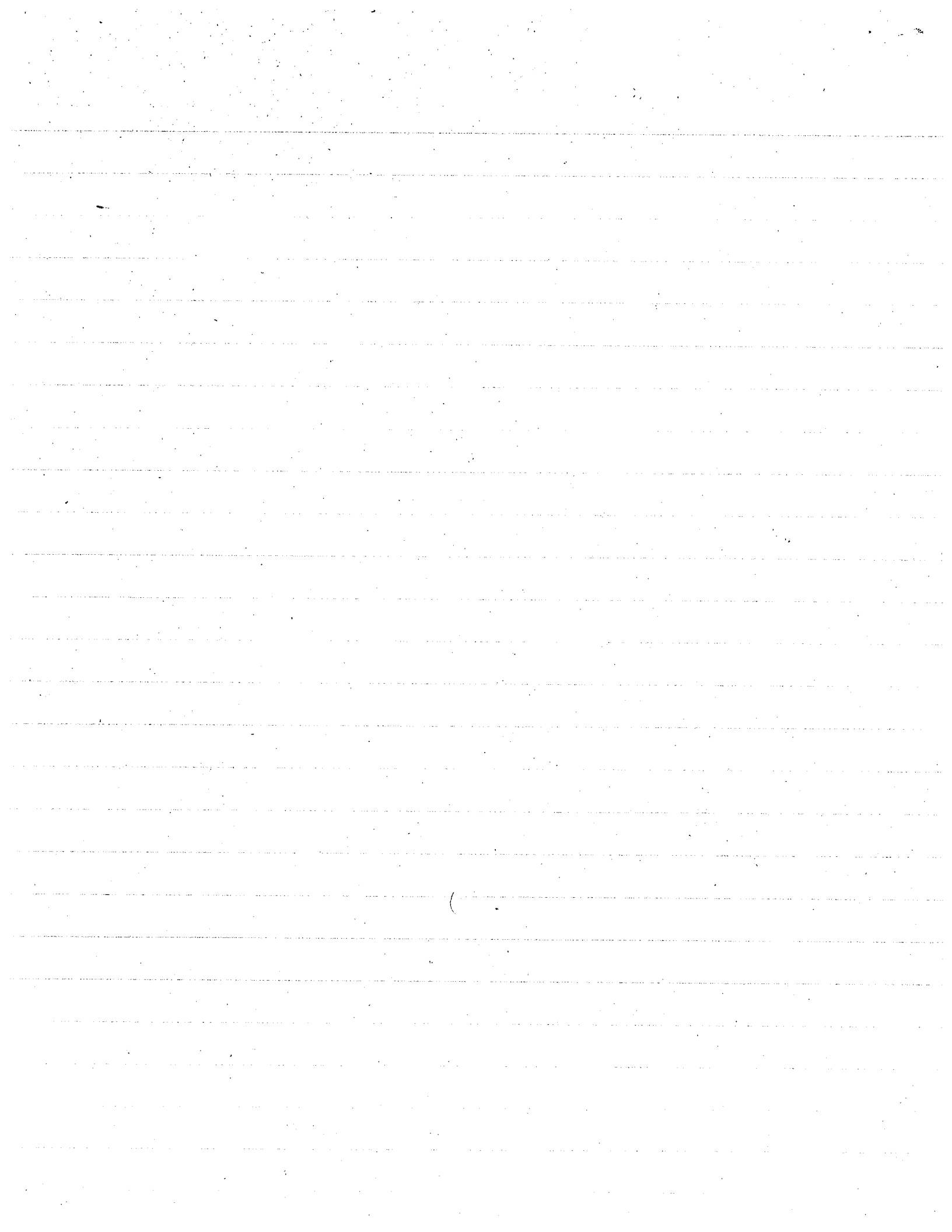
$$(x + L') = \sqrt{\frac{2L^2}{L'^2} (1 - \cos \theta) + \left(1 + \frac{2L}{L'} \sin \theta\right)}$$

$$\text{Use expansion } (1 + p)^{1/2} = 1 + \frac{1}{2}p - \frac{1}{8}p^2 + \frac{1}{16}p^3 \text{ where } p = \frac{2L}{L'} \sin \theta + \frac{2L^2}{L'^2} (1 - \cos \theta)$$

$$\therefore p \approx \frac{2L}{L'} \left(\theta - \frac{\theta^3}{3!} + \dots \right) + \frac{2L^2}{L'^2} \left(\frac{\theta^2}{2} - \frac{\theta^4}{4!} + \dots \right) = \frac{2L}{L'} \left(\theta + \frac{1}{2L'} \theta^2 - \frac{\theta^3}{3!} - \frac{1}{4!L'} \theta^4 \right)$$

$$= 1 + \frac{1}{L'} \left(\theta - \frac{\theta^3}{3!} \right) + \frac{1}{L'^2} \left(\frac{\theta^2}{2} - \frac{\theta^4}{4!} \right) - \frac{1}{8} \frac{4L^2}{L'^2} \left[\theta^2 + \frac{L^2}{4L'^2} \theta^4 + \frac{2L}{2L'} \theta^3 - \frac{2\theta^4}{3!} \right] + \dots$$

$$\therefore x = L \left(\theta - \frac{\theta^3}{3!} \right) + o(\theta^4) + \frac{1}{16} \left[\frac{8L^3}{L'^3} \left(\theta^3 + \frac{3L}{2L'} \theta^4 \right) + \dots \right] x^2 = L^2 \left(\theta^2 - \frac{2\theta^4}{6} + \text{h.o.t.} \right)$$



$$P_{\text{Tot}} = -mgL \left(1 - \frac{\theta^2}{2!} + \frac{\theta^4}{4!} + \dots \right) + \frac{1}{2} \cdot \frac{5mg}{L} \cdot L^2 \left[\theta^2 - \frac{\theta^4}{3} + \dots \right]$$

$$= -mgL + mgL \frac{\theta^2}{2!} - mgL \frac{\theta^4}{4!} + \frac{5}{2} mgL \theta^2 - \frac{5mgL \theta^4}{6} + \dots$$

$$P_{\text{Tot}} - mgL = mgL \left[\theta^2 \cdot 3 \right] - mgL \theta^4 \left[\frac{1}{24} + \frac{5}{6} \right] = mgL \left[3\theta^2 - \frac{7}{8}\theta^4 \right]$$

