

A hand-drawn diagram showing a circle. A point labeled 'M' is on the left side of the circle. A point labeled 'M'' is at the top of the circle. A point labeled 'A' is on the right side of the circle. A line segment connects M and A, passing through M'.

2 R/a- #1
Accepted

then $5^r = x m_1 + y m_2$

$$R^D = \varphi \ln_3 = \varphi a_3$$

$${}^R W^T = {}^R W^P + c \frac{R}{da_2} = \dot{x}_{m_1} + \dot{y}_{m_2} + c \frac{R}{da_2} = \dot{x}_{m_1} + \dot{y}_{m_2} + c [{}^R B^D \times a_2]$$

then $\|w_k^1\| = \|r^p + c a_2\|$

$$\frac{d}{dt} \left(\frac{1}{2} m \dot{q}^2 \right) = \frac{\partial L}{\partial q} + \frac{\partial L}{\partial \dot{q}} \dot{q} + \frac{\partial L}{\partial t}$$

$$= x \ln 1 + y \ln 2 - c \ln a_1$$

the radius vector from P to Q .

$$IV' = x m_1 + y m_2 + \varphi a a_2$$

W_1	$-S_1$	W_2	$-S_2$	W_3	$-S_3$
W_1	W_2	W_3	W_4	W_5	W_6

on W_1 in contact with R.

then $11^{w_1}_0 = 11^{w_1}_* - 11^{a_3}$ and let

$$R_{W_1} = R_D + (R_D)_{W_1} = 4000 + 5200 = 9200$$

$$= x m_1 + y m_2 - c \varphi a_1 + [c \varphi a_1 - c \varphi a_2 + \varphi a_3 + \varphi a_2] x (-r a_3)$$

$$r^2 = 1 - \rho^2$$

[Handwritten marks and scribbles]

percent of W_2 in contact with R_1 than

$$R_{W_2} = R_{W_2} - R_{W_2} + R_{W_2} = R_{W_2} + R_{W_2} = R_{W_2}$$

R_{W_2} & R_{W_1} is the replacement of a by $-a$. Then

$$(\psi^2 \zeta^2 x - \psi \phi \psi + \psi m \lambda + \psi m \lambda) = (\psi^2 \zeta^2 x - \psi \phi \psi) + (\psi m \lambda + \psi m \lambda) = \psi^2 \zeta^2 x - \psi \phi \psi + \psi m \lambda + \psi m \lambda$$

$${}^R W_2^0 = 0 = [-\dot{x} s_\varphi + \dot{y} c_\varphi + c\dot{\varphi} - r\dot{\Omega}_3] a_1 + [-\dot{x} c_\varphi - \dot{y} s_\varphi] a_2$$

thus the constraints are $[-\dot{x} s_\varphi + \dot{y} c_\varphi + c\dot{\varphi} - r\dot{\Omega}_3 = 0]$ and $[-\dot{x} c_\varphi - \dot{y} s_\varphi = 0]$ again.

$$\text{now } \begin{bmatrix} A & B \\ \omega & \end{bmatrix} = \dot{\theta} a_3 \quad {}^R W_3^* = {}^R V + \omega \times (b |b_1) = {}^R V + \omega \times r r^{W_3^*/\omega}$$

$${}^R \omega = {}^R \omega^A + \omega^B = [\dot{\varphi} a_3 + \dot{\theta} a_3 = {}^R \omega^B] \quad |b_1 = c_\theta a_1 + s_\theta a_2$$

$$\begin{aligned} \therefore {}^R W_3^* &= {}^R V + b[\dot{\varphi} + \dot{\theta}] a_3 \times (c_\theta a_1 + s_\theta a_2) \\ &= {}^R V + b[\dot{\varphi} + \dot{\theta}] c_\theta a_2 - b[\dot{\varphi} + \dot{\theta}] s_\theta a_1 \\ &= [-\dot{x} s_\varphi + \dot{y} c_\varphi - b(\dot{\varphi} + \dot{\theta}) s_\theta] a_1 + [\dot{\varphi} a - \dot{x} c_\varphi - \dot{y} s_\varphi + b(\dot{\varphi} + \dot{\theta}) c_\theta] a_2 \end{aligned}$$

$$\text{now } {}^B \omega W_3 = \dot{\Omega}_3 |b_2 = \dot{\Omega}_3 [c_\theta a_2 - s_\theta a_1] \quad \text{since } \text{all wheels have same}$$

magnitude of angular velocity

$$\begin{aligned} 0 &= {}^R W_3^0 = {}^R W_3^* + \omega W_3 \times (-r a_3) \quad \text{since } r^{W_3^0/W_3^*} = -r a_3 \\ &= {}^R W_3^* + [\omega^B + \omega W_3] \times (-r a_3) \quad |b_1 \\ &= {}^R W_3^* + [(\dot{\varphi} + \dot{\theta}) a_3 + \dot{\Omega}_3 [c_\theta a_2 - s_\theta a_1]] \times (-r a_3) \\ &= {}^R W_3^* + [-\dot{\Omega}_3 c_\theta r a_1 - \dot{\Omega}_3 s_\theta r a_2] \end{aligned}$$

$$\text{thus } 0 = \{-\dot{x} s_\varphi + \dot{y} c_\varphi - b(\dot{\varphi} + \dot{\theta}) s_\theta - \dot{\Omega}_3 c_\theta r\} a_1 + [\dot{\varphi} a - \dot{x} c_\varphi - \dot{y} s_\varphi + b(\dot{\varphi} + \dot{\theta}) c_\theta - \dot{\Omega}_3 s_\theta r] a_2$$

thus our constraint eqns are

$$\left\{ \begin{aligned} &[-\dot{x} s_\varphi + \dot{y} c_\varphi - b(\dot{\varphi} + \dot{\theta}) s_\theta - \dot{\Omega}_3 c_\theta r] = 0 \quad + \quad [\dot{\varphi} a - \dot{x} c_\varphi - \dot{y} s_\varphi + b(\dot{\varphi} + \dot{\theta}) c_\theta - \dot{\Omega}_3 s_\theta r] = 0 \end{aligned} \right.$$

Collecting our constraint eqns.

$$-\dot{x} s_\varphi + \dot{y} c_\varphi - c\dot{\varphi} - r\dot{\Omega}_3 = 0 \quad (1)$$

$$-\dot{x} c_\varphi - \dot{y} s_\varphi = 0 \quad (2)$$

$$-\dot{x} s_\varphi + \dot{y} c_\varphi + c\dot{\varphi} - r\dot{\Omega}_3 = 0 \quad (3)$$

$$-\dot{x} s_\varphi + \dot{y} c_\varphi - b(\dot{\varphi} + \dot{\theta}) s_\theta - \dot{\Omega}_3 c_\theta r = 0 \quad (4) \quad \text{same}$$

$$\dot{\varphi} a - \dot{x} c_\varphi - \dot{y} s_\varphi + b(\dot{\varphi} + \dot{\theta}) c_\theta - \dot{\Omega}_3 s_\theta r = 0 \quad (5)$$

this is 5 eqs in 7 unknowns $(\dot{x}, \dot{y}, \dot{\varphi}, \dot{\Omega}_1, \dot{\Omega}_2, \dot{\theta}, \dot{\Omega}_3)$

$$\text{now } {}^R V = (-\dot{x} s_\varphi + \dot{y} c_\varphi) a_1 + (\dot{\varphi} a - \dot{x} c_\varphi - \dot{y} s_\varphi) a_2$$

subtract (3) from (1) then $-2c\dot{\varphi} - r\dot{\Omega}_1 + r\dot{\Omega}_2 = 0$
 $\therefore \dot{\varphi} = r(\dot{\Omega}_2 - \dot{\Omega}_1)/2c \quad (6)$

sub (6) into (1) to get

$$-x\dot{\Omega}_1 + y\dot{\Omega}_2 = r(\dot{\Omega}_2 - \dot{\Omega}_1)/2 + 2r\frac{\dot{\Omega}_1}{2} = r(\dot{\Omega}_2 + \dot{\Omega}_1)/2 \quad (7)$$

using this and (2) in ${}^R W^A$

$${}^R W^A = \frac{r}{2}(\dot{\Omega}_2 + \dot{\Omega}_1)\hat{a}_1 + \dot{\varphi}\hat{a}_2 = \frac{r}{2}(\dot{\Omega}_2 + \dot{\Omega}_1)\hat{a}_1 + \frac{ar}{2c}(\dot{\Omega}_2 - \dot{\Omega}_1)\hat{a}_2 \quad (8)$$

$$\Rightarrow \left| u_1 = \frac{r}{2}(\dot{\Omega}_2 + \dot{\Omega}_1) \right| (9) \quad \left| u_2 = \frac{ar}{2c}(\dot{\Omega}_2 - \dot{\Omega}_1) \right| (10)$$

Now ${}^R \omega^A = \dot{\varphi}\hat{a}_3 = r(\dot{\Omega}_2 - \dot{\Omega}_1)/2c\hat{a}_3 = \frac{u_2}{a}\hat{a}_3$
 $\therefore \left| {}^R \omega_{u_1}^A = 0 \quad {}^R \omega_{u_2}^A = \hat{a}_3/a \right|$

$${}^R \omega^B = {}^R \omega^A + {}^A \omega^B = \frac{u_2}{a}\hat{a}_3 + \dot{\theta}\hat{a}_3$$

when $\theta = 180^\circ$ (4) becomes $-x\dot{\Omega}_1 + y\dot{\Omega}_2 = -\dot{\Omega}_3 r = r(\dot{\Omega}_2 + \dot{\Omega}_1)/2$
 $\therefore \dot{\Omega}_3 = -(\dot{\Omega}_2 + \dot{\Omega}_1)/2 = -u_1/r \quad (11)$

when $\theta = 180^\circ$ (5) becomes (using (2))

$$\dot{\varphi}a + b(\dot{\varphi} + \dot{\theta}) = 0 \quad \therefore \dot{\theta} = \dot{\varphi}\frac{(a-b)}{b} = \frac{r(\dot{\Omega}_2 - \dot{\Omega}_1)}{2c}\frac{(a-b)}{b}$$

$$\therefore \left| \dot{\theta} = \frac{u_2}{a}\frac{(a-b)}{b} \right|$$

$${}^R \omega^B = \frac{u_2}{a}\hat{a}_3 + \frac{u_2}{a}\frac{(a-b)}{b}\hat{a}_3 = u_2\hat{a}_3 \left[\frac{b}{ab} + \frac{a-b}{ab} \right] = \frac{u_2}{b}\hat{a}_3$$

$$\left| {}^R \omega_{u_1}^B = 0 \quad {}^R \omega_{u_2}^B = \frac{\hat{a}_3}{b} \right|$$

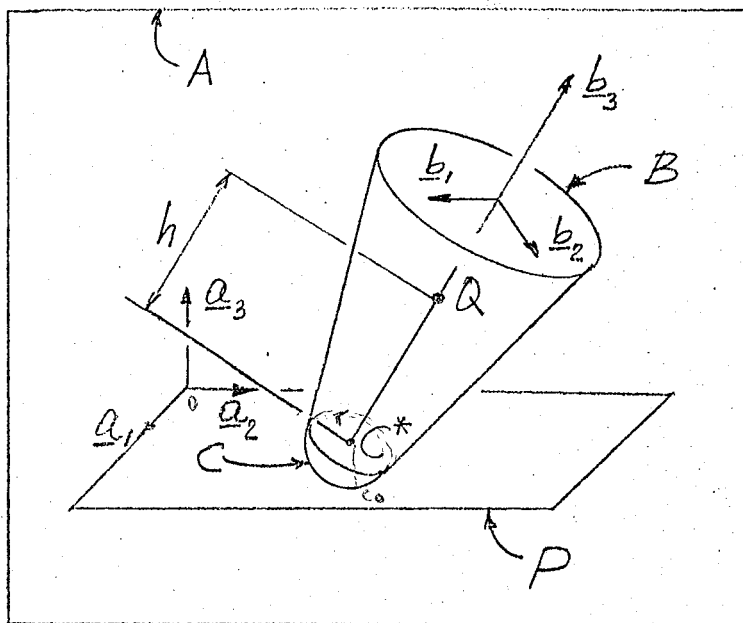
$${}^R \omega^W = {}^R \omega^D + {}^D \omega^W = {}^R \omega^A + {}^B \omega^W = \frac{u_2}{a}\hat{a}_3 + \dot{\Omega}_1\hat{a}_2$$

$$\left. \begin{aligned} u_1 \frac{a}{c} &= \frac{ra}{2c}(\dot{\Omega}_2 + \dot{\Omega}_1) \\ u_2 &= \frac{ra}{2c}(\dot{\Omega}_2 - \dot{\Omega}_1) \end{aligned} \right\} \quad u_1 \frac{a}{c} - u_2 = \frac{ra}{2c} \cdot 2\dot{\Omega}_1 = \frac{ra}{c}\dot{\Omega}_1$$

$$\therefore \dot{\Omega}_1 = \frac{u_1 a - u_2 c}{c} \cdot \frac{c}{ra} = \frac{u_1 a - u_2 c}{ra}$$

$${}^R \omega^W = \frac{u_2}{a}\hat{a}_3 + \frac{u_1 a - u_2 c}{ra}\hat{a}_2$$

$$\left| {}^R \omega_{u_1}^W = \frac{1}{r}\hat{a}_2 \quad {}^R \omega_{u_2}^W = \frac{\hat{a}_3}{a} - \frac{c}{ra}\hat{a}_2 \right|$$



A conical top B has a hemispherical base C that remains in contact with a plane P . (In general, C slips on P .) P is fixed in a reference frame A , and $\underline{a}_1, \underline{a}_2, \underline{a}_3$ are mutually perpendicular unit vectors fixed in A , with $\underline{a}_3$ normal to P and $\underline{a}_3 = \underline{a}_1 \times \underline{a}_2$. $\underline{b}_1, \underline{b}_2, \underline{b}_3$ are mutually perpendicular unit vectors fixed in B , with $\underline{b}_3 = \underline{b}_1 \times \underline{b}_2$ and parallel to the axis of B . The distance from the center C^* of C to a certain point Q that is fixed on the axis of B is equal to h , and C has a radius r .

To bring B into a general orientation in A , one can align $\underline{b}_1$ with $\underline{a}_1$ ($i=1,2,3$) and then subject B successively to right-handed rotations of amounts q_1, q_2 , and q_3 radians about lines parallel to $\underline{b}_1, \underline{b}_2$, and $\underline{b}_1$ (note the last subscript), respectively.

Defining generalized speeds $u_1, \dots, u_5$ as $u_i \triangleq \underline{\omega} \cdot \underline{b}_i$ ($i=1,2,3$), $u_4 \triangleq \underline{v} \cdot \underline{a}_1$, $u_5 \triangleq \underline{v} \cdot \underline{a}_2$, where $\underline{\omega}$ is the angular velocity of B in A and $\underline{v}$ is the velocity in A of that point of C that is in contact with P , determine the partial velocity of Q in A with respect to u_1 . Express the results in terms of $r, h, \underline{b}_1, \underline{b}_2, \underline{b}_3$, and trigonometric functions of q_1, q_2, q_3 , using the notations $s_i \triangleq \sin q_i$, $c_i \triangleq \cos q_i$ ($i=1,2,3$).

Figure 1 is a schematic representation of a spacecraft A that carries a boom B . B is attached to A by means of a screw S , so that B can rotate and translate relative to A simultaneously. S has a pitch p (i.e., when B performs one revolution relative to A , B advances a distance p); and the axes of B and S are perpendicular to each other.

At a certain time t , the velocity of point A^* , the acceleration of point A^* , the angular velocity of A , and the angular acceleration of A , all in a certain reference frame N , are equal to $3\mathbf{a}_1 \text{ ms}^{-1}$, 0 , $4\mathbf{a}_2 \text{ s}^{-1}$, and $5\mathbf{a}_3 \text{ s}^{-2}$, where $\mathbf{a}_1$, $\mathbf{a}_2$, $\mathbf{a}_3$ are mutually perpendicular unit vectors directed as shown; and the axis of B is parallel to $\mathbf{a}_1$, as indicated in Fig. 2. Assuming that the angular velocity of B in A is at all times equal to $6\mathbf{a}_3 \text{ s}^{-1}$, determine the velocity and the acceleration of P in N at time t , P being the tip of B . (B has a length of 7m , and the distance from A^* to B^* , the base of B , is equal to 8m at time t .)

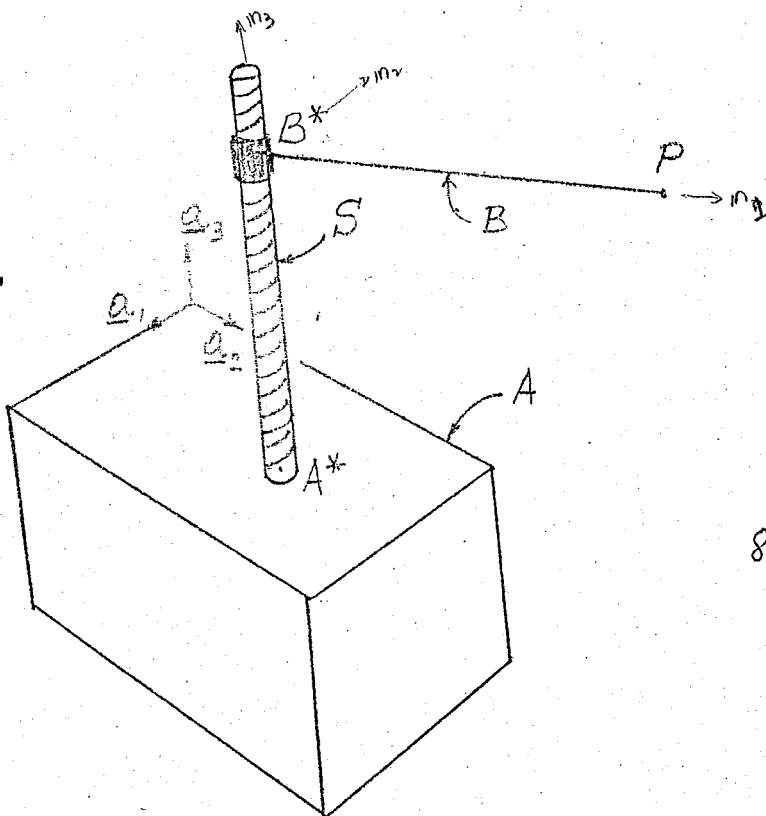


Fig. 1

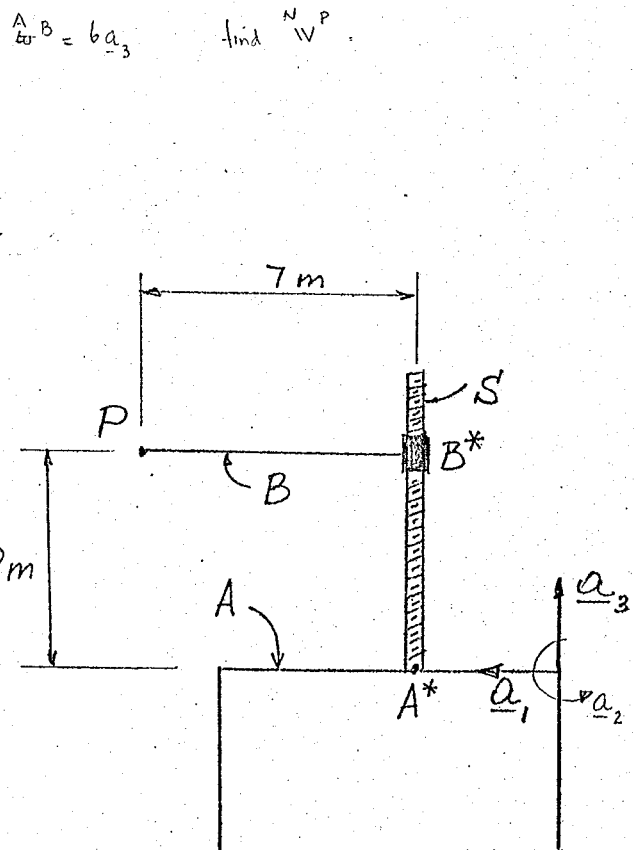
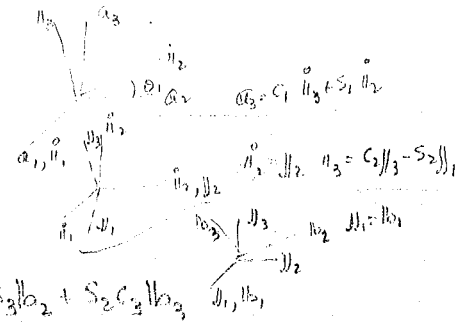
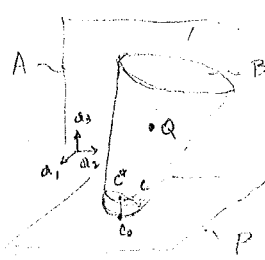


Fig. 2



Since $u_i = \omega^B \cdot b_i$ $\omega^B = u_i b_i$

$$v^{Q_0} = u_4 \bar{a}_1 + u_5 \bar{a}_2$$

$$v^Q = v^{Q_0} + \omega^B \times r^{Q/Q_0}$$

$$v^Q_{u_4} = \frac{v^{Q_0}}{u_4} + \frac{\omega^B}{u_4} \times r^{Q/Q_0}$$

Since $v^{Q_0} = f(u_4)$

$$= b_1 \times [h b_3 + r \{ -c_4 s_2 b_1 + (s_1 c_3 + c_2^e s_3) b_2 + (c_2^e s_3 - s_1 s_3) b_3 \}]$$

$$= -h b_2 + r (s_1 c_3 + c_2^e s_3) b_3 - r (c_2^e s_3 - s_1 s_3) b_2$$

$$\bar{a}_1 = c_2 b_1 + s_2 s_3 b_2 + s_2 c_3 b_3$$

$$\bar{a}_2 = s_2 s_1 b_1 + (c_1 c_3 - s_1 c_2 s_3) b_2 - (c_1 s_3 + s_1 c_2 c_3) b_3$$

$$r^{Q/Q_0} = h b_3 + r \bar{a}_3$$

$$\bar{a}_3 = -c_4 s_2 b_1 + (s_1 c_3 + c_2^e s_3) b_2 + (c_2^e s_3 - s_1 s_3) b_3$$

now $\omega^B = \dot{q}_1 \bar{a}_1 + \dot{q}_2 \bar{a}_2 + \dot{q}_3 \bar{a}_3$

	$\bar{a}_1$	$\bar{a}_2$	$\bar{a}_3$	b_1	b_2	b_3	b_1	b_2	b_3
$\bar{a}_1$	1	0	0	b_1	c_2	$-s_2$	b_1	1	0
$\bar{a}_2$	0	c_1	s_1	b_2	0	1	b_2	0	c_3
$\bar{a}_3$	0	$-s_1$	c_1	b_3	s_2	0	b_3	0	$-s_3$

$$v^{Q_0} = \dot{x} \bar{a}_1 + \dot{y} \bar{a}_2 - \dot{r} \bar{a}_3$$

$$v^{Q_0} = \frac{d}{dt}(-r \bar{a}_3) = -\dot{r} \bar{a}_3 - r (\omega^B \times \bar{a}_3) = -\dot{r} \bar{a}_3 - r (\omega^B \times \bar{a}_3) = -\dot{r} \bar{a}_3 + r (\omega^B \times \bar{a}_3)$$

$$v^{Q_0} = \dot{x} \bar{a}_1 + \dot{y} \bar{a}_2 \quad r \bar{a}_3 = x \bar{a}_1 + y \bar{a}_2$$

$$\omega^B = (\dot{q}_1 c_2 + \dot{q}_3) b_1 + (\dot{q}_1 s_2 s_3 + \dot{q}_2 c_3) b_2 + (\dot{q}_1 c_3 s_2 - \dot{q}_2 s_3) b_3 = u_4 b_1$$

$$u_1 = \dot{q}_1 c_2 + \dot{q}_3 \quad u_2 = \dot{q}_1 s_2 s_3 + \dot{q}_2 c_3 \quad u_3 = \dot{q}_1 s_2 c_3 - \dot{q}_2 s_3$$

$$v^{Q_0} = \dot{x} \bar{a}_1 + \dot{y} \bar{a}_2 - \dot{r} \bar{a}_3 + r (\omega^B \times \bar{a}_3)$$

$$v^{Q_0} \cdot \bar{a}_1 = \dot{x} + r (\omega^B \times \bar{a}_3 \cdot \bar{a}_1) = \dot{x} + r (\dot{q}_2 c_1 + s_2 s_1 \dot{q}_3) = u_4$$

$$\begin{vmatrix} \omega_1 & \omega_2 & \omega_3 \\ 0 & 0 & 1 \\ 1 & 0 & 0 \end{vmatrix} = \omega_2$$

$$\omega^B = \dot{q}_1 \bar{a}_1 + \dot{q}_2 (c_1 \bar{a}_2 + s_1 \bar{a}_3) + \dot{q}_3 (c_2 \bar{a}_1 - s_2 [-s_1 \bar{a}_2 + c_1 \bar{a}_3])$$

$$= (\dot{q}_1 + \dot{q}_3 c_2) \bar{a}_1 + (\dot{q}_2 c_1 + s_2 s_1 \dot{q}_3) \bar{a}_2 + (\dot{q}_2 s_1 - \dot{q}_3 s_2 c_1) \bar{a}_3$$

$$v^{Q_0} \cdot \bar{a}_2 = \dot{y} + r (\omega^B \times \bar{a}_3 \cdot \bar{a}_2) = \dot{y} + r (\dot{q}_1 + \dot{q}_3 c_2) = u_5$$

$$\begin{vmatrix} \omega_1 & \omega_2 & \omega_3 \\ 0 & 0 & 1 \\ 0 & 1 & 0 \end{vmatrix} = -\omega_1$$

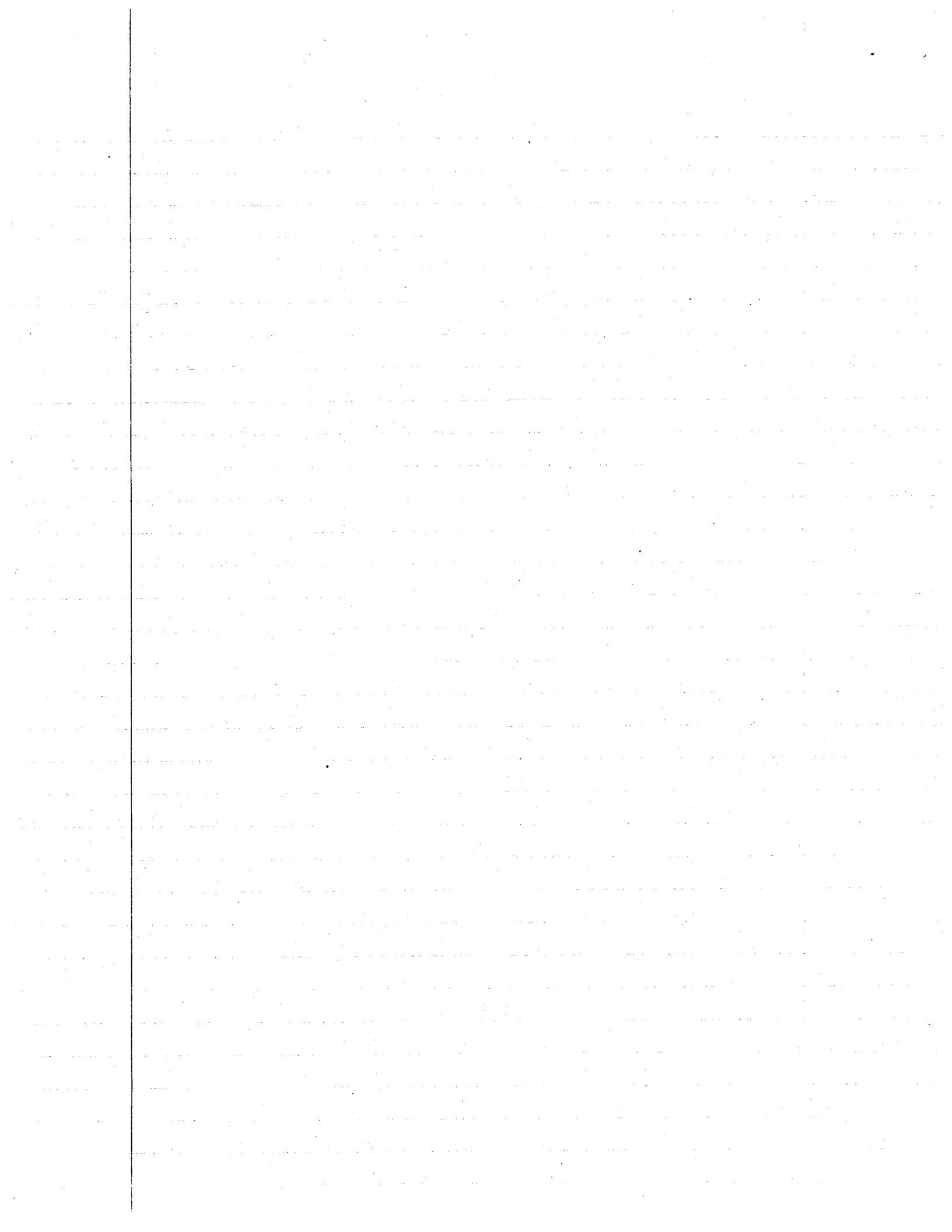
$$v^Q = v^{Q_0} + \omega^B \times r^{Q/Q_0} = \dot{x} \bar{a}_1 + \dot{y} \bar{a}_2 - \dot{r} \bar{a}_3 + r (\omega^B \times \bar{a}_3) + \omega^B \times h b_3$$

$$\bar{a}_1 = c_2 b_1 + s_2 s_3 b_2 + s_2 c_3 b_3$$

$$\bar{a}_2 = s_1 s_2 b_1 + (c_1 c_3 - s_1 s_3 c_2) b_2 + (-c_1 s_3 - s_1 c_2 c_3) b_3$$

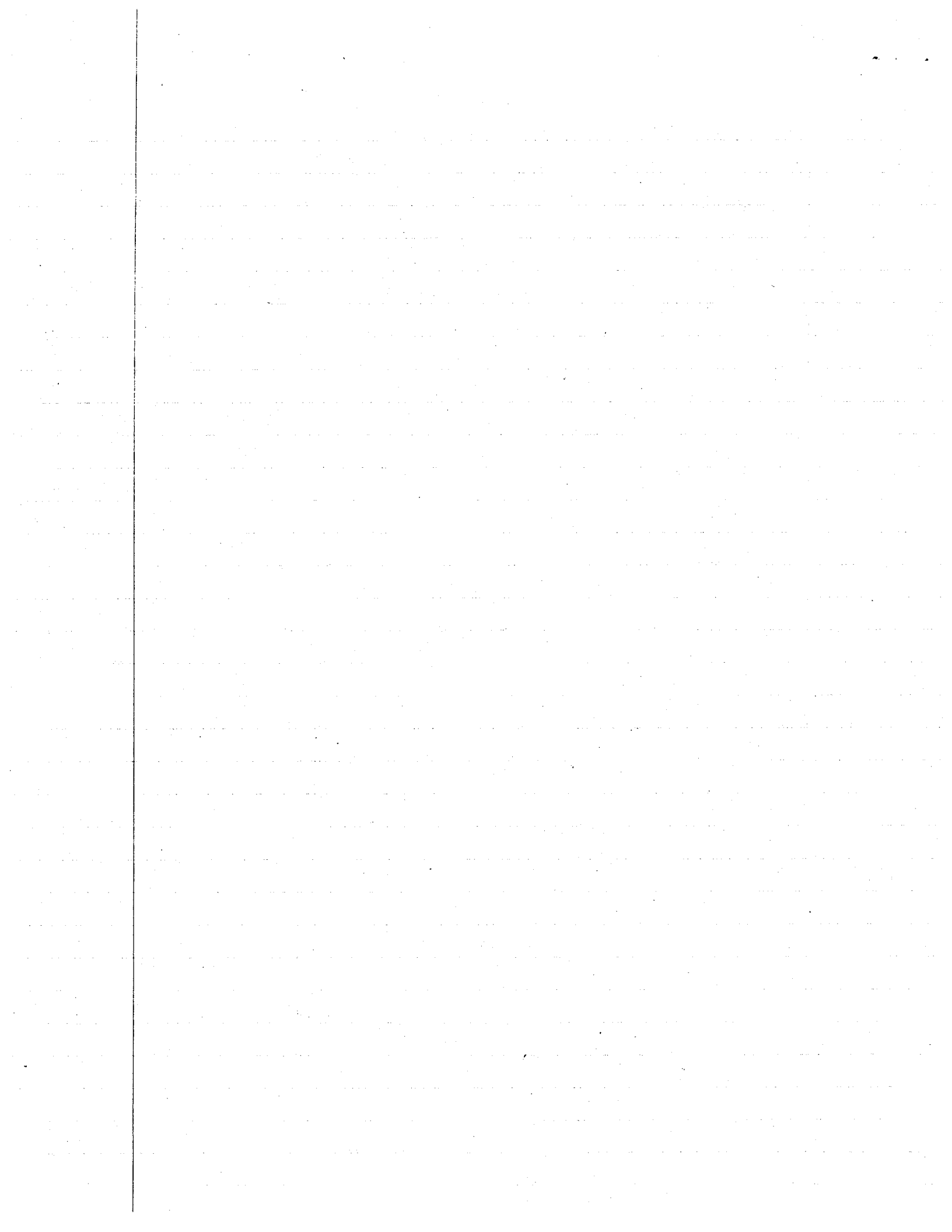
$$\bar{a}_3 = -c_4 s_2 b_1 + (s_1 c_3 + c_2^e s_3) b_2 + (c_2^e s_3 - s_1 s_3) b_3$$