$$a_1 = 0, a_2 = 3, a_3 = 0, a_4 = -\frac{7}{8}$$

Now $F_\theta = -\frac{Kx}{x+L} \cdot \{ L \cos \theta + L^2 \sin \theta \} - mgL \sin \theta = -\frac{KL(\theta - \theta^3/3!)}{1 + L} \cdot \left\{ (1 - \frac{\theta^2}{2!} + \frac{\theta^4}{4!} + \dots) + \frac{L}{L} (\theta - \frac{\theta^3}{3!} + \dots) \right\} \cdot \left\{ 1 - \frac{L}{L} (\theta - \frac{\theta^3}{3!}) + \frac{L^2}{L^2} (\theta - \frac{\theta^3}{3!})^2 + \dots \right\}$

$$- mgL \left\{ \theta - \frac{\theta^3}{3!} \right\} = -KL^2 (\theta - \frac{\theta^3}{3!}) \left\{ 1 + \frac{L}{L} \theta - \frac{\theta^2}{2!} - \frac{L}{L} \frac{\theta^3}{3!} \right\} \left\{ 1 - \frac{L}{L} \theta + \frac{L^2}{L^2} \theta^2 + \frac{L}{L} \frac{\theta^3}{3!} + \dots \right\} = -KL^2 (\theta - \frac{\theta^3}{3!}) \left\{ 1 + 2\frac{L}{L} \theta^2 - \frac{\theta^2}{2!} \right\} = -mgL \left\{ \theta - \frac{\theta^3}{3!} \right\}$$

$$= -6mgL\theta + 5\frac{mgL}{6}\theta^3 + mgL\frac{\theta^3}{6}$$

q: since $P = mgl \sum a_n \theta^n$ then

$$\frac{\partial P}{\partial \theta} = mgl \sum_{n=1}^{\infty} a_n n \theta^{n-1} = mgl [-6\theta + 3.5\theta^2 + \dots]$$

$$F_\theta = -\frac{\partial P}{\partial \theta} = -mgL [a_1 + a_2 \cdot 2\theta + a_3 \cdot 3\theta^2 + \dots]$$

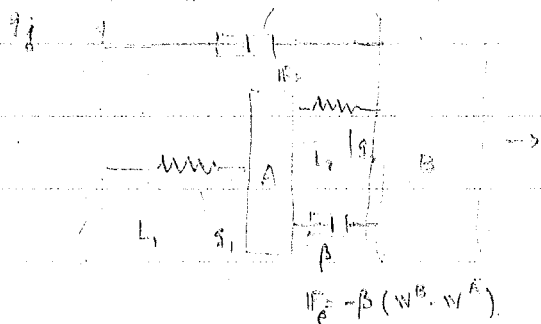
$$mgL [b_1 \theta + b_2 \theta^2 + \dots]$$

$$= mgL [-a_1 - a_2 \cdot 2\theta - a_3 \cdot 3\theta^2 + \dots]$$

$$a_1 = 0, b_1 = -2a_2 = -6, b_2 = -3a_3 = 0, b_3 = -4a_4 = +4 \cdot \frac{7}{8} = +3.5$$

easier to find the potential energy then differentiate. Both methods are cumbersome.

$$F_A = -\alpha W^B$$



$$r^B = (L_1 + q_1 + L_2 + q_2) \cdot m$$

$$r^A = (L_1 + q_1) \cdot m$$

$$W^B = (\dot{q}_1 + \dot{q}_2) \cdot m$$

$$W^B - W^A = \dot{q}_2 \cdot m$$

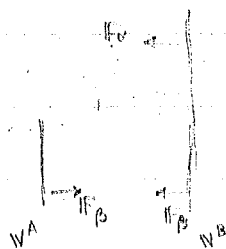
$$W^A = \dot{q}_1 \cdot m$$

$$F_B = -\beta \dot{q}_2 \cdot m$$

$$F_A = -\alpha [\dot{q}_1 + \dot{q}_2] \cdot m$$

$$W=0$$

$$F_A$$



$$F_1' = F_A \cdot W_{q_1}^B + F_B \cdot W_{q_1}^B + F_A \cdot 0 + F_B \cdot W_{q_1}^A$$

$$= -\alpha [\dot{q}_1 + \dot{q}_2] - \beta \dot{q}_2 + \beta \dot{q}_2 = -\alpha [\dot{q}_1 + \dot{q}_2]$$

$$F_2' = F_A \cdot W_{q_2}^B + F_B \cdot W_{q_2}^B + F_A \cdot 0 + F_B \cdot W_{q_2}^A$$

$$= -\alpha [\dot{q}_1 + \dot{q}_2] - \beta \dot{q}_2 = -(\alpha + \beta) \dot{q}_2 - \alpha \dot{q}_1$$