$$v^Q = \{ \dot{x} c_2 + \dot{y} s_1 s_2 - \dot{r} (-c_4 s_2) + r [(\dot{q}_1 + \dot{q}_3 c_2)(c_2^e s_3 - s_1 s_3) - (\dot{q}_1 c_3 s_2 - \dot{q}_2 s_3)(s_1 c_3 + c_2^e s_3)] \} b_1 +$$



$$\left\{ \dot{x} s_2 s_3 + \dot{y} (c_1 c_3 - s_1 s_3 c_2) - \dot{r} (s_1 c_3 + c_2^e s_3) + r \left[-(\dot{q}_1^e c_2 + \dot{q}_3)(c_2^e s_3 - s_1 s_3) \right. \right. \\ \left. \left. + (\dot{q}_1^e c_3 s_2 - \dot{q}_3 s_3)(c_2 s_2) \right] + h \left[-(\dot{q}_1^e c_2 + \dot{q}_3) \right] \right\} l_{b_2} + \\ \left\{ \dot{x} s_2 c_3 + \dot{y} (-c_1 s_3 - s_1 c_2 c_3) - \dot{r} (c_2^e s_3 - s_1 s_3) + r \left[(\dot{q}_1^e c_2 + \dot{q}_3)(s_1 c_3 + c_2^e s_3) \right. \right. \\ \left. \left. + (\dot{q}_1^e s_2 s_3 + \dot{q}_3 c_3) c_2 s_2 \right] \right\} l_{b_3}$$

$${}^A V_{u_1}^{(R)} = + r^2 (c_2^e s_2 - s_1 s_3) l_{b_1} - r (c_2^e s_3 - s_1 s_3) l_{b_2} - h l_{b_2} + r (s_1 c_3 + c_2^e s_3) l_{b_3}$$



1. Given ${}^N\omega^A = 4\mathbf{a}_2$ ${}^N\alpha^A = 5\mathbf{a}_3$ ${}^N\mathbf{v}^{A*} = 3\mathbf{a}_1$ ${}^N\mathbf{a}^{A*} = 0$

$m_1 = \mathbf{a}_1$ ${}^A\omega^B = 6\mathbf{a}_3$ at time t

find ${}^N\mathbf{v}^P$ and ${}^N\mathbf{a}^P$ P is moving in rigid body A

$${}^N\mathbf{v}^{B*} = {}^N\mathbf{v}^{\bar{A}} + {}^A\mathbf{v}^{B*}$$

$${}^N\mathbf{v}^{\bar{A}} = {}^N\mathbf{v}^{A*} + {}^N\omega^A \times \mathbf{r}^{\bar{A}/A*}$$

$${}^N\mathbf{v}^{\bar{A}} = 3\mathbf{a}_1 + 4\mathbf{a}_2 \times (8\mathbf{a}_3) = 35\mathbf{a}_1$$

if ${}^A\omega^B = \dot{\theta}\mathbf{a}_3 = 6\mathbf{a}_3$ $\dot{\theta} = 6$

$$\mathbf{r}^{B*} = \frac{\theta p}{2\pi}\mathbf{a}_3$$

$${}^A\mathbf{v}^{B*} = \frac{\dot{\theta} p}{2\pi}\mathbf{a}_3 = \frac{6p}{2\pi}\mathbf{a}_3$$

$${}^N\mathbf{v}^{B*} = 35\mathbf{a}_1 + \frac{6p}{2\pi}\mathbf{a}_3$$

$${}^N\mathbf{v}^P = {}^N\mathbf{v}^{B*} + {}^N\omega^B \times \mathbf{r}^{P/B*} = 35\mathbf{a}_1 + \frac{6p}{2\pi}\mathbf{a}_3 + \omega^B \times 7\mathbf{a}_1$$

$$\boxed{\mathbf{r}^{P/B*} = 7\mathbf{a}_1}$$

$$\boxed{{}^N\omega^B = {}^N\omega^A + {}^A\omega^B = 4\mathbf{a}_2 + 6\mathbf{a}_3}$$

$${}^N\mathbf{v}^P = {}^N\mathbf{v}^{\bar{A}} + {}^A\mathbf{v}^{B*} + {}^N\omega^B \times \mathbf{r}^{P/B*}$$

$${}^N\mathbf{v}^P = 35\mathbf{a}_1 + \frac{6p}{2\pi}\mathbf{a}_3 + [4\mathbf{a}_2 + 6\mathbf{a}_3] \times 7\mathbf{a}_1$$

$$= 35\mathbf{a}_1 + \frac{6p}{2\pi}\mathbf{a}_3 + (-28\mathbf{a}_3 + 42\mathbf{a}_2) = 35\mathbf{a}_1 + \left(\frac{6p}{2\pi} - 28\right)\mathbf{a}_3 + 42\mathbf{a}_2$$

$${}^N\mathbf{a}^P = {}^N\mathbf{a}^{B*} + {}^N\alpha^B \times \mathbf{r}^{P/B*} + {}^N\omega^B \times \frac{d\mathbf{r}^{P/B*}}{dt}$$

$$\frac{d\mathbf{r}^{P/B*}}{dt} = \frac{d\mathbf{r}^{P/B*}}{dt} + {}^N\omega^B \times \mathbf{r}^{P/B*} = (4\mathbf{a}_2 + 6\mathbf{a}_3) \times 7\mathbf{a}_1 = -28\mathbf{a}_3 + 42\mathbf{a}_2$$

$$\boxed{{}^N\mathbf{v}^{P/B*} = 42\mathbf{a}_2 - 28\mathbf{a}_3}$$

$${}^N\alpha^B = \frac{d}{{}^N\omega^B} = \frac{d}{{}^N\omega^A} + \frac{d}{{}^A\omega^B} = {}^N\alpha^A + \frac{d}{{}^A\omega^B} + {}^N\omega^A \times {}^A\omega^B$$

$$= 5\mathbf{a}_3 + 4\mathbf{a}_2 \times 6\mathbf{a}_3$$

$$\boxed{{}^N\alpha^B = 5\mathbf{a}_3 + 24\mathbf{a}_1}$$

to find ${}^N\mathbf{a}^{B*} \Rightarrow {}^N\mathbf{a}^{B*} = \frac{d}{{}^A\mathbf{v}^{B*}} + {}^N\omega^A \times {}^A\mathbf{v}^{B*} + 2{}^N\omega^A \times {}^A\omega^B$

$$= \frac{\ddot{\theta} p}{2\pi}\mathbf{a}_3 + \omega^A + 2[4\mathbf{a}_2] \times \frac{6p}{2\pi}\mathbf{a}_3$$

$${}^N\mathbf{a}^{\bar{A}} = {}^N\mathbf{a}^{A*} + {}^N\alpha^A \times \mathbf{r}^{\bar{A}/A*} + {}^N\omega^A \times \frac{d\mathbf{r}^{\bar{A}/A*}}{dt}$$

$${}^N\omega^A \times \mathbf{r}^{\bar{A}/A*}$$

$$= 0 + 5\mathbf{a}_3 \times 8\mathbf{a}_3 + 4\mathbf{a}_2 \times 4\mathbf{a}_2 \times 8\mathbf{a}_3$$

$${}^N\mathbf{a}^{\bar{A}} = 4\mathbf{a}_2 \times (+32\mathbf{a}_1) = -128\mathbf{a}_3$$

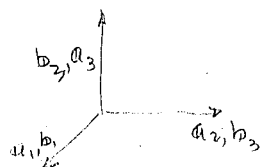
$$\therefore {}^N\mathbf{a}^{B*} = \frac{\ddot{\theta} p}{2\pi}\mathbf{a}_3 + 128\mathbf{a}_3 + \frac{48p}{2\pi}\mathbf{a}_1$$

$$\therefore {}^N\mathbf{a}^P = \frac{\ddot{\theta} p}{2\pi}\mathbf{a}_3 + 128\mathbf{a}_3 + \frac{48p}{2\pi}\mathbf{a}_1 + 35\mathbf{a}_1 + (4\mathbf{a}_2 + 6\mathbf{a}_3) \times (42\mathbf{a}_2 - 28\mathbf{a}_3)$$

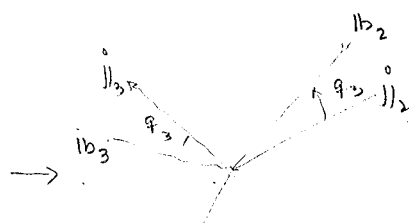
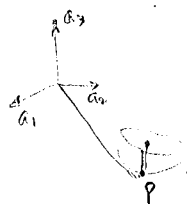
$$= \frac{\ddot{\theta} p}{2\pi}\mathbf{a}_3 + 35\mathbf{a}_1 - 128\mathbf{a}_3 - 112\mathbf{a}_1 - 252\mathbf{a}_1 + \frac{48p}{2\pi}\mathbf{a}_1 = \left(\frac{48p}{2\pi} - 364\right)\mathbf{a}_1 + 35\mathbf{a}_2 + \left(\frac{\ddot{\theta} p}{2\pi} - 128\right)\mathbf{a}_3$$

$${}^N\mathbf{a}^P = {}^N\mathbf{a}^{\bar{A}} + {}^A\mathbf{a}^{B*} + {}^N\omega^A \times {}^A\mathbf{v}^{B*} + {}^N\alpha^B \times \mathbf{r}^{P/B*} + {}^N\omega^B \times ({}^N\omega^B \times \mathbf{r}^{P/B*})$$

where $\mathbf{a}^{\bar{A}} = \mathbf{a} + \alpha \times \mathbf{r}^{\bar{A}/A*}$
 $+ {}^N\omega^A \times ({}^N\omega^A \times \mathbf{r}^{\bar{A}/A*})$



→

 $\dot{a}_1, \dot{a}_2$

 $\dot{b}_1, \dot{b}_2$


$$1. \quad \begin{aligned} \omega &= \omega + \omega + \omega \\ &= \dot{q}_1 \dot{a}_1 + \dot{q}_2 \dot{a}_2 + \dot{q}_3 \dot{a}_3 \end{aligned}$$

2. Need to find ω^P , ω^Q

$$\begin{aligned} \omega^P &= x a_1 + y a_2 + z a_3 \quad \text{where } x, y \text{ is location of common pt of } P \text{ \& } C \\ \omega^Q &= x a_1 + y a_2 + z a_3 \end{aligned}$$

$$\omega^P = x a_1 + y a_2$$

Let the point Q be the fixed point in B

$$\text{then } \omega^Q = -h b_3 \quad \text{where } \omega^{A/B} \text{ denotes vector from B to A}$$

$$\begin{aligned} \text{thus } \omega^Q &= \omega \times \omega^Q \\ &= (\dot{q}_1 \dot{a}_1 + \dot{q}_2 \dot{a}_2 + \dot{q}_3 \dot{a}_3) \times -h b_3 \end{aligned}$$

$$\text{now } \omega^P = \omega^Q - r a_3$$

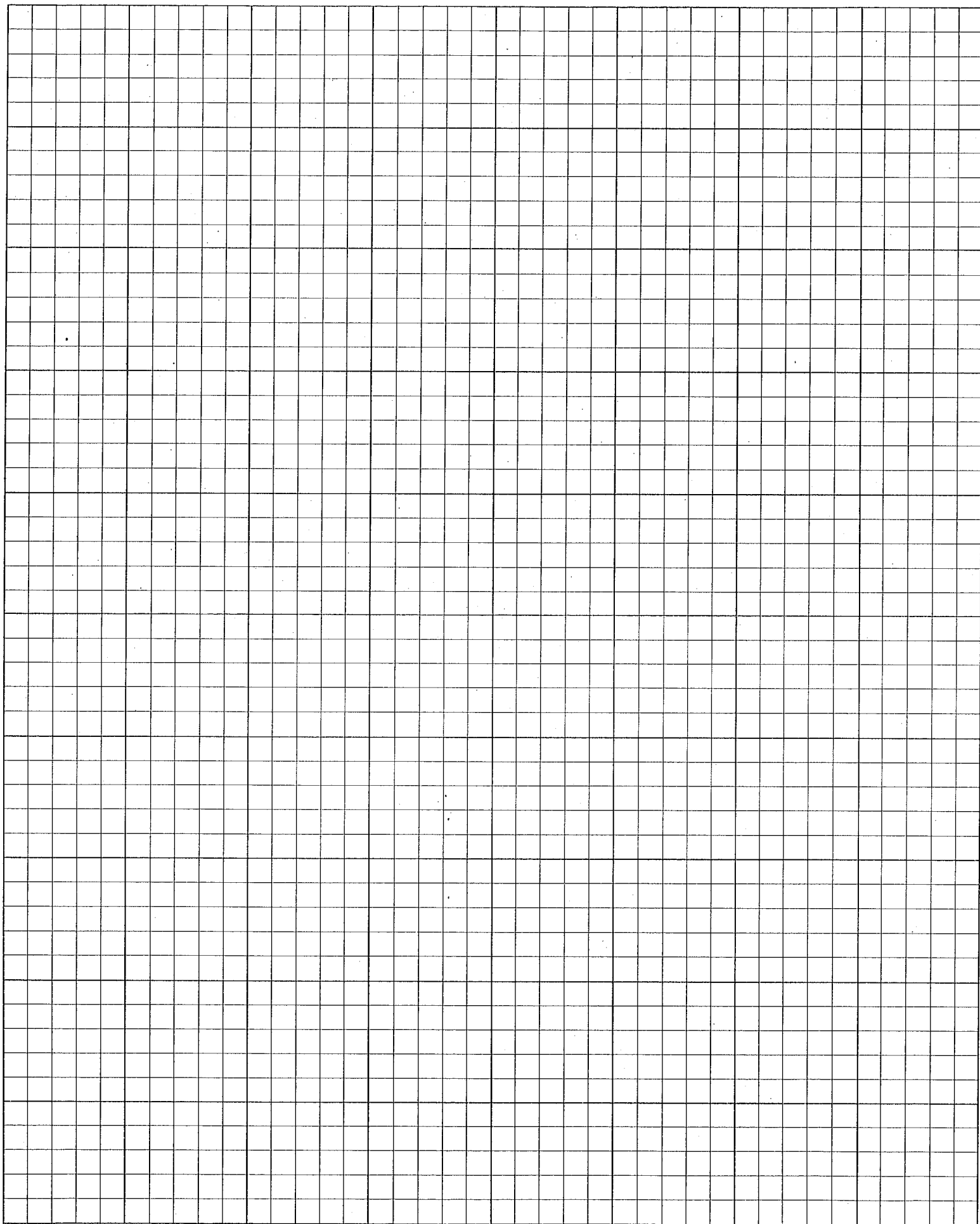
$$\therefore \omega^P = \omega^Q \quad \text{since } r a_3 \text{ is a fixed vector in A}$$

$$\text{now } \omega^Q = \omega + \omega \times \omega^Q$$

	$\dot{a}_1$	$\dot{a}_2$	$\dot{a}_3$
$\dot{a}_1$	1	0	0
$\dot{a}_2$	0	1	0
$\dot{a}_3$	0	0	1

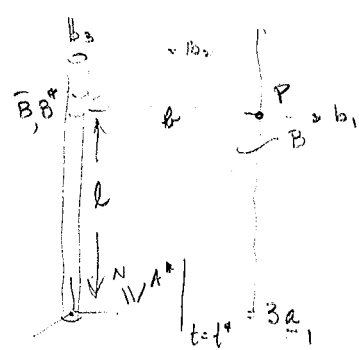
	$\dot{b}_1$	$\dot{b}_2$	$\dot{b}_3$
$\dot{b}_1$	1	0	0
$\dot{b}_2$	0	1	0
$\dot{b}_3$	0	0	1

not necessary



Gwen

poor



Let a plane which contains P & is $\perp$ to A
have fixed unit vectors b_1, b_2, b_3

How can something be perpendicular to a spacecraft?

given that $\vec{A}^B = 6a_3$: find $\left. \frac{N}{V} P \right|_t$ & $\left. \frac{N}{R} \right|_t$