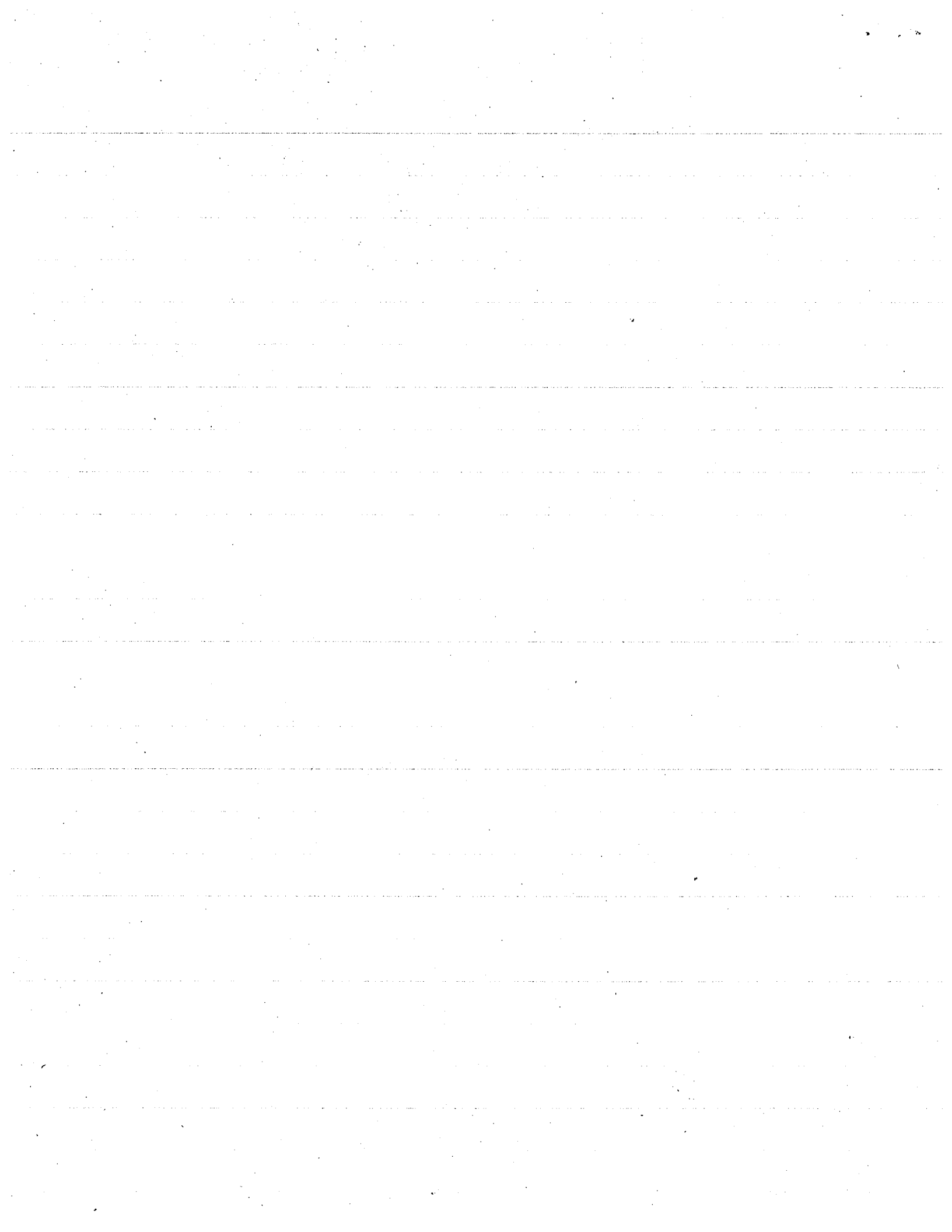
$$\frac{\partial \mathcal{L}}{\partial \dot{q}_1} = -F_1' = \alpha [\dot{q}_1 + \dot{q}_2] \quad \mathcal{L} = \frac{\alpha}{2} [\dot{q}_1^2 + \dot{q}_2^2] + f(\dot{q}_2, t)$$

$$\frac{\partial \mathcal{L}}{\partial \dot{q}_1} = \alpha \dot{q}_1 + f' = -F_2' = (\alpha + \beta) \dot{q}_2 + \alpha \dot{q}_1$$

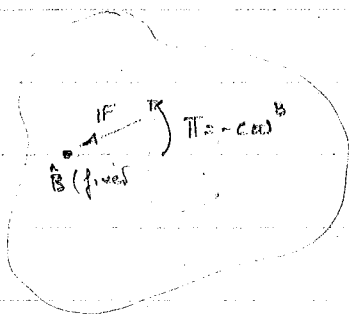
$$f' = (\alpha + \beta) \dot{q}_2$$

$$f = (\alpha + \beta) \frac{\dot{q}_2^2}{2} + g(t)$$

$$\therefore \mathcal{L} = \frac{\alpha}{2} \dot{q}_1^2 + \alpha \dot{q}_1 \dot{q}_2 + \frac{(\alpha + \beta)}{2} \dot{q}_2^2$$



9k



$$F_r = \hat{B} \cdot \nabla_{\hat{q}_r} \cdot \Pi + \omega \dot{q}_r \cdot \Pi \quad \Pi = -c\omega$$

$$\text{since } \hat{B} = 0 \quad (\hat{B} \text{ is fixed})$$

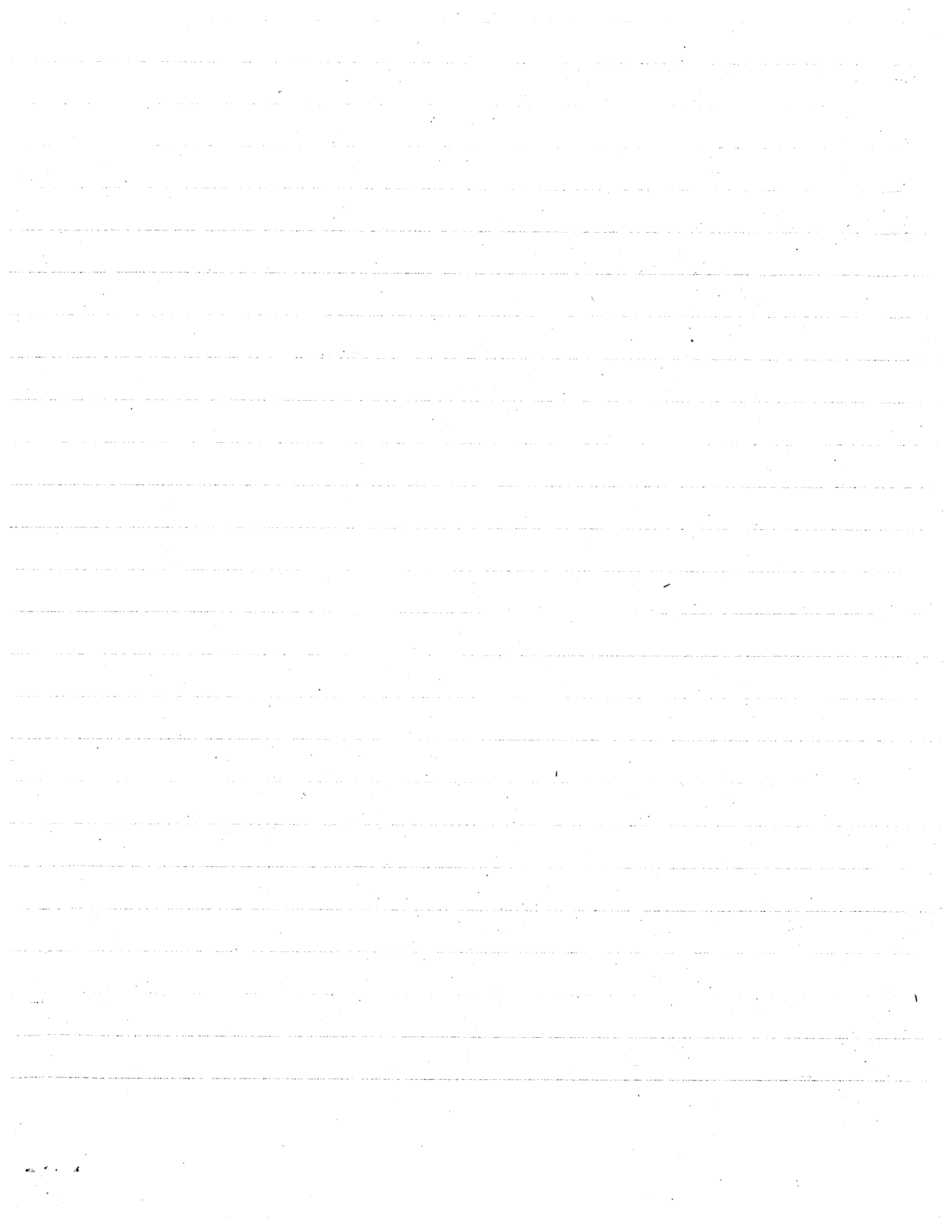
$$F_r = \omega \dot{q}_r \cdot \Pi = \omega \dot{q}_r \cdot (-c\omega)$$

$$\text{now if } \omega \dot{q}_r = \frac{\partial \omega}{\partial \dot{q}_r} \text{ then } \omega = \omega \dot{q}_r \cdot \dot{q}_r + \omega_t$$

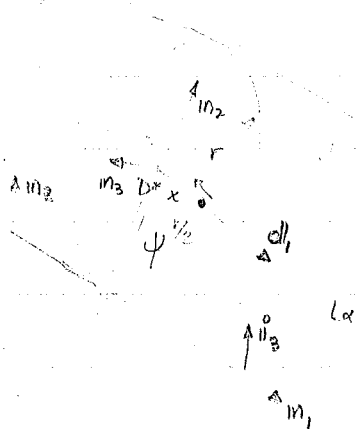
$$\text{hence } F_r = -c (\omega \cdot \omega \dot{q}_r) = c (\omega \dot{q}_r \cdot \omega_t)$$

$$F_r = -c \left(\frac{\partial}{\partial \dot{q}_r} \frac{\omega^2}{2} \right) = c \frac{\partial}{\partial \dot{q}_r} \left(\frac{d}{dt} \frac{\omega^2}{2} \right)$$

$$\text{Now } F_r = - \frac{\partial \mathcal{F}}{\partial \dot{q}_r} = -c \frac{\partial}{\partial \dot{q}_r} \frac{\omega^2}{2} \Rightarrow \mathcal{F} = c \frac{\omega^2}{2} + \text{fn of time.}$$



10a



$$V^{D*} = (\dot{\phi} m_2 - \dot{\theta} m_1 + \dot{\psi} m_3) \times r m_2 = r \dot{\phi} \cdot 0 - r \dot{\theta} m_3 - r \dot{\psi} m_1$$

$$V^P = V^{D*} + \omega \times dl_1$$

$$V^P = V^{D*} - v dl_1 + \omega \times dl_1$$

$$V^{D*} = \dot{\phi} m_2 - \dot{\theta} m_1 + \dot{\psi} m_3; \quad \omega \times dl_1 = \dot{\phi} m_3 + \dot{\psi} m_2$$

$$\text{when } \theta = 0, \psi = 1/2: m_3 \times dl_1 = m_2; \quad m_1 \times dl_1 = 0; \quad m_2 \times dl_1 = -m_3 \quad dl_1 = m_1$$

$$V^P{}^2 = V^{D*}{}^2 - 2v V^{D*} \cdot dl_1 + r V^{D*} \cdot (\omega \times dl_1) + v^2 + 2v r \frac{dl_1}{2} \cdot (\omega \times dl_1) + \frac{r^2}{4} (\omega \times dl_1)^2$$

$$= r^2 \dot{\theta}^2 + r^2 \dot{\psi}^2 - 2v(-r \dot{\psi}) + r[+r \dot{\theta} \dot{\phi}] + v^2 + \frac{r^2}{4} [\dot{\phi}^2 + \dot{\psi}^2]$$

$$= r^2 \left[\dot{\theta}^2 + \frac{5}{4} \dot{\psi}^2 + \frac{\dot{\phi}^2}{4} + 2v \frac{\dot{\psi}}{r} + \dot{\theta} \dot{\phi} + \frac{v^2}{r^2} \right]$$

$$V^P{}^2 = \left[r^2 (\dot{\theta}^2 + \frac{5}{4} \dot{\psi}^2 + \frac{\dot{\phi}^2}{4} + \dot{\theta} \dot{\phi}) + v(v + 2r \dot{\psi}) \right]$$

$$K = \frac{1}{2} m V^P{}^2 = \frac{1}{2} m \left[r^2 (\dot{\theta}^2 + \frac{5}{4} \dot{\psi}^2 + \frac{\dot{\phi}^2}{4} + \dot{\theta} \dot{\phi}) + v(v + 2r \dot{\psi}) \right]$$

$$10b \quad K = K_v + K_w$$

$$K_v = \frac{1}{2} m v^2$$

$$K_w = \frac{1}{2} \omega \cdot \mathbb{I} \cdot \omega$$

$$\omega^{R/D} = \dot{\theta} m_c - \Omega m_c$$

$$v^2 = (R^2 - L^2) [\dot{\theta}^2 + \Omega^2 \sin^2 \theta]$$

$$\mathbb{I} = \frac{m L^2}{3} m_o m_o + \frac{m L^2}{3} m_c m_c \quad m_o = c m_c + s m_b$$

$$\mathbb{I} \cdot \omega = \frac{m L^2}{3} \left\{ -\Omega c^2 m_c - \Omega s c m_b + \dot{\theta} m_c \right\}$$

$$\frac{1}{2} \omega \cdot \mathbb{I} \cdot \omega = \frac{\Omega^2 c^2 m L^2}{6} + \frac{m L^2}{6} \dot{\theta}^2$$

$$K = K_v + K_w = \frac{1}{2} m \left[(R^2 - L^2) \left\{ \dot{\theta}^2 + \Omega^2 \sin^2 \theta \right\} + \Omega^2 \omega^2 \frac{L^2}{3} + \frac{L^2}{3} \dot{\theta}^2 \right]$$

$$F_{\theta}^* = \frac{\partial K}{\partial \theta} - \frac{d}{dt} \frac{\partial K}{\partial \dot{\theta}} = \frac{1}{2} m \left[(R^2 - L^2) \Omega^2 \cdot 2 \sin \theta \cos \theta + 2 \Omega^2 \cos \theta \sin \theta \frac{L^2}{3} \right] - \frac{1}{2} m \left[(R^2 - L^2) \cdot 2 \dot{\theta} + \frac{2 L^2}{3} \ddot{\theta} \right]$$

$$= -\frac{m}{3} \left[\Omega^2 \sin \theta \cos \theta (4L^2 - 3R^2) + (3R^2 - 2L^2) \ddot{\theta} \right]$$

Same result. Much easier tho.