$\left. \frac{N}{V} A \right|_{t=t^*} = 3a_1$ $\left. \frac{N}{R} A \right|_{t=t^*} = 0$

$\left. \frac{N}{V} A \right|_{t=t^*} = 4a_2$ $\left. \frac{N}{R} A \right|_{t=t^*} = 5a_3$

$\left. \frac{N}{V} B \right|_t = \left. \frac{N}{V} A \right|_t + \left. \frac{A}{V} B \right|_t = 4a_2 + 6a_3$ at time t

$\frac{A}{V} B^* = \frac{A}{V} A^* + \frac{A}{V} B^* A$

$\frac{A}{V} B^* = \frac{A}{V} A^* + l b_3 + \omega l b_3$

$\frac{A}{V} = \frac{A}{V} + \omega \times l b_1$

$\left. \frac{N}{V} B^* \right|_{t=t^*} = \left. \frac{N}{V} A^* \right|_{t=t^*} + \left. \frac{N}{V} B^* A \right|_{t=t^*}$

$\left. \frac{N}{V} B^* \right|_{t=t^*} = 3a_1 + 4a_3 + 8a_3 = 3a_1 + 12a_3$

$\left. \frac{N}{V} B^* \right|_{t=t^*} = 3a_1 + 32a_3 = 35a_3$

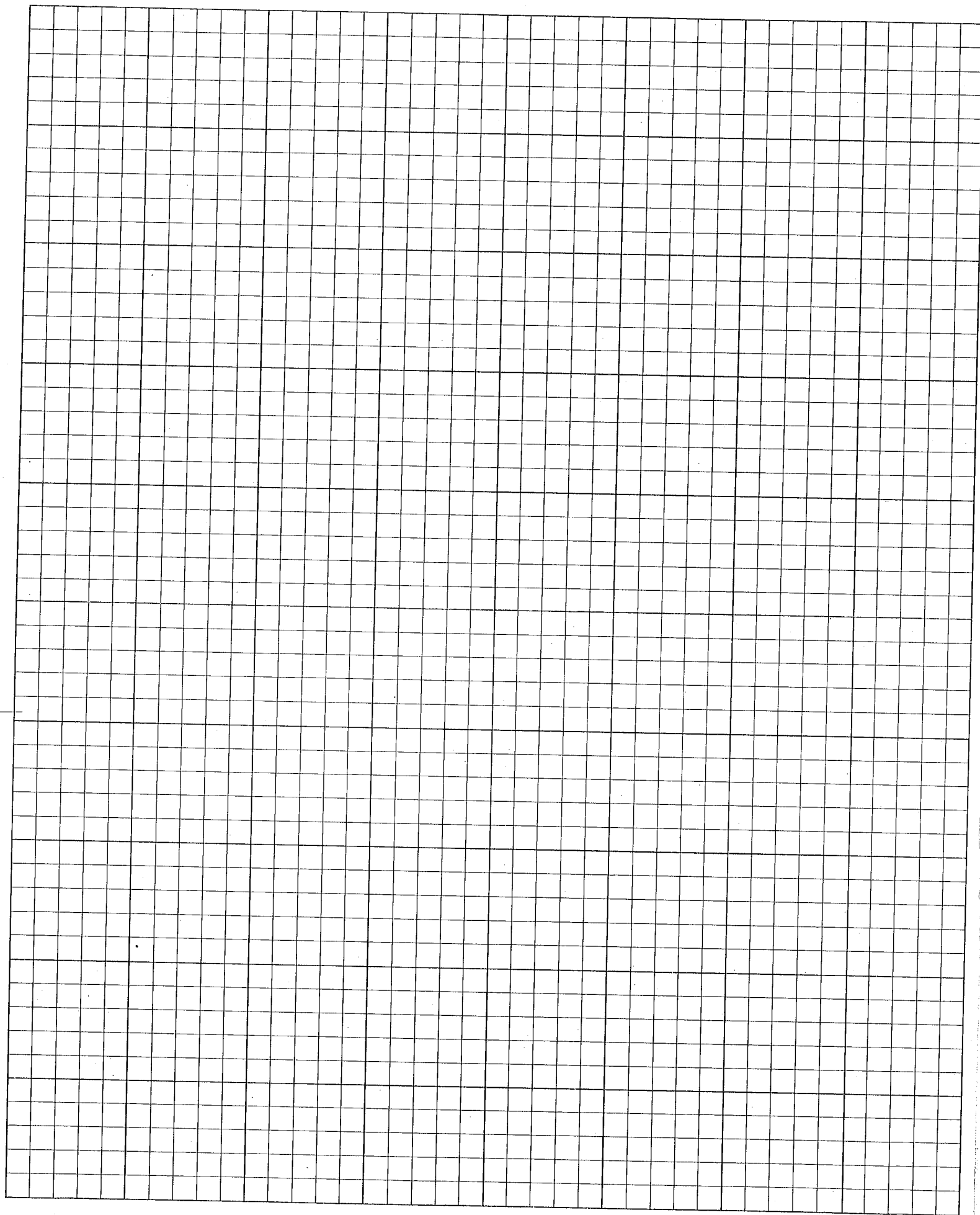
$\left. \frac{N}{V} P \right|_{t=t^*} = \left. \frac{N}{V} B^* \right|_{t=t^*} + \left. \frac{N}{V} B^* A \right|_{t=t^*}$

$\left. \frac{N}{V} P \right|_{t=t^*} = 35a_3 + [4a_2 + 6a_3] \times 7b_1$

is $\frac{A}{V} B^*$ the position vector of B^* relative to some point? Which point? What is $\frac{A}{V} B^* A$?

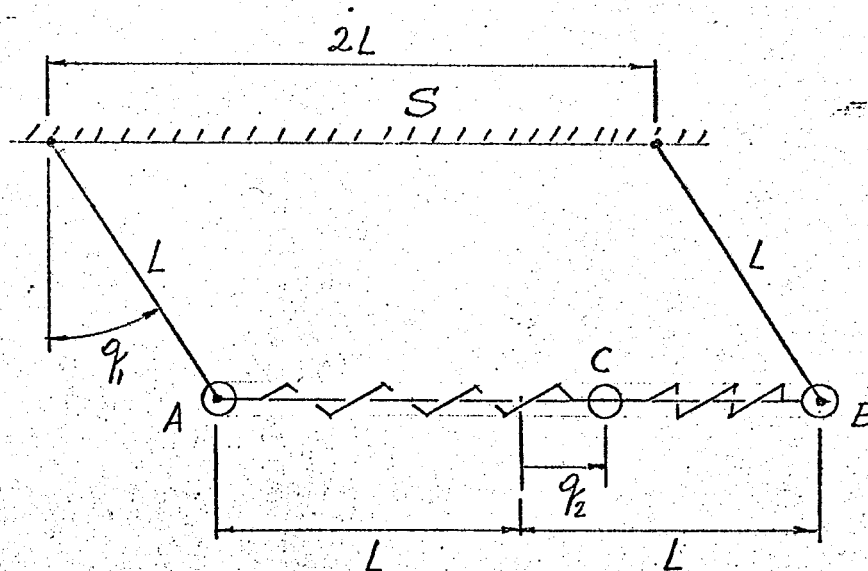
$\left. \frac{N}{V} B \right|_{t=t^*} = \left. \frac{N}{V} A \right|_{t=t^*} + \left. \frac{A}{V} B \right|_{t=t^*}$

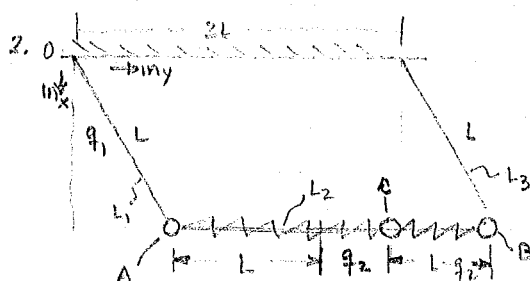
$\left. \frac{N}{V} B \right|_{t=t^*} = 4a_2 + 6a_3$



In the sketch, A, B and C designate particles of equal mass m . A and B are attached to light rods of length L , and these rods are pinned to a horizontal support S . C can slide on a smooth, light rod of length $2L$, which is pin-connected to A and B. Two light, linear springs, each of modulus k and natural length L , connect A and B to C.

Letting q_1 and q_2 measure an angle and a distance as indicated in the sketch, form two differential equations of motion for this system.





1. Since L_1, L_2, L_3 are pin connected at their common joints and L_1 & L_3 are pin connected to S , the contribution of the forces at the joints to the generalized forces are zero.
2. Since C slides on a smooth rod, the contact force of C on the rod L_2 (or vice versa) contributes nothing to the generalized active force.

We must get $F_r + F_r^* = 0$ (1)

To accomplish this we first assume that L_1, L_2, L_3 are inextensible and have no mass. We also assume that the springs have no mass. Thus

define $\cos q_1 \triangleq c_1$ $\sin q_1 \triangleq s_1$. Let R be reference frame & O be the reference point

$$r^A = Lc_1 m_x + Ls_1 m_y \quad (2)$$

$$v^A = -L\dot{q}_1 s_1 m_x + L\dot{q}_1 c_1 m_y \quad (3)$$

$$a_1^A = -Lm_x(\ddot{q}_1 s_1 + \dot{q}_1^2 c_1) + Lm_y(\ddot{q}_1 c_1 - \dot{q}_1^2 s_1) \quad (4)$$

$$r^B = Lc_1 m_x + (Ls_1 + 2L)m_y \quad (5)$$

$$v^B = -L\dot{q}_1 s_1 m_x + L\dot{q}_1 c_1 m_y \quad (6)$$

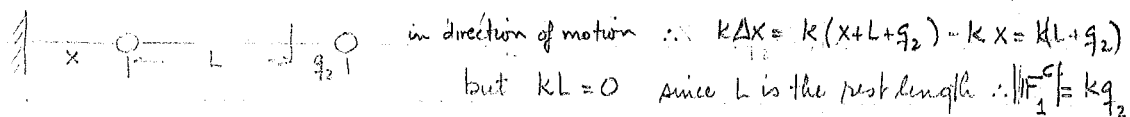
$$a_1^B = -Lm_x(\ddot{q}_1 s_1 + \dot{q}_1^2 c_1) + Lm_y(\ddot{q}_1 c_1 - \dot{q}_1^2 s_1) \quad (7)$$

$$r^C = Lc_1 m_x + (Ls_1 + L + q_2)m_y \quad (8)$$

$$v^C = -L\dot{q}_1 s_1 m_x + (L\dot{q}_1 c_1 + \dot{q}_2)m_y \quad (9)$$

$$a_1^C = -Lm_x(\ddot{q}_1 s_1 + \dot{q}_1^2 c_1) + (L\ddot{q}_1 c_1 - L\dot{q}_1^2 s_1 + \ddot{q}_2)m_y \quad (10)$$

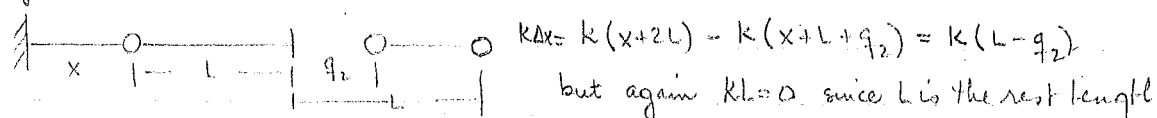
to find the force, on C due to A $F = -K\Delta x m$, where Δx is the change of length wrt rest length & m is



the direction of F^C is toward A . Thus $F_2^C = -Kq_2 m_y$ (11)

to find the force on A due to C : $F^A = -F_1^C = Kq_2 m_y$ (12)

to find the force on B due to C



$\therefore \|F^B\| = Kq_2$. The direction of F^B is toward B . Thus $F^B = Kq_2 m_y$ (13)

to find the force on C due to B : $F_2^C = -F^B = -Kq_2 m_y$ (14)

$$\text{Thus } F^C = -2Kq_2 m_y \quad (15)$$

1
2
3
4
5
6
7
8
9
10
11
12
13
14
15
16
17
18
19
20
21
22
23
24
25
26
27
28
29
30
31
32
33
34
35
36
37
38
39
40
41
42
43
44
45
46
47
48
49
50
51
52
53
54
55
56
57
58
59
60
61
62
63
64
65
66
67
68
69
70
71
72
73
74
75
76
77
78
79
80
81
82
83
84
85
86
87
88
89
90
91
92
93
94
95
96
97
98
99
100
101
102
103
104
105
106
107
108
109
110
111
112
113
114
115
116
117
118
119
120
121
122
123
124
125
126
127
128
129
130
131
132
133
134
135
136
137
138
139
140
141
142
143
144
145
146
147
148
149
150
151
152
153
154
155
156
157
158
159
160
161
162
163
164
165
166
167
168
169
170
171
172
173
174
175
176
177
178
179
180
181
182
183
184
185
186
187
188
189
190
191
192
193
194
195
196
197
198
199
200
201
202
203
204
205
206
207
208
209
210
211
212
213
214
215
216
217
218
219
220
221
222
223
224
225
226
227
228
229
230
231
232
233
234
235
236
237
238
239
240
241
242
243
244
245
246
247
248
249
250
251
252
253
254
255
256
257
258
259
260
261
262
263
264
265
266
267
268
269
270
271
272
273
274
275
276
277
278
279
280
281
282
283
284
285
286
287
288
289
290
291
292
293
294
295
296
297
298
299
300
301
302
303
304
305
306
307
308
309
310
311
312
313
314
315
316
317
318
319
320
321
322
323
324
325
326
327
328
329
330
331
332
333
334
335
336
337
338
339
340
341
342
343
344
345
346
347
348
349
350
351
352
353
354
355
356
357
358
359
360
361
362
363
364
365
366
367
368
369
370
371
372
373
374
375
376
377
378
379
380
381
382
383
384
385
386
387
388
389
390
391
392
393
394
395
396
397
398
399
400
401
402
403
404
405
406
407
408
409
410
411
412
413
414
415
416
417
418
419
420
421
422
423
424
425
426
427
428
429
430
431
432
433
434
435
436
437
438
439
440
441
442
443
444
445
446
447
448
449
450
451
452
453
454
455
456
457
458
459
460
461
462
463
464
465
466
467
468
469
470
471
472
473
474
475
476
477
478
479
480
481
482
483
484
485
486
487
488
489
490
491
492
493
494
495
496
497
498
499
500
501
502
503
504
505
506
507
508
509
510
511
512
513
514
515
516
517
518
519
520
521
522
523
524
525
526
527
528
529
530
531
532
533
534
535
536
537
538
539
540
541
542
543
544
545
546
547
548
549
550
551
552
553
554
555
556
557
558
559
560
561
562
563
564
565
566
567
568
569
570
571
572
573
574
575
576
577
578
579
580
581
582
583
584
585
586
587
588
589
590
591
592
593
594
595
596
597
598
599
600
601
602
603
604
605
606
607
608
609
610
611
612
613
614
615
616
617
618
619
620
621
622
623
624
625
626
627
628
629
630
631
632
633
634
635
636
637
638
639
640
641
642
643
644
645
646
647
648
649
650
651
652
653
654
655
656
657
658
659
660
661
662
663
664
665
666
667
668
669
670
671
672
673
674
675
676
677
678
679
680
681
682
683
684
685
686
687
688
689
690
691
692
693
694
695
696
697
698
699
700
701
702
703
704
705
706
707
708
709
710
711
712
713
714
715
716
717
718
719
720
721
722
723
724
725
726
727
728
729
730
731
732
733
734
735
736
737
738
739
740
741
742
743
744
745
746
747
748
749
750
751
752
753
754
755
756
757
758
759
760
761
762
763
764
765
766
767
768
769
770
771
772
773
774
775
776
777
778
779
780
781
782
783
784
785
786
787
788
789
790
791
792
793
794
795
796
797
798
799
800
801
802
803
804
805
806
807
808
809
810
811
812
813
814
815
816
817
818
819
820
821
822
823
824
825
826
827
828
829
830
831
832
833
834
835
836
837
838
839
840
841
842
843
844
845
846
847
848
849
850
851
852
853
854
855
856
857
858
859
860
861
862
863
864
865
866
867
868
869
870
871
872
873
874
875
876
877
878
879
880
881
882
883
884
885
886
887
888
889
890
891
892
893
894
895
896
897
898
899
900
901
902
903
904
905
906
907
908
909
910
911
912
913
914
915
916
917
918
919
920
921
922
923
924
925
926
927
928
929
930
931
932
933
934
935
936
937
938
939
940
941
942
943
944
945
946
947
948
949
950
951
952
953
954
955
956
957
958
959
960
961
962
963
964
965
966
967
968
969
970
971
972
973
974
975
976
977
978
979
980
981
982
983
984
985
986
987
988
989
990
991
992
993
994
995
996
997
998
999
1000

$$\text{Now } F_r = (F_r)_I + (F_r)_{G_1} \quad (16)$$

$$(F_r)_I = W \dot{q}_r^A \cdot IF^A + W \dot{q}_r^B \cdot IF^B + W \dot{q}_r^C \cdot IF^C \quad (17)$$

$$(F_r)_{G_1} = W \dot{q}_r^A \cdot W m_x + W \dot{q}_r^B \cdot W m_x + W \dot{q}_r^C \cdot W m_x \quad (18)$$