10c

$$V^{T*} = \dot{q}_1 s_2 \frac{a}{3} m_a + \dot{q}_1 s_2 \frac{b}{3} m_b + \left(\frac{2b}{3} \dot{q}_1 c_2 + \dot{q}_2 \frac{a}{3} \right) m_a$$

$$\omega^T = \dot{q}_1 \left[m_b \cos \theta_2 - m_c \sin \theta_2 \right] + \dot{q}_2 m_a$$

$$\mathbb{I}^{T/A} = \frac{m a^2}{18} m_a m_a + \frac{m a b}{36} (m_a m_b + m_b m_a) + \frac{m b^2}{18} m_b m_b + \frac{m (a^2 + b^2)}{18} m_c m_c$$

$$p \cdot m_a = -\frac{2}{3} b m_a + \frac{1}{3} a m_b$$

$$p \cdot m_b = -\frac{2}{3} b m_b$$

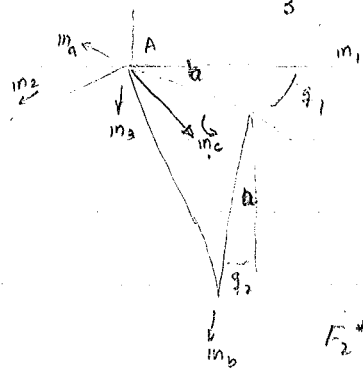
$$p \cdot m_b = \frac{1}{3} a m_a$$

$$p \cdot p = \frac{4}{9} b^2 + \frac{a^2}{9}$$

$$\mathbb{I}^{T/A} = \begin{bmatrix} \frac{m a^2}{18} + \frac{m a^2}{9} = \frac{m a^2}{6} & \frac{m a b}{4} & 0 \\ \frac{m a b}{4} & \frac{m b^2}{18} + \frac{8 m b^2}{18} = \frac{m b^2}{2} & 0 \\ 0 & 0 & \frac{m (a^2 + b^2)}{18} + \frac{m (b^2 + a^2)}{18} = \frac{m (a^2 + b^2)}{9} \end{bmatrix}$$

$$F_2^* = \frac{\partial K}{\partial q_2} - \frac{d}{dt} \frac{\partial K}{\partial \dot{q}_2}$$

$$\mathbb{I}_{lm}^{A/T*} = m \left[p^2 \delta_{lm} - (p \cdot m_l)(p \cdot m_m) \right]$$



$$\mathbf{I} \cdot \boldsymbol{\omega} = \frac{ma^2}{6} \dot{q}_2 m a + \frac{mab}{4} [\dot{q}_1 c_2 m a + \dot{q}_2 m b] + \frac{mb^2}{2} \dot{q}_1 c_2 m b - \dot{q}_1 s_2 m (3a^2 + \frac{9b^2}{18}) m c$$

$$\frac{1}{2} \boldsymbol{\omega} \cdot \mathbf{I} \cdot \boldsymbol{\omega} = \frac{ma^2}{12} \dot{q}_2^2 + \frac{mab}{8} [2\dot{q}_1 \dot{q}_2 c_2] + \frac{mb^2}{4} \dot{q}_1^2 c_2^2 + \dot{q}_1^2 s_2^2 m (3a^2 + \frac{9b^2}{36})$$

$$\frac{\partial K}{\partial \dot{q}_2} = -\frac{mab}{4} \dot{q}_1 \dot{q}_2 s_2 - \frac{9mb^2}{18} \dot{q}_1^2 c_2 s_2 + \dot{q}_1^2 m (3a^2 + \frac{9b^2}{18}) s_2 c_2 = \dot{q}_1^2 \frac{ma^2}{6} s_2 c_2$$

$$\frac{\partial K}{\partial \dot{q}_2} = \frac{ma^2}{6} \dot{q}_2 + \frac{mab}{4} \dot{q}_1 c_2; \quad -\frac{d}{dt} \left(\frac{\partial K}{\partial \dot{q}_2} \right) = -2 \frac{ma^2}{12} \ddot{q}_2 + \frac{mab}{4} \dot{q}_1 s_2 \dot{q}_2 - \frac{3mab}{12} \dot{q}_1 c_2$$

$$F_2^* = \frac{ma}{12} [2a(s_2 c_2 \dot{q}_2^2 - \ddot{q}_2) - 3b c_2 \dot{q}_1]$$

Method of Solution

10d. Find the velocity of the mass centers; $\mathbf{I} \cdot \boldsymbol{\omega} = I_3 \omega_3$ where I_3, ω_3 are components in direction

out of paper. Thus for each rod form $\sum \frac{1}{2} I_3 \omega_3^2 = K_\omega$ $I_3 = m l^2 / 12$ for rods length l .

square velocity & form $\sum \frac{1}{2} m_i v_i^2 = K_v$ now that $\frac{\partial K_T}{\partial \dot{q}_1} = \frac{d}{dt} \frac{\partial K_T}{\partial \dot{q}_1} = F_1^*$

where $K_T = K_\omega + K_v$

Better to use $\mathbf{F}^* \cdot \mathbf{v} \dot{q}_1 + \mathbf{T}^* \cdot \boldsymbol{\omega} \dot{q}_1 = F_1^*$

10e. (8d) is a non holonomic system.