Now using (3), (6), (7) we obtain

$$W \dot{q}_1^A = -L s_1 m_x + L c_1 m_y \quad W \dot{q}_2^A = 0 \quad (19a, b)$$

$$W \dot{q}_1^B = -L s_1 m_x + L c_1 m_y \quad W \dot{q}_2^B = 0 \quad (20a, b)$$

$$W \dot{q}_1^C = -L s_1 m_x + L c_1 m_y \quad W \dot{q}_2^C = m_y \quad (21a, b)$$

$$(F_1)_I = k q_2 l c_1 + k q_2 L c_1 - 2k L c_1 q_2 = 0 \quad (22)$$

$$(F_2)_I = 0 + 0 - 2k q_2 = -2k q_2 \quad (23)$$

$$(F_1)_G = -L s_1 W - L s_1 W - W L s_1 = -3W L s_1 \quad (24)$$

$$(F_2)_G = 0 + 0 + 0 = 0 \quad (25)$$

$$F_1 = (F_1)_I + (F_1)_G \stackrel{(22, 24)}{=} -3W L s_1 q_1 \quad (26)$$

$$F_2 = (F_2)_I + (F_2)_G \stackrel{(23, 25)}{=} -2k q_2 \quad (27)$$

Now

$$F_r^* = \sum W \dot{q}_r^{P_i} \cdot IF_i^* \quad \text{where } IF_i^* = -m_i \ddot{q}_i^{P_i} \quad (28)$$

$$\therefore F_1^* = W \dot{q}_1^A \cdot IF^A + W \dot{q}_1^B \cdot IF^B + W \dot{q}_1^C \cdot IF^C^* \\ = -3m \left[-L s_1 (-L \ddot{q}_1 s_1 - \dot{q}_1^2 L c_1) + L c_1 (\ddot{q}_1 L c_1 - \dot{q}_1^2 L s_1) \right] - m L c_1 \ddot{q}_2 \quad (29)$$

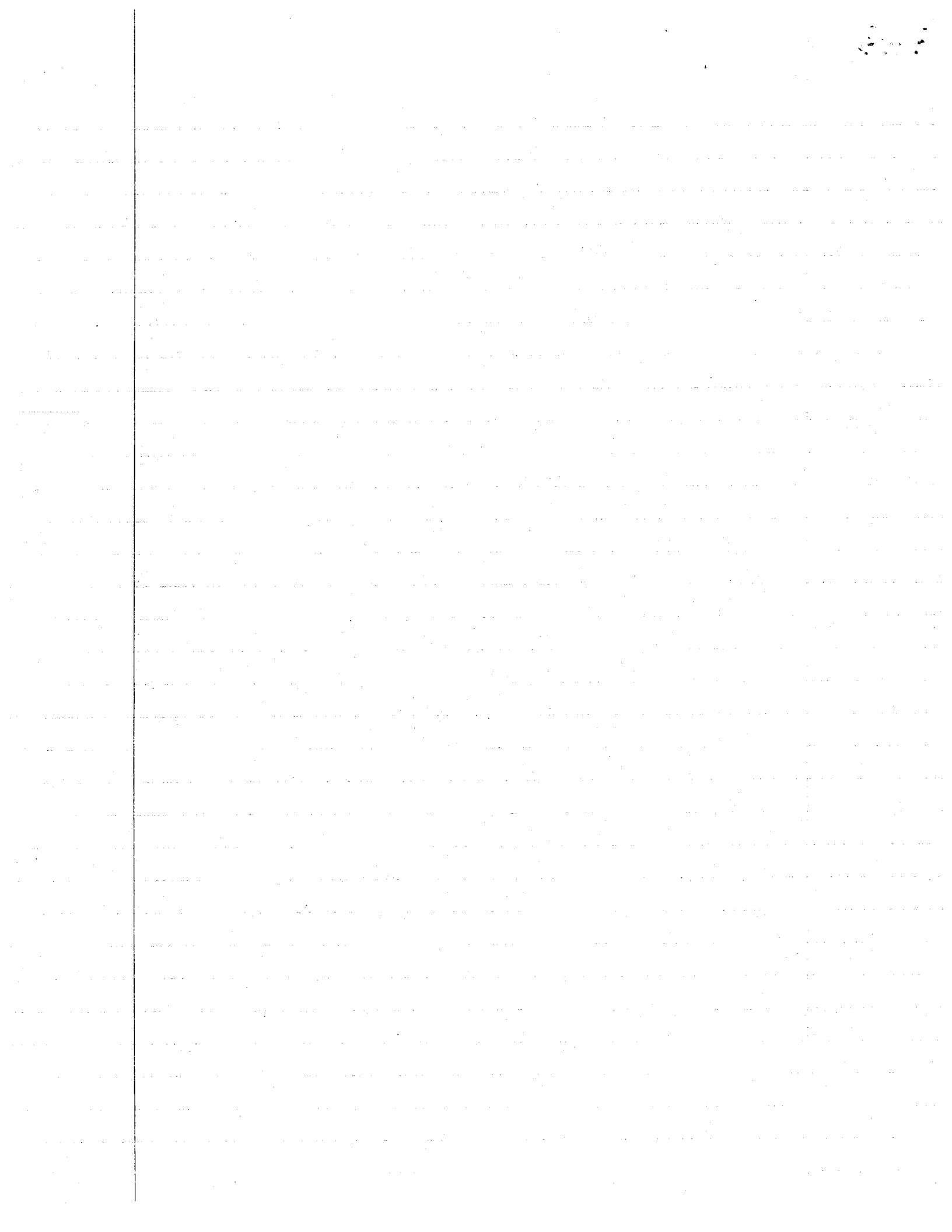
$$F_2^* = W \dot{q}_2^A \cdot IF^A + W \dot{q}_2^B \cdot IF^B + W \dot{q}_2^C \cdot IF^C^* \\ = 0 + 0 - m (L \ddot{q}_1 c_1 - L \dot{q}_1^2 s_1 + \ddot{q}_2) \quad (30)$$

Now F_1^* reduces to $F_1^* = -3m L^2 \ddot{q}_1 - m L \cos q_1 \ddot{q}_2$

Thus (1) becomes

$$F_1 + F_1^* = -3W L s_1 q_1 - 3m L^2 \ddot{q}_1 - m L \cos q_1 \ddot{q}_2 = 0 \quad (31)$$

$$F_2 + F_2^* = -2k q_2 - m (L \ddot{q}_1 \cos q_1 - L \dot{q}_1^2 \sin q_1 + \ddot{q}_2) = 0 \quad (32)$$



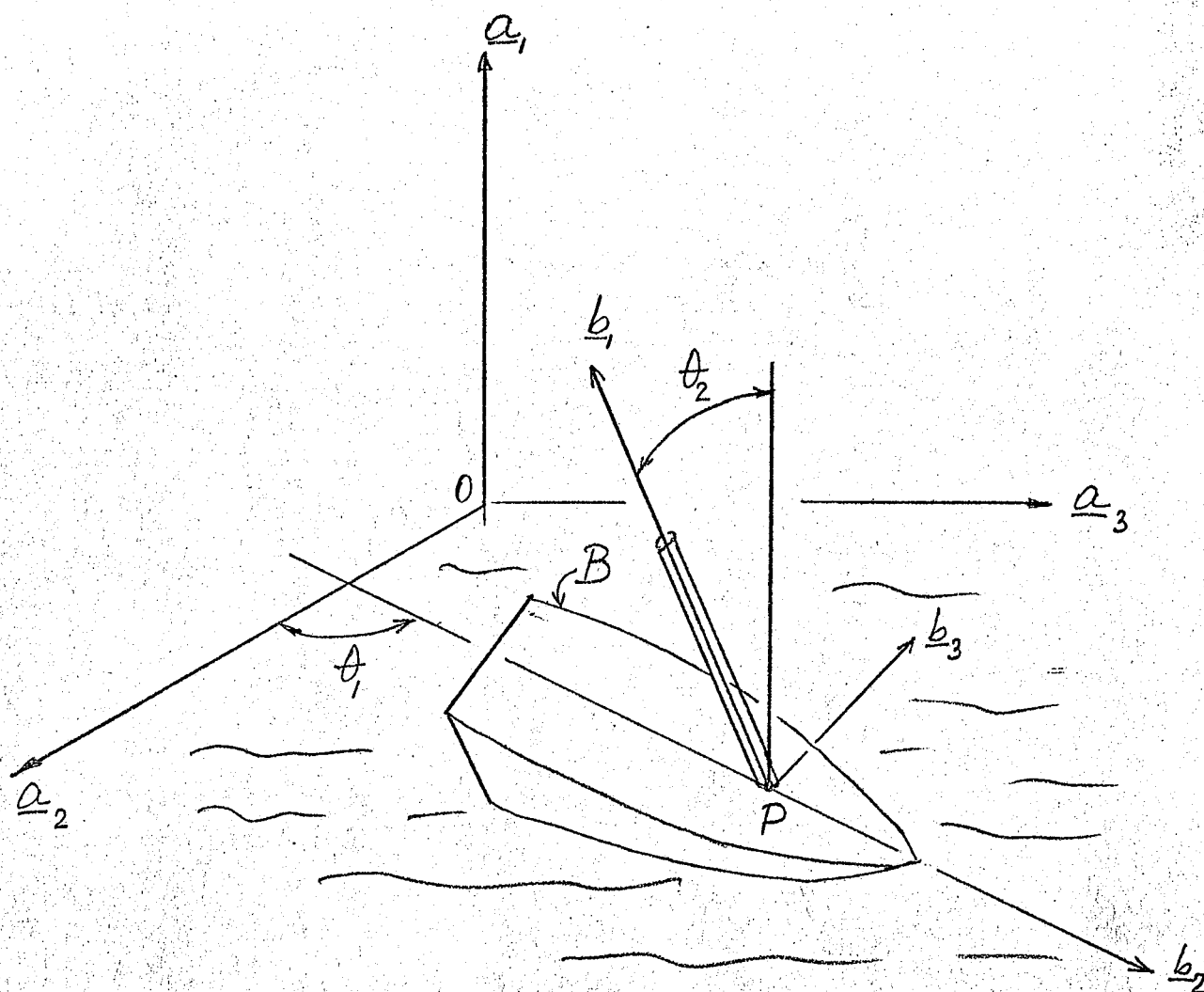
To analyze motions of a sailboat B approximately one may proceed as follows: Letting $\underline{a}_1, \underline{a}_2, \underline{a}_3$ be a dextral set of orthogonal unit vectors, with $\underline{a}_1$ pointing vertically upward, and letting $\underline{b}_1, \underline{b}_2, \underline{b}_3$ be a similar set fixed in B , with $\underline{b}_1$ parallel to the main mast and $\underline{b}_2$ pointing forward, align $\underline{b}_i$ with $\underline{a}_i$ ($i = 1, 2, 3$), and then subject B successively to a dextral $\underline{b}_1$ rotation of amount θ_1 (yaw) and a dextral $\underline{b}_2$ rotation of amount θ_2 (roll). Next, letting O be a point fixed in the surface of the water and P a point lying in the surface of the water and fixed in the plane of symmetry of B , express the position vector of P relative to O as $y\underline{a}_2 + z\underline{a}_3$. The quantities θ_1, θ_2, y , and z then characterize motions during which pitching and vertical translation are negligible.

The action of the keel (or centerboard) of B may be taken into account by assuming that the velocity of P is always parallel to $\underline{b}_2$. Under these circumstances the four quantities θ_1, θ_2, y , and z may be regarded as generalized coordinate $q_1, \dots, q_4$, respectively, of a nonholonomic system possessing three degrees of freedom.

Taking the system of all contact and body forces acting on B to be equivalent to a couple of torque $\underline{R}$ together with a force $\underline{S}$ applied to P , with

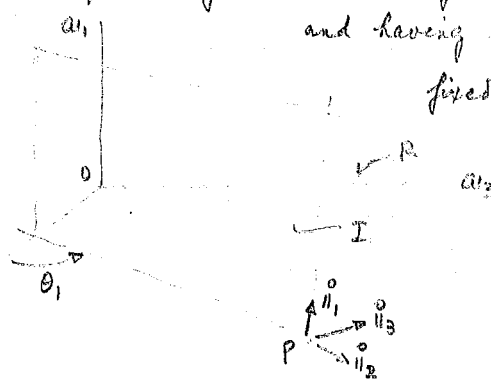
$$\underline{R} = R_1 \underline{b}_1 + R_2 \underline{b}_2 + R_3 \underline{b}_3, \quad \underline{S} = S_1 \underline{b}_1 + S_2 \underline{b}_2 + S_3 \underline{b}_3$$

determine the generalized forces F_1, F_2, F_3 associated with q_1, q_2, q_3 , respectively.



Final Examination

1. With the information given let us define the auxiliary frame I, which is $\perp$ to the planes of the water surface, containing the point P as shown below and having unit vectors $\hat{n}_1, \hat{n}_2, \hat{n}_3$ forming a right handed triad fixed in I. Thus



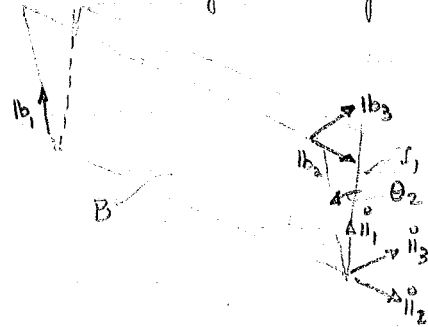
$${}^R \omega^I = \dot{\theta}_1 \hat{a}_1 \quad (1)$$

	$\hat{a}_1$	$\hat{a}_2$	$\hat{a}_3$
$\hat{n}_1$	1	0	0
$\hat{n}_2$	0	c_1	s_1
$\hat{n}_3$	0	$-s_1$	c_1

define $c_i \triangleq \cos \theta_i$; $s_i \triangleq \sin \theta_i$

$\hat{n}_1 \times \hat{n}_2 = \hat{n}_3$

We can then define the reference frame B as follows



$$\text{Thus } {}^I \omega^B = \dot{\theta}_2 \hat{b}_2 \quad (2)$$

and

	$\hat{n}_1$	$\hat{n}_2$	$\hat{n}_3$
$\hat{b}_1$	c_2	0	$-s_2$
$\hat{b}_2$	0	1	0
$\hat{b}_3$	s_2	0	c_2

Since W^P is parallel to $\hat{b}_2$ then $W^P = (W^P \cdot \hat{b}_2) \hat{b}_2 \quad (3)$

Thus starting with $W^P = \dot{y} \hat{a}_2 + \dot{z} \hat{a}_3$ then $\frac{d}{dt} W^P = W^P = \dot{y} \hat{a}_2 + \dot{z} \hat{a}_3 \quad (4)$