(8e) Because of generalized speeds we have problem

10f. must form $KE_\omega + KE_v$ $KE_v = \frac{1}{2} m v_{sp}^{*2} \cdot 4 + \frac{1}{2} m v_c^{*2} > 0$

$$KE_\omega = \frac{1}{2} \boldsymbol{\omega}^c \cdot \mathbf{I}^c \cdot \boldsymbol{\omega}^c + \frac{4}{2} \boldsymbol{\omega}^s \cdot \mathbf{I}^s \cdot \boldsymbol{\omega}^s = \frac{1}{2} J \omega^2 +$$

$$\mathbf{I}^s = \frac{2}{5} m r^2 (m_1 m_3 + m_2 m_2 + m_3 m_3)$$

$$\mathbf{I}^c = J m_2 m_2 + \dots \quad \boldsymbol{\omega}^c = \omega m_2$$

$$v_{s \cdot m_3}^* = v^s = \frac{\omega r \sin \theta}{\cos \theta - \sin \theta} \quad \text{for prob (3d)} \quad \text{at } 30^\circ \quad \frac{\omega \cdot \frac{1}{2} r}{\frac{\sqrt{3}}{2} - \frac{1}{2}} = \frac{\omega r}{\sqrt{3} - 1}$$

$$\omega^s = \omega^c + \omega^s = \omega m_2 + \left(-\frac{r}{V} \omega - \omega \right) m_2 + \left(\frac{r}{V} \right) m_1 = (m_1 - m_2) \frac{r}{V}$$

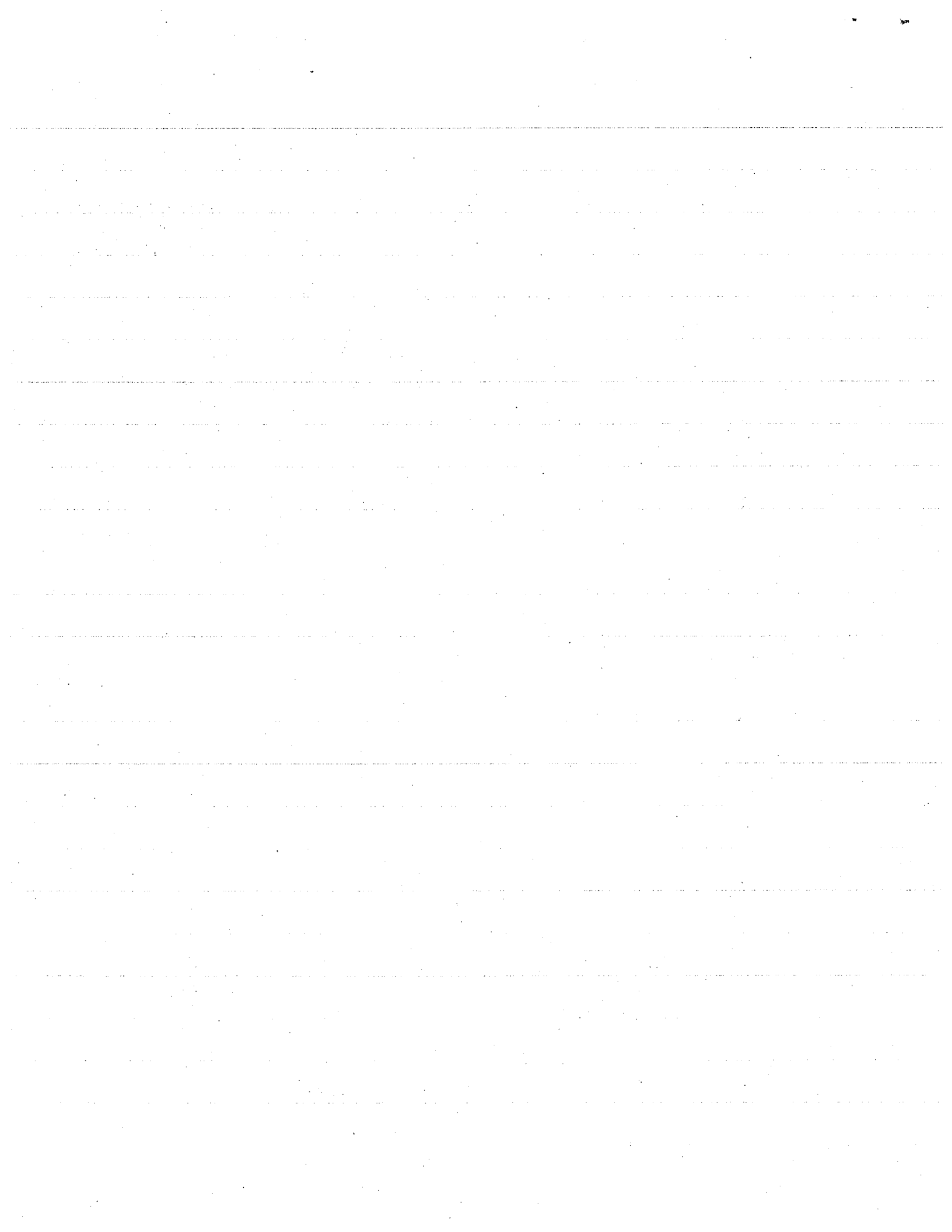
$$\omega^s = (m_1 - m_2) \frac{\omega}{\sqrt{3} - 1} \quad \omega^c = \omega m_2 \quad v^{*2} = \frac{\omega^2 r^2}{4 - 2\sqrt{3}}$$

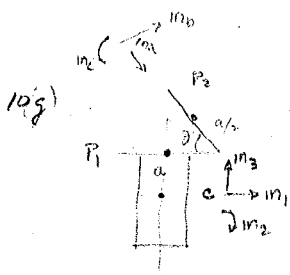
$$\therefore K = \frac{4}{2} m \frac{\omega^2 r^2}{4 - 2\sqrt{3}} + \frac{1}{2} J \omega^2 + \frac{4}{2} \frac{\omega^2}{4 - 2\sqrt{3}} \left(m_1 - m_2 \right) \cdot \left[\frac{2}{5} m r^2 (m_1 m_1 + m_2 m_2 + \dots) \right] (m_1 - m_2)$$

$$= \frac{4}{2} \frac{m \omega^2 r^2}{4 - 2\sqrt{3}} + \frac{1}{2} J \omega^2 + \frac{4}{2} \frac{\omega^2}{4 - 2\sqrt{3}} \cdot \frac{4}{5} m r^2 = \frac{2 m \omega^2 r^2}{4 - 2\sqrt{3}} + \frac{8}{5} \frac{\omega^2 m r^2}{4 - 2\sqrt{3}} + \frac{1}{2} J \omega^2$$

$$= \frac{m \omega^2 r^2}{2 - \sqrt{3}} + \frac{4}{5} \frac{m \omega^2 r^2}{2 - \sqrt{3}} + \frac{1}{2} J \omega^2$$

$$= \frac{9}{5} m \omega^2 r^2 (2 - \sqrt{3}) + \frac{1}{2} J \omega^2$$





I_{P_2}	m_a	m_b	m_c
m_a	$\frac{ma^2}{12}$	0	0
m_b	0	$\frac{ma^2}{12}$	0
m_c	0	0	$\frac{ma^2}{12}$

	m_a	m_b	m_c
m_1	c	3	0
m_2	0	0	0-1
m_3	-5	0	0

only need I_{33} .

$$I_{P_2} = \frac{ma^2}{12} (m_a m_a + 2m_b m_b + m_c m_c)$$

$$I_{P_2} = \frac{ma^2}{12} (s^2 m_3 m_3 + 0 + \frac{ma^2}{12} c^2 m_3 m_3 + \dots) \therefore \frac{1}{2} \frac{ma^2}{12} (s^2 + 2c^2) = \frac{1}{2} \omega^2 P_2 \cdot I_{P_2} \cdot \omega^2 P_2$$

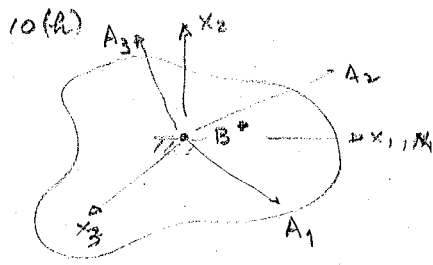
$$\therefore K_{TOT} = \frac{1}{2} \frac{ma^2}{12} \omega^2 + \frac{1}{2} m \omega^2 a^2 (1 - \cos \theta)^2 + \frac{1}{2} \omega^2 \frac{ma^2}{12} (s^2 + 2c^2) + \frac{1}{2} \omega^2 \frac{ma^2}{12}$$

$$\frac{\partial K}{\partial \theta} = 0 = 6(1 - \cos \theta)s + 2sc + 4cs$$

$$= 6s - 6cs + 2sc + 4cs = 6s - 8cs = 0 \therefore \frac{3}{4} = \cos \theta \quad \theta = \arccos \left(\frac{3}{4} \right)$$

$$\frac{\partial^2 K}{\partial \theta^2} = 6c - 8c^2 + 8s^2 = 6c - 8(c^2 - s^2) = 6c - 8(\cos 2\theta)$$

since $\theta < 90^\circ$ then $\cos 2\theta < 0 \therefore \frac{\partial^2 K}{\partial \theta^2} > 0 \therefore \min$



for the fixed point $\Rightarrow (mrs)_B = \omega^B_{f_r} \cdot I \cdot \omega^B_{f_s}$
of a rigid body

let I_1, I_2, I_3 be the value of the principal moments of inertia

$$\omega = \dot{q}_1 \hat{x}_1 + \dot{q}_2 \hat{x}_2 + \dot{q}_3 \hat{x}_3 = (\dot{q}_1 c_2 c_3 + s_2 \dot{q}_2) a_1 + (-\dot{q}_1 s_3 c_2 + c_2 \dot{q}_2) a_2 + (s_2 \dot{q}_1 + \dot{q}_3) a_3$$

$$\text{from the form of } \omega_{\dot{q}_1} = (c_2 c_3 a_1 - s_2 c_2 a_2 + s_2 a_3)$$

$$\omega_{\dot{q}_2} = (s_2 a_1 + c_2 a_2)$$

$$\omega_{\dot{q}_3} = a_3$$

$$\therefore m_{12} = c_2 c_3 s_2 I_1 - s_3 c_2 c_3 I_2$$

$$m_{13} = s_2 I_3$$

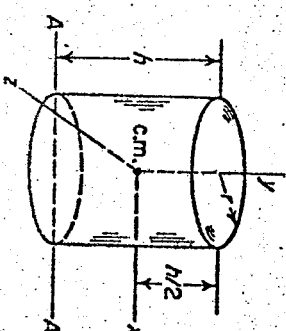
$$m_{23} = 0$$

$\therefore \dot{q}_1 \neq \dot{q}_2$ and $\dot{q}_1$ and $\dot{q}_3$ are coupled dynamically

Note that i, j, k, l are virtual work problems

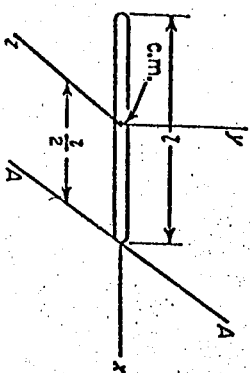
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APPENDIX A

INERTIAL PROPERTIES OF
HOMOGENEOUS BODIES

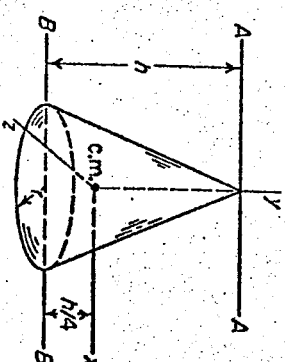
Right circular cylinder

$$\begin{aligned}
 V &= \pi r^2 h \\
 I_{xx} = I_{yy} &= \frac{m}{12} (3r^2 + h^2) \\
 I_{zz} &= \frac{1}{2} mr^2 \\
 I_{xx} &= \frac{m}{12} (3r^2 + 4h^2)
 \end{aligned}$$



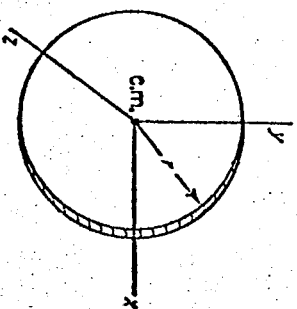
Thin rod

$$\begin{aligned}
 I_{xx} &= 0 \\
 I_{yy} = I_{zz} &= \frac{ml^2}{12} \\
 I_{xx} &= \frac{ml^2}{3}
 \end{aligned}$$



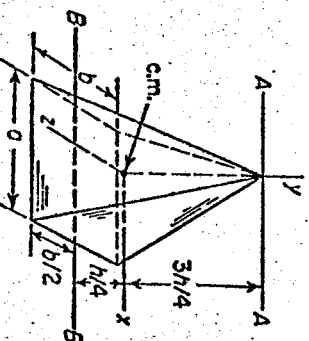
Right circular cone

$$\begin{aligned}
 V &= \frac{1}{3} \pi r^2 h \\
 I_{xx} = I_{yy} &= \frac{3m}{80} (4r^2 + h^2) \\
 I_{zz} &= \frac{3}{10} mr^2 \\
 I_{xx} &= \frac{3m}{20} (r^2 + 4h^2) \\
 I_{yy} &= \frac{m}{20} (3r^2 + 2h^2)
 \end{aligned}$$