But using the definitions of $\hat{n}_1, \hat{n}_2, \hat{n}_3$ and $\hat{b}_1, \hat{b}_2, \hat{b}_3$ we can convert the $\hat{a}_i$ bases in terms of the $\hat{b}_i$ bases. Hence

$$\begin{aligned} W^P &= \dot{y} \hat{a}_2 + \dot{z} \hat{a}_3 = (\dot{y} c_1 + \dot{z} s_1) \hat{n}_2 + (-\dot{y} s_1 + c_1 \dot{z}) \hat{n}_3 \\ &= (\dot{y} c_1 + \dot{z} s_1) \hat{b}_2 + (\dot{y} s_1 s_2 - \dot{z} c_1 s_2) \hat{b}_1 + (-\dot{y} s_1 c_2 + c_1 c_2 \dot{z}) \hat{b}_3 \end{aligned} \quad (5)$$

but by (3) we know that $\dot{y} s_1 s_2 - \dot{z} c_1 s_2 = 0 \quad (6a)$ and

$$-\dot{y} s_1 c_2 + c_1 c_2 \dot{z} = 0 \quad (6b)$$

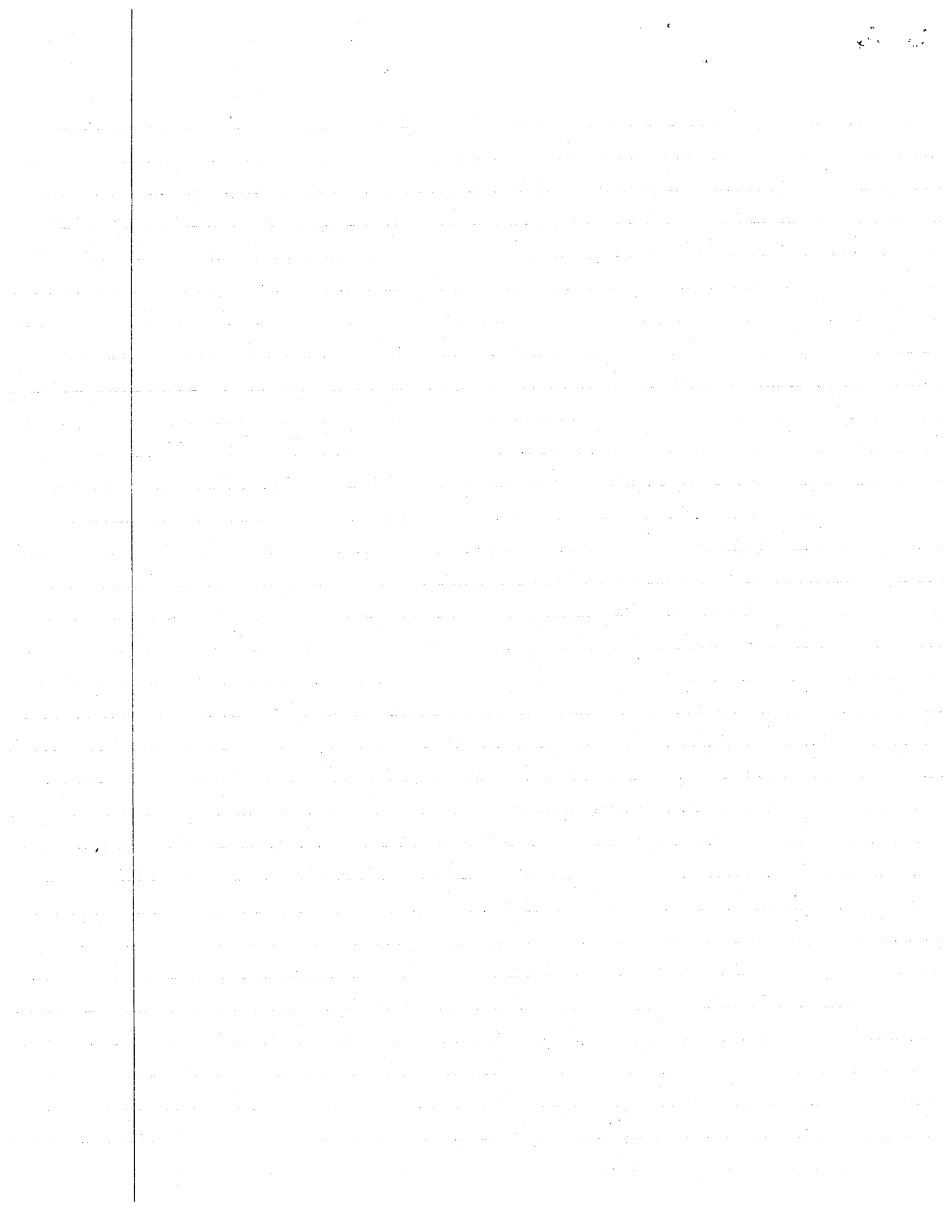
For nontrivial solutions of $\dot{y}$ & $\dot{z}$ we must have the determinant of the coefficients be zero. The determinant $= s_1 s_2 c_1 c_2 - s_1 s_2 c_1 c_2 \equiv 0 \quad \forall \theta_1$ and θ_2

We can then solve either (6a) or (6b) for $\dot{z}$ in terms of $\dot{y}$. Solving (6b) we find

$$\dot{z} = \frac{s_1}{c_1} \dot{y} \quad (7)$$

Now from (5) $W^P = (\dot{y} c_1 + \dot{z} s_1) \hat{b}_2$; using (7) in this relation gives

$$W^P = (\dot{y} c_1 + \dot{y} \frac{s_1^2}{c_1}) \hat{b}_2 = \frac{\dot{y}}{c_1} \hat{b}_2 \quad (8)$$



we also have from (1) and (2) That ${}^R_B \omega = {}^R_I \dot{\theta} + {}^I_B \omega = \dot{\theta}_1 a_1 + \dot{\theta}_2 b_2$ (9)

To find the F_r 's we employ the relationships for a rigid body

$$F_r = {}^R_P V_{\dot{q}_r} \cdot S + {}^R_B \omega \cdot R \quad (10)$$

Since we only have 3 degrees of freedom but 4 generalized coordinates, it was necessary to find a relationship between 2 of the generalized coordinates, i.e. eqn (7)

Therefore from (8) and (4) and employing $\theta_1 = q_1$, $\theta_2 = q_2$, $y = q_3$

$${}^R_P V_{\dot{q}_1} = 0 \quad {}^R_P V_{\dot{q}_2} = 0 \quad {}^R_P V_{\dot{q}_3} = b_2/c_1 \quad (11a, b, c)$$

$$\text{and } {}^R_B \omega \cdot \dot{q}_1 = a_1, \quad {}^R_B \omega \cdot \dot{q}_2 = b_2, \quad {}^R_B \omega \cdot \dot{q}_3 = 0 \quad (12a, b, c)$$

Hence with the definitions

$$R \hat{=} \sum_{i=1}^3 R_i b_i \quad S \hat{=} \sum_{i=1}^3 S_i b_i \quad (13a, b)$$

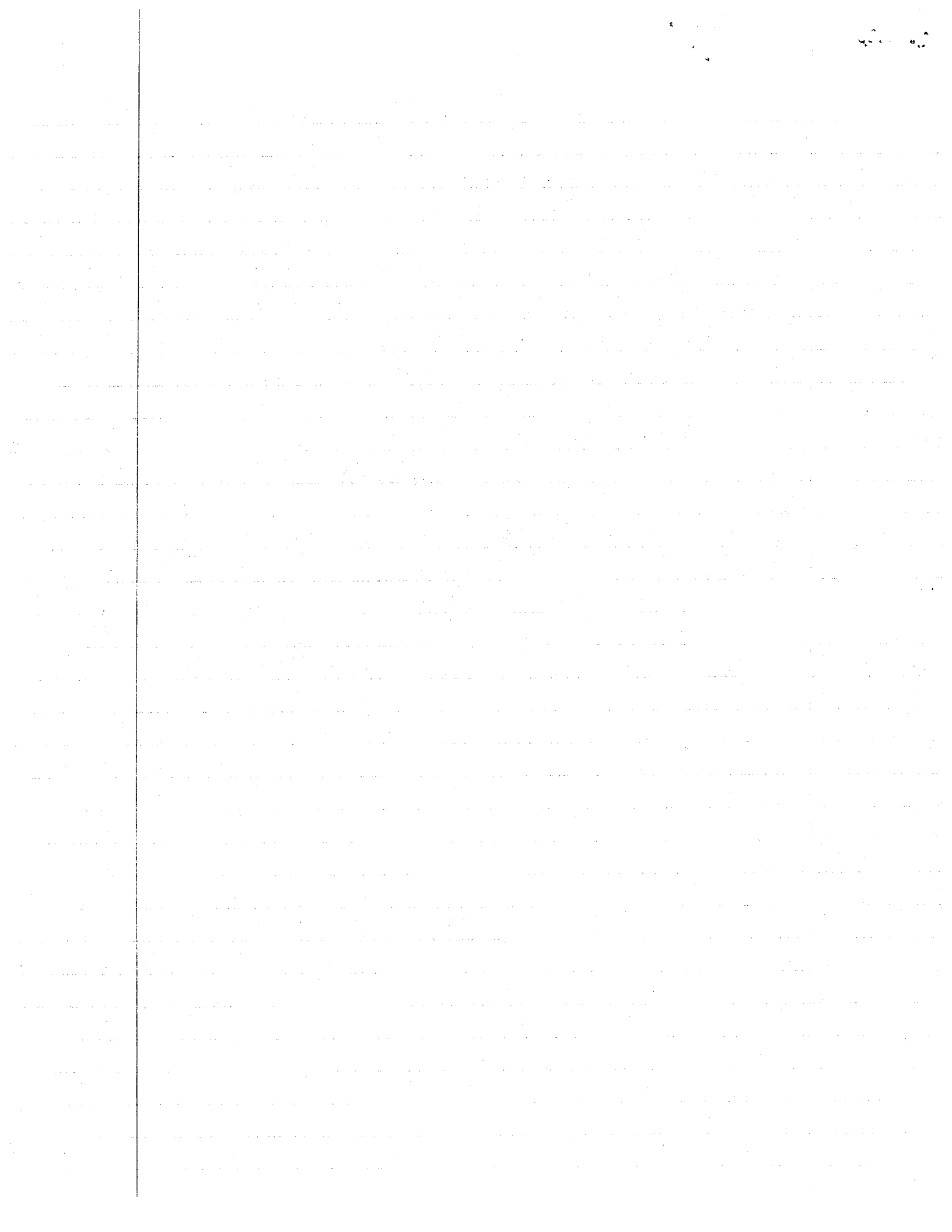
Then

$$F_1 \stackrel{(11a, 12a, 13)}{=} {}^R_B \omega \cdot \dot{q}_1 \cdot R + {}^R_P V_{\dot{q}_1} \cdot S = R_1 \cos \theta_2 + R_3 \sin \theta_2 \quad (14)$$

$$F_2 \stackrel{(11b, 12b, 13)}{=} {}^R_B \omega \cdot \dot{q}_2 \cdot R + {}^R_P V_{\dot{q}_2} \cdot S = R_2 \quad (15)$$

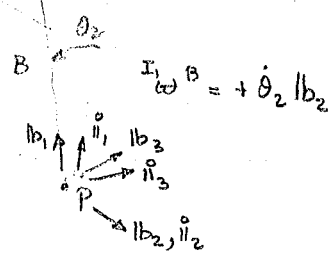
and

$$F_3 \stackrel{(11c, 12c, 13)}{=} {}^R_B \omega \cdot \dot{q}_3 \cdot R + {}^R_P V_{\dot{q}_3} \cdot S = \frac{S_2}{c_1} = S_2 \sec \theta_1 = S_2 \sqrt{1 + \tan^2 \theta_1} \quad (16)$$



Final Exam

$${}^R \omega^B = \dot{\theta}_1 a_1$$



$${}^R \omega^B = \dot{\theta}_1 a_1 + \dot{\theta}_2 b_2$$

	a_1	a_2	a_3
${}^0 \hat{a}_1$	1	0	0
${}^0 \hat{a}_2$	0	c_1	s_1
${}^0 \hat{a}_3$	0	$-s_1$	c_1

	b_1	b_2	b_3
${}^0 \hat{a}_1$	c_2	0	$-s_2$
${}^0 \hat{a}_2$	0	1	0
${}^0 \hat{a}_3$	s_2	0	c_2

$${}^R P = y a_2 + z a_3$$

no pitching, vertical translation

${}^W P = b b_2 \Rightarrow \theta_1 = q_1, \theta_2 = q_2, y = q_3, z = q_4$ but there are only 3 degrees of freedom. The fourth is removed by the fact that ${}^W P \parallel b_2$.

all contact & body forces can be made equivalent to $R = \sum R_i b_i$ & $S = \sum S_i b_i$ acting at P where R is a torque & S is the force.

What we use: $F_r = {}^R \omega^B \cdot R + {}^R \omega^P \cdot S$

we must find ${}^R \omega^B$, ${}^R \omega^P$ in terms of b_i for simplicity

$${}^R \omega^P = \dot{y} a_2 + \dot{z} a_3 = \dot{y} [c_1 \hat{a}_2 - s_1 \hat{a}_3] + \dot{z} [s_1 \hat{a}_2 + c_1 \hat{a}_3]$$

$$= (\dot{y} c_1 + \dot{z} s_1) \hat{a}_2 + (-\dot{y} s_1 + \dot{z} c_1) \hat{a}_3 = (\dot{y} c_1 + \dot{z} s_1) b_2 + (-\dot{y} s_1 + \dot{z} c_1) b_3$$

$$\therefore (\dot{y} s_1 s_2 - \dot{z} c_1 s_2) = 0 \text{ and } (-\dot{y} s_1 c_2 + \dot{z} c_1 c_2) = 0$$

$$\therefore \begin{bmatrix} s_1 s_2 & -c_1 s_2 \\ -s_1 c_2 & c_1 c_2 \end{bmatrix} \begin{pmatrix} \dot{y} \\ \dot{z} \end{pmatrix} = 0 \Rightarrow s_1 c_1 s_2 c_2 - s_1 c_1 s_2 c_2 = 0$$

for nontrivial $\dot{y}, \dot{z}$

$$\therefore {}^R \omega^P = (\dot{y} c_1 + \frac{s_1 c_2}{c_1 c_2} s_1 \dot{z}) b_2 = \dot{y} (c_1 + \frac{s_1^2 c_2}{c_1 c_2}) b_2 = \dot{y} c_2 (\frac{c_1^2 + s_1^2}{c_1 c_2}) b_2 = \dot{y} \frac{b_2}{c_1}$$

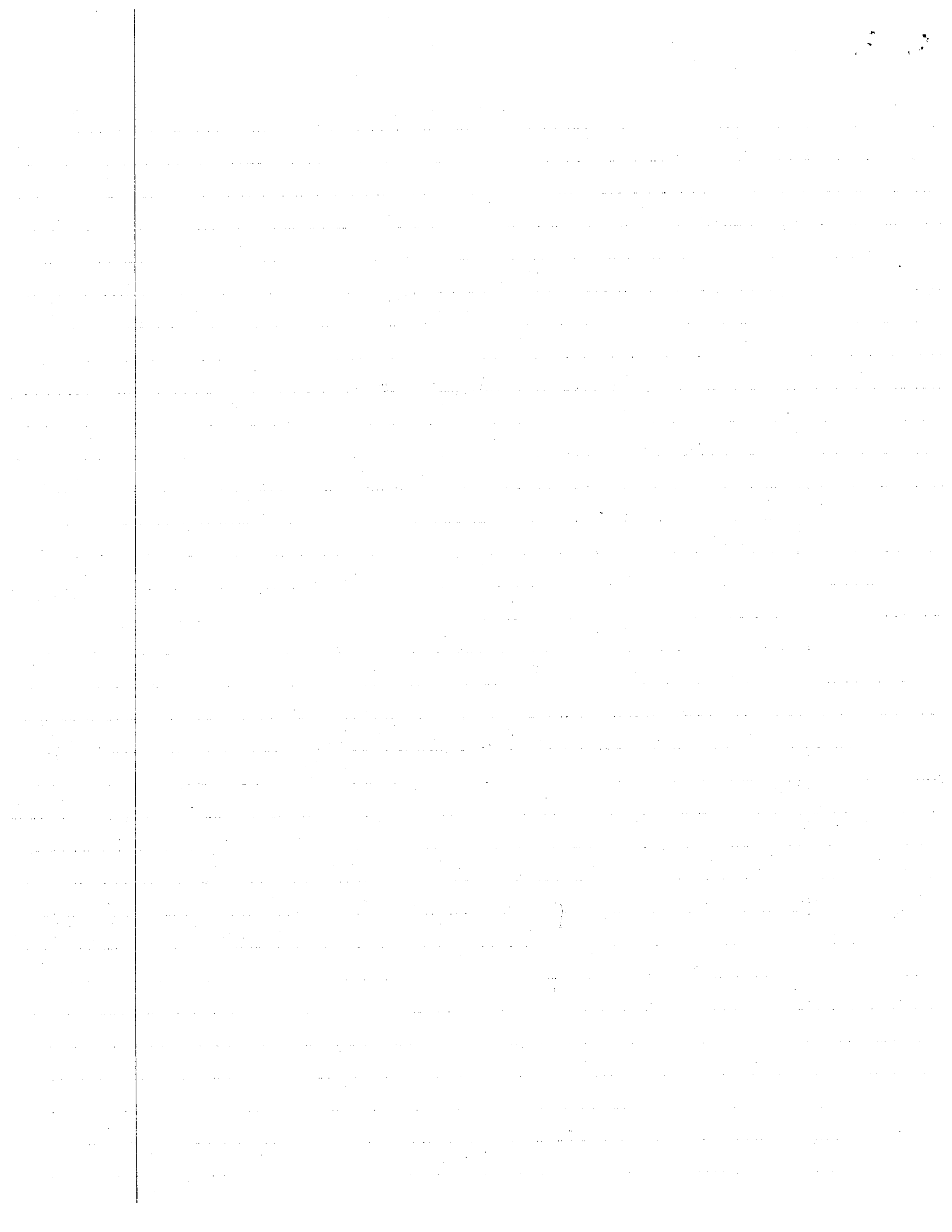
$${}^R \omega^P_{q_1} = 0 \quad {}^R \omega^P_{q_2} = 0 \quad {}^R \omega^P_{q_3} = \frac{b_2}{c_1}$$