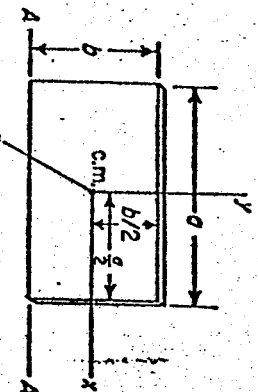
Thin circular disk

$$\begin{aligned}
 A &= \pi r^2 \\
 I_{xx} = I_{yy} &= \frac{mr^2}{4} \\
 I_{zz} &= \frac{mr^2}{2}
 \end{aligned}$$

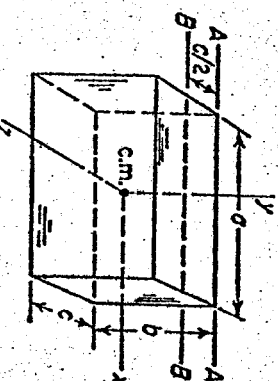


Right rectangular pyramid

$$\begin{aligned}
 V &= \frac{ab}{3} \\
 I_{xx} &= \frac{m}{80} (4b^2 + 3h^2) \\
 I_{yy} &= \frac{m}{20} (a^2 + b^2) \\
 I_{zz} &= \frac{m}{80} (4a^2 + 3h^2) \\
 I_{xx} &= \frac{m}{20} (b^2 + 12h^2) \\
 I_{yy} &= \frac{m}{20} (b^2 + 2h^2)
 \end{aligned}$$



$$\begin{aligned}
 A &= ab \\
 I_{xx} &= \frac{mb^3}{12} \\
 I_{yy} &= \frac{ma^3}{12} \\
 I_{zz} &= \frac{m}{12} (a^2 + b^2)
 \end{aligned}$$

See pg 108
for more

$$\begin{aligned}
 V &= abc \\
 I_{xx} &= \frac{m}{12} (b^2 + c^2) \\
 I_{yy} &= \frac{m}{12} (a^2 + c^2) \\
 I_{zz} &= \frac{m}{12} (a^2 + b^2) \\
 I_{xx} &= \frac{m}{3} (b^2 + c^2)
 \end{aligned}$$

 $I_{xx} = \frac{m}{12} (a^2 + b^2)$

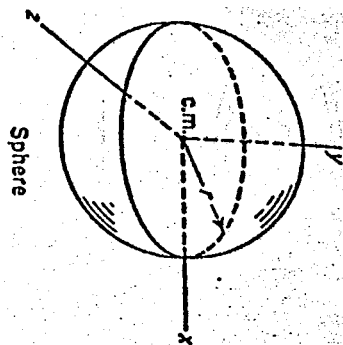
107

$$x = \frac{3}{2} \text{ cm}$$

$$y = \frac{3}{2} \text{ cm}$$

$$z = \frac{3}{2} \text{ cm}$$

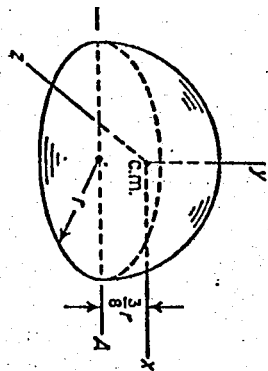
NOTE: The above values are for the case of a cube.



Sphere

$$V = \frac{4}{3} \pi r^3$$

$$I_{xx} = I_{yy} = I_{zz} = \frac{2}{5} m r^2$$



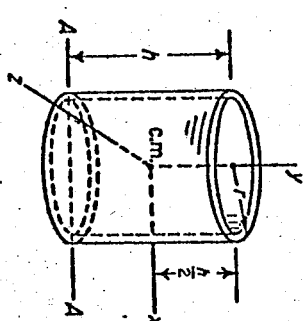
Hemisphere

$$V = \frac{2}{3} \pi r^3$$

$$I_{xx} = I_{zz} = \frac{83}{320} m r^2$$

$$I_{yy} = \frac{2}{5} m r^2$$

$$I_{xx} = \frac{2}{5} m r^2$$



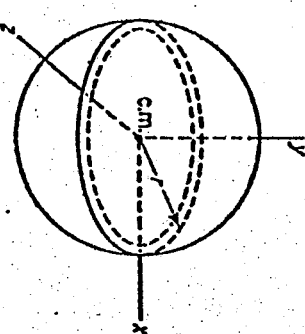
Thin circular cylindrical shell

$$A = 2 \pi r h$$

$$I_{xx} = I_{zz} = \frac{m}{12} (6r^2 + h^2)$$

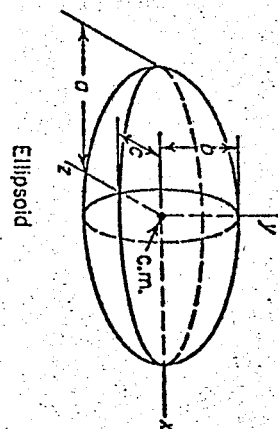
$$I_{yy} = m r^2$$

$$I_{xx} = \frac{m}{6} (3r^2 + 2h^2)$$



$$A = 4 \pi r^2$$

$$I_{xx} = I_{yy} = I_{zz} = \frac{2}{5} m r^2$$



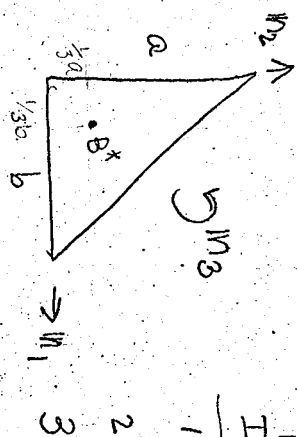
Ellipsoid

$$V = \frac{4}{3} \pi a b c$$

$$I_{xx} = \frac{m}{5} (b^2 + c^2)$$

$$I_{yy} = \frac{m}{5} (a^2 + c^2)$$

$$I_{zz} = \frac{m}{5} (a^2 + b^2)$$



I_{ij}	1	2	3
I_{11}	$\frac{m a^2}{18}$	$\frac{m a b}{36}$	0
I_{22}	$\frac{m a b}{36}$	$\frac{m b^2}{18}$	0
I_{33}	0	0	$\frac{m(a^2 + b^2)}{18}$



$$I_{xx} = \frac{m}{4} (a^2 + b^2)$$

$$I_{yy} = \frac{m}{4} (a^2 + b^2)$$

$$I_{zz} = \frac{m}{2} (a^2 + b^2)$$