$${}^R \omega^B = \dot{\theta}_1 a_1 + \dot{\theta}_2 b_2 \therefore {}^R \omega^B_{q_1} = a_1 \quad {}^R \omega^B_{q_2} = b_2 \quad {}^R \omega^B_{q_3} = 0$$

$${}^R \omega^B_{q_1} = \hat{a}_1 = c_2 b_1 + s_2 b_3$$

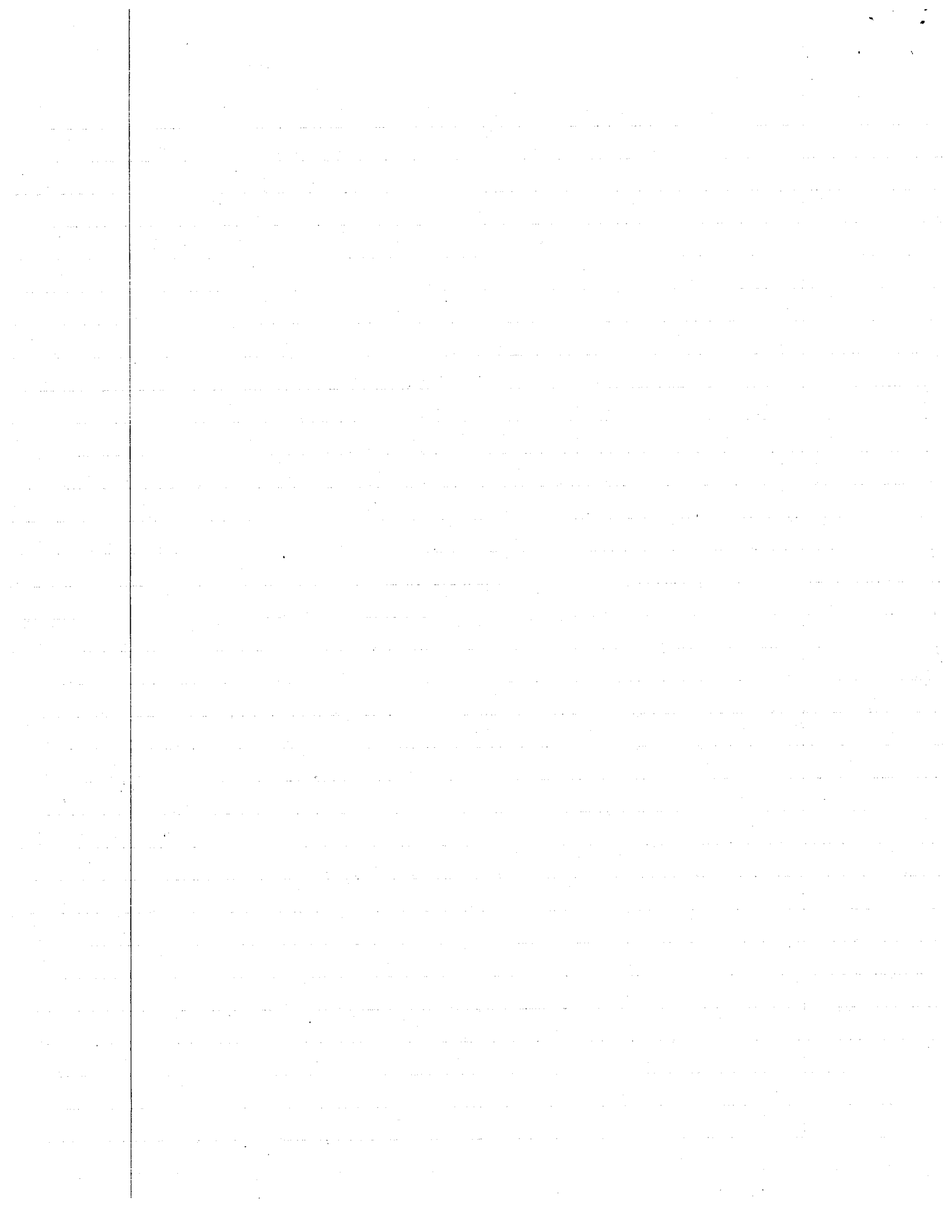
$$F_1 = [c_2 b_1 + s_2 b_3] \cdot [R_1 b_1 + R_2 b_2 + R_3 b_3] + [0] \cdot \sum S_i b_i = {}^R \omega^B \cdot R + {}^R \omega^P \cdot S$$

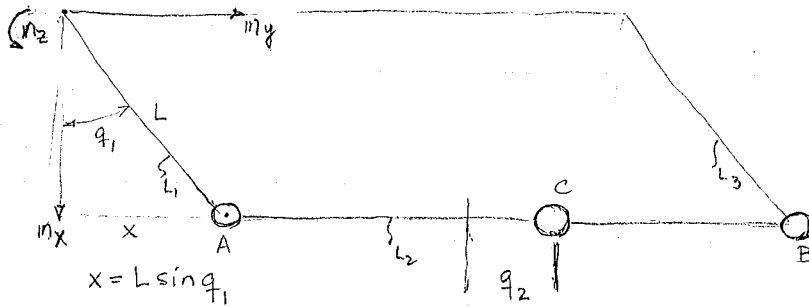
$$F_1 = c_2 R_1 + s_2 R_3 = R_1 \cos \theta_2 + R_3 \sin \theta_2$$



$$F_2 = (+1b_2) \cdot R + 0 \cdot S = +R_2 = \overset{R}{\omega} \overset{S}{g_2} \cdot R + \overset{R}{N} \overset{P}{g_2} \cdot S$$

$$F_3 = \overset{R}{\omega} \overset{S}{g_3} \cdot R + \overset{R}{N} \overset{P}{g_3} \cdot S = 0 \cdot \sum R_i 1b_i + \frac{1b_2}{c_1} \cdot (S_1 1b_1 + S_2 1b_2 + S_3 1b_3) = \frac{S_2}{c_1} = S_2 \sec \theta$$





Since C slides on a smooth rod the contact force of C on the rod (or vice versa) contributes nothing to the generalized active force F_r

$$r^A = L \cos q_1 m_x + L \sin q_1 m_y$$

$$r^{L_1^*} = r^A \frac{1}{2} \Rightarrow v^{L_1^*} = \frac{v^A}{2} \quad \omega^{L_1^*} = \frac{\dot{q}_1}{2}$$

$$v^A = -L \sin q_1 \dot{q}_1 m_x + L \cos q_1 \dot{q}_1 m_y$$

$$r^{L_2^*} = r^A + L m_x \quad v^{L_2^*} = v^A; \quad \omega^{L_2^*} = \dot{q}_1$$

$$r^C = L \cos q_1 m_x + (L \sin q_1 + L + q_2) m_y$$

$$v^C = -\dot{q}_1 L \sin q_1 m_x + (\dot{q}_1 L \cos q_1 + \dot{q}_2) m_y$$

$$r^{L_3^*} = r^{L_1^*} + 2L m_x \quad v^{L_3^*} = v^{L_1^*} = \frac{v^A}{2}; \quad \omega^{L_3^*} = \frac{\dot{q}_1}{2}$$

$$r^B = L \cos q_1 m_x + (2L + L \sin q_1) m_y$$

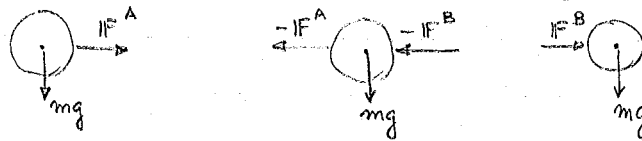
$$v^B = -L \dot{q}_1 \sin q_1 m_x + \dot{q}_1 L \cos q_1 m_y$$

$$a^A = (-\ddot{q}_1 L \sin q_1 + L \cos q_1 \dot{q}_1^2) m_x + (\ddot{q}_1 L \cos q_1 - \dot{q}_1^2 L \sin q_1) m_y$$

$$a^C = -(\ddot{q}_1 L \sin q_1 + \dot{q}_1^2 L \cos q_1) m_x + (\ddot{q}_1 L \cos q_1 - \dot{q}_1^2 L \sin q_1 + \ddot{q}_2) m_y$$

$$a^B = -(L \ddot{q}_1 \sin q_1 + L \dot{q}_1^2 \cos q_1) m_x + (\ddot{q}_1 L \cos q_1 - \dot{q}_1^2 L \sin q_1) m_y$$

the force on A is $F^A = +k q_2 m_y$ $F^C = -2k q_2 m_y$ $F^B = k q_2 m_y$



Since L_1, L_2, L_3 are pin connected at their common joints and L_1, L_3 are pin connected to S, the contribution of the forces at the joints to the generalized forces are zero.

$$\text{Now } (F_r)_I = \sum \tilde{W}_{\dot{q}_r}^{P_i} \cdot F_i = \tilde{W}_{\dot{q}_r}^A \cdot F^A + \tilde{W}_{\dot{q}_r}^B \cdot F^B + \tilde{W}_{\dot{q}_r}^C \cdot F^C$$

$$(F_r)_q = \sum \tilde{W}_{\dot{q}_r}^{L_i^*} \cdot m g m_x = \tilde{W}_{\dot{q}_r}^{L_1^*} \cdot M g m_x + \tilde{W}_{\dot{q}_r}^{L_2^*} \cdot 2M g m_x + \tilde{W}_{\dot{q}_r}^{L_3^*} \cdot M g m_x$$

$$\tilde{W}_{\dot{q}_1}^A = -L \sin q_1 m_x + L \cos q_1 m_y$$

$$\tilde{W}_{\dot{q}_2}^A = 0$$

$$\tilde{W}_{\dot{q}_1}^{L_1^*} = \frac{1}{2} \tilde{W}_{\dot{q}_1}^A$$

$$\tilde{W}_{\dot{q}_2}^{L_1^*} = 0$$

$$\tilde{W}_{\dot{q}_1}^B = -L \sin q_1 m_x + L \cos q_1 m_y$$

$$\tilde{W}_{\dot{q}_2}^B = 0$$

$$\tilde{W}_{\dot{q}_1}^{L_2^*} = \tilde{W}_{\dot{q}_1}^B$$

$$\tilde{W}_{\dot{q}_2}^{L_2^*} = 0$$

$$\tilde{W}_{\dot{q}_1}^C = -L \sin q_1 m_x + L \cos q_1 m_y$$

$$\tilde{W}_{\dot{q}_2}^C = m_y$$

$$\tilde{W}_{\dot{q}_1}^{L_3^*} = \frac{1}{2} \tilde{W}_{\dot{q}_1}^A$$

$$\tilde{W}_{\dot{q}_2}^{L_3^*} = 0$$

Assume
rods are massless
 $M=0$

$$(F_1)_I = k L q_2 \cos q_1 + k L q_2 \cos q_1 - 2k L q_2 \cos q_1 = 0$$

$$(F_2)_I = -2k q_2$$

$$(F_1)_q = -\frac{1}{2} M g L \sin q_1 + (-2L M g \sin q_1) + -\frac{1}{2} M g L \sin q_1 - m g L \sin q_1 - m g L \sin q_1 - m g L \sin q_1$$

$$(F_2)_G = 0 + 0 + 0 + 0 + 0 + 0 = 0$$

$$\therefore F_1 = -3mLg \sin q_1 - 3mLg \sin q_1$$

$$F_1 = -3mLg \sin q_1$$

$$F_2 = -2kq_2$$

$$F_2 = -2kq_2$$

$$F_r^* = \sum \tilde{W}_{q_r}^{P_i} \cdot IF_i^* \quad \text{where} \quad IF_i^* = -m_i a_i^{P_i} \quad IF^{A^*} = -m a^A \quad IF^{B^*} = -m a^B \quad IF^{C^*} = -m a^C$$

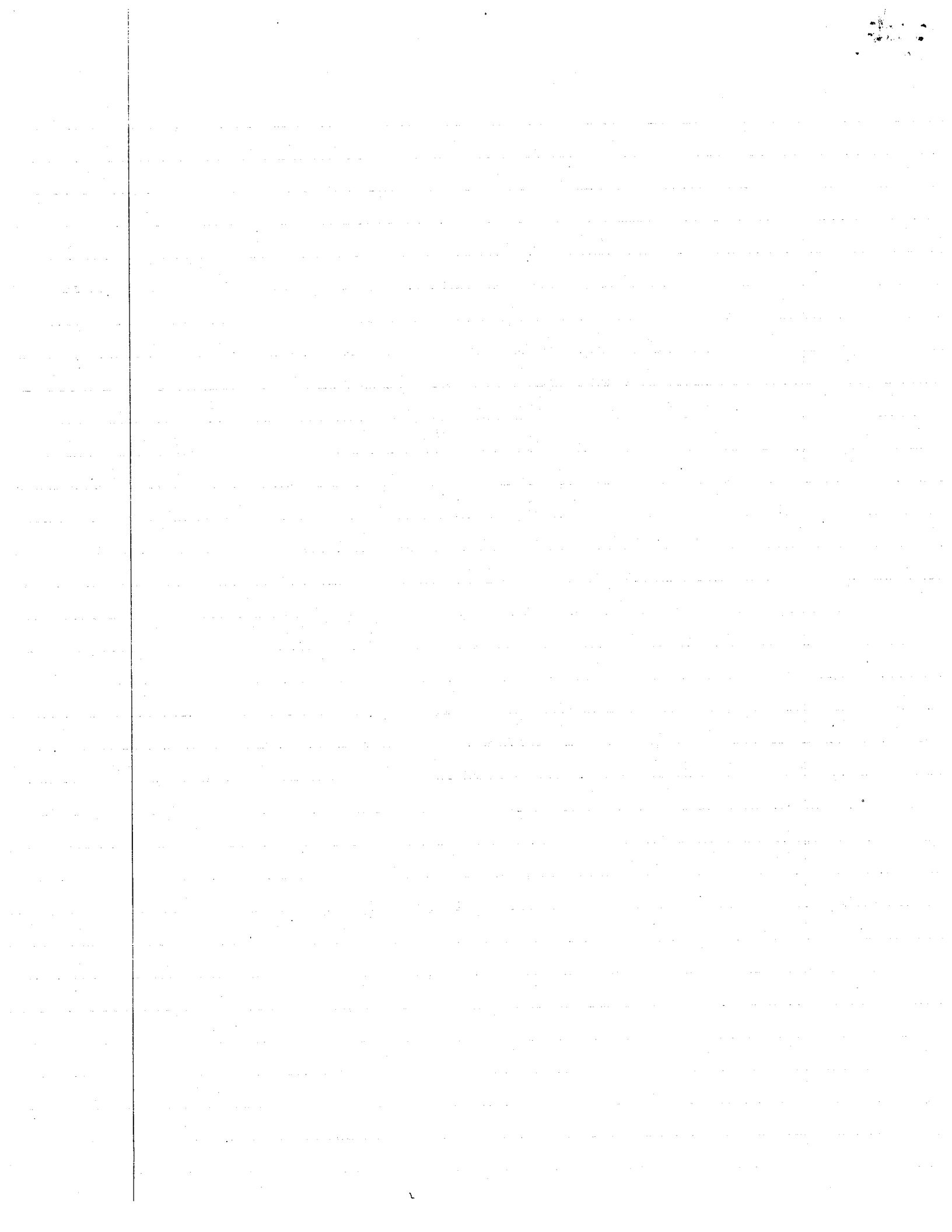
$$\begin{aligned} F_1^* &= W_{q_1}^A \cdot IF^{A^*} + W_{q_1}^B \cdot IF^{B^*} + W_{q_1}^C \cdot IF^{C^*} \\ &= -m \left[-L \sin q_1 (-\ddot{q}_1 L \sin q_1 + L \cos q_1 \dot{q}_1^2) + L \cos q_1 (\ddot{q}_1 L \cos q_1 - \dot{q}_1^2 L \sin q_1) \right] \\ &\quad - m \left[-L \sin q_1 (-L \ddot{q}_1 \sin q_1 + L \dot{q}_1^2 \cos q_1) + L \cos q_1 (\ddot{q}_1 L \cos q_1 - \dot{q}_1^2 L \sin q_1) \right] \\ &\quad - m \left[-L \sin q_1 (-\ddot{q}_1 L \sin q_1 - L \dot{q}_1^2 \cos q_1) + L \cos q_1 (\ddot{q}_1 L \cos q_1 - \dot{q}_1^2 L \sin q_1 + \ddot{q}_2) \right] \end{aligned}$$

$$\begin{aligned} F_2^* &= W_{q_2}^A \cdot IF^{A^*} + W_{q_2}^B \cdot IF^{B^*} + W_{q_2}^C \cdot IF^{C^*} \\ &= 0 + 0 + (-m)(\ddot{q}_1 L \cos q_1 - \dot{q}_1^2 L \sin q_1 + \ddot{q}_2) \end{aligned}$$

$$F_2 + F_2^* = -2kq_2 - m\ddot{q}_1 L \cos q_1 + m\dot{q}_1^2 L \sin q_1 - m\ddot{q}_2 = 0$$

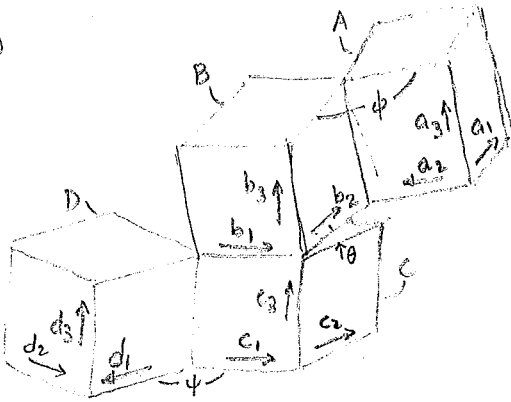
$$F_1 + F_1^* = -3mLg \sin q_1 - 3mL^2\ddot{q}_1 - mL \cos q_1 \ddot{q}_2 = 0$$

$$\begin{aligned} F_1^* &= \left[3mL \sin q_1 (-\ddot{q}_1 L \sin q_1 - L \cos q_1 \dot{q}_1^2) - 3mL \cos q_1 (\ddot{q}_1 L \cos q_1 - \dot{q}_1^2 L \sin q_1) \right] - mL \cos q_1 \ddot{q}_2 \\ &\quad - 3mL^2 \ddot{q}_1 \sin^2 q_1 - 3mL^2 \sin q_1 \cos q_1 \dot{q}_1^2 - 3mL \cos^2 q_1 \ddot{q}_1 + 3mL^2 \dot{q}_1^2 \sin q_1 \cos q_1 - mL \cos q_1 \ddot{q}_2 \\ &= -3mL^2 \ddot{q}_1 - mL \cos q_1 \ddot{q}_2 \end{aligned}$$



Problem Set #1

1(a)



$$\begin{aligned}
 \bullet \quad {}^B \underline{a}_1 &= \cos \phi \underline{b}_1 + \sin \phi \underline{b}_2 \quad \therefore \frac{\partial {}^B \underline{a}_1}{\partial \phi} = -\sin \phi \underline{b}_1 + \cos \phi \underline{b}_2 \\
 \text{and } \left| \frac{\partial {}^B \underline{a}_1}{\partial \phi} \right| &= (\cos^2 \phi + \sin^2 \phi)^{1/2} = 1 \\
 \bullet \quad {}^B \underline{b}_1 &= \underline{b}_1 \quad \therefore \frac{\partial {}^B \underline{b}_1}{\partial \phi} = 0 \quad \therefore \left| \frac{\partial {}^B \underline{b}_1}{\partial \phi} \right| = 0 \\
 \bullet \quad {}^B \underline{a}_3 &= \underline{b}_3 \quad \therefore \frac{\partial {}^B \underline{a}_3}{\partial \phi} = \frac{\partial \underline{b}_3}{\partial \phi} = 0 \quad \therefore \left| \frac{\partial {}^B \underline{a}_3}{\partial \phi} \right| = 0 \quad \text{a}_3 \text{ not in } 1^{\text{st}} \text{ frame} \\
 \bullet \quad {}^B \underline{b}_2 &= \underline{b}_2 \quad \therefore \frac{\partial {}^B \underline{b}_2}{\partial \theta} = 0 \quad \therefore \left| \frac{\partial {}^B \underline{b}_2}{\partial \theta} \right| = 0 \quad \text{b}_2 \text{ changes dir w/ } \theta \text{ but no } \phi \\
 \bullet \quad {}^C \underline{b}_2 &= \cos \theta \underline{c}_2 + \sin \theta \underline{c}_3 \quad \therefore \frac{\partial {}^C \underline{b}_2}{\partial \theta} = -\sin \theta \underline{c}_2 + \cos \theta \underline{c}_3 \\
 \text{and } \left| \frac{\partial {}^C \underline{b}_2}{\partial \theta} \right| &= (\sin^2 \theta + \cos^2 \theta)^{1/2} = 1
 \end{aligned}$$

$$\begin{aligned}
 \bullet \quad {}^D \underline{c}_2 &= \cos \psi \underline{d}_2 - \sin \psi \underline{d}_1 \quad {}^D \underline{c}_3 = \underline{d}_3 \\
 \therefore {}^D \underline{b}_2 &= \cos \theta \cos \psi \underline{d}_2 - \cos \theta \sin \psi \underline{d}_1 + \sin \theta \underline{d}_3 \\
 \therefore \frac{\partial {}^D \underline{b}_2}{\partial \theta} &= -\sin \theta \cos \psi \underline{d}_2 - \cos \theta \sin \psi \underline{d}_1 + \cos \theta \underline{d}_3 \Rightarrow \left| \frac{\partial {}^D \underline{b}_2}{\partial \theta} \right| = 1
 \end{aligned}$$

$$\bullet \quad {}^C \underline{b}_2 = \cos \theta \underline{c}_2 + \sin \theta \underline{c}_3 \quad \frac{\partial {}^C \underline{b}_2}{\partial \psi} = 0 \quad \therefore \left| \frac{\partial {}^C \underline{b}_2}{\partial \psi} \right| = 0$$

$$\bullet \quad {}^D \underline{b}_2 = \cos \theta \cos \psi \underline{d}_2 - \cos \theta \sin \psi \underline{d}_1 + \sin \theta \underline{d}_3 \quad \therefore \frac{\partial {}^D \underline{b}_2}{\partial \psi} = -\cos \theta \sin \psi \underline{d}_2 - \cos \theta \cos \psi \underline{d}_1 = -\cos \theta \underline{b}_1$$

$$\text{thus } \left| \frac{\partial {}^D \underline{b}_2}{\partial \psi} \right| = |\cos \theta| (\sin^2 \psi + \cos^2 \psi)^{1/2} = |\cos \theta|$$

$$\begin{aligned}
 \bullet \quad {}^D \underline{b}_1 &= \cos \psi \underline{d}_1 + \sin \psi \underline{d}_2 = {}^D \underline{c}_1 \quad \text{Now } {}^D \underline{a}_1 = \cos \phi {}^D \underline{b}_1 + \sin \phi {}^D \underline{b}_2 \\
 &= \cos \phi \cos \psi \underline{d}_1 + \cos \phi \sin \psi \underline{d}_2 - \sin \phi \cos \theta \sin \psi \underline{d}_2 + \sin \phi \cos \theta \cos \psi \underline{d}_1 + \sin \phi \sin \theta \underline{d}_3 \\
 &= (\cos \phi \cos \psi - \sin \phi \cos \theta \sin \psi) \underline{d}_1 + (\cos \phi \sin \psi + \sin \phi \cos \theta \cos \psi) \underline{d}_2 + \sin \phi \sin \theta \underline{d}_3
 \end{aligned}$$

$$\begin{aligned}
 \frac{\partial {}^D \underline{a}_1}{\partial \psi} &= (-\cos \phi \sin \psi - \sin \phi \cos \theta \cos \psi) \underline{d}_1 + (\cos \phi \cos \psi + \sin \phi \cos \theta \sin \psi) \underline{d}_2 \\
 \left| \frac{\partial {}^D \underline{a}_1}{\partial \psi} \right| &= (\cos^2 \phi \cdot 1 + \sin^2 \phi \cos^2 \theta \cdot 1)^{1/2}
 \end{aligned}$$

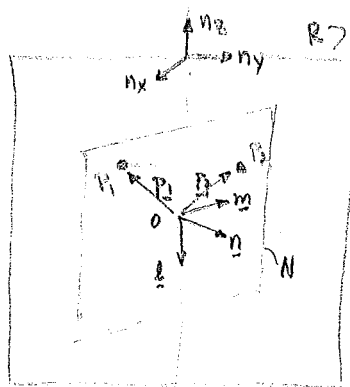
$$1(b) \quad {}^B \underline{a}_1 = \cos \phi \underline{b}_1 + \sin \phi \underline{b}_2 \quad {}^C \underline{b}_1 = \underline{c}_1 \quad {}^C \underline{b}_2 = \cos \theta \underline{c}_2 + \sin \theta \underline{c}_3$$

$$\begin{aligned}
 \therefore {}^B \underline{a}_1 &= \cos \phi \underline{c}_1 + \sin \phi (\cos \theta \underline{c}_2 + \sin \theta \underline{c}_3) \quad \frac{\partial {}^B \underline{a}_1}{\partial \theta} = \sin \phi (-\sin \theta \underline{c}_2 + \cos \theta \underline{c}_3) \\
 &= \sin \phi {}^C \underline{b}_3 = \sin \phi \underline{a}_3
 \end{aligned}$$

$$\therefore \frac{\partial {}^B \underline{a}_1}{\partial \theta} = \omega_1 \underline{a}_1 + \omega_2 \underline{a}_2 + \omega_3 \underline{a}_3 \Rightarrow \omega_1 = \omega_2 = 0 \quad \omega_3 = \sin \phi$$

$$\begin{aligned}
 1(c) \quad \left. \frac{d \underline{a}_1}{dt} \right|_P &= \left. \frac{\partial \underline{a}_1}{\partial \theta} \frac{d\theta}{dt} \right|_P + \left. \frac{\partial \underline{a}_1}{\partial \phi} \frac{d\phi}{dt} \right|_P + \left. \frac{\partial \underline{a}_1}{\partial \psi} \frac{d\psi}{dt} \right|_P \quad \text{Since } {}^C \underline{a}_1 = \cos \phi \underline{c}_1 + \sin \phi \cos \theta \underline{c}_2 + \sin \phi \sin \theta \underline{c}_3 \\
 & \quad \frac{\partial {}^C \underline{a}_1}{\partial \theta} = \sin \phi {}^C \underline{b}_3 \quad \frac{\partial {}^C \underline{a}_1}{\partial \phi} = -\sin \phi \underline{c}_1 + \cos \phi (\underline{b}_2) \\
 \text{but } \underline{c}_1 &= {}^C \underline{b}_1 = \cos \phi \underline{a}_1 - \sin \phi \underline{a}_2 \quad {}^C \underline{b}_2 = \cos \phi \underline{a}_2 + \sin \phi \underline{a}_1 \quad \therefore \frac{\partial {}^C \underline{a}_1}{\partial \phi} = \underline{a}_2 \\
 \text{and } \underline{a}_1 &\neq \text{fn of } \psi \Rightarrow \frac{d \underline{a}_1}{dt} = (\sin \phi \underline{a}_3) \cdot 6 \text{ rad/sec} + (\underline{a}_2) \cdot 4 \text{ rad/sec} = \frac{1}{2} \cdot 6 \underline{a}_3 + 4 \underline{a}_2 = 3 \underline{a}_3 + 4 \underline{a}_2
 \end{aligned}$$

11(d)



$$\underline{\ell} = -\underline{n}_z$$

$$\underline{n} = \underline{n}_x \cos \omega t + \underline{n}_y \sin \omega t$$

$$\underline{m} = \underline{n}_y \cos \omega t - \underline{n}_x \sin \omega t$$

$$\underline{p}_1 = a_{11} \underline{\ell} + a_{12} \underline{m} = -a_{11} \underline{n}_z + a_{12} \cos \omega t \underline{n}_y - a_{12} \sin \omega t \underline{n}_x$$

$$\underline{p}_2 = a_{21} \underline{\ell} + a_{22} \underline{m} = -a_{21} \underline{n}_z + a_{22} \cos \omega t \underline{n}_y - a_{22} \sin \omega t \underline{n}_x$$

$$\text{Now } \underline{p}_1 - \underline{p}_2 = (a_{11} - a_{21}) \underline{\ell} + (a_{12} - a_{22}) \underline{m}$$

$$x_1 = -a_{12} \sin \omega t \quad y_1 = a_{12} \cos \omega t \quad z_1 = -a_{11}$$

$$x_2 = -a_{22} \sin \omega t \quad y_2 = a_{22} \cos \omega t \quad z_2 = -a_{21}$$

$$\therefore |\underline{p}_1 - \underline{p}_2|^2 = (a_{11} - a_{21})^2 + (a_{12} - a_{22})^2 = L^2$$

$$\text{Now } \begin{cases} x_1 \cos \omega t + y_1 \sin \omega t = 0 = f_1(x_1, y_1, z_1, x_2, y_2, z_2, t) \\ x_2 \cos \omega t + y_2 \sin \omega t = 0 = f_2(x_1, y_1, z_1, x_2, y_2, z_2, t) \end{cases}$$

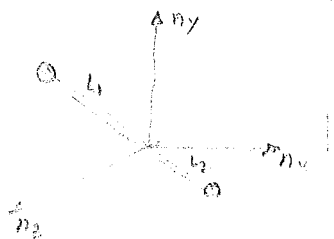
$$f_1 = 0$$

$$f_2 = 0$$

$$|\underline{p}_1 - \underline{p}_2|^2 = (a_{11} - a_{21})^2 + (a_{12} - a_{22})^2 = L^2$$

$$f_3 = |\underline{p}_1 - \underline{p}_2|^2 - L^2 = (z_1 - z_2)^2 + (x_1 - x_2)^2 + (y_1 - y_2)^2 - L^2 = 0$$

11(e)



Since P_1 and P_2 move in the plane of n_y and n_x then

$$z_1 = z_2 = 0 \quad \text{and} \quad \underline{p}_i = x_i \underline{n}_x + y_i \underline{n}_y \quad |\underline{p}_i|^2 = L_i^2$$

$$\therefore |x_i^2 + y_i^2 - L_i^2 = 0 \quad i=1,2|$$

$$\text{also } \underline{p}_1 - \underline{p}_2 = (x_1 - x_2) \underline{n}_x + (y_1 - y_2) \underline{n}_y \quad \therefore |\underline{p}_1 - \underline{p}_2|^2 = (L_1 + L_2)^2$$

$$\therefore (x_1 - x_2)^2 + (y_1 - y_2)^2 = (L_1 + L_2)^2 \Rightarrow x_1^2 + x_2^2 - 2x_1x_2 + y_1^2 + y_2^2 - 2y_1y_2 = L_1^2 + 2L_1L_2 + L_2^2$$

using the 3rd & 4th constraint

$$\therefore |x_1x_2 + y_1y_2 + L_1L_2 = 0| \quad \|\underline{p}_1\| \|\underline{p}_2\| = -|\underline{p}_1| |\underline{p}_2| \cos \pi$$

11(f) What are the degrees of freedom of $S = (\underline{p}_1, \underline{p}_2)$ in $\mathbb{R}^3$ in problem 11(d)

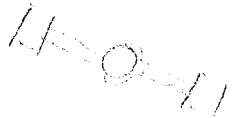
$\underline{p}_1$ & $\underline{p}_2$ have 3 degrees of freedom in $\mathbb{R}^3$ (in the n_x, n_y, n_z directions)

$$3 \cdot 2 \text{ part} - 3 \text{ constr} = 6 - 3 = 3.$$

11(g) Using problem 11(e) let q be the angle between $\underline{p}_2$ and $\underline{n}_x$ and rewrite the 5 constraint eqns in terms of q

$$z_1 = z_2 = 0 \quad x_2 = L_2 \cos q \quad y_2 = -L_2 \sin q \quad x_1 = -L_1 \sin(90 - q) = -L_1 \cos q \quad y_1 = L_1 \sin q$$

1 (k) Determine the number of degrees of freedom of the following

- (a)  since the ball & socket can rotate in any of 3 directions
 6 degrees of freedom - fixed particle & socket gives 3 constraints - $6 - 3 = 3$

- (b) ~~point on sun; 2 particles earth satellite & rocket, 3 constraints~~ 7 degrees of freedom (6 for translation, 1 for rotation)
 1 constraint $\phi - f(t) = 0$
 $n = 3 \cdot 3 - 3 = 6$

- (c) Same as (b) yet no 3rd constraint $n = 3 \cdot 3 - 2 = 7$

- (d) $2 \cdot 3 - 5 = 1$ Particular $P_1 \neq P_2$ of problem 1(c)

